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Article

Mesoscale Static and Fatigue Analysis of 2D Woven Roving Plates with Convex Holes and Subjected to Axial Tension

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Abstract: In the present paper the static and fatigue analysis of plates with holes is conducted. Structures are made of 2D woven roving composites. The parametrization of convex holes is proposed. Experimental results of specimens without holes and with different shapes of notches under static and fatigue tensile loads are presented. The fatigue behaviour of structures is analysed with the use of the low cycle fatigue method. The attention is focused on the nonlinear deformations of 2D woven roving structures leading to the scatter of results. The analysis is supplemented by the numerical finite element modeling. The agreement between experimental and finite element modeling is very good. Since we do not intend to repeat in the abstract the review of the existing literature I would like to emphasize only that the present work is a continuation and extension of the results discussed in the literature – see the next sections of the paper. The finite element modeling of plates with holes is presented in details by Muc [1], Muc, Ulatowska [2] and Muc, Romanowicz [3] including also the problems of the accuracy. The mesh correctness and accuracy is discussed in Ref [2] where the results are compared to the analytical results illustrated in the Soviet Union literature. The experimental results and the form of specimens are shown by Pandita et al. [4] and they are fundamentals for our present experimental investigations. Two parametrical description of static and fatigue behaviour of plates with holes characterizes the goal and the novelty of the paper. The analysis demonstrate how those parameters affect on the variations of static and fatigue failure loads. In addition, it should be emphasized that the fuzzy approach is employed herein to reduce the number of experimental data and to avoid statistical analysis of experimental results – see e.g., Muc [5]. In the mesoscale modeling and using the fuzzy approach it is possible to achieve a very good description of static and fatigue behavior of plates with convex holes subjected to tension.

Keywords: 2D woven roving composites; Notch; Stress concentrations; Low Fatigue Cycles; Nonlinear behaviour; Crack initiation; Fuzzy degradation of properties; Experimental results; Numerical modeling

1. Introduction

Nowadays, woven roving composites are very significant class of high performance composites materials due to various aspects dealing with the improved damage tolerance in the comparison of unidirectional composites (UD). The simplicity of their use in the closed mold process and of the possible applications in various, complicated engineering structures (e.g., aerospace, building, mechanical problems) prefer their use in different industrial problems. However, the stiffness and the strength of the yarn woven fabric composites are lower than those for UD composites. Therefore it is necessary to evaluate static and fatigue mechanical properties of woven roving composites to define and describe the sense of their possible applications – e.g., wind turbine blades, aerospace, automotive. Referring to the variety of manufacturing processes it is necessary to distinguish the various mechanical behaviour of 2-D, 2.5-D, and 3D woven roving composites.

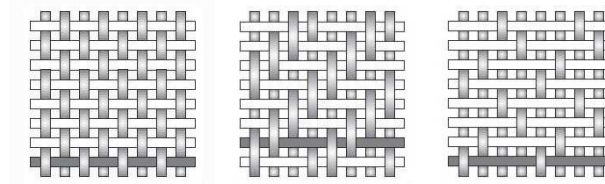
One of the possible classifications of composite materials refers to the form of the reinforcement – particulate reinforcements, whisker reinforcements, nanostructural reinforcements (nanoplatelets, nanoribbons, various forms of graphens – hexagonal nanostructures), continuous fiber laminated composites, and 2-D, 2.5-D or 3-D woven composites (braided or knitted fiber architecture).

To obtain more comprehensive information on woven roving composites the reader should study the following classical works dealing with Textile Structural Components : Bogdanovich, Pastore [6], Naik [7], Tong et al. [8]; books/monographs : Design and Manufacture [9], Composite Forming Technologies [10], Composite Reinforcement for Optimum Design – Boisse (edt) [11], Non-crimp Fabric Composites – Lomov (edt) [12], Fiber Architectures for Composite Applications – [13], online encyclopedias (Blokley, Shyy (Edts) [14]), (H. Altenbach, A. Ochsner (Edts) [15]), Handbook of Epoxy/FiberComposites [16], and proceedings of Textile Composites conferences.

1.1. A brief description of 2-D woven roving composites

The category of fiber architecture is that formed by 2-D, 2.5-D or 3-D weaving, braiding, or knitting the fiber bundles or “tows” to create interlocking fibers that often have orientations slightly or fully in an orientation orthogonal to the primary structural plane. This approach is taken for a variety of reasons, including the ability to have structural, thermal, or electrical properties in the third or “out-of-plane” dimension. Another often-cited reason for using these architectures is that the “unwetted” or dry fiber preforms (fibers before any matrix is added) are easier to handle, lower in cost, and conform to highly curved shapes more readily than the highly aligned, continuous fiber form.

Three different types of 2-D woven roving composites are distinguished – Figure 1.



a) plain weave b) twill weave c) satin weave

Figure 1. Types of 2-D woven roving composites.

Woven fiber-reinforce plastics are becoming increasingly important because they have the following advantages over laminates made from individual layers of unidirectional material:

- Their cost efficiency
- High processability, particularly, in lay-up manufacturing of large scale structures due to Improved formability and drape
- Bidirectional reinforcement in a single layer
- Improved impact resistance
- Balanced properties in the fabric plane

The woven composite is formed by interlacing two sets of threads, the warp and the weft, in a wide variety of weaves and balances.

The lay-up attributes of woven preregs are:

- Thicker (therefore fewer) layers and faster lay-up rate
- Much higher curvature conformability and hence lower susceptibility to wrinkling
- Greater material width of 1.25 or 1.7 m (4.1 or 5.6 ft) compared to 0.3 or 0.6 m (1 or 2 ft) for tape prepreg. (Tape prepreg is narrow since it has low conformability, and materials waste is high for wide tape.)
- Lay-up rates are therefore approximately 3 to 5 times higher than for unidirectional tape.
- No requirement to butt strip edges since fabrics are wider than the parts
- Less-precise ply orientation is required since the lay-up is less optimized; lay-up can therefore be faster.

