

## Article

# Investigation of Laminar-Turbulent Transition on a Rotating Wind Turbine Blade of Multi Megawatt Class with Thermography and a Microphone Array

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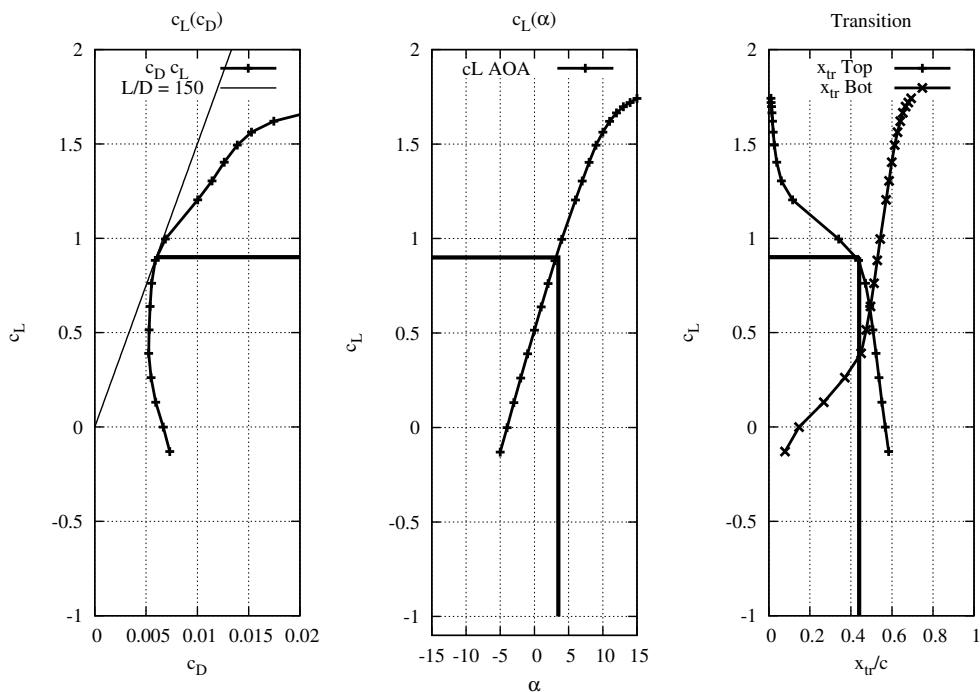
Version April 18, 2019 submitted to Energies

**Abstract:** Knowledge about laminar-turbulent transition on operating multi-megawatt wind turbine blades needs sophisticated equipment like hot-films or microphone arrays. Contrarily thermographic pictures can easily be taken from the ground and temperature differences indicate different states of the boundary layer. The accuracy however, still is an open question, so that an aerodynamic glove known from experimental research on aero-planes was used to classify the boundary-layer state of a 2 megawatt wind turbine blade operating in the northern part of Schleswig-Holstein, Germany. State-of-the-art equipment for measuring static surface pressure was used for monitoring the lift distribution. To distinguish laminar and turbulent parts of the boundary layer (suction side only) 48 microphones were applied together with ground-based thermographic cameras from two teams. Additionally, an optical camera mounted on the hub was used to survey vibrations. During start-up (from 0 to 9 rpm) extended, but irregularly shaped regions of a laminar boundary layer were observed which had the same extension measured both with microphones and Thermography. When an approximately constant rotor rotation (9 rpm corresponding to approximately 6 m/s wind-speed) was achieved, a flow transition was visible at the expected position of 40 % chord length on the rotor blade, which was fouled with dense turbulent wedges and an almost complete turbulent state on the glove was detected. In all observations, quantitative determination of the flow transition positions from thermography and microphones agree well within their accuracy.

**Keywords:** Boundary Layer Transition, Wind Turbine, Thermography, Aerodynamic Glove

## 1. Introduction

Wind energy [1] has been very successful over the last years with more than 50 GW newly added rated wind power each year. One important part in any design of wind turbines especially for wind turbine blades is aerodynamics [2]. It consists of defining chord, twist and appropriate 2D aerofoil sections. From inviscid fluid mechanics it is well-known that design rules, if formulated in terms of circulation, only give expressions for  $c \cdot c_L$  (chord times lift coefficient). As so-called profile losses can be estimated by 1.5 tip-speed-ratio / lift-to-drag [2] those with low drag are particularly interesting. From aeroplanes laminar aerofoils [3] are known and widely used to reduce fuel consumption during cruise flight and it is tempting to use these kind of aerofoils for wind turbines as well. A so-called operating-



**Figure 1.** Determination of an operating point for an aerodynamic profile. See table 1 for numbers.

28 or design-point ( $\alpha_{des}, c_{L,des}$ ) is found by constructing a tangent from the origin of the coordinate system  
 29 to the  $c_L(c_D)$  curve to reach for maximum lift over drag ( $c_L/c_D = L/D$ ), see Figs. 1. Data calculated by  
 Xfoil for aerofoil profile is shown in Fig. 1 and is summarised in Tab. 1. The aerofoil profile can be

**Table 1.** Operation point for profile at  $R = 35$

Quantity	Value	Unit
$c_{L,des}$	0.9	-
$c_{D,min}$	0.005	-
Maximum lift over drag (L/D)	150	-
angle-of-attack	3.5	deg
laminar part (suction or top side)	0.45	$x/c$

30 found at a radius of  $R = 35$  m on the investigated wind turbine rotor. Usually a newly designed aerofoil  
 31 has to be tested in a wind tunnel. See [4] for an impressive example of high-Reynolds-number wind  
 32 tunnel testing and comparison with computational fluid dynamics (CFD). Panel-codes like Xfoil as well  
 33 as genuine CFD-codes like DLR-TAU [5] were used. Here, Reynolds-number and inflow turbulence  
 34 level were varied and both effects could be well simulated. Furthermore, measured results agreed up  
 35 to few percent in lift-to-drag ratio.  
 36

37 As for an aeroplane and much more pronounced for wind turbines, ambient conditions may  
 38 differ substantially, especially with regard to inflow turbulence from wind tunnel settings. Therefore  
 39 field tests are more or less inevitable. In fact one of the authors of this paper was inspired by [6] for a  
 40 comparable experiment on wind turbine blades. In [7] a state-of-the-art version of an aerodynamic  
 41 glove used on a small aeroplane is described. Unfortunately this state of sophistication in measurement  
 42 equipment is not reached - at the moment - for wind turbine blades.

43 First investigation with emphasis on laminar-turbulent transition in the boundary layer are  
 44 reported from the Energy research Centre of the Netherlands (ECN) (now part of the Netherlands  
 45 Organisation for Applied Scientific Research (TNO) [8,9] for a 12-meter long blade. Much later the wind

46 energy department of the Danish Technical University (DTU) [10–12] investigated transition on a 38.8  
 47 m blade. A much more in-depth investigation is part of the current IEAwind Task 29 trans-European  
 48 project [13].

49 A blade of 14 meter length was investigated with respect to laminar-turbulent transition by  
 50 using of hot-films by [14–16]. Tab. 2 summarises all published measurements of conducted and  
 51 reported laminar-turbulent transition measurements in field experiments. Nevertheless, an open  
 52 question remains concerning the mechanism of transition (triggered by Tollmien-Schlichting waves  
 53 or by the so-called bypass mechanism) and the accuracy of the detection of the location. This paper  
 54 focuses on results from the most recent experiment on a 45.3 meter long blade and by combining two  
 55 different methods (thermographic imaging and microphones) we establish quantitative reliabilty of  
 56 both methods-

57 The structure is as following: Firstly we describe the site, the wind turbine used, its blade and the  
 58 measurement equipment. Then, a summary of 2D- and 3D-CFD investigations is presented. Results  
 59 are reported for pressure distribution, sound-pressure-level (SPL) in combination with thermographic  
 60 pictures. Finally a summary and some conclusions are given. The paper is finalised by our conclusions  
 61 and an outlook to possible further research.

