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## Article

# Robust Nonlinear Control of a Wind Turbine with a Permanent Magnet Synchronous Generator

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**Abstract:** In this paper, a robust nonlinear dynamic controller is designed for a wind turbine with a permanent magnet synchronous generator. The wind turbine is subject to variations in all the parameters appearing in its mathematical model. Furthermore, the wind velocity is considered unavailable for direct measurement. This situation is of particular interest in practical applications, where only the nominal parameter values are known, and accurate wind velocity measurement is challenging due to perturbations caused by the turbine itself, even with appropriate sensors. The problem addressed in this work involves tracking the reference angular velocity corresponding to the wind velocity. To achieve accurate tracking, appropriate compensation of the perturbation terms resulting from parameter uncertainties and wind estimation errors is required. To address this problem, an estimator of the wind velocity is utilized, along with high-order sliding mode parameter estimators, to ensure high-performance operation of the turbine.

**Keywords:** wind turbine; nonlinear control; parameter variations; high-order sliding mode

## 1. Introduction

Recently, Wind Turbine Conversion Systems (WTCSs) have received great attention by the scientific community for the exploitation of renewable energies. Many works available in the literature focus on the nonlinear control of small and large scale wind energy generation systems. In particular, Permanent Magnet Synchronous Generators (PMSGs) are used as wind turbine generators due to their advantages regarding the efficient and reliable performance. These advantages are linked to their properties of self-excitation and of low speed, which results in direct-drive WTCSs. In fact, PMSGs have a simpler mechanical structure, may have direct coupling to the wind turbine shaft avoiding gears, and do not need an external excitation system, so avoiding the corresponding copper losses.

The aim is to maximize the power generated by the Wind Turbine (WT) under the condition of variable wind speed. This is achieved maintaining the so-called tip-speed ratio, i.e. the ratio between the blade tip linear velocity and the wind velocity, at an optimal value. For a given wind velocity, this optimal value of the tip-speed ratio corresponds to a certain angular velocity, which is considered as a reference to be tracked. Therefore, the problem of the optimal energy extraction can be expressed as a tracking problem of the reference angular velocity corresponding to the given wind velocity.

This tracking problem is here solved in the presence of some difficulties that normally arise in practical cases. The first difficulty is that the measurement of the real wind velocity value is unrealistic for industrial WTs. In fact, the wind velocity can not be measured with accuracy since, even in the presence of appropriate sensors, the wind velocity field is perturbed by the turbine itself. A second difficulty arises from the fact that the parameters appearing in the mechanical and electrical equations

are usually subject to parameter variations or uncertainties. Examples are the winding resistance, air density, etc., which are subject to variations, and the PMSG rotor inertia, friction, etc., which are known with a certain uncertainty. In practice, only the nominal values of these parameters can be assumed known. In order to solve the tracking problem in the presence of these two difficulties, in this work an estimator of the wind velocity is used. Furthermore, high-order sliding mode estimators are used to estimate the perturbation terms arising in the dynamical equations due to the parameter uncertainties. This allows a compensation of the perturbation terms due to the parameter uncertainties and of the wind estimation error, ensuring an accurate tracking of the reference angular velocity.

Regarding parameter variations and uncertainties, the First Order Sliding Mode (FOSM) technique constitutes an interesting methodology that can be used to compensate their effects. This is achieved thanks to the inherent robustness of this technique [1]. Examples of FOSM control of WT are given in [2–4] which presented robust controllers designed for variable speed WTs with doubly fed induction generators. A FOSM control is also designed in [5] for a turbine model considering the mechanical dynamics only, along with a modified Newton–Raphson algorithm to estimate the wind speed. Another FOSM controller, using the blade pitch as input, was designed in [6] in the presence of uncertainties, in order to regulate the rotor speed to a fixed rated value. A FOSM control for a variable speed WT with PMSG was considered in [7], under the assumption of measurability of the wind speed. In [8], an algorithm based on the combination of FOSM and fractional-order SM was proposed, where the latter ensures finite-time convergence to zero of the angular velocity tracking error. In [9], a novel robust FOSM control was proposed, using nonlinear perturbation observers for WTs with a doubly-fed induction generator. Finally, an integral terminal FOSM controller was proposed in [10] to enhance the power quality of WTs under unbalanced voltage conditions.

FOSM control schemes may suffer from the well-known chattering problem, i.e. high frequency oscillations due to actuator bandwidth limitations, unable to reproduce exactly the sign “function” used in FOSM, which has clear negative consequences. In order to overcome this difficulty, and to consider SM surfaces with relative degree greater than one [11], High-Order Sliding-Mode (HOSM) techniques can be used. These techniques, also called Super-Twisting Sliding-Mode (STSM) techniques, have other appealing properties, since they ensure finite-time convergence to the origin, and robustness with respect to perturbation acting on the system. Such techniques were developed starting from the seminal paper of [12] and successively fully developed in successive papers [13–19]. HOSM techniques were first used for smooth control systems, and successively as finite-time differentiators. They were also applied successfully to a number of applications [11,20–27]. Recently, extensive research has been undertaken to control PMSG for wind energy conversion applications using HOSM techniques. In [28], an efficient controller based on a HOSM was proposed and applied to a PMSG. [29] presented an output HOSM power control of a Wind Energy Conversion System (WECS) based on a PMSG, which integrates a stand-alone hybrid system for the optimum power conversion and power regulation operational mode. In [4,30], a HOSM control scheme was developed for a PMSG, consisting of a robust aerodynamic torque observer based on the super-twisting algorithm in order to eliminate the sensors for the wind speed measurement, and a robust wind turbine speed control to regulate the rotor currents. [31] proposed an adaptive sliding mode speed control algorithm with an integral-operation sliding surface for a variable speed wind energy experimental system. Besides, the proposed controller includes an estimator that deals with the unknown turbine torque and inaccuracies in the mathematical model of the system and attempts to achieve zero steady-state error. [32] present a Super Twisting (ST) algorithm using an Adaptive Second-Order Sliding Mode Control (SOSMC) to derive a robust and fast current control for PMSG-based WECS. The aim of the controller is to force the PMSG to deliver the requested power to the grid via controlling the winding current. A conventional sliding mode (SMC) and second-order sliding mode (SOSMC) control schemes based on pulse width modulation (PWM) for the rotor side converter (RSC) and grid side converter (GSC) feeding a doubly fed induction generator (DFIG) are presented in [33,34]. It is worth nothing that all these works do not deal with parameter uncertainties in the PMSG dynamics.