Manufacturing disadvantages of woven preregs are:

- Higher proportion of waste from the wider material
- Higher cost of low-thickness fabric prepreg since the weaving process preceding prepregging is an added cost. Thicker woven prepreg with a fiber areal weight (FAW) of 370 g/m² has become

standard since the weaving cost is around half that of the conventional 285 g/ m² fabric. These thick prepregs reduced stiffness to the resultant components.

1.2. Design, Tooling, and Manufacturing Interaction

A draping simulation is used to generate fiber paths, identify areas of wrinkling and bridging, develop flat patterns, and allow the calculation of local ply orientations and the resulting laminate mechanical properties, such as stiffness, permeability, volume fraction, and thickness. The existing models for draping simulation fall into one of the following two categories: mapping models and mechanical models. Mapping models assume a geometric mechanism to transform an initial unit square of fabric into the corresponding draped shape. Mechanical models use the equilibrium equations that balance the internal forces in the fabric with external applied loads.

The above problems are very significant in manufacturing curved structures made of 2-D woven roving fabrics. For instance considering hemispherical shells the length of the meridian is equal to πR , and the maximal length of the circumference to $2\pi R$, where R denotes the shell radius. The excess of the material along the circumference leads to the wrinkling of the produced curved structures. Those problems are discussed for shallow ellipsoidal domes by Gohari et al. [17], curved structures by Wang et al. [18], Potluri et al. [19], Sharma, Sutcliffe [20], torispherical shells by Galletly, Muc [21] or hemispherical domes by Muc [22]. For all cases discussed above the rectangular form of 2-D woven roving composites (e.g., Figure 1a) changes to rhombs. It leads to the unsymmetric mechanical description of the axisymmetric shell constructions. It should be included in the numerical analysis.

The use of filament winding method can eliminate the appearance of wrinkles along shells circumference both for opened or closed shells but it does not eliminate the unsymmetric deformations of axisymmetric domes.

1.3. Notches/holes in structures made of woven roving composites.

In general for composites, the notch sensitivity and stress concentration around the holes problems can be divided into four categories:

- Strength performance of laminated unidirectional constructions [23–28]
- 2-D braided composite structures [29–32]
- 2.5-D woven roving composites Guo et al. [33], Song et al. [34]
- 3-D woven roving composites [35–39]

The investigations in each of the above mentioned areas tend to introduce the more accurate description of numerical modeling to experimental results. The investigation on 2-D, 2.5-D, and 3-D textile composite structures with various shapes of notches is very rare. Pandita et al. [4] demonstrate experimental and theoretical static strain concentrations results for circular, and vertical, and horizontal ellipsoidal centrally located holes. Guo et al. [33] describe the results for centrally located waist, slit, octagon, square, hexagon and rhombus holes.

As it is pointed by Guo et al. [33] the present studies are mainly directed in the analysis of structures made of 3-D woven roving materials. The identical conclusions can be drawn in the literature concerning with the fatigue reliability.

1.4. The goals of the present investigations

The aim of the present work is to present the comparison of experimental and theoretical (numerical) studies for plates (specimens) made of 2-D woven roving composites. The analysis deals with constructions of centrally located holes and subjected to tensile loading. Since the static experimental studies demonstrate an evident material nonlinear (σ - ϵ) characteristics the fatigue numerical (finite element) description is carried out with the use of the low fatigue cycle (LFC) method. The experimental (static and fatigue) results illustrate essentially the occurrence of the random (probabilistic) nature of both static and fatigue cracks. The classical approach to such problems is based on the statistical (Weibull) distributions. However, in such a description, a lot of

experimental data is required. Therefore, we propose to use herein the limit data (lower and higher bounds) to characterize the LFC behaviour of structures.

In the present paper we intend to discuss similarities and differences for static and fatigue behaviour of plates made of 2D woven roving composites having central holes and subjected to uniaxial tension. The analysis is conducted with the application of experimental results with the finite element computations.

2. Static and Fatigue Analysis of 2D Woven Roving Specimens

Due to manufacturing process textile composites have specific geometrical forms and possess specific periodic geometrical patterns – see Figure 2. The analysis deals with the plain 2D woven fabric. The geometry of the cross-sections of 2D woven fabric have a crucial influence of the derivation of mechanical properties. The homogenized mechanical properties can be determined with the use of experimental tension tests or by the numerical finite element analysis.

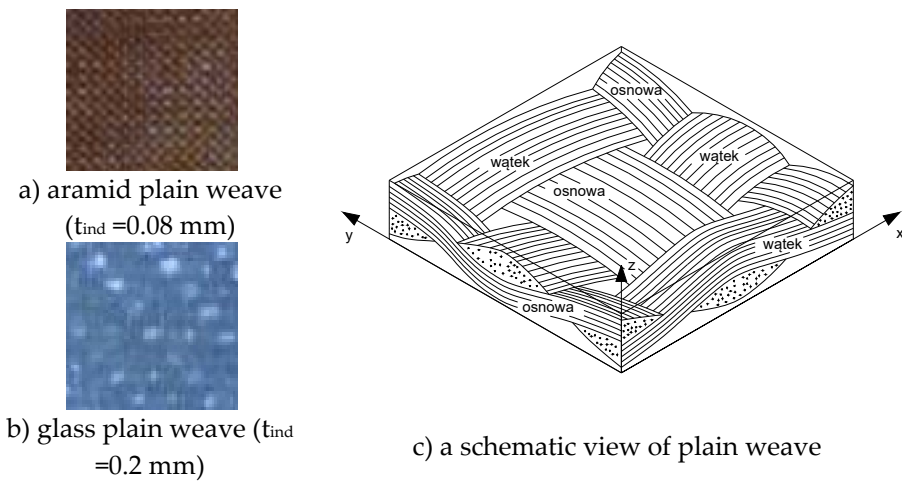


Figure 2. Textile 2D woven fabric (plain weave).

Figures 2a and 2b depict the photographs of 2D woven composites having different thicknesses of bundles corresponding to packing density of fibres.