Table 2. Summary of field experiments on wind turbine blades dedicated to investigations of laminar  
 to turbulent transition

blade length/meter	year	Reynolds number/million	source
12.05	1983	3 (estimated)	[9]
38.5	2009	3 - 5	[11]
14.5	2011	1 - 2	[15]
2.2	2014	0.5 - 1	[17]
45.3	2018	1 - 5	this paper

## 62 2. Experimental Set-Up

### 63 2.1. Aerodynamic glove

#### 64 Turbine and Site

65 The measurements were performed at a Senvion (formerly: REpower) MM92 wind turbine. This  
 66 wind turbine has a rated power of 2 MW at rated wind speed of 12.5 m/s and the hub height is 100 m.  
 67 This type of a wind turbine was firstly delivered in 2005 and since then more than 3000 units have  
 68 been manufactured. The rotor blades are of type LM45.3p. Due to a non-disclosure agreement with  
 69 the blade manufacturer LM, we obtained a Computer Aided Design (CAD) file of the blade's outer  
 70 surface which allowed CFD simulations of the blade for comparison with the measurements.

71 The test site is located at Eggebek, a former military airport, approximately 100 km to the North  
 72 of Hamburg and approximately 50 km to both the north Sea (to the west) and Baltic Sea (to the east).  
 73 According to the European Wind Atlas[18] the roughness class is 1.5. If a logarithmic wind profile  
 74 is fitted then  $z_0 = 0.055$  m has to be used as roughness length. Assuming a wind speed of 6 m/s  
 75 the difference by wind-shear from  $z_1 = 65$  m (glove at bottom position) to  $z_2 = 135$  m (glove at top  
 76 position) then is 0.38 m/s or 6.3% only. Data acquisition took place on 13 July 2018 from 9 a.m. to  
 77 approximately 1 p.m. local time.

#### 78 Overview of the Experimental Set-Up

79 The experimental set up consists of different measuring techniques. First, an aerodynamic glove  
 80 is used for measuring the pressure distribution and the acoustic properties of the boundary layer

81 together with environmental parameters like temperature, humidity and local acceleration of the glove.  
 82 In addition, a five hole probe is mounted onto the glove to determine the local flow velocity. Second,  
 83 thermographic flow visualization was applied by two teams (Deutsche WindGuard Engineering,  
 84 DWGE together with Bremer Institute for Metrology, Automation and Quality Science, BIMAQ,  
 85 and German Aerospace Center, DLR) to optically determine the laminar-turbulent transition of the  
 86 boundary layer. The thermographic set-ups are described in detail in section 2.2.0.1

87 In addition, a video camera was installed at the spinner and directed at the aerodynamic glove to  
 88 measure vibrations of the blade.

89 Furthermore, the SCADA data from the wind turbine were given to us for a complete data  
 90 set during the measurement period. All clock-times given in this article refer to local wind turbine  
 91 (SCADA) time.

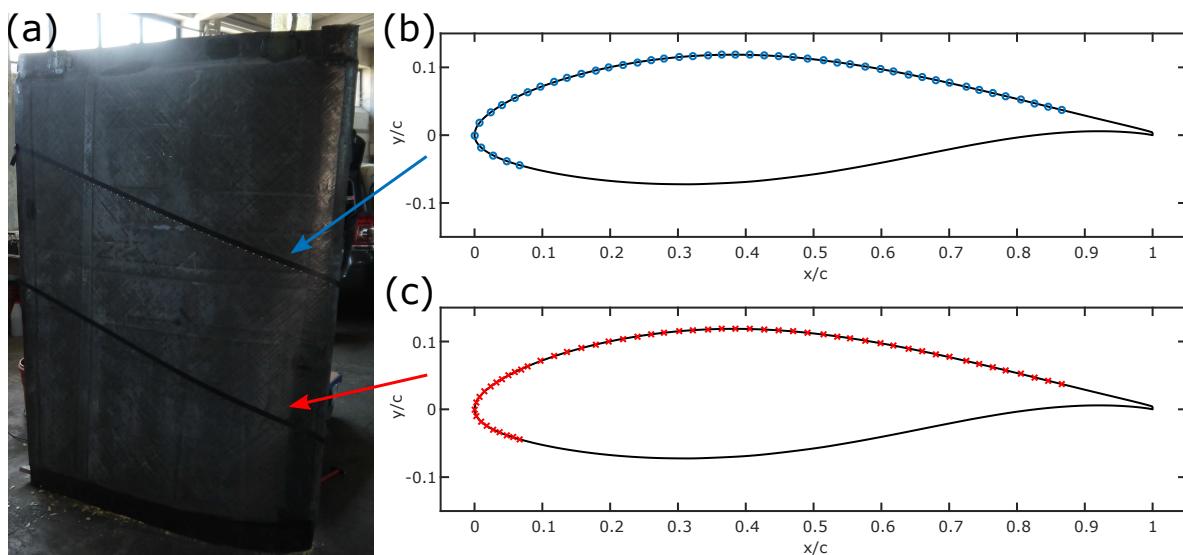
## 92 Aerodynamic glove

93 The principle of the aerodynamic glove is well known in aviation experiments for measuring the  
 94 local flow conditions around the wings of an airplane [6]. For wind turbine measurements, only a few  
 95 experiments were conducted [10,11,16,19,20]. For this study, a local company that is specialized in  
 96 wind turbine maintenance, was commissioned to build the outer shell of the glove.

97 The surface consists of carbon fiber fabrics, laminated onto 20 mm thick foam plates that were  
 98 aligned to a blade that is identical to the investigated one to ensure the best fit possible. The foam plates  
 99 allow the installation of tubes and cables in trenches below the glove's surface without damaging the  
 100 blade while maintaining the blade's profile.

101 In order to determine the local condition of the boundary layer, i.e., laminar or turbulent, 48  
 102 microphones were equally spaced along a straight line with an inclination of 20 degrees with respect  
 103 to the chord to minimize the influence of microphones upstream. Four microphones were positioned  
 104 on the pressure side, one at the leading edge of the glove and the remaining 43 on the suction side.

105 For the determination of the pressure distribution along the suction side, 57 pressure holes were  
 106 drilled into the glove. 48 holes were distributed exactly like the microphones and the remaining  
 107 nine holes were drilled at the center between two holes at the front of the glove resulting in a higher  
 108 spatial resolution around the leading edge of the glove. For the acquisition of the pressure data a  
 109 DTC (Digital Temperature Compensation) Initium from Esterline Pressure System was used. Fig. 2  
 shows the distribution of measurement points along the profile of the glove. The local flow velocity is



110 **Figure 2.** Position of sensors: (a) positions on the glove; (b) position of microphones; (c) positions of  
 pressure holes

111 measured by a five hole probe. The data is also recorded by the Initium system. The five hole probe  
 112 was calibrated in the wind tunnel of the University of Oldenburg, Germany.

113 For the data acquisition, a fan-less computer was used. It was equipped with a wifi access point to  
 114 allow control of the system from the ground. The microphone data was recorded at 25 kHz and stored  
 115 locally on four solid-state disks (SSD). Acceleration, temperature and humidity were also recorded  
 116 at 25 kHz and stored on a separate SSD. The Initium data was recorded at 250 Hz and stored on the  
 117 internal SSD of the computer via a LabVIEW program. This program was also used to synchronize all  
 118 measurements. The power supply was realized by a battery with 24 V and 30 Ah. Fig. 3 shows the  
 119 data acquisition system mounted on an aluminum plate ready to be mounted onto the pressure side of  
 120 the glove. Obviously, there is an influence of the box on the pressure side on the flow of the suction  
 121 which was investigated. Using 2D and 3D CFD (see section 3), we were able to investigate the different  
 122 pressure distribution on the suction side. The differences seen there were within the accuracy of the  
 observed values, see Fig. 11.