In this paper, a robust nonlinear dynamic controller is designed for a WT with a PMSG, which guarantees the maximization of the power extracted by the WT from the wind. The control objective is to track an appropriate angular velocity signal, ensuring the asymptotic stability of the closed-loop system despite variations in all the parameters present in the WT mathematical model. The original contributions of this paper are centered around solving the tracking problem in the presence of the following factors:

1. Unknown wind velocity, which is estimated based on measurements of the produced power.
2. Parameter variations and uncertainties in the WT model, estimated using high-order sliding mode estimators.

It is worth noting that the unavailability of wind velocity measurement is not considered in the available literature.

The paper is organized as follows. In Section 2, the mathematical model of a WT is recalled, and the control problem is formulated. In Section 3, a robust nonlinear dynamic controller is presented. In Section 4, the proposed controller is tested via simulations. Some comments conclude the paper.

## 2. Mathematical Model of a WT

The power extracted by the WT from the wind is given by [35]

$$P_w = \frac{1}{2} \rho \pi R^2 C_p(\lambda, \beta) v_w^3 = k_w \omega^3, \quad k_w = \frac{1}{2} \rho \pi R^5 \frac{C_p(\lambda, \beta)}{\lambda^3}$$

where  $\rho$  is the air density,  $R$  is the WT rotor radius,  $v_w$  is the wind speed,  $\omega$  is the turbine shaft speed, and  $\lambda = \omega R / v_w$  is the tip speed ratio, with  $\omega R$  the tip speed of the turbine blade. Moreover,  $C_p(\lambda, \beta)$  is the power coefficient, depending on the blade design. The power coefficient is function of the blade pitch angle  $\beta$  and of the tip speed ratio  $\lambda$ . It represents the turbine efficiency to convert the kinetic energy of the wind into mechanical energy. The power coefficient  $C_p$  is a nonlinear function of  $\lambda$  which, for  $\beta = \beta^\circ = 0$ , can be approximated as [36,37]

$$C_p(\lambda, \beta^\circ) = c_1 \left( c_2 \frac{1 - c_3 \lambda}{\lambda} - c_4 \right) e^{-c_5 \frac{1 - c_3 \lambda}{\lambda}} + c_6 \lambda$$

where  $c_1, \dots, c_6$  are experimental coefficients depending on the shape of the blade and on its aerodynamic performance.

To maximize the power  $P_w$  generated by the WT for a given wind velocity  $v_w$ , one has to maximize the power coefficient  $C_p$ . Since this latter depends on  $\lambda$  and  $\beta$ , also its maximum depends on these two variables. Nevertheless, usually  $\beta$  is changed occasionally, and in the following it will be considered constant and equal to a value  $\beta^\circ = 0$  deg. Therefore, the maximization of  $P_w$  is achieved maximizing  $C_p(\lambda, \beta^\circ)$ , which has its maximum  $C_{p,\max}$  for  $\lambda_{\max}$ . Obviously, the same reasoning can be followed for other values of the blade pitch angle  $\beta$ .

The model of a PMSG in the  $(d, q)$  reference frame, rotating synchronously with the generator rotor, is [37]

$$\begin{aligned} \frac{di_d}{dt} &= -\frac{R_s}{L} i_d + p\omega i_q + \frac{1}{L} v_d \\ \frac{di_q}{dt} &= -\frac{R_s}{L} i_q - p\omega i_d - p \frac{\phi_m}{L} \omega + \frac{1}{L} v_q \\ \dot{\omega} &= -\frac{k_m}{J} i_q - \frac{f}{J} \omega + \frac{T_w}{J} = -\frac{k_m}{J} i_q - \frac{f}{J} \omega + \frac{k_w}{J} \omega^2 \end{aligned} \quad (1)$$

where  $i_d, i_q$  are the  $d, q$ -axis stator currents,  $v_d, v_q$  are the  $d, q$ -axis stator voltages,  $R_s$  is the winding resistance and  $L = L_d = L_q$  is the winding inductance on axis  $d$  and  $q$ ,  $\psi_m$  is the flux linkage of the permanent magnet,  $\omega_e = p\omega$  is the electrical angular speed of the generator rotor, and  $p$  is the number of pole pairs. In the mechanical equation,  $T_e = k_m i_q$  is the electrical torque, with  $k_m = 3p\phi_m/2$

the torque constant. The developed torque of the motor is proportional to the  $i_q$  current because of the assumption that there is no reluctance torque in the considered PMSG. Furthermore,  $J$  is the mechanical inertia, and  $f$  is the coefficient of the viscous friction. The torque extracted from the wind is  $T_w = P_w/\omega = k_w\omega^2$ . It summarizes the effect of the aerodynamic torque, generally known with large inaccuracy.

In this work, all the parameters appearing in the electric and mechanical dynamics (1) are assumed to be subject to variation. Let  $R_s^\circ, L^\circ, \phi_m^\circ, k_m^\circ = 3p\phi_m^\circ/2, J^\circ, f^\circ$  be the nominal values of  $R_s, L, \phi_m, k_m, J, f$ , and

$$k_w^\circ = \frac{1}{2}\varrho^\circ\pi(R^\circ)^5\frac{C_p^\circ(\lambda^\circ, \beta^\circ)}{(\lambda^\circ)^3}$$

the nominal value of  $k_w$ , where  $\varrho^\circ$  is the nominal value of  $\varrho$ ,  $R^\circ$  is the nominal value for  $R$ ,  $\lambda^\circ = \omega R^\circ/v_w$ , and  $C_p^\circ$  is the nominal function

$$C_p^\circ(\lambda^\circ, \beta^\circ) = c_1^\circ \left( c_2^\circ \frac{1 - c_3^\circ \lambda^\circ}{\lambda^\circ} - c_4^\circ \right) e^{-c_5^\circ \frac{1 - c_3^\circ \lambda^\circ}{\lambda^\circ}} + c_6^\circ \lambda^\circ$$

with  $c_1^\circ, \dots, c_6^\circ$ , the nominal values of  $c_1, \dots, c_6$ . Finally, the nominal value of  $T_w$  is  $T_w^\circ = k_w^\circ\omega^2$ .

In what follows it will be assumed that the parameter variations  $|R_s - R_s^\circ|, |L - L^\circ|, |\phi_m - \phi_m^\circ|, |J - J^\circ|, |f - f^\circ|, |\varrho - \varrho^\circ|, |R - R^\circ|$  are bounded by certain values  $\Delta_{R_s}, \Delta_L, \Delta_{\phi_m}, \Delta_J, \Delta_f, \Delta_\varrho, \Delta_R$ , respectively. This is an acceptable assumption from a physical point of view.

The *control problem* consists of designing a dynamic controller to track a reference angular velocity  $\omega_{\text{ref}}$ , which will be determined later, in the presence of uncertainties on all the parameters appearing in the model.