Using the experimental results plotted in Figure 3 the average Young’s and Kirchhoff’s moduli are written in Table 1- see Ref [40].

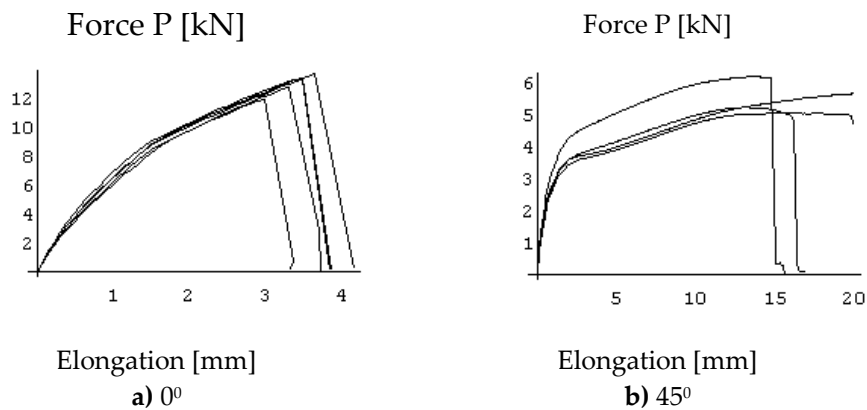


Figure 3. Static strength of rectangular specimens made of woven roving composites (plain 2D glass) for different warp orientations.

To model mechanical properties of 2-D woven roving composites two possible approaches can be used – see e.g., L. Wang et al. [41]:

- Two level modeling where at the first level the fiber bundle (tow) is represented in the microscale and at the second level the representative volume element (RVE) is illustrated and modeled in the mesoscale
- One level modeling (mesoscale) where the mesh of composites (RVE) contains three parts: resin pocket, warp tows and fill tows – each of the part is represented by 3-D (hexahedron) finite elements

The homogenization can be also carried out for the unit cell presented in Figure 4 – see Refs [42,43]. They are dependent on the form of the cross-sections that are demonstrated in Figure 4. The agreement between experimental and numerical results is very good. In addition, let us note that the values of the Young’s moduli for warp and weft directions are almost the same.

Table 1. Static mechanical properties of plain 2D glass.

	Young’s modulus in the warp (longitudinal) direction linear part of the curves plotted in Figure 3a in [GPa]	Young’s modulus in the weft (transverse) direction [GPa]	Kirchhoff’s modulus linear part of the curves plotted in Figure 3b in [GPa]
Experimental	13.142	13.004	9.621
Finite Element modeling	12.958	12.930	9.143

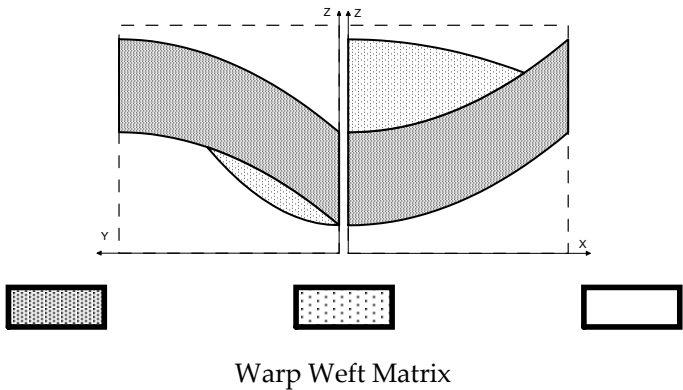


Figure 4. Cross-sections of the unit cell for plain 2D glass composites.

The failure modes of rectangular specimens are presented in Figure 5. The damage occurs in the perpendicular direction to the tension load. Since the thickness of the aramid specimen is very low (Figure 2) its mechanical behaviour (σ - ϵ curve) is almost linear and the failure is characterized by a brittle fracture. For glass woven roving the deformation (Figure 3) is nonlinear and can be described by elastic-plastic behaviour. The failure is not localized along one line (compare Figs 5a and 5b). It is observed that failure forms is associated with delaminations illustrated by white zones.

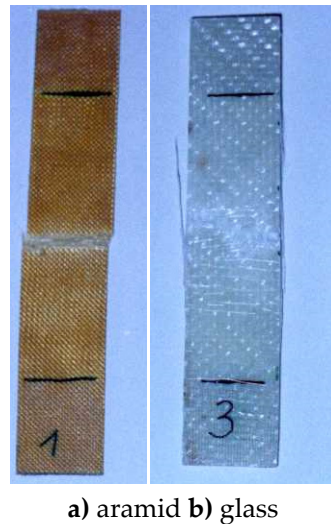


Figure 5. Failure modes of 2D woven roving composites subjected to tension.

It is mainly caused by the non-uniform thickness distributions of glass specimens. It has a significant influence on the load distributions along specimens at the particular points as it is plotted in Figure 6. The broader discussion of those problems is presented by Muc [44]. For woven roving composites the scatter of results is usually observed.

Usually, as it discussed in Ref [45] for 2D woven composites (fabric formed by interlacing the longitudinal yarns (warp) and the transverse yarns (weft)), such as plain, twill or satin), four types of damage mechanisms occur under static and fatigue loadings: intra-yarn cracks in yarns oriented transversely to the loading direction, inter-yarn decohesion between longitudinal and transverse yarns, fiber failure in longitudinal yarns and yarn failures. The failure forms plotted in photographs in Figs. 5 are identical for fatigue problems.

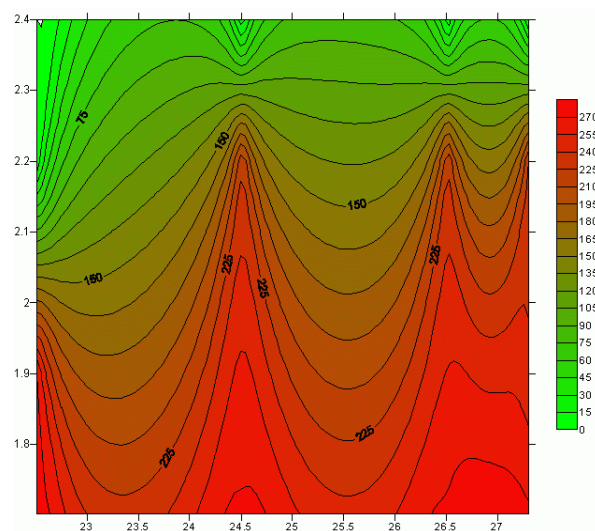


Figure 6. Distributions of the stresses (the vertical axes warp, the horizontal axes weft– the width) – the force 11.25 [kN] – see [5].

