

1 Article

## 2 Effect of Geometric Nonlinearity on Aerodynamic 3 Stability of Membrane Roofs

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9 **Abstract:** Membrane materials are most widely applied in construction engineering with  
10 small mass and high flexibility, it presents strong geometric nonlinearity in the process of  
11 vibration. In the paper, an improved multi-scale perturbation method is used to solve the  
12 aeroelastic stability of closed and open membrane roofs for quantify the effect of geometric  
13 nonlinearity on the single-mode aeroelastic instability wind speed of membrane roofs. The  
14 results show that the critical wind speed values of the two models are small when the  
15 geometrical nonlinearity of membrane material is neglected. In addition, under normal wind  
16 load, the influence of geometrical nonlinearity of membrane on the aerodynamic stability of  
17 roof can be neglected, However, under strong wind load, when the roof deformation reaches  
18 3% of the span, the influence of geometric nonlinearity should be considered and the influence  
19 increases with the decrease of transverse and downwind span of membrane roof. The results  
20 obtained in this paper have important theoretical reference value for the design the membrane  
21 structures.

22 **Keywords:** geometric nonlinearity; improved multis-scale method; orthotropic membrane;  
23 aeroelastic instability  
24

### 25 1. Introduction

26 Fabric membrane is the most widely used membrane material in construction engineering. It  
27 has the characteristics of high tensile strength and good flexibility. Fabric membranes are mainly  
28 composed of substrates and coatings. The substrates are usually braided by orthogonal fibers, which  
29 results in the orthotropic properties of the membranes, That is to say, the elastic modulus and  
30 Poisson's ratio in the two orthogonal directions are different. The building which is made up of  
31 membrane material covered on the structure skeleton or tensioned as a whole has beautiful  
32 appearance, good transparency, environmental protection and energy saving [1, 2]. Therefore, it is  
33 widely used in large-scale stadiums, exhibition venues and other public buildings. Because of the  
34 small mass and flexibility, it is easy to vibration under external disturbance. and the stiffness of the  
35 membrane material is small, which results in the large vibration deformation of the membrane  
36 structure under wind load, showing strong geometric non-linearity. Many research results shows  
37 that the single-mode aeroelastic instability can easily occur in membrane structures when the  
38 pre-tension of membrane materials is small [3, 4].

39 In the mathematical analysis of aeroelastic instability of flexible membrane structures, Yang  
40 et al. [5, 6] established the wind-induced dynamic coupling equation of hyperbolic parabolic  
41 membrane roofs with small sag by using elastic shallow shell theory and ideal fluid potential flow  
42 theory in 2006, and determined the critical wind speed of aeroelastic instability according to  
43 Routh-Hurwitz stability criterion. The influence of geometric nonlinearity of membranes was not  
44 taken into account when establishing the mathematical model. In 2011, Zheng et al. [7, 8] studied  
45 the non-linear aerodynamic stability of orthotropic tensioned membrane structures in rectangular

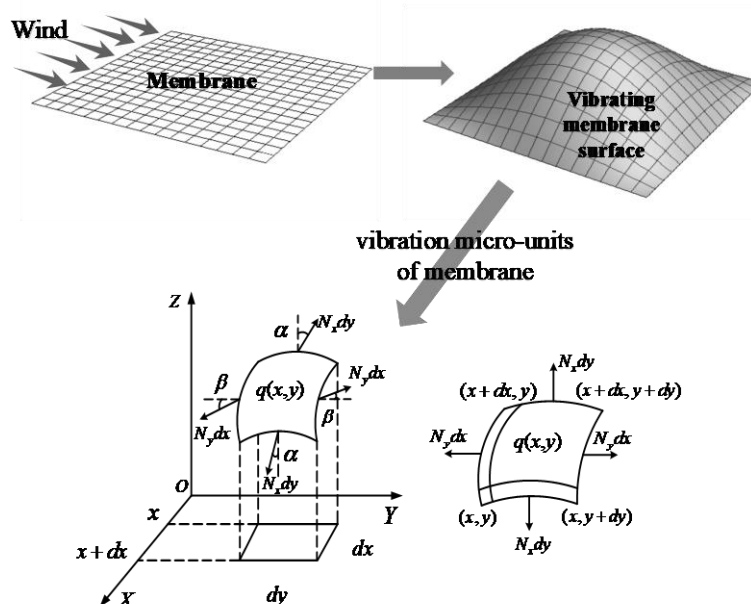
46 plane and hyperbolic paraboloid respectively. The critical wind velocities of single-mode instability  
 47 of two membrane structures were determined by assuming the solution of vibration equation. In  
 48 2017, Liu et al. [9] studied the aerodynamic stability of closed tensioned membrane structures by  
 49 Galerkin method. The geometric nonlinearity of membrane vibration is weakened, and the critical  
 50 wind speed of instability is obtained by using the weak nonlinearity solution method.

51 In order to investigate the influence of geometric nonlinearity on the aeroelastic stability of  
 52 membrane materials, the nonlinear wind-induced dynamic equations of membrane roofs are  
 53 established based on Von Kamen's large deflection theory and Darumbel's principle, taking the flat  
 54 rectangular orthotropic tensioned membrane roofs with fixed supports on the four sides of open  
 55 and closed structures as analytical models and considering the effects of geometric nonlinearity and  
 56 air damping of membrane materials. An improved multi-scale method which suitable for strong  
 57 geometric nonlinearity is used to solve the vibration equation. The critical wind speed of instability  
 58 obtained is compared with the results without considering geometric nonlinearity. The effect of  
 59 geometric nonlinearity on the wind speed of single-mode aeroelastic instability of membrane  
 60 material is obtained quantitatively.

## 61 2. Analytical deduction of single-mode instability of orthotropic membrane aeroelasticity

### 62 2.1. Establishment of Basic Equations

63 Let the length and width of the orthotropic rectangular flexible membrane with four sides  
 64 fixed be  $a$  and  $b$  respectively; the pre-tension along the length direction is  $N_{0x}$  and the width  
 65 direction is  $N_{0y}$ . The wind blows parallel to the roof and toward the membrane surface, which  
 66 makes the membrane surface vibrate. For flexible membranes, the research results show that  
 67 the shear stress has little influence on the vibration process of the membranes and can be  
 68 considered as zero[10]. Assuming that the planar membrane is in the  $xoy$  plane when  
 69 equilibrium, and the pre-tension in the  $x$  direction is  $N_x$  and in the  $Y$  direction is  $N_y$ , When the  
 70 membrane is disturbed by external forces perpendicular to the  $xoy$  plane, it will deform and  
 71 then produce transverse vibration perpendicular to the membrane surface under the action of  
 72 tension. Taking a vibrating micro-units on the vibrating membrane surface as shown in Figure  
 73 1.



