

## Article

# The Effect of Negative Mass in Gravitating System

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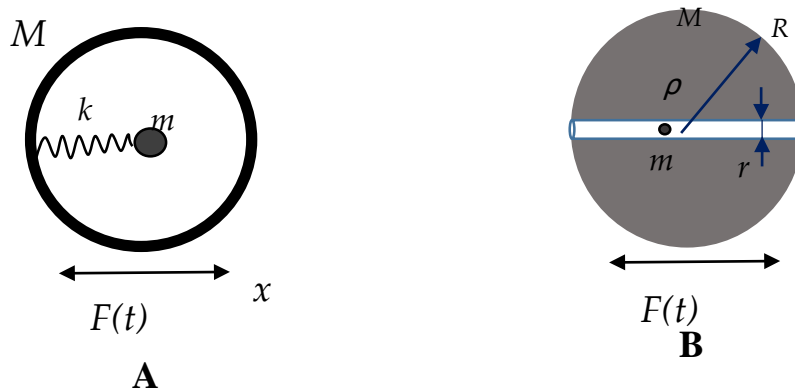
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**Abstract:** The effect of the negative effective mass emerging in the gravitating core-spring-shell system is considered. The effect appears when the entire system is exerted to the external harmonic force and the frequency of external force approaches to the critical frequency  $\omega_0$  from above. The critical frequency  $\omega_0$  depends on the density of the self-gravitating system only. The scaling law predicting the value of  $\omega_0$  for condensed phases is derived as:  $\omega_0 = A^{\frac{1}{2}}\tilde{\omega}$ , where  $A$  is the atomic weight and  $\tilde{\omega}$  is the fundamental frequency. The generalization of the effect for the Coulomb-like forces is reported.

**Keywords:** core-spring-shell system; gravitating system; negative mass; Coulomb-like forces

## 1. Introduction

The broadly discussed effect of the “negative effective” mass emerges when the core-spring-shell system is substituted by the single effective mass  $m_{eff}$  [1]. Pseudo-negative mass introduced in Ref. 1 is quite different from the hypothetic “negative mass” suggested for the solutions of equations of general relativity [2-6] and enabling reasonable, however debatable, explanation for the “dark matter” problem [7]. The negative effective mass emerging in the core-spring-shell system, depicted in **Figure 1A** reflects the fact that the acceleration of the shell at certain conditions (to be discussed in detail below) is in the opposite direction to the applied force [1, 8-10]. Consequently, the chain of the core-spring-shell elements, depicted in **Figure 1A** demonstrates the paradoxical effective negative density [8, 11-14]. The effect of the negative effective density, in turn, opens the new pathways for the development of novel acoustic metamaterials [15]. Acoustic metamaterials are defined as engineered, artificial, periodic composites enabling altering mechanic properties of materials; thus, allowing properties that are not registered naturally [16]. In particular, metamaterials with controllable and negative bulk modulus were demonstrated, exploiting the effect of the negative mass [10, 11-13, 17]. A number of experimental exemplifications of the “negative mass effect” were already demonstrated [11-13, 18-19]. Use of the plasma oscillations of the free electron gas for the realization of the negative mass effect was suggested [20-21]. We demonstrate, how the gravitating system gives rise to the effect of the negative effective mass.



**Figure 1. A.** Mechanical scheme giving rise to the effect of the negative mass. The core-shell system, in which the core mass  $m$  is connected to the shell  $M$  via ideal Hookean spring  $k$ . **B.** Drilled gravitating ball  $R$  is depicted; density of the ball  $\rho$  is constant; the point mass  $m$  is located within the drilled channel  $r$ ;  $r \ll R$  is adopted. Both of systems are exerted to the harmonic external force  $F(t) \sim e^{-j\omega t}$ .