Considering the maximization of  $P_w$ , which is obtained imposing the value  $\lambda_{\text{max}}$ , and using the definition of the tip speed ratio, one obtains the reference angular velocity  $\lambda_{\text{max}}v_w/R$ , that should be imposed. However, one observes that such a function is not known, as  $\lambda_{\text{max}}$  and  $v_w$  are not known. Therefore, in the following an estimation of the wind velocity will be determined, while  $\lambda_{\text{max}}$  will be substituted by its (known) nominal value  $\lambda_{\text{max}}^\circ$ .

### 3. Design of the Dynamic Controller for the Tracking of the Angular Velocity Reference

The first step in the design of the desired controller is to estimate the wind velocity, in order to obtain a known angular velocity reference.

#### 3.1. Design of the Wind Velocity Estimator

As already commented, the measurement of the real velocity value  $v_w$  is unrealistic for industrial WTs. In the following, the wind velocity  $v_w$  can be estimated from the measurement of the produced power  $P_m$ . In fact, considering the nominal value  $k_w^\circ$  and the function

$$f(\lambda^\circ) = P_m - k_w^\circ\omega^3 = P_m - \frac{1}{2}\varrho^\circ\pi(R^\circ)^5\frac{C_p^\circ(\lambda^\circ, \beta^\circ)}{(\lambda^\circ)^3}\omega^3$$

it is possible to determine the value of  $\lambda^\circ$  imposing  $f(\lambda^\circ) = 0$ . To this aim, among the other methods, a Newton–Raphson method can be used [38]. Hence, one can look for the solutions  $\hat{\lambda}$  of the equation  $f(\lambda^\circ) = 0$  considering the iterative estimations

$$\lambda_k^\circ = \lambda_{k-1}^\circ - \frac{f(\lambda^\circ)}{f'(\lambda^\circ)} \Big|_{\lambda^\circ = \lambda_{k-1}^\circ}, \quad k = 1, 2, \dots, N$$

for a fixed integer  $N \in \mathbb{Z}$ , where

$$f'(\lambda^\circ) = -\frac{1}{2}q^\circ\pi(R^\circ)^5\omega^3\frac{d(C_p^\circ(\lambda^\circ, \beta^\circ)/(\lambda^\circ)^3)}{d\lambda^\circ}.$$

From the definition of  $\lambda^\circ$ , and using the estimation  $\hat{\lambda}$ , one computes the wind velocity estimation

$$\hat{v}_w = \frac{R^\circ}{\hat{\lambda}}\omega \quad (2)$$

and the following reference angular velocity

$$\omega_{\text{ref}} = \frac{\lambda_{\text{max}}^\circ}{R^\circ}\hat{v}_w = \frac{\lambda_{\text{max}}^\circ}{\hat{\lambda}}\omega \quad (3)$$

can be defined, with  $\lambda_{\text{max}}^\circ$  the (known) maximum value of  $C_p^\circ(\lambda^\circ, \beta^\circ)$ . This is the reference angular velocity that the controller has to track, in the presence of parameter uncertainties.

### 3.2. Design of the Dynamic Controller for the Tracking of the Angular Velocity Reference

Given the model (1), and considering the nominal values  $R_s^\circ, L^\circ, \phi_m^\circ, k_m^\circ, f^\circ, k_w^\circ$ , one obtains the following model

$$\begin{aligned} \frac{di_d}{dt} &= -\frac{R_s^\circ}{L^\circ}i_d + p\omega i_q + \frac{1}{L^\circ}v_d + \Delta_1 \\ \frac{di_q}{dt} &= -\frac{R_s^\circ}{L^\circ}i_q - p\omega i_d - p\frac{\phi_m^\circ}{L^\circ}\omega + \frac{1}{L^\circ}v_q + \delta_2 \\ \dot{\omega} &= -\frac{k_m^\circ}{J^\circ}i_q - \frac{f^\circ}{J^\circ}\omega + \frac{k_w^\circ}{J^\circ}\omega^2 + \Delta_3 \end{aligned} \quad (4)$$

with the perturbation terms

$$\begin{aligned} \Delta_1 &= -\left(\frac{R_s}{L} - \frac{R_s^\circ}{L^\circ}\right)i_d + \left(\frac{1}{L} - \frac{1}{L^\circ}\right)v_d \\ \delta_2 &= -\left(\frac{R_s}{L} - \frac{R_s^\circ}{L^\circ}\right)i_q - p\left(\frac{\phi_m}{L} - \frac{\phi_m^\circ}{L^\circ}\right)\omega + \left(\frac{1}{L} - \frac{1}{L^\circ}\right)v_q \\ \Delta_3 &= -\left(\frac{k_m}{J} - \frac{k_m^\circ}{J^\circ}\right)i_q - \left(\frac{f}{J} - \frac{f^\circ}{J^\circ}\right)\omega + \left(\frac{k_w}{J} - \frac{k_w^\circ}{J^\circ}\right)\omega^2. \end{aligned}$$

The final perturbation term  $\Delta_2$  for the dynamics of  $i_q$  will be given by the sum of  $\delta_2$  and additional terms arising from the derivative of the reference imposed on  $i_q$ . This will become clearer in the subsequent computations.

The following result solves the posed control problem for the system (4).