74

75 **Figure 1.** Vibration micro-units of membrane

76 Taking the element  $dx dy$  on the membrane surface. When the micro-facets are deformed,  
 77 the edges of the micro-facets are subjected to the tension of the adjacent facets. In the

78 X-direction, we can regard the surface element as composed of countless chord elements with  
 79 length  $dx$  and width of one unit. The tension acting on the chord element is consistent with its  
 80 tangent direction. The tension  $N_x$  is at an angle  $\alpha$  with the  $x$  coordinate axis. Therefore, the  
 81 vertical component of the tension acting on the chord element at one end of  $X$  is  $N_x \sin \alpha$ .  
 82 because  $\alpha$  is small,  $\sin \alpha \approx \tan \alpha$ . Let  $w$  be the vertical displacement of a point on the membrane  
 83 away from the equilibrium position. Therefore:

$$84 \quad N_x \sin \alpha = N_x \tan \alpha = N_x \left( \frac{\partial w}{\partial x} \right)_x \quad (1)$$

85 The vertical force acting on the  $dy$  edge is:  $N_x \left( \frac{\partial w}{\partial x} \right)_x dy$ ; and the vertical force at the  $x$   
 86 edge should be:  $N_x \left( \frac{\partial w}{\partial x} \right)_{x+dx} dy$ . Thus, the resultant force in the vertical direction on the  $x$  and  
 87  $x+dx$  sides of the panel is as follows:

$$88 \quad N_x \left( \frac{\partial w}{\partial x} \right)_{x+dx} dy - N_x \left( \frac{\partial w}{\partial x} \right)_x dy = N_x \frac{\partial^2 w}{\partial x^2} dx dy \quad (2)$$

89 Similarly, the resultant force of the vertical component of the tension acting on the  $Y$   
 90 direction can be obtained as follows:

$$91 \quad N_y \left( \frac{\partial w}{\partial y} \right)_{y+dy} dx - N_y \left( \frac{\partial w}{\partial y} \right)_y dx = N_y \frac{\partial^2 w}{\partial y^2} dx dy \quad (3)$$

92 So the total vertical force acting on the whole panel is:

$$93 \quad F_z = N_x \frac{\partial^2 w}{\partial x^2} dx dy + N_y \frac{\partial^2 w}{\partial y^2} dx dy + q(x, y) dx dy \quad (4)$$

94 Where,  $N_x$  is the tension in the  $x$  direction (longitude),  $N_y$  is the tension in the  $y$  direction  
 95 (latitude),  $w$  is the deflection of the membrane, and  $q(x, y)$  is the external load acting on the  
 96 unit area of the projection surface of the membrane. According to the force balance, we can  
 97 obtained that:

$$98 \quad N_x \frac{\partial^2 w}{\partial x^2} dx dy + N_y \frac{\partial^2 w}{\partial y^2} dx dy + q(x, y) dx dy = 0 \quad (5)$$

$$99 \quad q(x, y) + N_x \frac{\partial^2 w}{\partial x^2} + N_y \frac{\partial^2 w}{\partial y^2} = 0 \quad (6)$$

100 The generalized external loads of flexible membrane roof under wind load include the  
 101 wind load acting on the membrane surface, structural damping force and inertial force [10]. If  
 102 the aerodynamic term is defined as  $p$ , then the generalized external load per unit area  
 103  $q(x, y)$  is:

$$104 \quad q(x, y) = p(x, y, t) - 2\rho c \frac{\partial w(x, y, t)}{\partial t} - \rho \frac{\partial^2 w(x, y, t)}{\partial t^2} \quad (7)$$

105 For membrane material, the stiffness of membrane surface comes from the initial  
 106 pre-tension of membrane material, so the initial pre-tension should be added in formula 6.  
 107 Finally, the differential equations of motion of flexible membranes are obtained as follows:

$$108 \quad p(x, y, t) - 2\rho c \frac{\partial w(x, y, t)}{\partial t} - \rho \frac{\partial^2 w(x, y, t)}{\partial t^2} + (N_{0x} + N_{xt}) \frac{\partial^2 w}{\partial x^2} + (N_{0y} + N_{yt}) \frac{\partial^2 w}{\partial y^2} = 0 \quad (8)$$

109 Introducing the stress function  $\varphi(x, y)$ ,  $N_x = h \frac{\partial^2 \varphi}{\partial y^2}$ ,  $N_y = h \frac{\partial^2 \varphi}{\partial x^2}$ , Then equation (6)  
 110 becomes:

$$111 \quad \left( N_{0x} + h \frac{\partial^2 \varphi}{\partial y^2} \right) \frac{\partial^2 w}{\partial x^2} + \left( N_{0y} + h \frac{\partial^2 \varphi}{\partial x^2} \right) \frac{\partial^2 w}{\partial y^2} + p - 2\rho c \frac{\partial w}{\partial t} - \rho \frac{\partial^2 w}{\partial t^2} = 0 \quad (9)$$

112 After deformation, the membrane surface strain is composed of linear and non-linear parts.  
 113 The linear strain is caused by in-plane displacement  $u$  and  $v$ , and the non-linear strain is  
 114 caused by deflection  $w$ . After ignoring shear stress, the total strain is as follows:

$$115 \quad \left. \begin{aligned} \varepsilon_x &= \frac{\partial u}{\partial x} + \frac{1}{2} \left( \frac{\partial w}{\partial x} \right)^2 \\ \varepsilon_y &= \frac{\partial v}{\partial y} + \frac{1}{2} \left( \frac{\partial w}{\partial y} \right)^2 \end{aligned} \right\} \quad (10)$$

116 Where,  $\varepsilon_x$  is the strain in the X direction,  $\varepsilon_y$  is the strain in the Y direction.

117 By eliminating  $u$  and  $v$  in equation (10), the continuous deformation conditions satisfying  
 118 the strain and deflection of the film surface can be obtained.

$$119 \quad \frac{\partial^2 \varepsilon_x}{\partial y^2} + \frac{\partial^2 \varepsilon_y}{\partial x^2} = \left( \frac{\partial^2 w}{\partial x \partial y} \right)^2 - \frac{\partial^2 w}{\partial x^2} \frac{\partial^2 w}{\partial y^2} \quad (11)$$

120 The membrane is orthotropic, and the direction of the fiber is the main direction of  
 121 elasticity, so that it is consistent with the direction of coordinate system X and Y. Assuming  
 122 that the direction of fiber is the same as the direction of coordinate system X and Y. The  
 123 Young's modulus of elasticity in X and Y directions is  $E_1$  and  $E_2$ , respectively. The longitudinal  
 124 Poisson's ratio and the latitudinal Poisson's ratio is  $\mu_1$  and  $\mu_2$ , respectively. The relationship  
 125 between elastic modulus and Poisson's ratio is as follows.

$$126 \quad \frac{\mu_1}{E_1} = \frac{\mu_2}{E_2} \quad (12)$$

127 The stress-strain relationship is as follows:

$$128 \quad \left\{ \begin{array}{l} \sigma_x \\ \sigma_y \end{array} \right\} = \left( \begin{array}{cc} \frac{E_1}{1 - \mu_1 \mu_2} & \frac{\mu_1 E_2}{1 - \mu_1 \mu_2} \\ \frac{\mu_2 E_1}{1 - \mu_1 \mu_2} & \frac{E_2}{1 - \mu_1 \mu_2} \end{array} \right) \left\{ \begin{array}{l} \varepsilon_x \\ \varepsilon_y \end{array} \right\} \quad (13)$$

129 Where  $\sigma_x$  and  $\sigma_y$  is the normal stresses in the X direction and Y direction respectively.  $h$  is  
 130 the thickness of membrane.