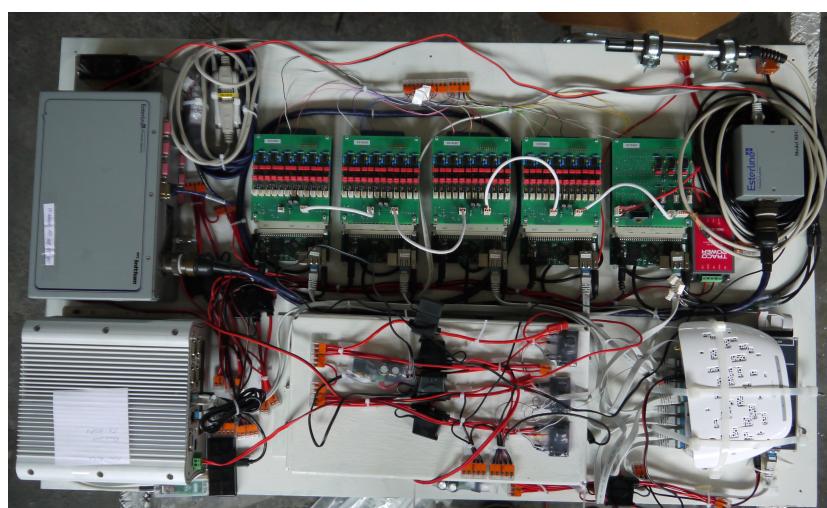


Figure 3. data acquisition system. Length about 1 meter

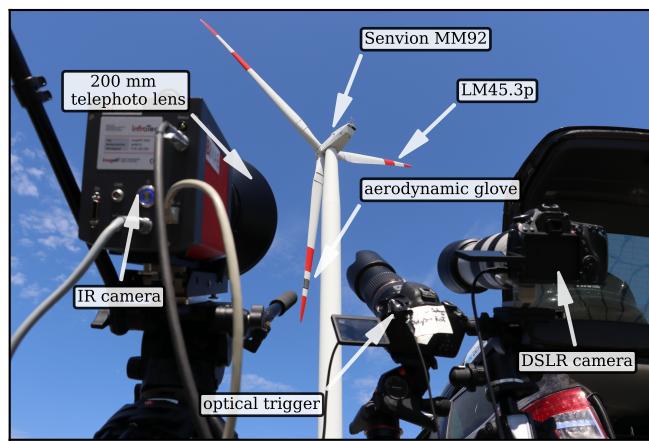
123

## 124 2.2. Thermography

125 A thermographic measurement of the rotor blade surface enables a non-invasive spatial resolved  
 126 determination of the boundary layer state. The thermographic flow visualization relies on an initial  
 127 temperature difference between the rotor blade and the flow. The rotor blade surface temperature  
 128 depends on the local heat transfer coefficient which is a function of the surface friction in the different  
 129 areas of the boundary layer flow [21,22]. As a result, the surface areas with a laminar boundary  
 130 layer flow appear warmer than surface areas with a turbulent boundary layer flow. The transition  
 131 from laminar to turbulent flow can be determined by a steep change in surface temperature. Image  
 132 processing algorithms enable the localization of the laminar-turbulent transition and any existing flow  
 133 separations [23–25].

134 Thermographic measurements have been established in wind tunnel experiments for decades to  
 135 visualize the boundary layer state in ultra- and hyper-sonic as well as in trans- and subsonic flows  
 136 [26]. In wind tunnel experiments the position of the laminar-turbulent transition as well as areas with  
 137 laminar or turbulent flow separation can be visualized [27,28]. Outside the wind tunnel, thermographic  
 138 measurements for the localization of the laminar-turbulent transition in flight experiments on aircraft  
 139 wings [29], helicopter rotors [30] and wind turbine rotor blades in operation [31] are reported.

140 In this study different thermographic cameras were used in order to determine the  
 141 laminar-turbulent transition on the aerodynamic glove and to assign the position to the  
 142 three-dimensional geometry. This allows a comparison with the other sensors by chord-based position  
 143 information.



**Figure 4.** Experimental set-up for the thermographic flow visualization measurements. In the foreground the thermographic camera with the 200 mm telephoto lens, the optical trigger and a DSLR-camera for documentation purposes are shown.

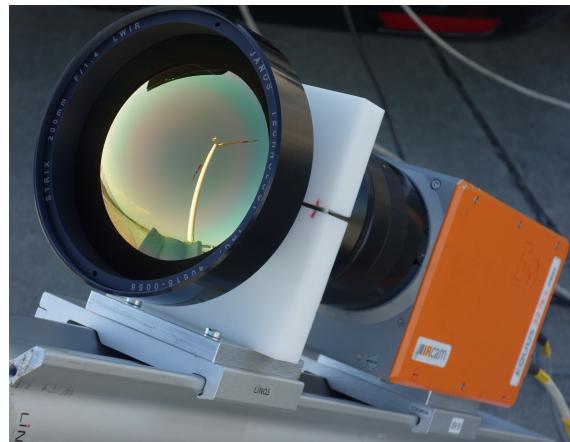
**144** Thermographic set-up provided by BIMAQ and DWGE

**145** The thermographic measurements were acquired with a cooled thermographic camera with  
**146** a global shutter (snap-shot detector) from the manufacturer InfraTec. The detector is an  
**147** InSb focal plane array with  $640 \times 512$  pixels which is sensitive between 2 and  $5 \mu\text{m}$  (model ImageIR  
**148** 8300). The dynamic range of the camera is 14 bit and the integration time can be set between 1  
**149** and 20,000  $\mu\text{s}$ . InfraTec specifies the noise equivalent temperature difference (NETD) as less than  
**150** 25 mK @  $30^\circ\text{C}$  after a standard calibration routine with measurements of a black body radiator at  
**151** different temperatures. For the presented measurements a one point non-uniformity correction is  
**152** performed before the measurements. A 200 mm telephoto lens with an instantaneous field of view (IFOV)  
**153** of 0.08 mrad is used. This results in a spatial resolution 11.2 mm for the given measurement distance of  
**154** 140 m. The thermographic camera is triggered by an optical trigger mechanism to enable the acquisition  
**155** of an image every time one of the blades is passing the camera's field of view. The post-processing of the  
**156** images is done in Python. Fig. 4 shows the experimental set-up for the thermographic measurements  
**157** at the Senvion wind turbine.

**158** The signal processing for the localization of the laminar-turbulent is based on the approximation  
**159** of the chord wise temperature profile with a Gaussian error function, which results in a determination  
**160** of the transition position with subpixel accuracy [24]. By taking into account the position of the  
**161** aerodynamic glove in relation to the thermographic camera, the visible surface area can be assigned  
**162** to its geometry. With the known measurement distance and the local rotor blade angle, a series of  
**163** coordinate transformations is carried out to enable the determination of the angle of aperture that  
**164** results in the imaging of the investigated surface in the thermographic image. The consideration of the  
**165** position of the transition in the image plane of the thermographic image allows the determination of  
**166** the transition position on the surface of the aerodynamic glove and subsequently the projection of this  
**167** position onto the rotor blade chord [31].