**Theorem 1.** If the signals  $i_d$ ,  $i_q$ ,  $\omega$  are measurable, the dynamic controller

$$\begin{aligned}
 \dot{\xi}_{1,1} &= -\frac{R^\circ}{L^\circ} i_d + p\omega i_q + \frac{1}{L^\circ} v_d - \alpha_{1,1} [\xi_{1,1} - (i_d - i_{d,\text{ref}})]^{1/2} + \xi_{1,2} \\
 \dot{\xi}_{1,2} &= -\alpha_{1,2} [\xi_{1,1} - (i_d - i_{d,\text{ref}})]^0 \\
 \dot{\xi}_{2,1} &= -\frac{R^\circ}{L^\circ} i_q - p\omega i_d - p\frac{\phi_m^\circ}{L^\circ} \omega + \frac{1}{L^\circ} v_q - D_{i_{q,\text{ref}}}^\circ - \alpha_{2,1} [\xi_{2,1} - (i_q - i_{q,\text{ref}})]^{1/2} + \xi_{2,2} \\
 \dot{\xi}_{2,2} &= -\alpha_{2,2} [\xi_{2,1} - (i_q - i_{q,\text{ref}})]^0 \\
 \dot{\xi}_{3,1} &= c \left( -\frac{k_m^\circ}{J^\circ} i_q - \frac{f^\circ}{J^\circ} \omega + \frac{k_w^\circ}{J^\circ} \omega^2 \right) - \alpha_{3,1} [\xi_{3,1} - (\omega - \omega_{\text{ref}})]^{1/2} + c\xi_{3,2} \\
 \dot{\xi}_{3,2} &= -\frac{\alpha_{3,2}}{c} [\xi_{3,1} - (\omega - \omega_{\text{ref}})]^0 \\
 \dot{e}_{i_d} &= i_d - i_{d,\text{ref}} \\
 \dot{e}_{i_q} &= i_q - i_{q,\text{ref}} \\
 \dot{e}_\omega &= \omega - \omega_{\text{ref}} \\
 v_d &= L^\circ \left( -k_{1,p}(i_d - i_{d,\text{ref}}) - k_{1,i} e_{i_d} + \frac{R_s^\circ}{L^\circ} i_d - p\omega i_q + \frac{d}{dt} i_{d,\text{ref}} - \xi_{1,2} \right) \\
 v_q &= L^\circ \left( -k_{2,p}(i_q - i_{q,\text{ref}}) - k_{2,i} e_{i_q} + \frac{R_s^\circ}{L^\circ} i_q + p\omega i_d + p\frac{\phi_m^\circ}{L^\circ} \omega + D_{i_{q,\text{ref}}}^\circ - \xi_{2,2} \right)
 \end{aligned} \tag{5}$$

$[\cdot]^\alpha = |\cdot|^\alpha \text{sign}(\cdot)$ , with  $c = 1 - \lambda_{\max}^\circ / \hat{\lambda}$ ,  $k_{j,p}, k_{j,i} > 0$ ,  $j = 1, 2, 3$ , and

$$\begin{aligned}
 \alpha_{1,1} &> 0, \quad \alpha_{1,2} > 2 \frac{\theta_1^2}{\alpha_{1,1}^2} + 3\theta_1 \\
 \alpha_{2,1} &> 0, \quad \alpha_{2,2} > 2 \frac{\theta_2^2}{\alpha_{2,1}^2} + 3\theta_2 \\
 \alpha_{3,1} &> 0, \quad \alpha_{3,2} > 2 \frac{c^2 \theta_3^2}{\alpha_{3,1}^2} + 3c\theta_3
 \end{aligned} \tag{6}$$

for some finite positive reals  $\theta_i$ ,  $i = 1, 2, 3$ ,  $\omega_{\text{ref}}$  given by (3),

$$\begin{aligned}
 i_{d,\text{ref}} &= \varphi(t) \\
 i_{q,\text{ref}} &= \frac{J^\circ}{k_m^\circ} \left( \frac{k_{3,p}}{c} (\omega - \omega_{\text{ref}}) + \frac{k_{3,i}}{c} e_\omega + \xi_{3,2} - \frac{f^\circ}{J^\circ} \omega + \frac{k_w^\circ}{J^\circ} \omega^2 \right)
 \end{aligned} \tag{7}$$

$\varphi(t)$  bounded function with bounded derivative, and with

$$\begin{aligned}
 D_{i_{q,\text{ref}}}^\circ &= \frac{J^\circ}{k_m^\circ} \left[ \frac{k_{3,p}}{c} \left( -\frac{k_m^\circ}{J^\circ} c e_{i_q} - k_{3,p} e_\omega - k_{3,i} e_\omega - c \xi_{3,2} \right) \right. \\
 &\quad \left. + \frac{k_{3,i}}{c} e_\omega - \frac{\alpha_{3,2}}{c} [e_{3,1}]^0 + \left( 2 \frac{k_w^\circ}{J^\circ} \omega - \frac{f^\circ}{J^\circ} \right) \left( -\frac{k_m^\circ}{J^\circ} i_q - \frac{f^\circ}{J^\circ} \omega + \frac{k_w^\circ}{J^\circ} \omega^2 \right) \right] \\
 D_{i_{d,\text{ref}}} &= \frac{J^\circ}{k_m^\circ} \left( \frac{k_{3,p}}{c} c + 2 \frac{k_w^\circ}{J^\circ} \omega - \frac{f^\circ}{J^\circ} \right) \Delta_3
 \end{aligned} \tag{8}$$

ensures that the tracking error  $\omega - \omega_{\text{ref}}$ , converges to zero asymptotically. Moreover, the errors  $i_d - i_{d,\text{ref}}$ ,  $i_q - i_{q,\text{ref}}$  converge asymptotically to zero.  $\diamond$

**Proof.** Let us define the angular velocity tracking error  $e_\omega = \omega - \omega_{\text{ref}} = c\omega$ . The dynamics of  $e_\omega$  are

$$\begin{aligned}\dot{e}_\omega &= c\dot{\omega} = c\left(-\frac{k_m^\circ}{J^\circ}i_q - \frac{f^\circ}{J^\circ}\omega + \frac{k_w^\circ}{J^\circ}\omega^2 + \Delta_3\right) \\ &= c\left(-\frac{k_m^\circ}{J^\circ}e_{i_q} - \frac{k_m^\circ}{J^\circ}i_{q,\text{ref}} - \frac{f^\circ}{J^\circ}\omega + \frac{k_w^\circ}{J^\circ}\omega^2 + \Delta_3\right) \\ &= -\frac{k_m^\circ}{J^\circ}ce_{i_q} - k_{3,p}e_\omega - k_{3,i}I_{e_\omega} - e_{3,2}\end{aligned}\quad (9)$$

where  $e_{i_q} = i_q - i_{q,\text{ref}}$ , with  $i_{q,\text{ref}}$  as in (7), and  $e_{3,2} = c(\xi_{3,2} - \Delta_3)$ . The dynamics of  $e_{i_q}$  are

$$\begin{aligned}\dot{e}_{i_q} &= -\frac{R_s}{L}i_q - p\omega i_d - p\frac{\phi_m}{L}\omega + \frac{1}{L}v_q - \frac{di_{q,\text{ref}}}{dt} \\ &= -\frac{R_s^\circ}{L^\circ}i_q - p\omega i_d - p\frac{\phi_m^\circ}{L^\circ}\omega + \frac{1}{L^\circ}v_q + \delta_2 - D_{i_{q,\text{ref}}}^\circ - D_{i_{q,\text{ref}}} \\ &= -\frac{R_s^\circ}{L^\circ}i_q - p\omega i_d - p\frac{\phi_m^\circ}{L^\circ}\omega + \frac{1}{L^\circ}v_q - D_{i_{q,\text{ref}}}^\circ + \Delta_2\end{aligned}$$

with the derivative of  $i_{q,\text{ref}}$  given by

$$\frac{d}{dt}i_{q,\text{ref}} = \frac{J^\circ}{k_m^\circ}\left(\frac{k_{3,p}}{c}\dot{e}_\omega + \frac{k_{3,i}}{c}e_\omega + \xi_{3,2} + \left(2\frac{k_w^\circ}{J^\circ}\omega - \frac{f^\circ}{J^\circ}\right)\dot{\omega}\right) = D_{i_{q,\text{ref}}}^\circ + D_{i_{q,\text{ref}}}$$