## 2. The effect of the negative effective mass in the gravitating system

Let us acquaint the contra-intuitive effect of the “negative mass”. Consider the core-ideal-spring-shell units, shown in **Figure 1A**. The spring is supposed to be Hookean and massless. This system, as demonstrated in refs. 1, 8-9, gives rise to the “negative effective mass behavior”. Assume that the mass of the shell is  $M$ , the mass of the core is  $m$  and the constant of the linear spring is  $k$ . When the core-shell system, depicted in **Figure 1A**, is subjected to the complex harmonic force  $\hat{F} = \hat{F}_0 e^{-j\omega t}$  the entire system may be substituted with a single mass  $m_{eff}$  defined according to Eq. 1, as suggested in refs 1, 8-9.

$$m_{eff} = M + \frac{m\omega_0^2}{\omega_0^2 - \omega^2} \quad (1)$$

where  $\omega_0 = \sqrt{\frac{k}{m}}$ . It is easily seen from Eq. 1 that, when the frequency  $\omega$  approaches  $\omega_0$  from above the effective mass  $m_{eff}$  will be negative [11-16]. Eq. 1 arises from the Eq. 2, which enables to present the entire complex moment  $\hat{P}$  of the core-spring-shell system, as follows (see Ref. 1):

$$\hat{P} = m_{eff} \hat{V} \quad , \quad (2)$$

where  $\hat{V}$  is the complex velocity of the shell  $M$ , and  $m_{eff}$  is given by Eq. 1. Huang *et al.* in ref. 8 noted that actually there is no, of course, the “negative mass”, and the negativity of the effective mass is the result of the attempt to use a single mass to represent a complex core-spring-shell system.

Consider now the drilled gravitating ball  $R$ , shown in **Figure 1B**, in which cylindrical channel  $r$  goes through the center of the ball and  $r \ll R$  takes place. Assume that the density of the ball is constant  $\rho =$

*const*, the mass of the ball is  $M$ . Small spherical mass  $m$  is placed in the channel, as shown in **Figure 1B**. We demonstrate that the aforementioned system, depicted in **Figure 1B** is mechanically equivalent to the core-spring-shell system shown in **Figure 1A**. Indeed, the gravitational force acting within the channel on the mass  $\vec{F}_{gr}$  is given by Eq. 3:

$$\vec{F}_{gr}(r) = m\vec{g}(r), \quad (3)$$

where  $\vec{g}$  is the gravitational field within the field. The Gauss's law for gravity is supplied by Eq. 4 (see ref. 22):

$$\iint_S \vec{g}(r) \cdot d\vec{S} = -4\pi GM, \quad (4)$$

where  $G \cong 6.67 \times 10^{-11} m^3 s^{-2} kg^{-1}$  is the gravitational constant. Considering  $r \ll R$  (keeping the spherical symmetry of the mass distribution with the ball  $R$ );  $M = \frac{4}{3}\pi\rho R^3$  and integration of Eq. 4 yields:

$$g(r) = -\frac{4}{3}\pi\rho Gr \quad (5)$$

Thus, the gravitational force  $\vec{F}_{gr}(r)$  acting on the point mass  $m$  is given by Eq. 6:

$$\vec{F}_{gr}(r) = -\frac{4}{3}\pi\rho Gmr\hat{r}, \quad (6)$$

where  $\hat{r}$  is the unit radius-vector. Eq. 6 represents the Hookean linear elastic force,  $F_{el} = -kr$ , where  $k$  is the stiffness coefficient, emerging immediately from Eq. 6:

$$k = \frac{4}{3}\pi\rho Gm \quad (7)$$

Hence, the point mass  $m$  oscillates within the channel according to the law:  $r \sim r_0 e^{-j\omega_0 t}$ , where the "critical frequency"  $\omega_0$  is given by Eq. 8:

$$\omega_0 = \sqrt{\frac{4}{3}\pi\rho G} \quad (8)$$

Thus, the gravitating system, presented in **Figure 1B**, is mechanically equivalent to the core-spring-shell system, shown in **Figure 1A**, in which the spring stiffness  $k$  and the frequency of oscillations  $\omega_0$  are given by Eq. 7 and Eq. 8. It should be emphasized that the frequency of oscillations  $\omega_0$  is independent of the mass  $m$ , thus, illustrating the Einstein Equivalency Principle, postulating the equivalence of gravitational and inertial mass [23-24]. It is also noteworthy that the frequency of oscillations  $\omega_0$  is independent of the radius

of the ball, being dependent on its density  $\rho$  only. Thus, we conclude that the critical frequency  $\omega_0$  depends on the intensive parameters of the self-gravitating system only.