**168** Thermographic setup provided by DLR

**169** The second infrared camera set-up was provided by the German Aerospace Center (DLR) and  
**170** installed at a sufficient distance of approx. 80-85 m downstream of the wind-turbine mast to have  
**171** an unobstructed view of the suction side of the aerodynamic glove at the minimum height above  
**172** the ground level (Fig. 5). Thermal images taken shortly before the blade disappears behind the mast  
**173** offer the best possible spatial resolution at given test conditions. Moreover, any distracting sunshine  
**174** reflections can simply be avoided by this way due to nearly vertical blade alignment.



**Figure 5.** Thermographic setup provided by DLR: thermographic *IRCAM Equus 327k L MCT* camera equipped with a *STRIX 200mm LWIR* telephoto lens

175 The measurements were conducted with the *IRCAM Equus 327k L MCT* camera, which has a  
 176 spectral range of 8.0–9.4  $\mu\text{m}$  detected by a Cadmium-Mercury-Telluride FPA-detector with  $640 \times 512$   
 177 pixels (pixel pitch:  $24\mu\text{m}$ ). The nominal exposure time of  $100\ \mu\text{s}$  at highest frame rate of 105 Hz in  
 178 free-running mode was applied without image triggering or averaging. To improve spatial resolution  
 179 an infrared *STRIX 200 mm LWIR* telephoto lens with a focal length of 200mm and a lens aperture of  
 180 F/1.4 was used enabling a maximum resolution of approx. 125 pixel per chord width ( $= 83.3\ \text{px}/\text{m}$  or  
 181  $12\ \text{mm}/\text{px}$ ). This spatial resolution is marginally satisfactory for transition detection. A higher spatial  
 182 resolution is simply invaluable at these long distances, because the prices for IR lenses are increasing  
 183 exponentially with the focal length. The temperature difference between the glove surface and the flow  
 184 needed for the transition detection was facilitated by naturally preheating the blades in the sunshine.

185 *2.3. Optical camera*

186 At the spinner of the WT, a video camera was mounted and directed towards the pressure side of  
 187 the aerodynamic glove. Due to the curvature of the blade, it was not possible to observe the suction  
 188 side of the glove. The camera was intended to visually detect vibrations of the blade, which in turn  
 189 could be correlated to oscillations of the local acceleration. Potential disturbance from precipitation,  
 190 insects or birds could be observed. Heating up of the blade and the aerodynamic glove from varying  
 191 sun radiation could be assessed, also in the blade positions that are out of sight for the stationary  
 192 thermographic cameras. The optical camera is a JVC GZ-RX115BE camcorder. The camera was  
 193 mounted onto the spinner with a rack that keeps the camera firmly in place. Contained in this rack  
 194 is also an additional battery pack that allows about 24 hours of continuous operation. The camera  
 195 is not remotely controlled. Instead the recording is started before the WT is started and the camera  
 196 records during the whole test. The optical camera records with a rate of 50 frames per second. The  
 197 resolution of the pictures is  $1920 \times 1080$  pixels. Fig. 6 shows that in standstill of the WT (pitch angle  
 198 approx. 90 deg) the camera points towards the pressure side of the blade. In operation at low wind  
 199 speeds (pitch angle approx. 0 deg) the camera points towards the trailing edge and the pressure side  
 200 of the blade. At low pitch angles the spoiler on the pressure side of the blade root becomes visible.  
 201 Since the blade is flexible the view varies with the out of plane bending of the blade (driven by thrust  
 202 forces) and the in-plane bending of the blade (mainly driven by the gravitational force, i.e. determined  
 203 by the rotor position). The optical camera has a focal length range from 2.9 mm to 116 mm. Therefore,  
 204 the aerodynamic glove could have been observed a lot closer. However, the intention was to observe  
 205 also the blade around the aerodynamic glove. The high resolution of the pictures allows zooming into  
 206 the videos if necessary for assessing certain scenarios, for example during start-up phase.



**Figure 6.** View from the optical camera on the spinner to the pressure side of the blade at  $R = 35$  m. The black box contains all the data recording system including power supply. Chord there is 1.5 m. In addition, orientation of the 5-hole-probe relative to vertical axis (approx. rotor plane) indicates the instantaneous pitch to be compared with values from the SCADA system. Left: WT in standstill, right: WT in operation.

### 207 3. Computational Fluid Dynamics

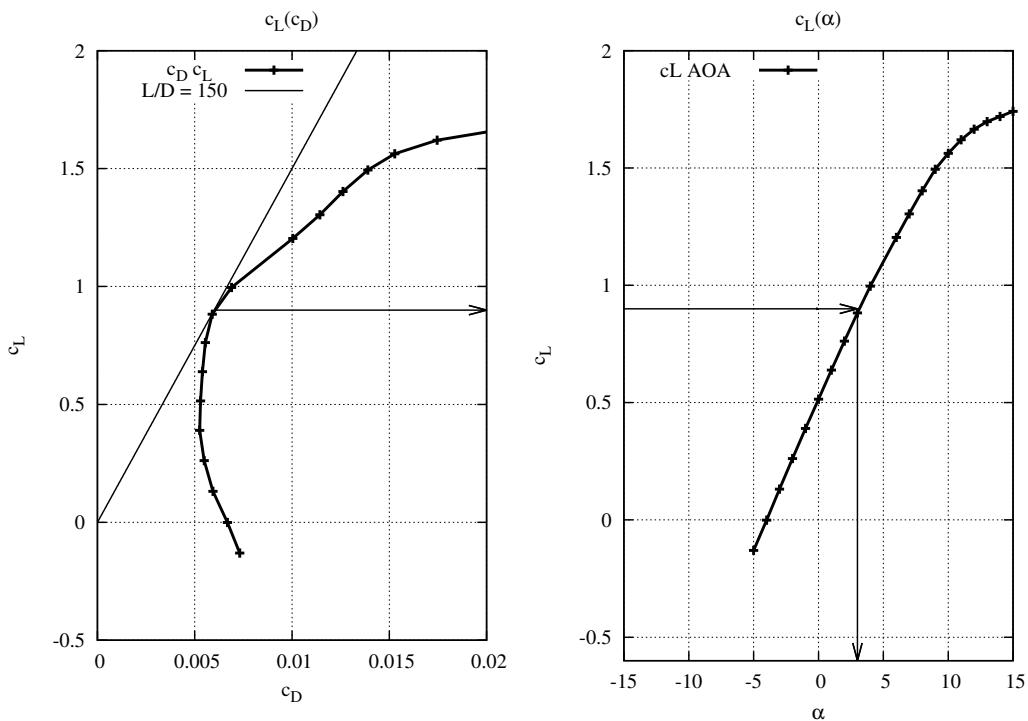
Parallel to the preparation of the experiments, 2D and 3D CFD simulations were performed. First, a 2D simulation was performed using XFOIL [32]. In Fig. 7 the resulting Lilienthal polar of that section is shown. A so-called laminar nose, a region of small drag,  $c_D < 0.008$ , is clearly visible. Using a tangent, the operating point of the airfoil ( $\alpha_{des}, c_{L,des}$ ) is determined at the largest lift-to-drag ratio  $c_{L,des} / c_{D,des}$ . Fig. 8 and 9, right side show the (2D-)pressure distribution on the surface of the profile for an angle of attack of  $\alpha_{des} = 3.5^\circ$  at operating conditions. It is clearly seen that no flow separation is present there. It has to be noted, that for every CFD calculation including laminar to turbulent transition an additionally quantity, the N-factor has to be provided. From wind tunnel measurements it is common practice to relate N to the inflow turbulence TI by Mack's relation