$D_{i_{q,\text{ref}}}^\circ, D_{i_{q,\text{ref}}}$  as in (8), and with the (unknown) perturbation term  $\Delta_2$  given by

$$\begin{aligned}\Delta_2 &= \delta_2 - D_{i_{q,\text{ref}}} = -\left(\frac{R_s}{L} - \frac{R_s^\circ}{L^\circ}\right)i_q - p\left(\frac{\phi_m}{L} - \frac{\phi_m^\circ}{L^\circ}\right)\omega + \left(\frac{1}{L} - \frac{1}{L^\circ}\right)v_q \\ &\quad + \frac{J^\circ}{k_m^\circ}\left(\frac{k_{3,p}}{c} + 2\frac{k_w^\circ}{J^\circ}\omega - \frac{f^\circ}{J^\circ}\right)\left[\left(\frac{k_m}{J} - \frac{k_m^\circ}{J^\circ}\right)i_q + \left(\frac{f}{J} - \frac{f^\circ}{J^\circ}\right)\omega - \left(\frac{k_w}{J} - \frac{k_w^\circ}{J^\circ}\right)\omega^2\right].\end{aligned}$$

Using the controller (5), one works out

$$\dot{e}_{i_q} = -k_{2,p}e_{i_q} - k_{2,i}I_{e_{i_d}} - e_{2,2}\quad (10)$$

with  $e_{2,2} = \xi_{2,2} - \Delta_2$ . Considering (9), (10), one finally gets the coupled dynamics

$$\begin{aligned}\begin{pmatrix} \dot{I}_{e_\omega} \\ \dot{e}_\omega \end{pmatrix} &= \begin{pmatrix} 0 & 1 \\ -k_{3,i} & -k_{3,p} \end{pmatrix} \begin{pmatrix} I_{e_\omega} \\ e_\omega \end{pmatrix} - \begin{pmatrix} 0 & 0 \\ 0 & \frac{k_m^\circ}{J^\circ}c \end{pmatrix} \begin{pmatrix} I_{e_{i_q}} \\ e_{i_q} \end{pmatrix} - \begin{pmatrix} 0 & 0 \\ 0 & 1 \end{pmatrix} \begin{pmatrix} e_{3,1} \\ e_{3,2} \end{pmatrix} \\ \begin{pmatrix} \dot{I}_{e_{i_q}} \\ \dot{e}_{i_q} \end{pmatrix} &= \begin{pmatrix} 0 & 1 \\ -k_{2,i} & -k_{2,p} \end{pmatrix} \begin{pmatrix} I_{e_{i_q}} \\ e_{i_q} \end{pmatrix} - \begin{pmatrix} 0 & 0 \\ 0 & 1 \end{pmatrix} \begin{pmatrix} e_{2,1} \\ e_{2,2} \end{pmatrix}\end{aligned}\quad (11)$$

where the dynamics of the errors  $e_{3,1}, e_{3,2}, e_{2,1}, e_{2,2}$ , are

$$\begin{aligned}\dot{e}_{3,1} &= -\alpha_{3,1} [e_{3,1}]^{1/2} + e_{3,2} \\ \dot{e}_{3,2} &= -\alpha_{3,2} [e_{3,1}]^0 - c\dot{\Delta}_3 \\ \dot{e}_{2,1} &= -\alpha_{2,1} [e_{2,1}]^{1/2} + e_{2,2} \\ \dot{e}_{2,2} &= -\alpha_{2,2} [e_{2,1}]^0 - \dot{\Delta}_2.\end{aligned}\quad (12)$$



The errors  $e_{3,1}$ ,  $e_{3,2}$  tend to zero in finite time. The proof of the global convergence to the origin in finite time of  $e_{3,1}$ ,  $e_{3,2}$  can be demonstrated with Lyapunov arguments [39,40], considering the Lyapunov candidate

$$V_3 = \frac{1}{2} E_3^T P_3 E_3, \quad P_3 = \begin{pmatrix} \alpha_{3,1}^2 + 4\alpha_{3,2} & -\alpha_{3,1} \\ -\alpha_{3,1} & 2 \end{pmatrix} = P_3^T > 0, \quad E_3 = \begin{pmatrix} |e_{3,1}|^{1/2} \\ e_{3,2} \end{pmatrix}.$$

This candidate is continuous but not differentiable. As explained in [39], the use of the classical version of Lyapunov's theory [41], instead of nonsmooth versions [42,43], is allowed since  $V_3$  is differentiable almost everywhere, i.e., except when  $e_{3,1} = 0$ . The fact that the trajectories of (12) may pass through  $e_{3,1} = 0$  for time instants constituting a set of null measure before reaching the origin from a certain finite-time instant on allows using the classical version of Lyapunov's theory. Therefore, considering the necessary modifications with respect to the proof given in [39]

$$\dot{V}_3 = -\frac{\alpha_{3,1}}{2|e_{3,1}|^{1/2}} E_3^T Q_{3,1} E_3 - \frac{c\dot{\Delta}_3}{2|e_{3,1}|^{1/2}} |e_{3,1}|^0 E_3^T Q_{3,0} E_3$$

where

$$Q_{3,1} = \frac{P_3 \Lambda_{3,1} + \Lambda_{3,1}^T P_3}{2\alpha_{3,1}} = \begin{pmatrix} \alpha_{3,1}^2 + 2\alpha_{3,2} & -\alpha_{3,1} \\ -\alpha_{3,1} & 1 \end{pmatrix}$$

$$Q_{3,0} = P_3 \Lambda_{3,0} = \begin{pmatrix} -2\alpha_{3,1} & 2 \\ 2 & 0 \end{pmatrix}.$$