3. Stress Concentration around Convex Holes – Static Behaviour.

The existence of holes in woven roving composite constructions leads to the significant stress concentrations around notches. The continuum damage mechanics of unidirectional laminates is illustrated by Muc, Ulatowska in Ref [2]. The static and fatigue description of constructions with holes made of woven roving composites is discussed in Refs [45–51]. The authors compared the static and fatigue behaviour of unnotched and notched composites

3.1. Definition of Convex Holes

A lot of planar convex curves can be expressed in the form of the super-ellipse defined in the following way – see Muc [1]:

$$\left(\frac{x}{a}\right)^n + \left(\frac{y}{b}\right)^n = 1 \quad (1)$$

where a , b and n are positive rational numbers. A supercircle will obviously correspond to setting $a = b$. For each value of n – the supercircular exponent – we obtain a different convex curve. The shapes of the supercircles for different values of n are shown in Figure 7. Evidently, $n = 2$ corresponds to a circle whereas $n = 1$ corresponds to a triangle with its sides rotated by an angle of $\pi/4$. The case $n \rightarrow \infty$ also corresponds to a square with its sides parallel to the axes. Finally, the curve characterizing the convex hole (the supercircle) can be represented in the form of the Fourier series:

The above relation exploits the symmetry with respect to the OX and OY axes.

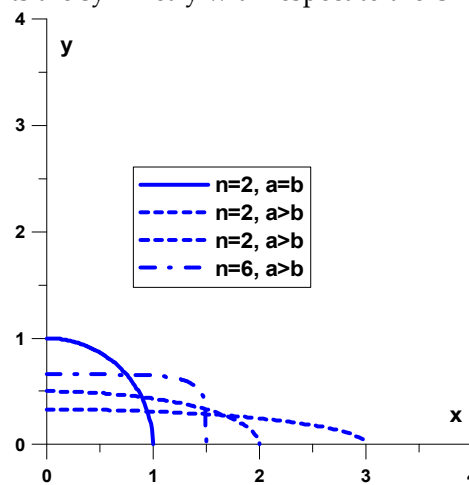


Figure 7. Shape of the supercircle for different values of n (the constant area).

Applying the form of the curve (1) one quarter of the surface closed by the superellipse can be written in the following form:

$$A_C = ab \frac{\Gamma^2(1+1/n)}{\Gamma(1+0.5/n)} \quad (2)$$

where $\Gamma(\dots)$ denotes the Euler gamma function.

3.2. Static failure investigations of specimens with holes

A series of experiments were carried out to study damage of specimens with circular holes. The plates made of 2D woven roving glass (see Table 1 and Figures 2 and 3) were subjected to the uniaxial tension. The geometry of the specimens is presented in Figure 8.

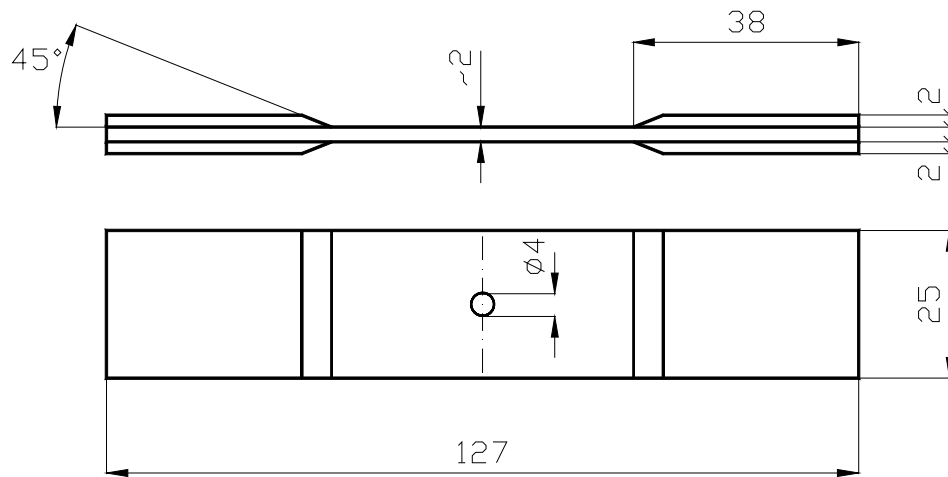


Figure 8. The geometry of the specimens subjected to the uniaxial tension.

Let us note that the ratio: the width to the hole diameter is equal to $25/4 = 6.25$. For such values of the ratio the influence of the hole can be neglected at the end of the width as it demonstrated for unidirectional laminates [3]. For 2D woven roving composites the analogous effects are also observed and are computed below with the aid of the finite element method.

Figure 9 demonstrates the forms of the final damage of the specimens. As it may be seen the failures occur always in the perpendicular direction to the tensile loads similarly as for isotropic materials. However, the failure mode is associated with the inelastic deformations of the matrix – white zones in Figure 9.