$$N = 2.13 - 6.18 \cdot \log_{10}(TI) . \quad (1)$$

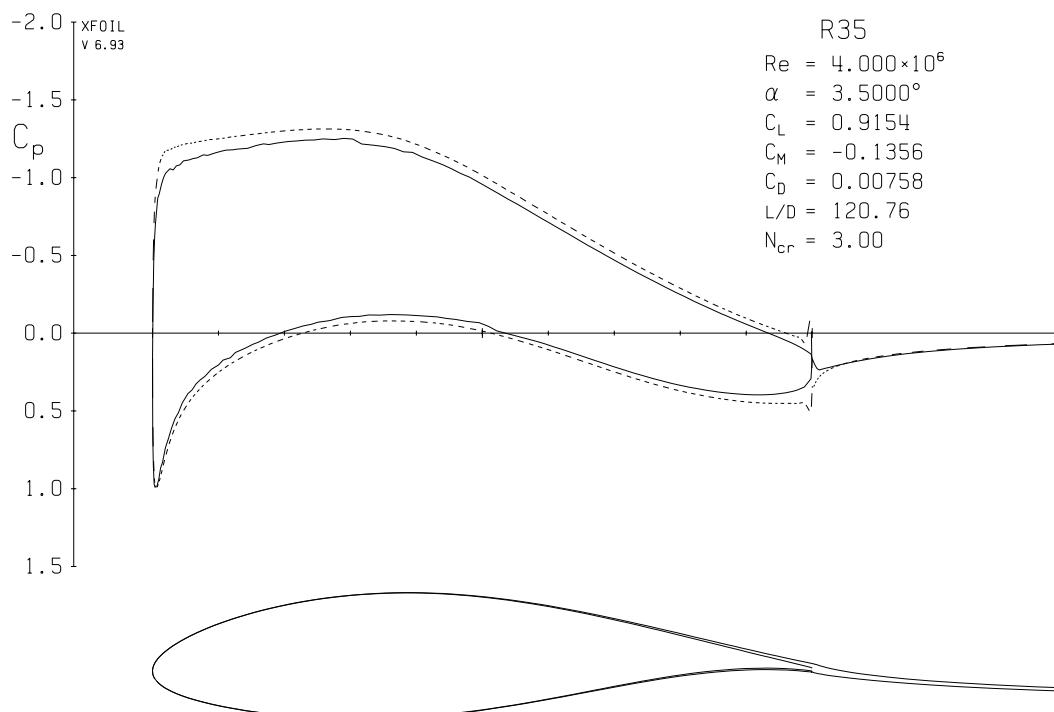
208 Mack [33,34] recommended to use it in the region  $0.1\% < TI < 1\%$ . Here,  $TI$  comes from  $TI =$   
 209  $\sigma/\bar{v} = (\int v^2 P(v) dv)^{1/2} / \int v P(v) dv$ , or in words: standard deviation divided by average wind speed  
 210 for some ensemble. Note that Eq. 1 cannot be used in atmospheric flow without further assumptions.  
 211 A procedure how to extend Mack's correlation, was proposed in [20].

212 Furthermore, full 3D simulations were used to estimate the position of transition from laminar to  
 213 turbulent flow in the boundary layer on the suction side of the glove. Fig. 9, left shows the expected  
 214 flow state regions obtained via DLR-TAU code [5], a CFD code by the German Aerospace Center. Blue  
 215 corresponds to laminar flow, red indicates a fully turbulent flow. A transition line is seen around  
 216  $x/c = 0.3$  ( $c$  = total chord = 1.5 m) in the central region of the glove. Due to boundary effects transition  
 217 is forced more to the leading edge at the border of the glove.

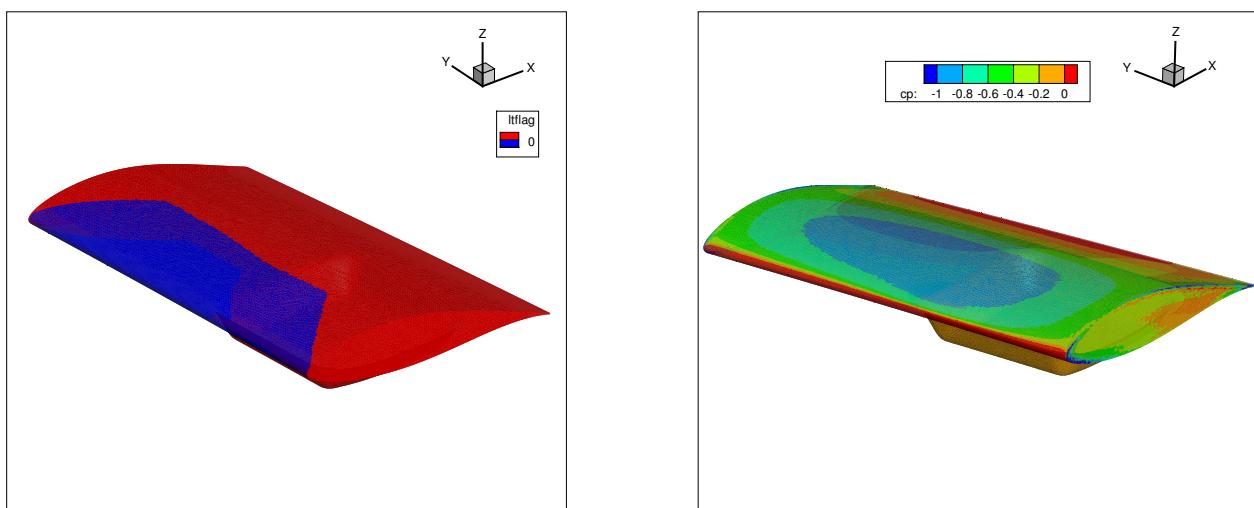
218 In addition, the perturbation of the flow due to the box on the lower side of the glove was  
 219 investigated and has been seen a posterior to be within the variation due to surface imperfections.  
 220 Fig. 9, right shows the 3D-pressure distribution. Again, as for transition, the central part, where the  
 221 pressure sensors and the microphones had been placed, smooth variations as expected from 2D are  
 222 visible. From these investigation we can conclude that even under ideal conditions a laminar extension  
 223 of the suction side not longer than 40 % can occur.



**Figure 7.** Lilenthal-polar of the profile at the location of the glove (Reynolds number 4 million). A so-called *laminar nose*, a region of small drag  $c_d < 0.008$ , is clearly visible. Arrows indicate determination of operation point using a tangent. The N-factor was chosen arbitrarily to 3. By Mack's correlation this corresponds to an inflow turbulence of 0.7%.



**Figure 8.** Static pressure on profile from Fig. 7 using Xfoil. AOA  $3.5^\circ$



**Figure 9.** Left side: Estimated flow state regions from 3D transitional CFD (DFL-TAU code). Blue: laminar flow, red: fully turbulent flow. Right side: Static pressure coefficient.