We will proof later that, with the proposed controller,  $\dot{\Delta}_3$  remains bounded, with  $|\dot{\Delta}_3| \leq \theta_3$  for all time instants. To take into account the perturbation  $\dot{\Delta}_3$ ,  $\dot{V}_3$  can be bounded as follows

$$\begin{aligned} \dot{V}_3 &\leq -\frac{\alpha_{3,1}}{2|e_{3,1}|^{1/2}} \mathcal{E}_3^T Q_{3,1} \mathcal{E}_3 - \frac{c\theta_3}{2|e_{3,1}|^{1/2}} \mathcal{E}_3^T Q_{3,0} \mathcal{E}_3 = -\frac{\alpha_{3,1}}{2|e_{3,1}|^{1/2}} \mathcal{E}_3^T Q_3 \mathcal{E}_3 \\ &\leq -\frac{\alpha_{3,1}}{2|e_{3,1}|^{1/2}} \lambda_{\min}^{Q_3} \|E_3\|_2^2 \leq -\gamma_3 V_3^{1/2}, \quad \gamma_3 = \alpha_{3,1} \frac{\sqrt{\lambda_{\min}^{P_3}}}{\sqrt{2} \lambda_{\max}^{P_3}} \lambda_{\min}^{Q_3} \\ \mathcal{E}_3 &= \begin{pmatrix} |e_{3,1}|^{1/2} \\ |e_{3,2}| \end{pmatrix}, \quad Q_3 = \begin{pmatrix} \alpha_{3,1}^2 + 2\alpha_{3,2} - 2c\theta_3 & -\left(\alpha_{3,1} + \frac{2c\theta_3}{\alpha_{3,1}}\right) \\ -\left(\alpha_{3,1} + \frac{2c\theta_3}{\alpha_{3,1}}\right) & 1 \end{pmatrix} \end{aligned}$$

with  $\alpha_{3,1} > 0$ ,  $Q_3 > 0$  under the conditions (6), and  $\|\mathcal{E}_3\|_2^2 = \|E_3\|_2^2 = |e_{3,1}|^{1/2}|^2 + |e_{3,2}|^2 = |e_{3,1}| + e_{3,2}^2$ . Also here, slight modifications with respect to the proof of [39] were necessary. This proves the finite-time convergence of  $E_3$  to the origin.

With the same arguments, also the errors  $e_{2,1}$ ,  $e_{2,2}$  can be proved to tend to zero in finite time, considering the Lyapunov candidate

$$V_2 = \frac{1}{2} E_2^T P_2 E_2, \quad P_2 = \begin{pmatrix} \alpha_{2,1}^2 + 4\alpha_{2,2} & -\alpha_{2,1} \\ -\alpha_{2,1} & 2 \end{pmatrix}, \quad E_2 = \begin{pmatrix} |e_{2,1}|^{1/2} \\ e_{2,2} \end{pmatrix}. \quad (13)$$

Also in this case it will be shown that  $\dot{\Delta}_2$  remains bounded, with  $|\dot{\Delta}_2| \leq \theta_2$ .

Since  $e_{2,1}$ ,  $e_{2,2}$ ,  $e_{3,1}$ ,  $e_{3,2}$  tend to zero in finite time, the tracking error dynamics (11) has the origin globally asymptotically stable, namely  $e_\omega$ ,  $I_{e_\omega}$ ,  $e_{i_q}$ ,  $I_{e_{i_q}}$  tend to zero asymptotically.

Finally, considering the expression of the reference  $i_{d,\text{ref}}$  as in (7) and the control  $v_d$  as in (5), for the error  $e_{i_d} = i_d - i_{d,\text{ref}}$  one obtains

$$\begin{aligned}\dot{e}_{i_d} &= -\frac{R_s}{L}i_d + p\omega i_q + \frac{1}{L}v_d - \frac{d}{dt}i_{d,\text{ref}} \\ &= -\frac{R_s^\circ}{L^\circ}i_d + p\omega i_q + \frac{1}{L^\circ}v_d - \frac{d}{dt}i_{d,\text{ref}} + \Delta_1 \\ &= -k_{1,p}e_{i_d} - k_{1,i}I_{e_{i_d}} - e_{1,2}\end{aligned}\quad (14)$$

where

$$\begin{aligned}\dot{e}_{1,1} &= -\alpha_{1,1} [e_{1,1}]^{1/2} + e_{1,2} \\ \dot{e}_{1,2} &= -\alpha_{1,2} [e_{1,1}]^0 - \dot{\Delta}_1.\end{aligned}\quad (15)$$

With the same arguments used before,  $e_{1,1}$ ,  $e_{1,2}$  tend to zero in finite time, since  $|\dot{\Delta}_1| \leq \theta_1$ . As a consequence, the closed-loop dynamics

$$\begin{pmatrix} \dot{I}_{e_{i_d}} \\ \dot{e}_{i_d} \end{pmatrix} = \begin{pmatrix} 0 & 1 \\ -k_{1,i} & -k_{1,p} \end{pmatrix} \begin{pmatrix} I_{e_{i_d}} \\ e_{i_d} \end{pmatrix} - \begin{pmatrix} 0 & 0 \\ 0 & 1 \end{pmatrix} \begin{pmatrix} e_{1,1} \\ e_{1,2} \end{pmatrix}$$

has the origin globally asymptotically stable, namely  $e_{i_d}$ ,  $I_{e_{i_d}}$  tend to zero asymptotically.

We can now show what stated before, namely, that  $\dot{\Delta}_1, \dot{\Delta}_2, \dot{\Delta}_3$  remain bounded for all time instants. In fact, since the initial tracking errors  $e_{i_d}(0)$ ,  $e_{i_q}(0)$ ,  $e_\omega(0)$  are finite, so are

$$\begin{aligned}\Delta_1(0) &= -\left(\frac{R_s}{L} - \frac{R_s^\circ}{L^\circ}\right)i_d(0) + \left(\frac{1}{L} - \frac{1}{L^\circ}\right)v_d(0) \\ \Delta_2(0) &= -\left(\frac{R_s}{L} - \frac{R_s^\circ}{L^\circ}\right)i_q(0) - p\left(\frac{\phi_m}{L} - \frac{\phi_m^\circ}{L^\circ}\right)\omega + \left(\frac{1}{L} - \frac{1}{L^\circ}\right)v_q(0) - D_{i_{q,\text{ref}}}(0) \\ \Delta_3(0) &= -\left(\frac{k_m}{J} - \frac{k_m^\circ}{J^\circ}\right)i_q(0) - \left(\frac{f}{J} - \frac{f^\circ}{J^\circ}\right)\omega(0) + \left(\frac{k_w}{J} - \frac{k_w^\circ}{J^\circ}\right)\omega^2(0)\end{aligned}$$