131 Letting  $N_x = h \cdot \sigma_x$ ,  $N_y = h \cdot \sigma_y$ , substituting Equation (13) into Equation (11), the  
 132 compatibility equation is obtained as follows:

$$\begin{aligned}
 & \frac{1}{E_1 h} \frac{\partial^2 N_x}{\partial y^2} - \frac{\mu_2}{E_2 h} \frac{\partial^2 N_y}{\partial y^2} - \frac{\mu_1}{E_1 h} \frac{\partial^2 N_x}{\partial x^2} + \frac{1}{E_2 h} \frac{\partial^2 N_y}{\partial x^2} \\
 & = \left( \frac{\partial^2 w}{\partial x \partial y} \right)^2 - \frac{\partial^2 w}{\partial x^2} \frac{\partial^2 w}{\partial y^2}
 \end{aligned} \tag{14}$$

By substituting the stress function into Equation (14), it can be transformed into:

$$\begin{aligned}
 & \frac{1}{E_1} \frac{\partial^4 \varphi}{\partial y^4} - \frac{\mu_2}{E_2} \frac{\partial^4 \varphi}{\partial x^2 \partial y^2} - \frac{\mu_1}{E_1} \frac{\partial^4 \varphi}{\partial x^2 \partial y^2} + \frac{1}{E_2} \frac{\partial^4 \varphi}{\partial x^4} \\
 & = \left( \frac{\partial^2 w}{\partial x \partial y} \right)^2 - \frac{\partial^2 w}{\partial x^2} \frac{\partial^2 w}{\partial y^2}
 \end{aligned} \tag{15}$$

## 2.2. Modified multiscale solutions of governing equations

The initial surface function of rectangular planar membrane  $z_0(x, y) = 0$ , then the surface equation of flexible membrane under wind load is as follows:

$$z(x, y, t) = w(x, y, t) \tag{16}$$

According to the Bubnov-Galerkin method, assuming the solution of the governing equation is [9, 10].

$$\begin{cases} w(x, y, t) = \sum_{i=1}^n T_i(t) W_i(x, y) \\ \varphi(x, y, t) = \sum_{i=1}^n U_i(t) \phi_i(x, y) \end{cases} \tag{17}$$

Where  $W_i(x, y)$  is the mode function,  $\phi_i(x, y)$  is the unknown stress function about the coordinates,  $T_i(t)$  and  $U_i(t)$  are the time-dependent function.

Because the membrane is fixed on four sides, the vertical deflection at the boundary of the membrane is zero, and the vibration mode function satisfying the conditions is assumed to be:

$$W(x, y) = \sin \frac{m\pi x}{a} \sin \frac{n\pi y}{b} \tag{18}$$

where m and n are positive integer, which denote sinusoidal half wave number.

substituting Equation (18) into Equation (17), the following equation is obtained:

$$w(x, y, t) = T(t) \sin \frac{m\pi x}{a} \sin \frac{n\pi y}{b} \tag{19}$$

Substituting Equation (19) into Equation (15), yields:

$$\frac{1}{E_1} \frac{\partial^4 \varphi}{\partial y^4} + \frac{1}{E_2} \frac{\partial^4 \varphi}{\partial x^4} = \frac{m^2 n^2 \pi^4}{2a^2 b^2} T^2(t) \left( \cos \frac{2m\pi x}{a} + \cos \frac{2n\pi y}{b} \right) \tag{20}$$

Assuming that the solution of stress function in formula (20) is

$$\begin{cases} \varphi(x, y, t) = T^2(t) \phi(x, y) \\ \phi(x, y) = \alpha \cos \frac{2m\pi x}{a} + \beta \cos \frac{2n\pi y}{b} \end{cases} \tag{21}$$

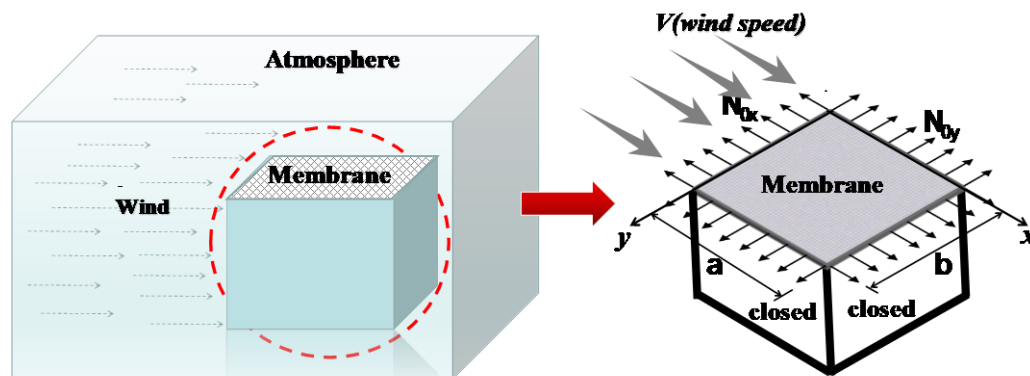
substituting Equation (21) into Equation (20), yields:

156

$$157 \quad \alpha = \frac{E_2 a^2 n^2}{32 b^2 m^2}, \quad \beta = \frac{E_1 b^2 m^2}{32 a^2 n^2} \quad (22)$$

### 158 2.1.1. Solution of Flexible Membrane Roof on Closed Structure

159 Flexible membrane covers the top of closed structure as roof, and rigid wall around the  
 160 structure as vertical bearing member. Because its stiffness is far greater than the membrane's  
 161 stiffness, it is assumed that the stiffness of vertical component is infinite in the process of  
 162 theoretical derivation in this paper. The membrane roof on closed structure is shown in Figure  
 163 2.



164 **Figure 2.** The membrane roof on closed structure

165 For the membrane roof on closed structure, the aerodynamic force acting on the unit area  
 166 of the membrane projection surface is expressed as follows[8]:

$$168 \quad p = -\frac{\rho_0}{2\pi} \left( -V \iint_{Ra} \frac{\left( V \frac{\partial w}{\partial x} + \frac{\partial w}{\partial t} \right)_{x=\xi, y=\eta} (x-\xi)}{\left( \sqrt{(x-\xi)^2 + (y-\eta)^2} \right)^3} d\xi d\eta + \iint_{Ra} \frac{\left( V \frac{\partial^2 w}{\partial x \partial t} + \frac{\partial^2 w}{\partial t^2} \right)_{x=\xi, y=\eta}}{\sqrt{(x-\xi)^2 + (y-\eta)^2}} d\xi d\eta \right) \quad (23)$$

169 Substituting Equation (23) into Equation (9), yields:

$$170 \quad \begin{aligned} & \left( h \frac{\partial^2 \varphi}{\partial y^2} + N_{0x} \right) \frac{\partial^2 w}{\partial x^2} + \left( h \frac{\partial^2 \varphi}{\partial x^2} + N_{0y} \right) \frac{\partial^2 w}{\partial y^2} - 2\rho c \frac{\partial w}{\partial t} \\ & - \rho \frac{\partial^2 w}{\partial t^2} - \frac{\rho_0}{\pi} \iint_{Ra} \frac{1}{r} \left( \frac{\partial^2 w}{\partial t^2} \right)_{x=\xi, y=\eta} d\xi d\eta - \frac{\rho_0 V}{2\pi} \iint_{Ra} \frac{1}{r} \left( \frac{\partial^2 w}{\partial x \partial t} \right)_{x=\xi, y=\eta} d\xi d\eta \\ & + \frac{\rho_0 V^2}{2\pi} \iint_{Ra} \frac{1}{r^3} \left( \frac{\partial w}{\partial x} \right)_{x=\xi, y=\eta} (x-\xi) d\xi d\eta + \frac{\rho_0 V}{2\pi} \iint_{Ra} \frac{1}{r^3} \left( \frac{\partial w}{\partial t} \right)_{x=\xi, y=\eta} (x-\xi) d\xi d\eta = 0 \end{aligned} \quad (24)$$

171 Where  $r = \sqrt{(x-\xi)^2 + (y-\eta)^2}$ , the integral region  $Ra \in \{0 \leq \xi \leq a, 0 \leq \eta \leq b\}$ .