## 224 4. Measurement results

### 225 4.1. Measurements

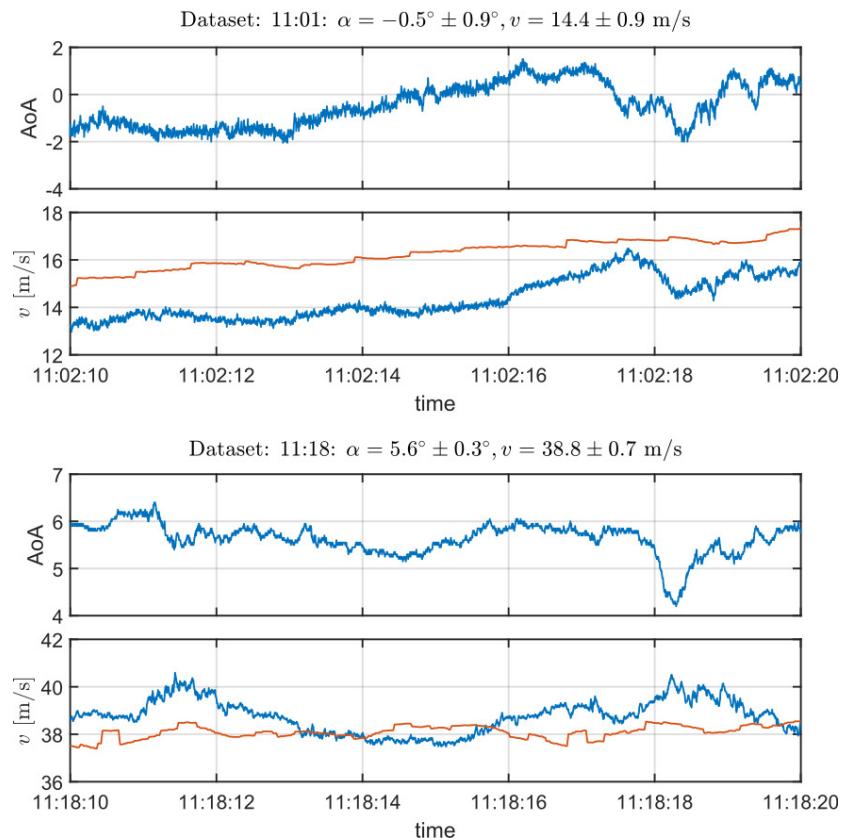
226 Now, the results are discussed in more detail in order to present a complete picture of the  
 227 aerodynamic properties of the boundary layer flow around the glove.

228 Two different data sets are compared, the start-up (SU) of the wind turbine (dataset 11:01) and  
 229 the steady-state (SS) of the wind turbine (dataset 11:18). These two data sets were chosen exemplarily  
 230 to illustrate the different operation states. All measurements were performed on a single sunny day  
 231 over a time period of about four hours. First, six sequences, each 60 seconds long, were recorded  
 232 in a stand-still configuration with the brakes of the turbine tightened. After that, the brakes were  
 233 loosened and the turbine was started. 31 sequences with 60 seconds length and a delay of 30 seconds  
 234 in between were recorded. During the measurements the turbine was stopped and started again in  
 235 order to measure the start-up of the turbine as well as the steady-state. The goal was to find differences  
 236 between the different operation states to study the temporal evolution of the boundary flow conditions.

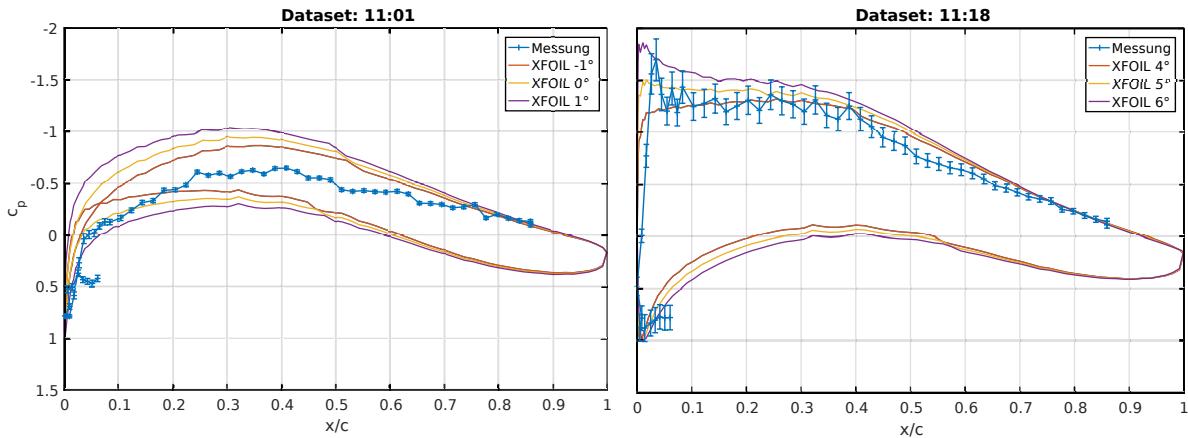
237 While the measurement system in the glove recorded data continuously, the thermography  
 238 cameras recorded stationary images of the glove at a fixed position of the blade's revolution.

### 239 4.2. Inflow

240 First, the inflow obtained by the five hole probe is discussed. Due to the calibration of the probe in  
 241 the wind tunnel of the University of Oldenburg, Germany, it was possible to obtain the inflow velocity  
 242 together with the angle of attack (AOA). Fig. 10 shows both quantities of the flow for the different  
 operation states, respectively. It can be seen that for SU the AOA is increasing from negative to positive



243 **Figure 10.** Inflow properties of different operation states. Top: start-up, bottom: steady-state. Shown  
 is AOA (upper row) and inflow velocity (lower row). Blue: velocity obtained by five hole probe, red:  
 velocity obtained by SCADA data.



**Figure 11.** Pressure ( $c_p$ ) distribution on the glove. Left: Start-up; right: steady-state.

values as expected. In addition, an offset between the inflow velocity obtained by the probe (blue) and by SCADA data (red) is seen which is explained by the different locations where the wind speed is measured.

For SS, the variation of the AOA is much smaller and no drift is observed. The offset between the inflow velocities is no longer observed. Furthermore, the inflow velocity varies along the revolution of the blade which is a direct confirmation of the atmospheric layering.

#### 4.3. Pressure distribution

Furthermore, the pressure distribution on the glove was measured. For a better comparison, the same 10 s sequence as in Fig. 10 was used. The recorded pressure was averaged and using

$$c_p = \frac{2(p - p_0)}{\rho v^2} \quad (2)$$

the lift coefficient  $c_p$  was calculated. Fig. 11 shows the resulting distribution of  $c_p$  for both operating states. For comparison, the CFD results of XFOIL calculations with different AOA are also shown.

For SU, a poor agreement is found which might result from the transient character of the sequence. AOA increases from  $-1^\circ$  to  $+2^\circ$  which makes a comparison with a stationary AOA somewhat difficult.

For SS, a better agreement between measurement and CFD is observed. The measured AOA of the five hole probe of  $5.6^\circ$  (cf. Fig. 10) is in good agreement with the observed  $c_p$  distribution.

It is worth noting that for both states the pressure distribution on the pressure side of the glove differs from the CFD calculations. Since the pressure distribution is strongly dependent on the local curvature at the leading edge, small deviations of the glove's shape result in large changes of the lift coefficient.

Too high production costs for a glove with wind tunnel (or even commercial blade) quality prevented a smoother surface. Hence, part of the observed deviations might be explained by insufficient surface quality.

#### 4.4. Detection of transition position

In addition to the pressure distribution the determination of the flow character of the boundary layer was a main goal of this project. Therefore, the noise emission on the suction side was recorded by the 48 microphones.