and

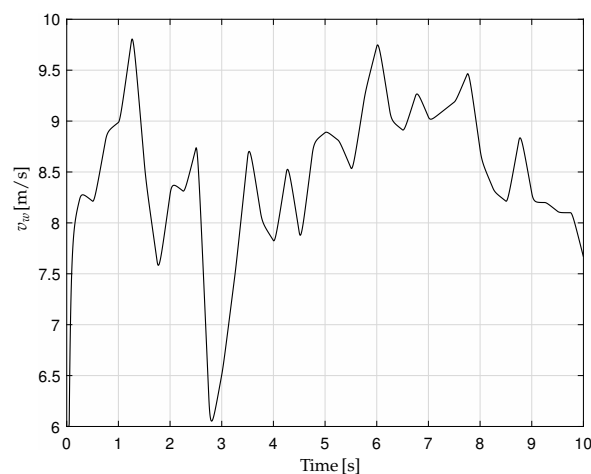
$$\begin{aligned}\dot{\Delta}_1(0) &= -\left(\frac{R_s}{L} - \frac{R_s^\circ}{L^\circ}\right)\frac{di_d}{dt}\Big|_{t=0} + \left(\frac{1}{L} - \frac{1}{L^\circ}\right)\dot{v}_d|_{t=0} \\ \dot{\Delta}_2(0) &= -\left(\frac{R_s}{L} - \frac{R_s^\circ}{L^\circ}\right)\frac{di_q}{dt}\Big|_{t=0} - p\left(\frac{\phi_m}{L} - \frac{\phi_m^\circ}{L^\circ}\right)\dot{\omega}|_{t=0} + \left(\frac{1}{L} - \frac{1}{L^\circ}\right)\dot{v}_q|_{t=0} - \dot{D}_{i_{q,\text{ref}}}|_{t=0} \\ \dot{\Delta}_3(0) &= -\left(\frac{k_m}{J} - \frac{k_m^\circ}{J^\circ}\right)\frac{di_q}{dt}\Big|_{t=0} - \left(\frac{f}{J} - \frac{f^\circ}{J^\circ}\right)\dot{\omega}|_{t=0} + \left(\frac{k_w}{J} - \frac{k_w^\circ}{J^\circ}\right)2\omega(0)\dot{\omega}|_{t=0}.\end{aligned}$$

and the right sides of the equations (4) for  $t = 0$ . Since the functions involved are also locally Lipschitz, this remains true for finite time intervals  $[0, \delta)$ ,  $\delta > 0$ . This implies that the dynamics (12), (15) converge to zero for  $t \in [0, \delta)$ . This reasoning can be iterated, so that the local Lipschitzianity implies that  $\dot{\Delta}_1, \dot{\Delta}_2, \dot{\Delta}_3$  remain bounded for all time instants.

This concludes the proof.  $\square$

#### 4. Simulation results

The proposed dynamic controller (5) has been tested by simulation using the mathematical model (1). The simulation of the wind speed profile was considered from realistic measurements [44], shown in Figure 1. The main parameters of the considered WT are shown in Table 1. Variations of some of these mechanical and electrical parameters have been considered, as indicated in Table 2.



**Figure 1.** Wind velocity profile  $v_w$ .

**Table 1.** WT nominal values.

$R^\circ$	Rotor radius	46.6 m
$R_s^\circ$	Winding resistance	$0.821 \Omega$
$L^\circ$	Windings inductance	$1.5731 \times 10^{-3} \text{ H}$
$\phi_m^\circ$	Flux linkage	5.8264 Wb
$p$	Poles number	26
$J^\circ$	Mechanical inertia	$34.6 \times 10^3 \text{ Kg m}^2$
$f^\circ$	Coefficient of viscous friction	$1.5 \times 10^{-3} \text{ Kg m}^2/\text{s}$
$\rho^\circ$	Normal air density	1.225

**Table 2.** WT real values.

$R = R^\circ$	$R_s = 1.2 R_s^\circ$	$L = 0.95 L^\circ$
$\phi_m = 0.98 \phi_m^\circ$	$J = 1.05 J^\circ \text{ Kg m}^2$	$f = 0.8 f^\circ \text{ Kg m}^2/\text{s}$

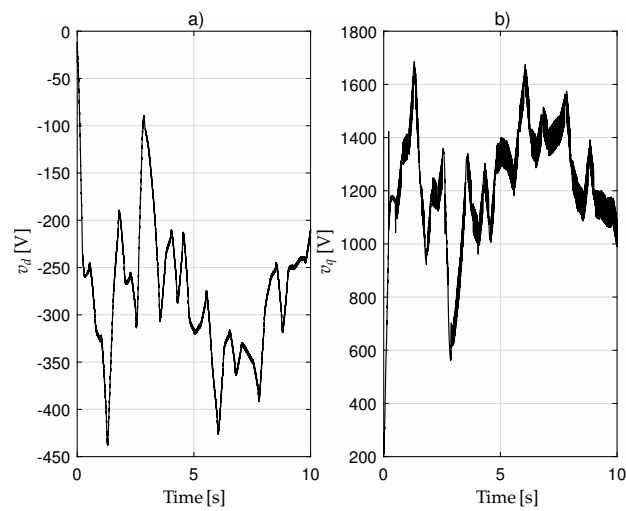
Some of the performances achieved by the proposed controller (5) using the gains given in Table 3 have been reported in Figures 2–4 with the initial conditions  $i_{d,0} = 0.5 \text{ A}$ ,  $i_{q,0} = 18 \text{ A}$ ,  $\omega_0 = 10.47 \text{ rad/s}$ . Figure 2 show the stator voltages  $v_d$  and  $v_q$ , while Figure 3 shows the behavior of the  $d$ -axis stator current  $i_d$  and its reference  $i_{d,\text{ref}} = \varphi(t) = 0$ , the  $q$ -axis stator current  $i_q$  and its reference  $i_{q,\text{ref}}$ , and the tracking error  $e_{i_q} = i_q - i_{q,\text{ref}}$ . Finally, Figure 4 shows the angular velocity  $\omega$  and its reference  $\omega_{\text{ref}}$ , along with the tracking error  $e_\omega = \omega - \omega_{\text{ref}}$ . The parameter estimations, shown in Figures 5 and 6, have been determined considering the gains reported in Table 4. The reported simulation results demonstrate the accurate tracking achieved by the proposed controller. These performances ensure the maximization of the power extracted by the WT from the wind.