172 Substituting Equations (19), (21) and (22) into Equation (24), yields:

$$173 \quad \begin{aligned} & \left( \rho W + \frac{\rho_0}{\pi} \gamma_1 \right) \frac{d^2 T(t)}{dt^2} + \left[ \frac{\rho_0 V}{2\pi} (\gamma_2 - \gamma_4) + 2\rho c W \right] \frac{dT(t)}{dt} \\ & - \left( N_{0x} \frac{\partial^2 W}{\partial x^2} + N_{0y} \frac{\partial^2 W}{\partial y^2} + \frac{\rho_0 V^2}{2\pi} \gamma_3 \right) T(t) - h \left( \frac{\partial^2 \Phi}{\partial y^2} \frac{\partial^2 W}{\partial x^2} + \frac{\partial^2 \Phi}{\partial x^2} \frac{\partial^2 W}{\partial y^2} \right) T^3(t) = 0 \end{aligned} \quad (25)$$

174 Where:

$$\gamma_1 = \iint_{Ra} \frac{1}{r} (W)_{x=\xi, y=\eta} d\xi d\eta = \iint_{Ra} \frac{1}{r} \sin \frac{m\pi\xi}{a} \sin \frac{n\pi\eta}{b} d\xi d\eta$$

$$\gamma_2 = \iint_{Ra} \frac{1}{r} \left( \frac{\partial W}{\partial x} \right)_{x=\xi, y=\eta} d\xi d\eta = \frac{m\pi}{a} \iint_{Ra} \frac{1}{r} \cos \frac{m\pi\xi}{a} \sin \frac{n\pi\eta}{b} d\xi d\eta$$

$$\gamma_3 = \iint_{Ra} \frac{1}{r^3} \left( \frac{\partial W}{\partial x} \right)_{x=\xi, y=\eta} (x-\xi) d\xi d\eta = \frac{m\pi}{a} \iint_{Ra} \frac{1}{r^3} (x-\xi) \cos \frac{m\pi\xi}{a} \sin \frac{n\pi\eta}{b} d\xi d\eta$$

$$\gamma_4 = \iint_{Ra} \frac{1}{r^3} (W)_{x=\xi, y=\eta} (x-\xi) d\xi d\eta = \iint_{Ra} \frac{1}{r^3} (x-\xi) \sin \frac{m\pi\xi}{a} \sin \frac{n\pi\eta}{b} d\xi d\eta$$

Using Bubnov-Galerkin method to integral Equation (25), yields:

$$\iint_S \left\{ \left( \rho W + \frac{\rho_0}{\pi} \gamma_1 \right) \frac{d^2 T(t)}{dt^2} + \left[ \frac{\rho_0 V}{2\pi} (\gamma_2 - \gamma_4) + 2\rho c W \right] \frac{dT(t)}{dt} - \left( N_{0x} \frac{\partial^2 W}{\partial x^2} + N_{0y} \frac{\partial^2 W}{\partial y^2} + \frac{\rho_0 V^2}{2\pi} \gamma_3 \right) T(t) - h \left( \frac{\partial^2 \Phi}{\partial y^2} \frac{\partial^2 W}{\partial x^2} + \frac{\partial^2 \Phi}{\partial x^2} \frac{\partial^2 W}{\partial y^2} \right) T^3(t) \right\} W(x, y) dx dy = 0 \quad (26)$$

Where  $S \in \{0 \leq x \leq a, 0 \leq y \leq b\}$ .

Simplifying Formula (26), yields:

$$A \frac{d^2 T(t)}{dt^2} + B \frac{dT(t)}{dt} - CT(t) - DT^3(t) = 0 \quad (27)$$

Where:

$$\begin{aligned} A &= \iint_S \left( \rho W + \frac{\rho_0}{\pi} \gamma_1 \right) W dx dy \\ &= \rho \iint_S \left( \sin \frac{m\pi x}{a} \sin \frac{n\pi y}{b} \right)^2 dx dy + \frac{\rho_0}{\pi} \iint_{Ra} \left( \iint_{Ra} \frac{1}{r} \sin \frac{m\pi\xi}{a} \sin \frac{n\pi\eta}{b} d\xi d\eta \right) \sin \frac{m\pi x}{a} \sin \frac{n\pi y}{b} dx dy \\ &= \frac{\rho ab}{4} + \frac{\rho_0}{\pi} \alpha_1 \\ \alpha_1 &= \iint_S \left( \iint_{Ra} \frac{1}{r} \sin \frac{m\pi\xi}{a} \sin \frac{n\pi\eta}{b} d\xi d\eta \right) \sin \frac{m\pi x}{a} \sin \frac{n\pi y}{b} dx dy \\ B &= \frac{\rho_0 V}{2\pi} \iint_S (\gamma_2 - \gamma_4) W dx dy + 2\rho c \iint_S W^2 dx dy \\ &= \frac{\rho_0 m V}{2a} \iint_S \left( \iint_{Ra} \frac{1}{r} \cos \frac{m\pi\xi}{a} \sin \frac{n\pi\eta}{b} d\xi d\eta \right) \sin \frac{m\pi x}{a} \sin \frac{n\pi y}{b} dx dy \\ &\quad - \frac{\rho_0 V}{2\pi} \iint_S \left[ \iint_{Ra} \frac{1}{r^3} (x-\xi) \sin \frac{m\pi\xi}{a} \sin \frac{n\pi\eta}{b} d\xi d\eta \right] \sin \frac{m\pi x}{a} \sin \frac{n\pi y}{b} dx dy \\ &\quad + 2\rho c \iint_S \left( \sin \frac{m\pi x}{a} \sin \frac{n\pi y}{b} \right)^2 dx dy \\ &= \frac{\rho_0 m V}{2a} \alpha_2 - \frac{\rho_0 V}{2\pi} \alpha_4 + \frac{\rho cab}{2} \end{aligned}$$