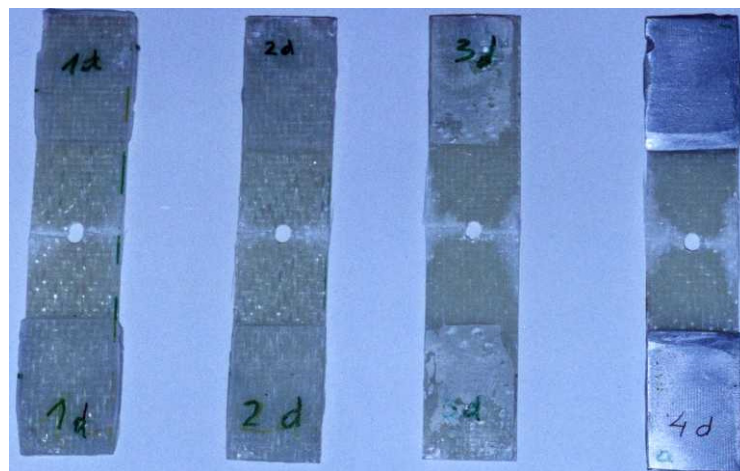
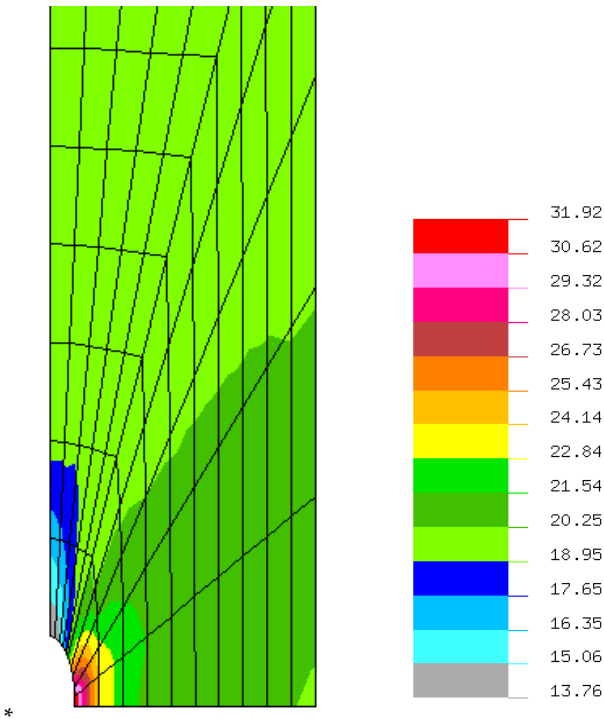


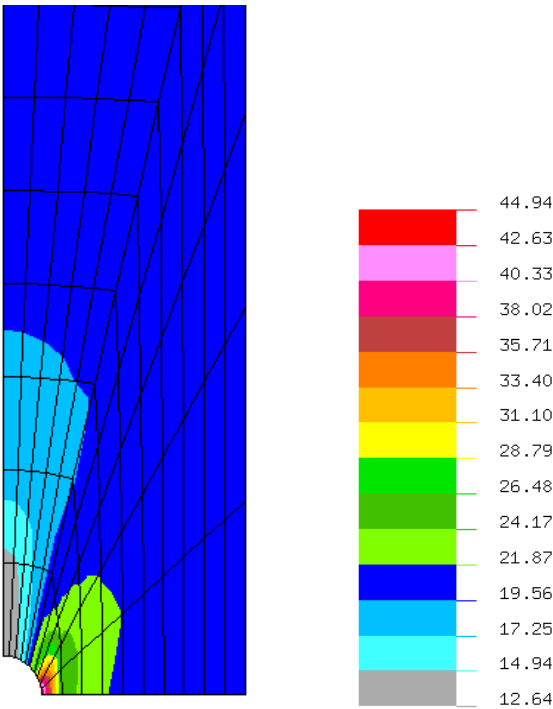
Figure 9. Final failure of specimens subjected to uniaxial tensile load – woven roving glass.

The 2-D solid elements (plane stress elements $NKTP = 1$) in the NISA finite element package are included in the present analysis. The element may be shaped as a 4 to 12 node quadrilateral, or as a 3 or 6 node triangle depending on the selected approach. Each node has two degrees of freedom (two displacements u_x, u_y) and the state of stress is characterized by three components: σ_x , σ_y and σ_{xy} . In addition, the nonuniform thickness distributions at each of the nodes can be introduced. The influence of the meshing on the accuracy of computations will be discussed and illustrated later.

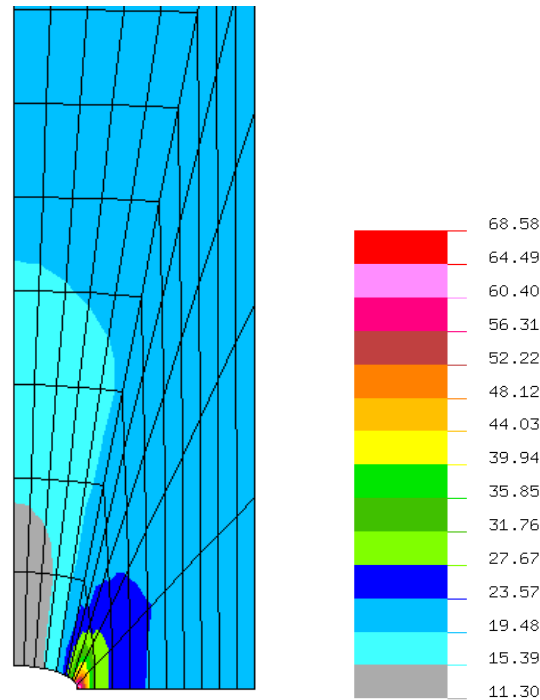
Using the planar finite elements it is possible to evaluate numerically the stress distributions around the holes. Figure 10 illustrates the plots of the static stress concentrations around the convex holes (1). The maximal stress concentrations occurs at the tip of the notch. The stresses are related to the value: $P/(Wt)$.



a) a vertical ellipse



b) a circle



c) a horizontal ellipse

Figure 10. Distributions of dimensionless the Huber-Mises-Hencky stresses for $n=2$ and the constant area A_c .

As it may be noticed in Figure 11 the stress concentrations increase with the growth of the ratio a/b (see Eq (1)). The identical behavior occurs for the higher values of the n parameter (i.e., $n=6$), keeping the same values of the area A_c . The similar effects are obtained for plates with holes made of woven roving composites, isotropic materials or laminated composites as it is discussed by Pandita ,Nishiyabu, Verpoest [4] and Muc, Romanowicz [3]. In general, various stress criteria can be employed in the analysis. We are searching for the maximal values of:

- The circumferential stresses
- The Hill criterion
- *The Huber-Mises-Hencky criterion
- The Tsai-Wu criterion
- The Hashin 3-D or 2-D criterion

The detailed description of the above criteria is presented e. g. in Ref. [52].