**Table 3.** Controller gains.

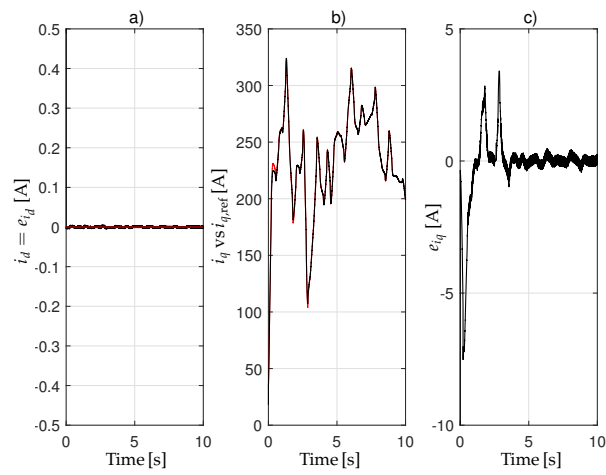
$\gamma_{1,1} = 1380$	$\gamma_{2,1} = 1820$	$\gamma_{3,1} = 26$
$\gamma_{1,2} = 1320$	$\gamma_{2,2} = 1790$	$\gamma_{3,2} = 23$

**Table 4.** HOSM estimator gains.

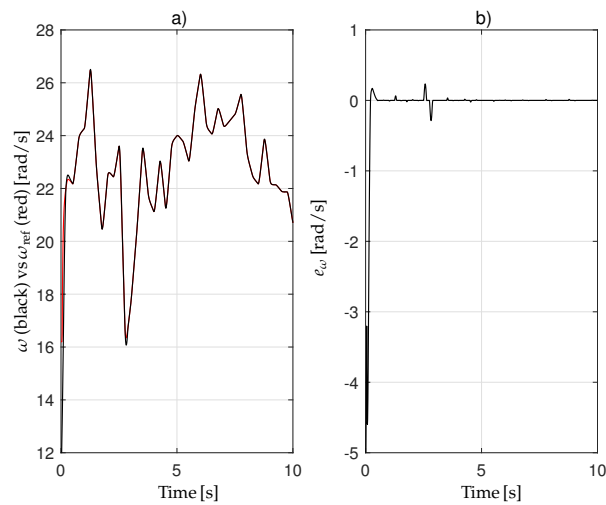
$\alpha_{1,1} = 1200$	$\alpha_{2,1} = 5650$	$\alpha_{3,1} = 380$
$\alpha_{1,2} = 1180$	$\alpha_{2,2} = 5600$	$\alpha_{3,2} = 320$



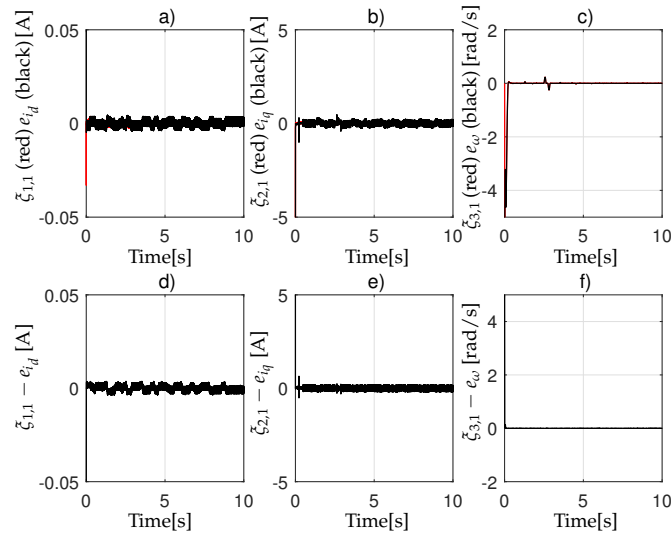
**Figure 2.** a) Stator voltage  $v_d$ ; b) Stator voltage  $v_q$ .



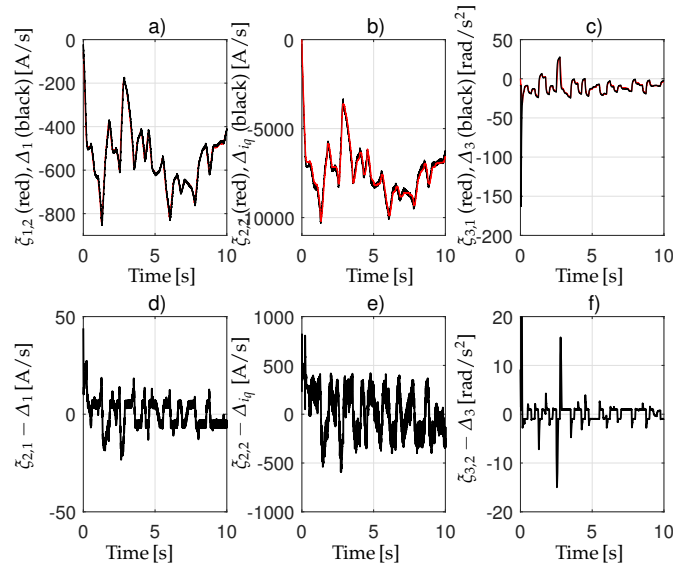
**Figure 3.** a) Stator current  $i_d$  and its reference  $i_{d,ref} = 0$ ; b) Stator current  $i_q$  (black) and reference  $i_{q,ref}$  (red); c) Tracking error  $e_{i_q}$ .



**Figure 4.** a) Angular velocity  $\omega$  (black) and reference  $\omega_{ref}$  (red); b) Tracking error  $e_\omega$ .



**Figure 5.** Error estimations. a)  $\zeta_{1,1}$  (red) and tracking error  $e_{i_d}$  (black); b)  $\zeta_{2,1}$  (red) and tracking error  $e_{i_q}$  (black); c)  $\zeta_{3,1}$  (red) and tracking error  $e_\omega$  (black); d)  $\zeta_{1,1} - e_{i_d}$ ; e)  $\zeta_{2,1} - e_{i_q}$ ; f)  $\zeta_{3,1} - e_\omega$ .



**Figure 6.** Velocity error estimations. a)  $\zeta_{1,2}$  (red) and  $\Delta_1$  (black); b)  $\zeta_{2,2}$  (red) and  $\Delta_{i_q}$  (black); c)  $\zeta_{3,1}$  (red) and  $\Delta_3$  (black); d)  $\zeta_{2,1} - \Delta_1$ ; e)  $\zeta_{2,2} - \Delta_{i_q}$ ; f)  $\zeta_{3,2} - \Delta_3$ .

## 5. Conclusions

In this work, a robust nonlinear dynamic controller has been designed for a WT with a PMSG. This controller ensures the maximization of the power extracted by the WT from the wind. To design the controller, a tracking problem was solved, ensuring the accurate tracking of an angular velocity reference for the WT blades. The wind velocity was assumed to be unavailable for direct measurement, and an estimator of the wind velocity was employed. Furthermore, a High-Order Sliding Mode (HOSM) parameter estimator was utilized to compensate for the unknown perturbation terms, resulting in high-performance control of the WT. The simulations demonstrate the good performance of the controller.

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