$$\begin{aligned}
184 \quad \alpha_2 &= \iint_S \left( \iint_{Ra} \frac{1}{r} \cos \frac{m\pi\xi}{a} \sin \frac{n\pi\eta}{b} d\xi d\eta \right) \sin \frac{m\pi x}{a} \sin \frac{n\pi y}{b} dx dy \\
\alpha_4 &= \iint_S \left[ \iint_{Ra} \frac{1}{r^3} (x-\xi) \sin \frac{m\pi\xi}{a} \sin \frac{n\pi\eta}{b} d\xi d\eta \right] \sin \frac{m\pi x}{a} \sin \frac{n\pi y}{b} dx dy \\
C &= \iint_S \left( N_{0x} \frac{\partial^2 W}{\partial x^2} + N_{0y} \frac{\partial^2 W}{\partial y^2} + \frac{\rho_0 V^2}{2\pi} \gamma_3 \right) W dx dy \\
185 \quad &= \iint_S N_{0x} \frac{\partial^2 W}{\partial x^2} W dx dy + \iint_{Ra} N_{0y} \frac{\partial^2 W}{\partial y^2} W dx dy + \frac{\rho_0 V^2}{2\pi} \iint_{Ra} \gamma_3 W dx dy \\
&= -\frac{m^2 \pi^2 b N_{0x}}{4a} - \frac{n^2 \pi^2 a N_{0y}}{4b} + \frac{\rho_0 m V^2}{2a} \alpha_3 \\
186 \quad \alpha_3 &= \iint_S \left( \iint_{Ra} \frac{1}{r^3} (x-\xi) \cos \frac{m\pi\xi}{a} \sin \frac{n\pi\eta}{b} d\xi d\eta \right) \sin \frac{m\pi x}{a} \sin \frac{n\pi y}{b} dx dy \\
187 \quad D &= \iint_S h \left( \frac{\partial^2 \Phi}{\partial y^2} \frac{\partial^2 W}{\partial x^2} + \frac{\partial^2 \Phi}{\partial x^2} \frac{\partial^2 W}{\partial y^2} \right) W dx dy \\
&= -\frac{hm^2 n^2 \pi^4 (\alpha + \beta)}{2ab}
\end{aligned}$$

188 It can be obtained from numerical calculation that only when  $b/a \ll 0.1$ ,  $A \leq 0$ ; this will  
189 not happen in practical engineering. Letting  $u = B/(\varepsilon A)$ ,  $\omega_0^2 = -C/A$ ,  $\varepsilon = -D/A$ . Then the  
190 equation (27) is transformed into:

$$191 \quad \ddot{T} + \omega_0^2 T + \varepsilon (\mu \dot{T} + T^3) = 0 \quad (28)$$

192 Letting  $\omega$  is the vibration frequency of membrane material and expanding  $\omega^2$  to the  
193 power series of  $\varepsilon$  near  $\omega_0^2$  as follows:

$$194 \quad \omega^2 = \omega_0^2 + \varepsilon \omega_1 + \varepsilon^2 \omega_2 + \dots \quad (29)$$

195 The transformation parameters are introduced as follows.

$$196 \quad \alpha = \frac{\varepsilon \omega_1}{\omega_0^2 + \varepsilon \omega_1} \quad (30)$$

$$197 \quad \varepsilon = \frac{\omega_0^2 \alpha}{\omega_1 (1 - \alpha)} \quad (31)$$

$$198 \quad \omega_0^2 + \varepsilon \omega_1 = \frac{\omega_0^2}{1 - \alpha}$$

Expanding  $\omega^2$  to the power series of  $\varepsilon$  as follows:

$$199 \quad \omega^2 = \omega_0^2 + \varepsilon \omega_1 \left[ 1 + \frac{1}{\omega_0^2 + \varepsilon \omega_1} (\varepsilon^2 \omega_2 + \varepsilon^3 \omega_3 + \dots) \right] = \frac{\omega_0^2}{1 - \alpha} (1 + \delta_2 \alpha^2 + \delta_3 \alpha^3 + \dots) \quad (32)$$

$$200 \quad \omega = \omega_0 \left[ 1 + \frac{1}{2} \alpha + \left( \frac{3}{8} + \frac{\delta^2}{2} \right) \alpha^2 + \dots \right] \quad (33)$$

201 The form of perturbation solution of the equation (28) can be:

$$202 \quad T(t, \alpha) = T_0(t_0, t_1) + \alpha T_1(t_0, t_1) + \alpha^2 T_2(t_0, t_1) + \dots \quad (34)$$

203 Where  $t_0 = t$ ,  $t_1 = \alpha t$ .

204 The differential operators are obtained as follows.



$$\frac{d}{dt} = D_0 + \alpha D_1 + \alpha^2 D_2 + \dots, \quad (35)$$

$$\frac{d^2}{dt^2} = D_0^2 + 2\alpha D_0 D_1 + \alpha^2 (D_1^2 + 2D_0 D_2) + \dots$$

Substituting Equations (30), (31), (32) and (34) into Equation (28), yields:

$$\begin{aligned} & (1-\alpha) \left[ D_0^2 + 2\alpha D_0 D_1 + \alpha^2 (D_1^2 + 2D_0 D_2) \right] (T_0 + \alpha T_1 + \alpha^2 T_2 + \dots) \\ & + (1-\alpha) \omega_0^2 (T_0 + \alpha T_1 + \alpha^2 T_2 + \dots) + \frac{\alpha \omega_0^2}{\omega_1^2} \left[ (T_0 + \alpha T_1 + \alpha^2 T_2 + \dots)^3 + (D_0 + \alpha D_1 + \alpha^2 D_2) \cdot \right. \\ & \left. (T_0 + \alpha T_1 + \alpha^2 T_2 + \dots) \right] \\ & = 0 \end{aligned} \quad (36)$$

$$\begin{aligned} \alpha^0 \quad & D_0^2 T_0 + \omega_0^2 T_0 = 0 \\ \alpha^1 \quad & D_0^2 T_1 + \omega_0^2 T_1 + 2D_0 D_1 T_0 + \frac{\omega_0^2}{\omega_1} (D_0 T_0 + T_0^3) = 0 \end{aligned} \quad (37)$$

$$\alpha^2 \quad D_0^2 T_2 + T_2 + 2D_0 D_1 T_1 + (D_1^2 + 2D_0 D_2) T_0 + \frac{\omega_0^2}{\omega_1} 3T_0^2 T_1 = 0$$

The solution of the first equation in the system of equations (37) can be as follows:

$$T_0 = A(t_1) e^{i\omega_0 t_0} + \bar{A}(t_1) e^{-i\omega_0 t_0} \quad (38)$$

Substituting Equation (38) into The Second Equation in Equation (37), yields:

$$D_0^2 T_1 + \omega_0^2 T_1 + \left( 2i\omega_0 D_1 A + 3 \frac{\omega_0^2}{\omega_1} A^2 \bar{A} + i\mu \frac{\omega_0^3}{\omega_1} A \right) e^{it_0} + \frac{1}{\omega_1} A^3 e^{3it_0} + cc = 0 \quad (39)$$

Where  $cc$  is the conjugate complex term. Eliminating the term of immortality in equation (39), yields:

$$2D_1 A + 3 \frac{\omega_0}{\omega_1} A^2 \bar{A} + i\mu \frac{\omega_0^3}{\omega_1} A = 0 \quad (40)$$

Solving equation (40), we can obtain:

$$T_1 = \frac{1}{8\omega_1} (A^3 e^{3it_0} + \bar{A}^3 e^{-3it_0}) \quad (41)$$

Letting  $A = \frac{1}{2} f e^{i\phi}$ , bring it into equation (41) and separating the imaginary part from the real part.