268 The determination of the transition position was performed as described in Ref. [10]. Firstly, the  
 269 power spectral density of the pressure signal from the microphones  $\widetilde{p(\omega)}$  was calculated from the  
 270 time-series  $p(t)$  by applying

$$\widetilde{p(\omega)} := \frac{1}{2\pi} \int_{-\infty}^{+\infty} r_{pp}(t) \cdot e^{-i\omega t} dt, \quad (3)$$

$$r_{pp}(t) = \lim_{T \rightarrow \infty} \frac{1}{2T} \int_{-T}^{+T} p(\tau)p(\tau + t) \cdot d\tau. \quad (4)$$

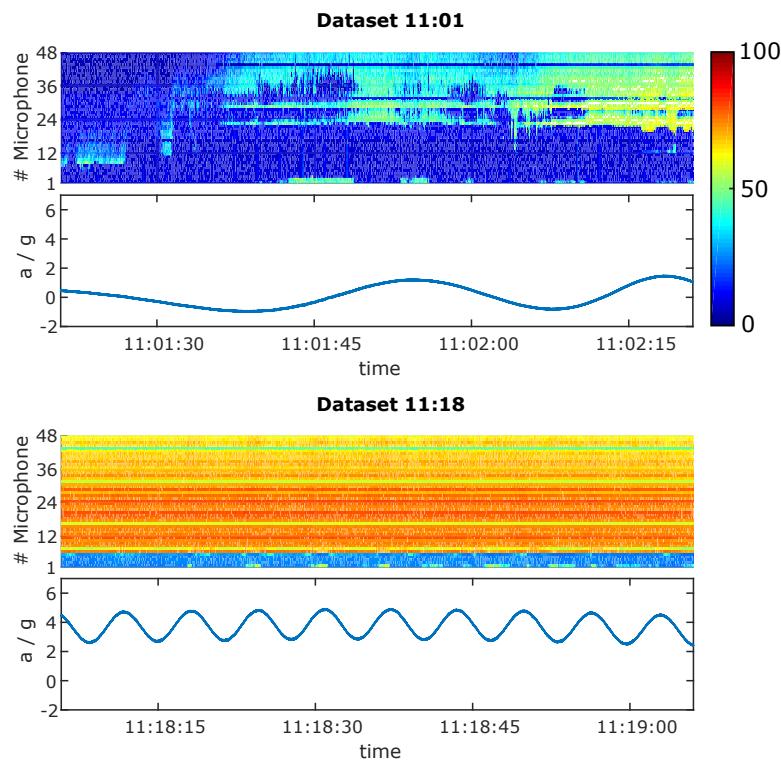
270 assuming a real time signal. Furthermore, [10] used only a finite part of the spectrum

$$p_{cut}^2 = 2 \int_{\omega_1}^{\omega_2} \widetilde{p(\omega)} d\omega \quad (5)$$

with  $[f_1; f_2] = [2 \text{ kHz}; 6 \text{ kHz}]$ ,  $\omega = 2\pi f$  and  $p_{ref} = 20 \mu\text{Pa}$ . The relevant sound pressure level (SPL)  $L_p^{cut}$  then is determined by

$$L_p^{cut} = 20 \log_{10} \left( \frac{P_{cut}}{p_{ref}} \right) (\text{dB}). \quad (6)$$

271 By definition SPL is given in pseudo-units of dB (Decibel). In order to study the temporal evolution of  
 272 the SPL, sequences of 2048 data points were averaged and the SPL for this sequence was calculated  
 273 resulting a time resolution of 12 Hz. Fig. 12 shows the temporal evolution of the SPL for both operating  
 states, together with the local radially outwards directed acceleration.



**Figure 12.** Detection of the transition position. Top: Start-up; bottom: steady-state. Upper row: sound pressure level in dB, Color map: Dark blue correspond to 0 dB and red to 80 dB. lower row: local acceleration.

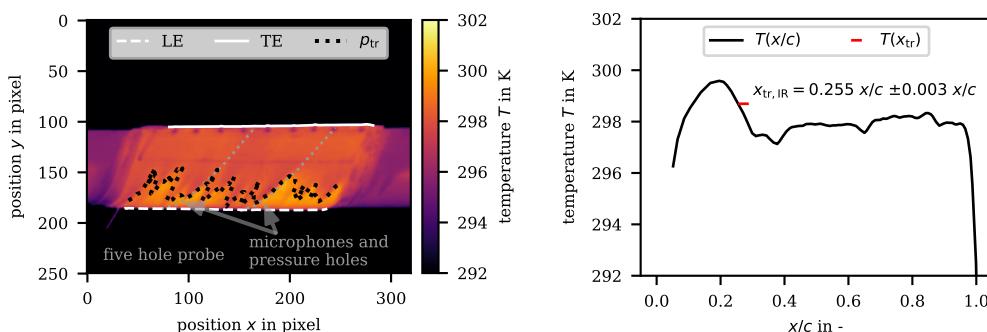
274 The color map changes from blue via green and yellow to red with increasing values of SPL. The  
 275 overall value of the SPL is lower in the SU than in the SS. Furthermore, areas with higher SPL can be  
 276 found at the back part of the suction side that travel in the direction of the leading edge.

277 In the SS, the overall value of SPL is higher. In addition, a distinct region at the leading edge can  
 278 be recognized with much lower values of SPL. This is a characteristic of a laminar boundary layer  
 279 while most of the suction side has a turbulent boundary layer resulting in high values of SPL.

280 Furthermore, a small periodic variation of the transition position can be observed. Due to the  
 281 periodic variation of the local acceleration the blade position relative to the ground can be determined.  
 282 At the lowest point of the revolution gravity and centrifugal force are pointing in the same direction,  
 283 resulting in a maximum acceleration. At this position, the transition position is slightly moving in  
 284 direction of the leading edge. Since the blade is passing through the wind shadow of the tower a  
 285 higher inflow turbulence is expected, thus resulting in a more turbulent boundary layer.

286 *4.5. Thermography Team BIMAQ and DWGE*

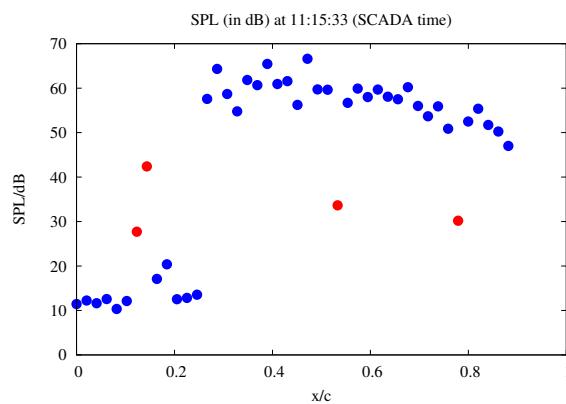
287 As an example the results from 11:15:32 and 10:16:45 (SCADA time) are shown. Fig. 13 shows the  
 288 transition position derived by the image processing algorithm from Dollinger et al. [24] for the start up  
 289 phase of the wind turbine. On the left side the thermographic image with the visible leading (LE) and  
 290 trailing edge (TE) as well as the derived relative transition position  $p_{tr}$  are shown. On the right side,  
 291 the surface temperature  $T$  of the glove at the location of the microphones is shown as a function of the  
 292 position on the chord  $x/c$  determined by a previously performed geometric assignment [31]. The span  
 293 wise average transition position along the glove is  $\bar{x}_{tr, IR}/c = 0.337$  with large local fluctuations due to  
 294 turbulence wedges as a result of the high surface roughness of the aerodynamic glove. Locally at the  
 295 location of the microphones the transition position determined by the thermographic flow visualization  
 296 is  $x_{tr, IR}/c = 0.255 \pm 0.003$ . The uncertainty of the measurement is estimated according to [31]. This  
 297 result is consistent to the transition location deduced by the microphone data  $x_{tr, M} = 0.26 \pm 0.01$   
 (cf. Fig. 14). In Fig. 15 the results for the turbine in regular operation mode are shown. Again, on



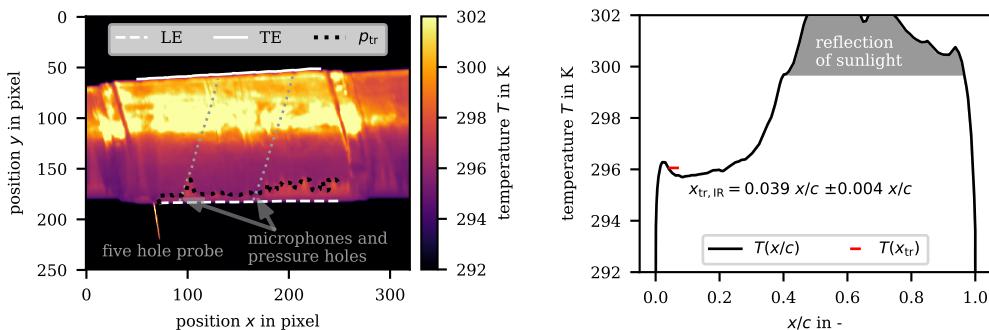
298 **Figure 13.** Thermographic flow visualization measurement at 11:15:32 (SCADA time)