Combining Eq. 1 and Eq. 8 yields for the effective mass of the gravitating system, shown in Figure 1B the following equation:

$$m_{eff} = M + \frac{\frac{4}{3}\pi\rho Gm}{\frac{4}{3}\pi\rho G - \omega^2} \quad (9)$$

It will be instructive to calculate the explicit value of  $\omega_0$  arising from Eq. 8. Assuming  $\rho \cong 1.0 \times 10^3 \frac{kg}{m^3}$  yields  $\omega_0 \cong 5.0 \times 10^{-4} s^{-1}$ . It is noteworthy, that the densities of condensed phases of different material are close, and they are confined within a relatively narrow range, as demonstrated in ref. 25. Victor Weisskopf in ref. 25 suggested the following expression for estimation of the density of the condensed phases:

$$\rho = \frac{A^{\frac{2}{5}} m_{pr}}{1.5^{\frac{4}{3}} \pi a_0^3}, \quad (10)$$

where  $A$  is the atomic or molecular weight,  $a_0$  is the Bohr radius and  $m_{pr}$  is the mass of the proton. Substitution of Eq. 10 into Eq. 8 yields:

$$\omega_0 = A^{\frac{1}{5}} \sqrt{\frac{m_{pr} G}{(1.5 a_0)^3}} = A^{\frac{1}{5}} \tilde{\omega}, \quad (11)$$

where  $\tilde{\omega} = \sqrt{\frac{m_{pr} G}{(1.5 a_0)^3}} \cong 4.4 \times 10^{-4} s$  is the fundamental frequency, which is built of the fundamental physical constants only. It is useful to supply the following explicit expression for the fundamental frequency  $\tilde{\omega}$ :

$$\tilde{\omega} = \sqrt{\frac{m_{pr} m_e^3 c^3 \alpha^3 G}{(1.5 \hbar)^3}}, \quad (12)$$

where  $m_e, c, \alpha$  and  $\hbar$  are the mass of electron, the speed of the light in vacuum, the fine structure constant and the reduced Planck constant correspondingly. Scaling Eq. 11 deserves the close inspection; indeed, it predicts a very weak dependence of the critical frequency  $\omega_0$  on the atomic weight, namely  $\omega_0 \sim A^{\frac{1}{5}}$ . In other words, the values of the critical frequency  $\omega_0$  are well-expected to be very close for very different condensed materials.

### 3. The effect of negative mass for homogeneous Coulomb-like spherically symmetrical systems

The aforementioned considerations are true for the similar systems governed by the Coulomb-like forces, which scale according to  $F(r) \sim \frac{1}{r^2}$ . Let us exemplify this fact with the electrically charged

ball  $R$  drilled through its center, as shown in **Figure 2B**. The mass of the ball is  $M$ . Assume that the ball is homogeneously electrically charged with a volume density  $\rho_{el} = \text{const}$ ;  $[\rho_{el}] = \frac{C}{m^3}$ . The point electrical charge  $q$  is placed within the cylindrical channel  $r$  as shown in **Figure 2B**; the mass of the charge carrier is  $m$ . Again, we adopt that the condition  $r \ll R$  takes place, which keeps the spherical symmetry of the electrical charge distribution within the ball. The electrical force  $\vec{F}_{el}(r)$  acting on the point charge  $q$  is given by Eq. 13:

$$\vec{F}_{el}(r) = q\vec{E}(r), \quad (13)$$

where  $\vec{E}(r)$  is an electrical field within the homogeneously charged ball  $R$ . The electrical field within the ball emerges from the Gauss theorem, and it is given by Eq. 14:

$$\vec{E}(r) = -\frac{\pi}{3\varepsilon_0}\rho_{el}r\hat{r}, \quad (14)$$

where  $\varepsilon_0 = 8.854 \frac{C}{Vm}$  is the vacuum permittivity. Thus, the Hookean linear electrical force acting on the charge is supplied by Eq. 15:

$$\vec{F}_{el}(r) = -\frac{\pi}{3\varepsilon_0}q\rho_{el}r\hat{r} \quad (15)$$

The stiffness coefficient  $k$  is now given by Eq. 16:

$$k = \frac{\pi}{3\varepsilon_0}q\rho_{el} \quad (16)$$

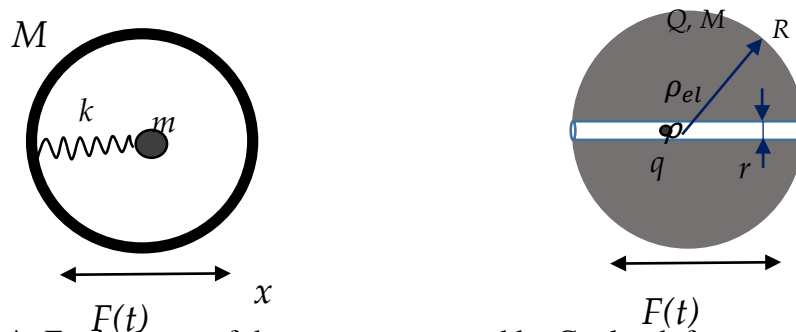
Thus, the frequency  $\omega_0$  is supplied by Eq. 17:

$$\omega_0 = \sqrt{\frac{\pi}{3\varepsilon_0} \frac{q\rho_{el}}{m}} \quad (17)$$

It is noteworthy, that the critical frequency  $\omega_0$  depends on the  $q/m$  ratio. The mechanical equivalence of the systems depicted in **Figures 2A** and **2B** is easily recognized, if the stiffness of the spring is given by Eq. 16. Now we exert the system shown in Figure 2B to the complex harmonic force  $\hat{F} = \hat{F}_0 e^{-j\omega t}$ . The effective mass of the system is given by Eq. 18:

$$m_{eff} = M + \frac{\frac{\pi}{3\varepsilon_0}q\rho_{el}}{\frac{\pi}{3\varepsilon_0} \frac{q\rho_{el}}{m} - \omega^2} \quad (18)$$

And, again, the effective mass of the system becomes negative when the frequency of the external force  $\omega$  approaches the critical frequency  $\omega_0 = \sqrt{\frac{\pi}{3\varepsilon_0} \frac{q\rho_{el}}{m}}$  from above and in principle in the frictionless system it is unrestricted.



**Figure 2.** A. Equivalence of the system governed by Coulomb force to the core-spring-shell system is illustrated. The core-shell system, in which the core mass  $m$  connected to the shell  $M$  via ideal Hookean spring  $k$  is shown. B. Drilled through it center electrically charged ball  $R$  is depicted; density of the electrical charge  $\rho_{el} = \text{const}$ ; the point charge  $q$  is located within the drilled channel  $r$ ;  $r \ll R$  is assumed. Both of systems are exerted to the harmonic external force  $F(t) \sim e^{-j\omega t}$ .

#### 4. Conclusions

The broadly discussed effect of the “negative effective mass” emerges in the core-spring-shell systems exerted to harmonic excitation [1, 8-10]. We addressed the effect of the negative effective mass emerging in the gravitating system, which is mechanically equivalent to the core-spring-shell system. The effect appears when the entire gravitating system is exerted to the external harmonic force and the frequency of external force approaches to the critical frequency  $\omega_0$  from above. The critical frequency  $\omega_0 = \sqrt{\frac{4}{3}\pi\rho G}$  depends on the density of the self-gravitating system  $\rho$  only. The scaling law predicting the value of  $\omega_0$  for condensed phases is derived as:  $\omega_0 = A^{\frac{1}{5}}\tilde{\omega}$ , where  $A$  is the atomic weight and  $\tilde{\omega} = \sqrt{\frac{m_{pr}G}{(1.5 a_0)^3}}$  is the fundamental frequency, built of the fundamental physical constants. The values of the critical frequency  $\omega_0$  are close for different condensed materials. The generalization of the effect for the Coulomb-like forces is reported. In this case, the effective mass becomes negative, when  $\omega$  approached to  $\omega_0 = \sqrt{\frac{\pi}{3\epsilon_0} \frac{q\rho_{el}}{m}}$  from above, and in principle, it is unrestricted.

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