$$\frac{df}{dt_1} = -\frac{1}{2} \mu \frac{\omega_0^2}{\omega_1} f, \quad f \frac{d\phi}{dt_1} = -\frac{3\omega_0}{8\omega_1} f^3 \quad (42)$$

Substituting  $A = \frac{1}{2} f e^{i\phi}$  into Equation (38), yields:

$$T_0 = f \cos(\omega_0 t_0 + \phi) = f \cos(\omega t + \phi_0) \quad (43)$$

Comparing the angular frequency in equation (33) with equation (43), we can get the equation under the first order approximation condition as follows.

$$\frac{d\phi}{dt_1} = \frac{\omega_0}{2} \quad (44)$$

According to the equations (42) and (43), yields:

$$\omega_1 = \frac{3}{4} f^2 \quad (45)$$

229

Substituting Equation (45) into Equation (42) and omitting higher order terms, yields:

230

$$\omega = \sqrt{\omega_0^2 + \frac{3}{4} \varepsilon f^2} \quad (46)$$

231

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233

The results show that when the critical wind speed of single mode instability is reached, the frequency of the characteristic equation of the system approaches zero, which is equivalent to static equilibrium instability [9]. That is:

234

$$\omega_0^2 + \frac{3}{4} \varepsilon f^2 = 0 \quad (47)$$

235

Where  $f$  is the vibration amplitude of membrane,

$$\omega_0^2 = \frac{2\pi}{\rho ab\pi + 4\rho_0\alpha_1} \cdot \left( \frac{m^2\pi^2b^2N_{0x} + n^2\pi^2a^2N_{0y}}{2ab} - \frac{\rho_0m\alpha_3}{a} V^2 \right)$$

236

$$\alpha_1 = \iint_S \left( \iint_{Ra} \frac{1}{r} \sin \frac{m\pi\xi}{a} \sin \frac{n\pi\eta}{b} d\xi d\eta \right) \sin \frac{m\pi x}{a} \sin \frac{n\pi y}{b} dx dy$$

$$\alpha_3 = \iint_S \left( \iint_{Ra} \frac{1}{r^3} (x-\xi) \cos \frac{m\pi\xi}{a} \sin \frac{n\pi\eta}{b} d\xi d\eta \right) \sin \frac{m\pi x}{a} \sin \frac{n\pi y}{b} dx dy$$

237

$$\varepsilon = \frac{2hm^2n^2\pi^5(\alpha + \beta)}{ab(\rho ab\pi + 4\rho_0\alpha_1)}$$

238

239

Solving the equation, the critical wind speed for single mode instability of closed membrane roof is obtained as follows:

240

$$V_{cr} = \sqrt{\frac{4\pi(m^2\pi^2b^2N_{0x} + n^2\pi^2a^2N_{0y}) + 3\varepsilon f^2(\rho\pi a^2b^2 + 4\rho_0aba\alpha_1)}{8\pi b\rho_0m\alpha_3}} \quad (48)$$

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where  $f$  is the vibration amplitude corresponding to the instability critical wind speed. It shows that the critical wind speed is related to the vibration amplitude. Considering the influence of geometric nonlinearity of membrane, the stiffness of membrane will change with the change of amplitude in the process of vibration, which will affect the aerodynamic stability of membrane roof to a certain extent, which is consistent with the conclusions of previous research. When  $f \rightarrow 0$ , the critical wind speed of instability can be obtained according to the theory of small deflection.

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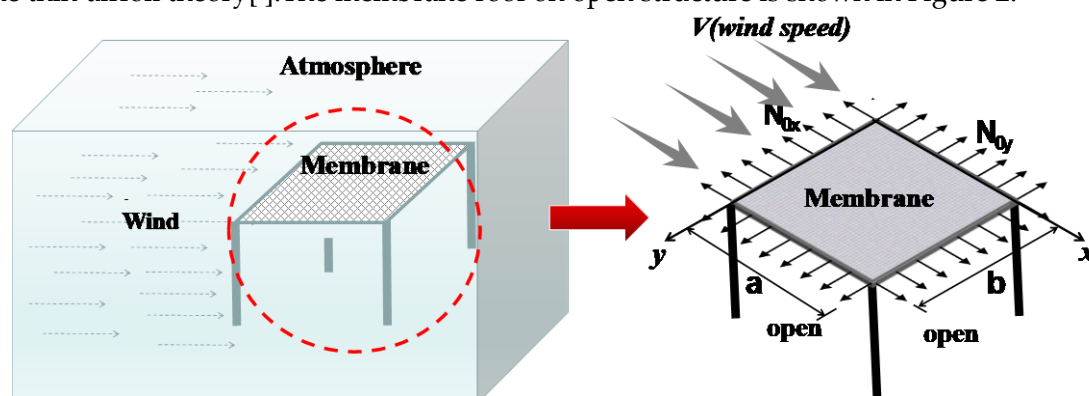
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### 2.1.2. Solution of Flexible Membrane Roof on Open Structure

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251

For membrane roofs open open structures, air flows from both sides of the membrane surface due to the smaller thickness of the membrane, which can be approximately determined by the thin-airfoil theory [ ]. The membrane roof on open structure is shown in Figure 2.



252

253

Figure 3. The membrane roof on open structure

254

255

For the membrane roof on open structure, the aerodynamic force acting on the unit area of the membrane projection surface is expressed as follows[7]:

256

$$p = \rho_0 \frac{\partial}{\partial t} \int_0^x \gamma_c(\xi, y, t) d\xi + \rho_0 V \gamma_c \quad (49)$$

257

Where  $\gamma_c$  is the density of vortices.

258

$$\gamma_c = aV\gamma_j = aV \left( a_{1j} T(t) + a_{2j} \frac{T'(t)}{V} \right), \quad (j = 1, 2, \dots, M \times N) \quad (50)$$

259

Substituting Equation (50) into Equation (9), yields:

260

$$\left( h \frac{\partial^2 \phi}{\partial y^2} + N_{0x} \right) \frac{\partial^2 w}{\partial x^2} + \left( h \frac{\partial^2 \phi}{\partial x^2} + N_{0y} \right) \frac{\partial^2 w}{\partial y^2} - 2\rho \xi_0 \frac{\partial w}{\partial t} + \rho_0 V \gamma_c + \rho_0 \int_0^y \frac{\partial \gamma_c}{\partial t} d\eta = \rho \frac{\partial^2 w}{\partial t^2} \quad (51)$$

261

262

Solutions of stress functions in compatible equations such as Equation (21), Substituting Equations (19) and (21) into Equation (51), yields:

263

$$\begin{aligned} & \rho W \frac{d^2 T(t)}{dt^2} + 2\rho \xi_0 W \frac{dT(t)}{dt} - \left( N_{0x} \frac{\partial^2 W}{\partial x^2} + N_{0y} \frac{\partial^2 W}{\partial y^2} \right) T(t) \\ & - h \left( \frac{\partial^2 \phi}{\partial y^2} \frac{\partial^2 W}{\partial x^2} + \frac{\partial^2 \phi}{\partial x^2} \frac{\partial^2 W}{\partial y^2} \right) T^3(t) - \rho_0 V \gamma_c - \rho_0 \int_0^y \frac{\partial \gamma_c}{\partial t} d\eta = 0 \end{aligned} \quad (52)$$