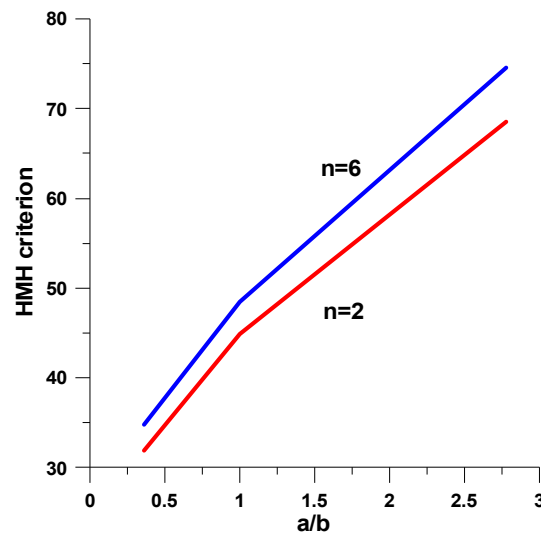


Figure 11. Variations of the stress concentration around convex hole for constant area A_c .

Since in the present analysis and modeling 2-D bidirectional woven roving plain composite materials can be treated as isotropic materials (see Table 1 – the mesoscale analysis) the strain concentration factors can be approximated by the results valid for isotropy. The ratio between the local stresses around the hole and the nominal stresses is characterized by the stress concentration factor k_{tl} , i.e.:

$$\sigma_{tip} = k_{tl} \sigma_{\infty} \quad (3)$$

where the nominal stress $\sigma_{\infty} = P/(Wt)$, and the local stress σ_{tip} corresponds to the value of the σ_y stress component at $x = a$ and $y = 0$ (the edge of the hole). The theoretical stress concentration factor is a function only of geometry and the type of loading, and can be determined either analytically (using the theory of elasticity), through finite element analysis, or through experiments.

For isotropic materials the stress concentration factor of an elliptical notch can be described in the following way:

$$k_{tl} = \left(1 + 2 \frac{b}{a} \right) \quad (4)$$

Considering the values of the coefficients $1+2b/a$ one can notice that the horizontal ellipses correspond to the high concentration effects, whereas the vertical position of ellipses corresponds to the low concentration effects. The comparisons of the stress concentration factors (4) are demonstrated in Table 2.

Table 2. The comparison of the stress concentration effects for the ellipsoidal notches – the initiation of cracks.

Stress Concentration Factor k_{tl}	Theoretical	Numerical Analysis (FE)	Percentage Error $(k_{tl}^{numer} - k_{tl}^{theoret}) / k_{tl}^{theoret}$
b/a=2.812	6.624	6.943	10.24
b/a=1.000	3.000	3.211	12.51
b/a=0.336	1.672	1.745	13.71

The accuracy of the FE approximations is a function of the accuracy of meshing, however, in the present case is quite good. Pandita et al. [4] obtained the unsymmetric behaviour of failure modes. It is difficult to verify such results since it depends on too many factors.

Let us note that for the parameter n greater than 6 the area A_c Eq. (2) can be approximated by the value $ab(1-0.5/n)$. Therefore, the growth of the ratio a/b and the parameter n corresponds to the increase of the stress concentration factors and/or the HMF factor (Table 2 and Figure 11).

4. Fatigue behaviour of notched woven roving 2D composites

Nowadays, the fatigue of structures can be approached using the following concepts – Figure 12:

- Low cycle fatigue (LCF)
- High (Mega) cycle fatigue (HCF) from A. Woehler – both infinite and finite cyclic life can be analyzed, where the small strain increment results in large stress increment

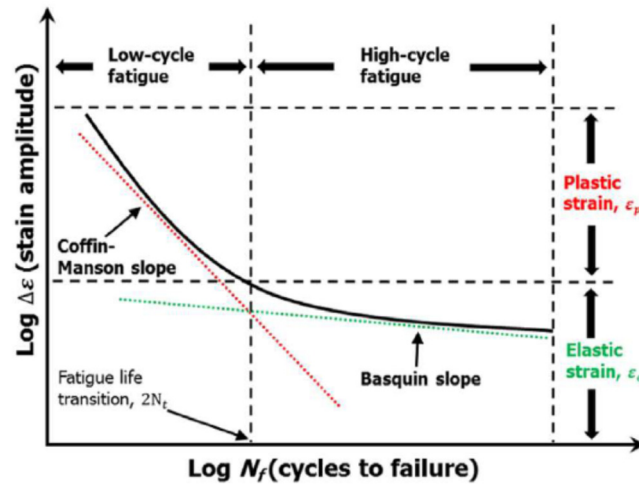


Figure 12. ε - N_f curve – elastic and plastic strain curves.

4.1. Low cycle fatigue (LCF)

In the further part of the paper the LCF problems are investigated.

Low cycle fatigue (LCF) from Coffin [53] and Manson [54] – only finite fatigue life is possible and can be considered using the LCF criteria in such a case a small strain increment corresponds to a large stress increment – Figure 13; according to this approach, the fatigue life of a member subjected to fully reversed constant amplitude strain-controlled loading is given by an equation of the form:

$$\varepsilon = \varepsilon_a + \varepsilon_p = \frac{\sigma'_f}{E} (N_f)^b + \varepsilon'_f (N_f)^c \quad (5)$$

where: ε - total strain amplitude, σ'_f - fatigue strength coefficient, b - fatigue strength exponent, ε'_f - fatigue ductility coefficient, c - fatigue ductility exponent, E - modulus of elasticity, N_f - reversals to failure; the determination of coefficients is presented schematically in Figure 14;

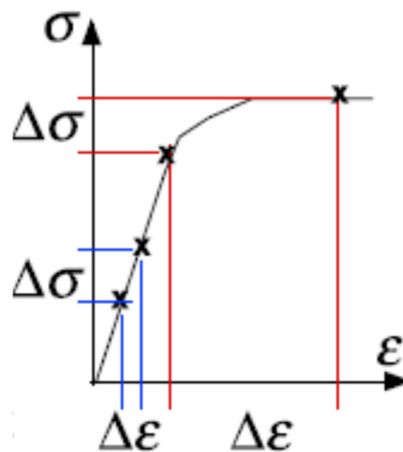


Figure 13. Stress-strain curves for LCF.

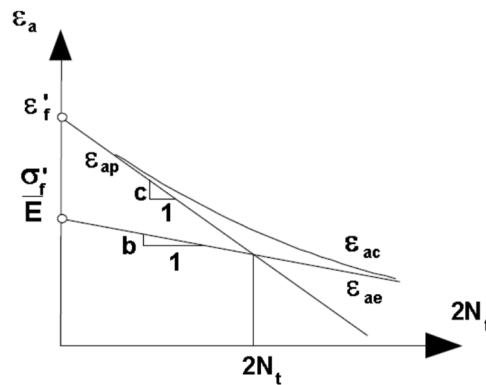


Figure 14. Strain life ε -N curve.

The LCF curve is defined by five coefficients. Manson proposed the method of universal slope where the equation for the method can be given as follows:

$$\Delta\varepsilon / 2_{total} = 1.75(UTS / E)N_f^{-0.12} + D^{0.6}N_f^{-0.6} \quad (6)$$

where UTS denotes the ultimate tensile strength, D is the ductility, i.e., the percentage reduce of the area.

4.2. Experimental Analysis

Since the static stress-strain curves demonstrate the evident nonlinear behaviour (Figure 3a) the fatigue analysis is carried out with the use of the LCF method where the static load carrying capacity is not exceeded. The form of the tensile load is illustrated in Figure 15 where $P_m = 0.8 P_{static}$ and $\Delta P = 0.1 P_m$.

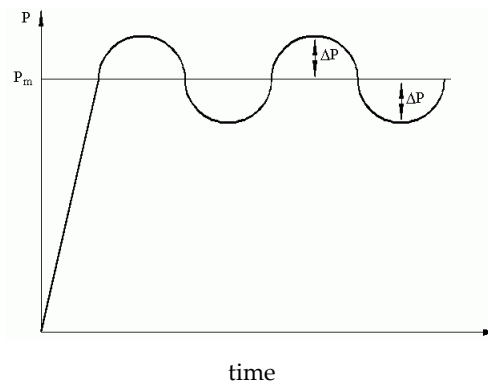


Figure 15. Fatigue tensile load distributions.

The typical fatigue final form is illustrated in Figure 16. It is analogous to fatigue behaviour of isotropic (metallic) structures.

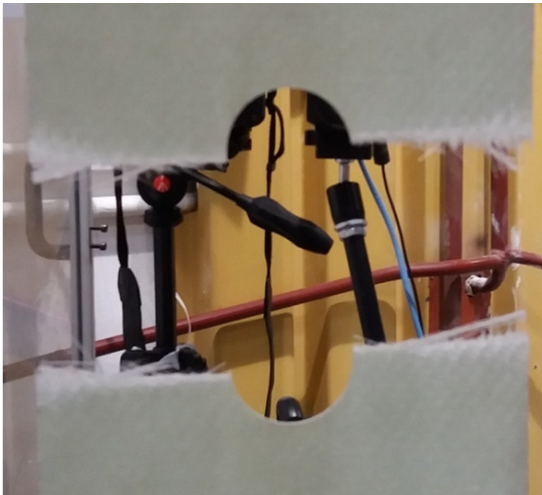


Figure 16. Fatigue failure modes of stretched plates made of woven roving glass/epoxy – circular hole [3].

The number of cycles corresponding to the final fatigue failure are presented in Table 3. They show the scatter of results caused by the nonuniform distributions of thickness for woven roving composites that have also a great influence on distributions of stresses along the specimen length – see Figure 5.

Table 3. The critical number of cycles.

Specimen	1	2	3	4
The length [mm]	125.0	125.0	125.05	125.04
The average thickness [mm]	2.48	2.39	2.43	2.47
Number of cycles N_f	11012	10945	15004	14617
Average number of cycles	12894.5			

The scatter of the critical number of cycles should be represented by the statistical or fuzzy set analysis – see the discussion presented by Muc [5]. Using the fuzzy set analysis the experimental fatigue degradation analysis (the decrease of the secant Young modulus) can be illustrated in the form drawn in Figure 17. The results are represented in the form of straight lines being the approximation (linearization) of the experimental data.