264

Using Bubnov-Galerkin method to integral Equation (52), yields:

265

$$\iint_S \left[ \begin{aligned} & \rho W \frac{d^2 T(t)}{dt^2} + 2\rho \xi_0 W \frac{dT(t)}{dt} - \left( N_{0x} \frac{\partial^2 W}{\partial x^2} + N_{0y} \frac{\partial^2 W}{\partial y^2} \right) T(t) \\ & - h \left( \frac{\partial^2 \phi}{\partial y^2} \frac{\partial^2 W}{\partial x^2} + \frac{\partial^2 \phi}{\partial x^2} \frac{\partial^2 W}{\partial y^2} \right) T^3(t) - \rho_0 V \gamma_c - \rho_0 \int_0^y \frac{\partial \gamma_c}{\partial t} d\eta \end{aligned} \right] W(x, y) dx dy = 0 \quad (53)$$

266

267

Where  $S \in \{0 \leq x \leq a, 0 \leq y \leq b\}$ .

Integrating Equation (53) and simplifying it, yields:

268

$$A \frac{d^2 T(t)}{dt^2} + B \frac{dT(t)}{dt} - CT(t) - DT^3(t) - E = 0 \quad (54)$$

269

Where:

270

$$A = \iint_S \rho W^2 dx dy = \frac{\rho ab}{4}$$

271

$$B = 2\rho \xi_0 \iint_S W^2 dx dy = 2\rho \xi_0 \iint_S \left( \sin \frac{m\pi x}{a} \sin \frac{n\pi y}{b} \right)^2 dx dy = \frac{\rho \xi_0 ab}{2}$$

272

$$\begin{aligned} C &= \iint_S \left( N_{0y} \frac{\partial^2 W(x, y)}{\partial y^2} + N_{0x} \frac{\partial^2 W(x, y)}{\partial x^2} \right) W(x, y) dx dy = \iint_S N_{0x} \frac{\partial^2 W}{\partial x^2} W dx dy + \iint_S N_{0y} \frac{\partial^2 W}{\partial y^2} W dx dy \\ &= -\frac{m^2 \pi^2 b N_{0x}}{4a} - \frac{n^2 \pi^2 a N_{0y}}{4b} \end{aligned}$$

273

$$D = \iint_S h \left( \frac{\partial^2 \phi}{\partial y^2} \frac{\partial^2 W}{\partial x^2} + \frac{\partial^2 \phi}{\partial x^2} \frac{\partial^2 W}{\partial y^2} \right) W dx dy = -\frac{hm^2 n^2 \pi^4 (\alpha + \beta)}{2ab}$$

274

275

$$E = \iint_S \left( \rho_0 V \gamma_c + \rho_0 \int_0^y \frac{\partial \gamma_c}{\partial t} d\eta \right) W(x, y) dx dy$$

Substituting Equation (50) into Equation (54), yields:

276

$$A_1 \frac{d^2 T(t)}{dt^2} + B_1 \frac{dT(t)}{dt} - C_1 T(t) - DT^3(t) = 0 \quad (55)$$

277

Where:

$$A_1 = A - \rho_0 a \iint_S \left( \int_0^y a_{2j} d\eta \right) W(x, y) dx dy$$

$$B_1 = B - \rho_0 a V \iint_S \left( \int_0^y a_{1j} d\eta \right) W(x, y) dx dy - \rho_0 a V \iint_S a_{2j} W(x, y) dx dy$$

$$C_1 = C + \rho_0 a V^2 \iint_S a_{1j} W(x, y) dx dy$$

279

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The composition of equation (55) is consistent with the vibration control equation of closed membrane roof, and the solving process is consistent with the above, which is not discussed here. Thus, the expression of critical wind speed for single mode instability of open flexible membrane roof can be obtained as follows.

283

$$V_{cr} = \pi \sqrt{\frac{m^2 b N_{0x} / 4a + n^2 a N_{0y} / 4b + 3hm^2 n^2 \pi^2 (\alpha + \beta) f^2 / 8ab}{\rho_0 a \frac{ab}{MN} \sum_{j=1}^{M \times N} a_{1j} \sin \frac{m\pi x_j}{a} \sin \frac{n\pi y_j}{b}}} \quad (56)$$

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### 3. Analysis of the Effect of Geometric Nonlinearity on Critical Wind Speed

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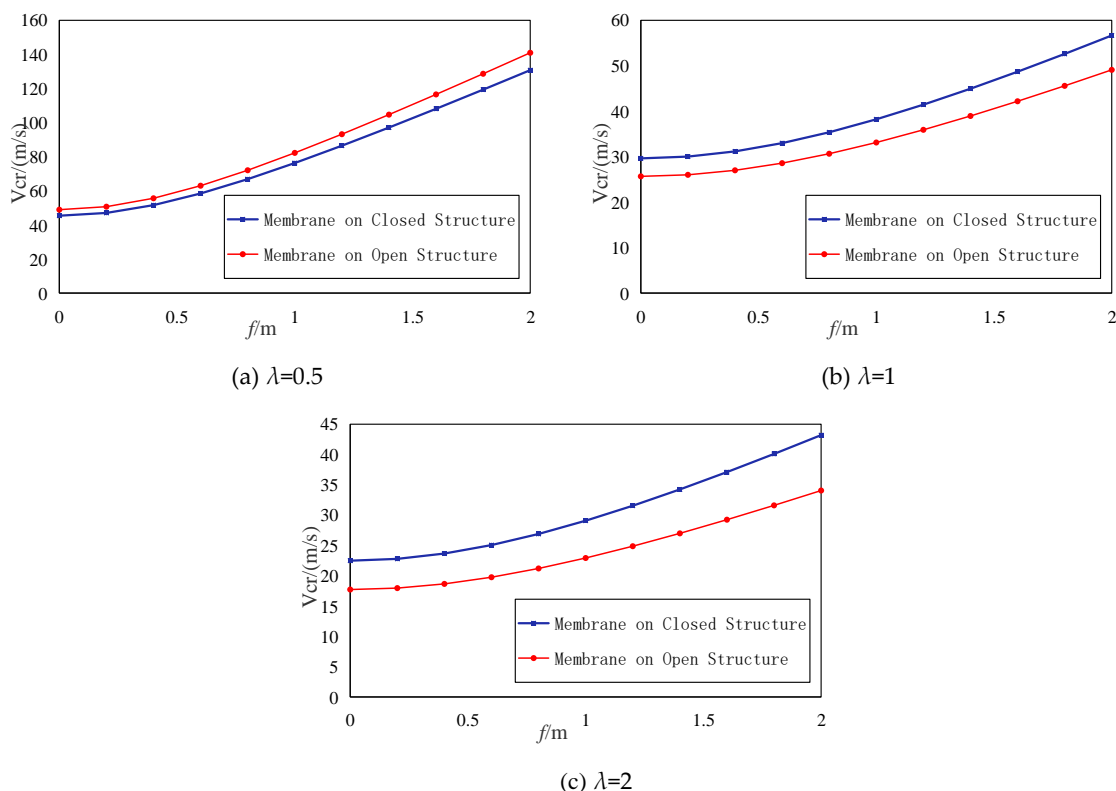
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295

Assuming that the wind speed is in the X direction,  $m = n = 1$ ,  $b = 20m$ ,  $N_{0x} = 2kN/m$ ,  $N_{0y} = 2kN/m$ . Next, the difference between the results of considering and not considering the geometrical nonlinearity of thin films is discussed, and the necessity of considering the geometrical nonlinearity of thin films in the design of such structures is given.