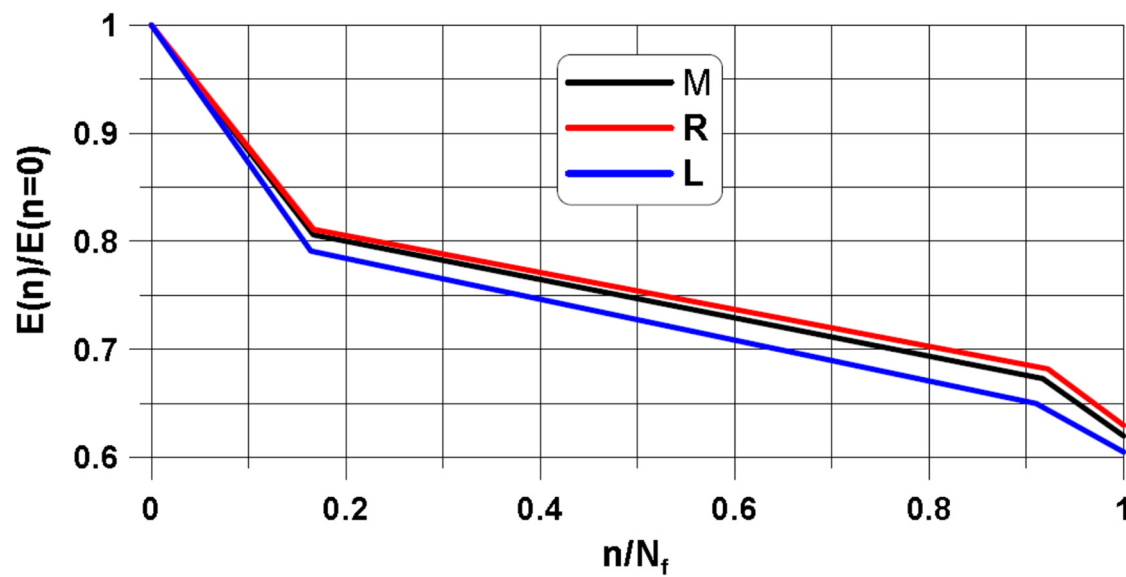


Figure 17. The degradation of the stiffness for 2D woven roving composites (M – the average value, R – the highest value treated as the upper bound, L – the lowest value treated as the lower bound).

In our opinion the results presented in Figure 17 are much more convenient than the classical statistical analysis since it requires less of experimental data.

4.3. Finite Element Analysis

For low cycle fatigue problems the numerical modeling is carried out in two steps:

1. Finite element modeling of structures with convex holes
2. Derivation of final number of cycles using the Coffin, Manson relation (5)

The analysis is conducted with the use of the numerical package NISA II Endure version 17.

The lowest value occurs at the tip of the circular notch (the grey colour – see Figure 18) and it corresponds to the experimental value – Table 2. The crack initiation is always characterized by the lowest value of the number of cycles. The maximal value of N_f exists in the direction of the remote tension – the axis y.

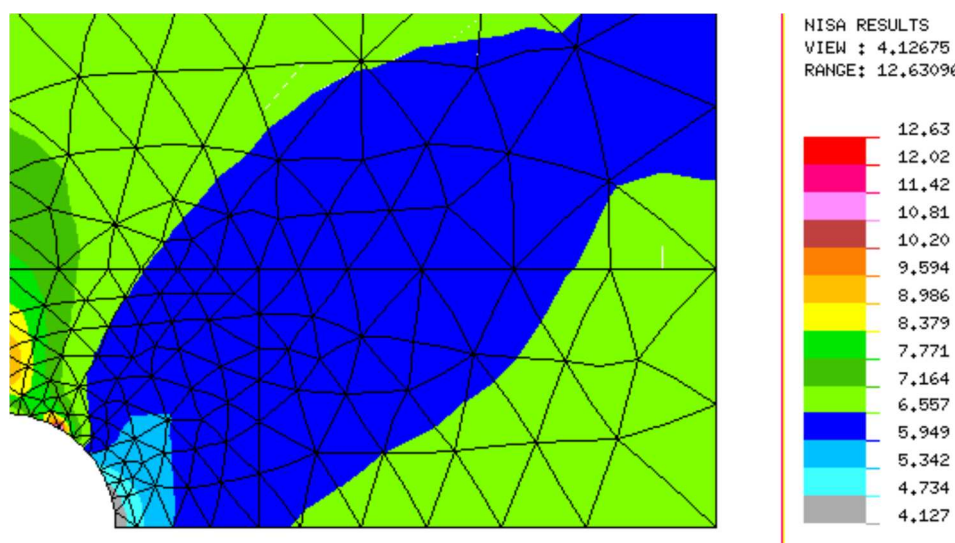


Figure 18. Fatigue crack initiation life contours (the logarithmic scale).

For different shapes of central notches the identical numerical analysis can be repeated. The results are illustrated in Figure 19. As it may be seen the distributions are almost identical to the curves plotted in Figure 11 describing the stress concentration effects for static loads.

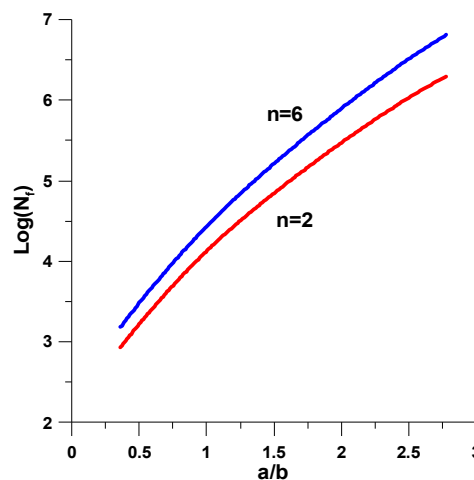


Figure 19. Variations of the critical number of cycles N_f around convex hole for constant area A_c .

Similarly as for the stress concentration factors the increase of the parameters a/b and n results in the increase of the critical number of cycles.

5. Conclusions

The influence of notches in 2D woven fabric composites is investigated in the mesoscale based on the ultimate stress and on fatigue loads. The stress/strain concentration under static and fatigue loads in 2D woven fabric composites with holes is influenced by the tensile loading direction and by the hole geometry and its dimension relative to the unit cell of the plain woven fabrics. For different shape parameters the strain concentration is located at the tip of holes.

The fatigue load behaviour is described by the low fatigue cycle method that is described herein.

The 2D woven roving composites have nonlinear behaviour and it has a significant influence on the final failure of static and fatigue failure. It results in the necessity of the use of statistical and fuzzy methods.

The computational finite element methods are also implemented in the paper. The comparison of experimental and numerical results is very good.

The presented results show the necessity of the use of at least two parameters (convex hole) to compare the static and fatigue behaviour of plates subjected to tension. The growth of the parameters a/b and n leads to the increase of the critical values characterizing the static and the LCS fatigue failure.

6. The novelty of the paper

1. The form of the convex holes can be approximated by two-parametrical description, i.e. the a/b ratio and the parameter n
2. The mesoscale analysis is sufficient for the characterization of static and low fatigue cycles failure damage – the comparison is presented with the use of FE and experimental analysis
3. The scatter of the experimental results is characterized by the fuzzy set analysis

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