In this paper, the expression of critical wind speed derived from mathematics is related to the amplitude of membrane vibration. Because the geometric nonlinearity of membrane is considered, the stiffness of membrane will change with the change of amplitude in the process of vibration, which will affect the aerodynamic stability of membrane roof to a certain extent.

Defining  $\lambda$  is the ratio of cross (Y) to along (X) wind direction span ratio. The variation of critical wind speed with vibration amplitude for two types of membrane roofs with different span ratios is shown in Figure 4.



**Figure 4.** Curves of critical wind speed with amplitude for two types roof models

296

297

298

Taking membrane roof on closed structure as an example, considering the geometric nonlinearity of membrane, the critical wind speed for single mode instability of closed membrane roof is obtained by solving the equation.

299

$$V_{cr} = \sqrt{\frac{4\pi(m^2\pi^2b^2N_{0x} + n^2\pi^2a^2N_{0y}) + 3\varepsilon f^2(\rho\pi a^2b^2 + 4\rho_0ab\alpha_1)}{8\pi b\rho_0m\alpha_3}} \quad (57)$$

300

301

By substituting  $f=0$  into Equation (57), the results of critical wind speed calculation without considering geometric nonlinearity can be obtained.

302

$$V_{cr,L} = \pi\sqrt{\frac{a(m^2bN_{0x}/a + n^2aN_{0y}/b)}{2\rho_0m\alpha_3}} \quad (58)$$

303

304

305

Comparing the critical wind speed of membrane roof on closed structure with and without considering the geometric nonlinearity of membrane, the following results are obtained:

306

$$\frac{V_{cr}}{V_{cr,L}} = \sqrt{1 + \frac{3\varepsilon f^2(\rho\pi a^2b^2 + 4\rho_0ab\alpha_1)}{4\pi(m^2\pi^2b^2N_{0x} + n^2\pi^2a^2N_{0y})}} \quad (59)$$

307

308

#### 4. Discussion

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According to Equation (59), the critical wind speed of single mode instability obtained by considering the geometrical nonlinearity of membrane is larger than the linear one. It shows that the critical wind speed of instability obtained by neglecting the geometrical nonlinearity of membrane is on the small side. It is conservative for structural design. By substituting the specific values, the critical wind speed ratio  $\frac{V_{cr}}{V_{cr,L}}$  of the membrane roof on closed structure can be obtained as shown in Table 1.

314

315

Table 1. Critical wind speed ratio of membrane roof on closed structure with and without geometric nonlinearity.

	$f=0.1m$	$f=0.2m$	$f=0.4m$	$f=0.6m$	$f=0.8m$	$f=1m$	$f=1.2m$	$f=1.4m$
$\lambda=0.5$	1.00	1.04	1.14	1.29	1.47	1.68	1.90	2.14
$\lambda=1$	1.00	1.01	1.05	1.11	1.19	1.29	1.40	1.52
$\lambda=2$	1.00	1.01	1.05	1.11	1.20	1.29	1.40	1.52

316

317

318

Similarly, the critical wind speed ratio  $\frac{V_{cr}}{V_{cr,L}}$  of the membrane roof on open structure is obtained as shown in Table 2.

319

320

Table 2. Critical wind speed ratio of membrane roof on open structure with and without geometric nonlinearity.

	$f=0.1m$	$f=0.2m$	$f=0.4m$	$f=0.6m$	$f=0.8m$	$f=1m$	$f=1.2m$	$f=1.4m$
$\lambda=0.5$	1.00	1.04	1.14	1.29	1.47	1.68	1.90	2.14
$\lambda=1$	1.00	1.01	1.05	1.11	1.19	1.29	1.40	1.52
$\lambda=2$	1.00	1.01	1.05	1.11	1.20	1.29	1.40	1.52

321 From Table 1 and Table 2, it can be seen that when the cross-wind span ratio is small, the  
322 geometric nonlinearity has a greater impact on the critical wind speed of single-mode  
323 instability of membrane roof, and increases with the increase of vibration amplitude. When the  
324 span ratio is greater than 1, the effect is relatively small. Article 5.3.4 of China's Technical  
325 Regulations for Membrane Structures (CECS 158:2015)[12] stipulates that for integral tensioned  
326 and cable-supported membrane structures, the deformation of the membrane structure should  
327 not be greater than 1/200 of the span when considering the combination of wind load effects. In  
328 this example, the normal displacement is limited to 0.1m, and the critical wind speed ratio is  
329 1.00 when geometric nonlinearity is considered or not. Therefore, under normal wind loads,  
330 the influence of membrane geometric nonlinearity on the aerodynamic stability of roofs can be  
331 neglected. However, under strong wind loads, the deformation of roofs will exceed the norm  
332 limit. At this time, the influence of geometric nonlinearity should be considered.

## 333 5. Conclusions

334 In this paper, the aerodynamic stability of orthotropic rectangular planar membranes on closed  
335 and open structures is studied by mathematical analytic method. The governing equations of  
336 wind-induced nonlinear vibration of tensioned membrane roofs are established by using the theory  
337 of large deflection of membrane and Darumbell's principle. According to Bubnov-Galerkin method,  
338 the governing equations of aerodynamic coupling are transformed into second-order non-linear  
339 differential equations with constant coefficients and their periodic solutions are obtained by using  
340 the improved multi-scale method. By judging the stability of the periodic solutions of the equations,  
341 critical wind speed for single mode instability of the membrane roof considering geometric  
342 nonlinearity is obtained. The influence of geometrical nonlinearity on the critical wind speed of  
343 single-mode aeroelastic instability of membrane material is quantitatively obtained by comparing  
344 the results with those without consideration. The main conclusions can be summarized as follows:

345 Considering the geometric nonlinearity of membrane vibration, the critical wind speed of  
346 single mode instability of membrane roof increases nonlinearly with the increase of transverse  
347 vibration displacement of membrane.

348 The critical wind speed of single mode instability with the geometrical nonlinearity of  
349 membrane considered is larger than that of linear results. It shows that the critical wind speed of  
350 membrane is small when the geometrical nonlinearity of membrane is neglected. For structural  
351 design, it is conservative. When the span along the wind direction is small, the geometrical  
352 nonlinearity has a great influence on the critical wind speed of single mode instability of membrane  
353 roof, and with the amplitude of vibration. When the span ratio is greater than 1, the influence is  
354 relatively small.

355 Under normal wind loads, the influence of membrane geometric nonlinearity on the  
356 aerodynamic stability of roofs can be neglected. However, under strong wind loads, the  
357 deformation of roofs may exceed the norm limit and reach about 3% of the span, the influence of  
358 geometric nonlinearity should be considered.

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