299 the left the thermographic image of the glove is shown and on the right the surface temperature  $T$  is  
 300 depicted as a function of the chord wise position  $x/c$ . Beside the relative position of the transition in  
 301 the image  $p_{tr}$ , the visible leading and trailing edge are shown. Due to a reflection of sunlight on the  
 302 surface of the glove, the area towards the trailing edge appears very warm. This area is not included in  
 303 the image processing for the evaluation of the transition position. The span wise average transition  
 304 position for this measurement is  $\bar{x}_{tr, IR}/c = 0.048$  and the transition position at the location of the  
 305 microphones is  $x_{tr, IR}/c = 0.039 \pm 0.004$ . From microphone data we here get a slightly larger value of  
 306  $x_{tr, M}/c = 0.05 \pm 0.01$ . Note that the spacing of the microphones is 0.02 (in units of  $x/c$ ).

307 Obviously, the rough surface leads to a premature laminar-turbulent transition near the leading  
 308 edge during these operational conditions.



**Figure 14.** Development of SPL (in dB) vs chord at 11:15:32 (SCADA time). If one discards outliers (in red)  $x_{tr,M}/c$  may be deduced to  $x_{tr,M}/c = 0.26 \pm 0.01$



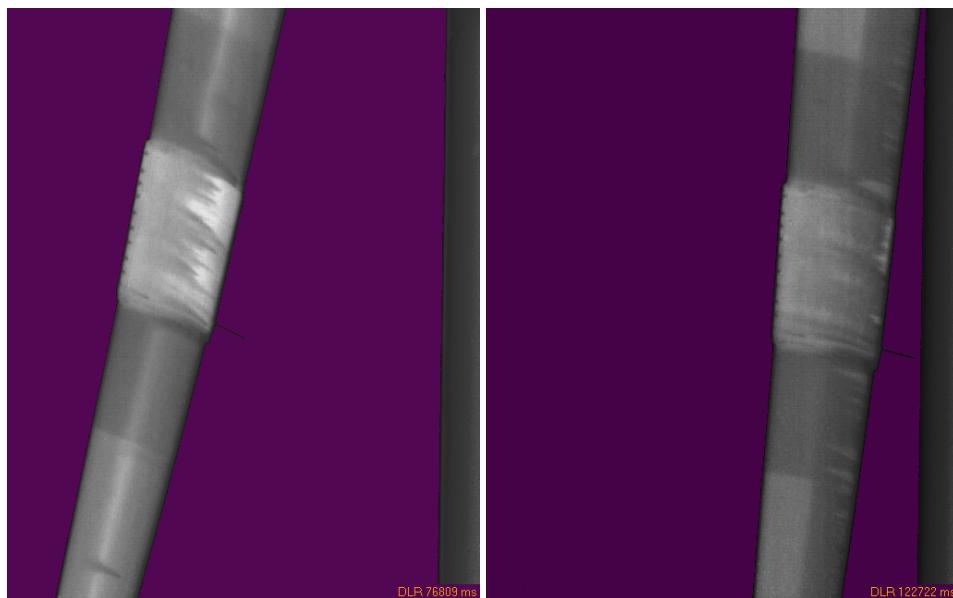
**Figure 15.** Thermographic flow visualization measurement at 10:16:45 (SCADA time)

#### 4.6. Results Team DLR

As described above, the detection of boundary layer transition on rotating blades relies on the increase in the heat transfer rate between the flow and the blade surface that occurs when the boundary layer state changes from laminar to turbulent. The change in the heat transfer rate causes differences in the blade surface temperature that was measured by an infrared camera.

Thus, the transition detection was based on the analysis of single instantaneous thermal images of the rotating blade in the start up phase, indicating different stages of surface-temperature evolution from the initially quasi uniform to a final locally-equilibrium distribution. The image pre-processing includes here the bad-pixels elimination, the sky-background separation and the contrast enhancement. Fig. 16 shows two typical thermal images obtained correspondingly at 11:22:27 and 11:23:13 (SCADA time) during a single wind-turbine run preprocessed accordingly. The rotation direction is here from left to right and the cold-sky background is colored maroon to enhance the visibility of the wind-turbine parts presented in graduations of gray. The vertical gray-scaled area on the right side of the picture represents the mast of the facility. The areas with laminar (warm/bright) and turbulent (cold/dark) boundary layers on the glove surface can be clearly distinguished in these pictures. The variations of the rotation rate (from 4.0 rpm to 8.5 rpm) and of the blade pitch angle (from 20 degrees down to 1 degree) taking place between these both time instances during the start up phase causes the shifting of the transition region in time documented thereby.

In the next step the DLR software *HeatFIT* was used for the final conditioning and automatic marker-based 3D-reconstruction of the thermal images with subsequently mapping of the data values onto the 3D grid of the aerodynamic glove (Fig. 17). The geometric information necessary for the markers is provided by the CFD-model. A full computerized analysis of the temperature distribution on the test surface in accordance with the method introduced by Schülein [35] allowed the distinction



**Figure 16.** Two typical thermal images obtained by DLR-setup after image processing including bad-pixels elimination, sky-background replacing and contrast enhancement. Images are from 11:22:27 (left) and 11:23:13 (right, both SCADA time). The left picture shows more laminar parts during startup and the right one only very small extended regions during constant RPM (approx. 9) operation mode.

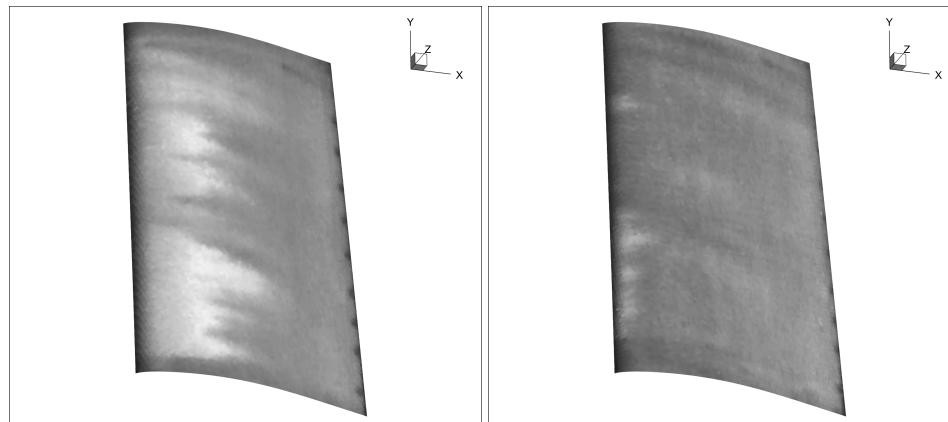
332 between the starting points and the end points of the transitional region. Additionally, in a simplified  
 333 approach the transition zone between laminar and turbulent flows was detected as a border line,  
 334 which corresponds to the locations with the highest wall-temperature gradients (highest gradients  
 335 of the thermal image intensity) measured in streamline direction (areas with detected laminar flow  
 336 are highlighted in yellow in Fig. 18). Fig. 19 shows a typical surface temperature distribution (here  
 337 thermal image intensity) over the normalized coordinate  $x/c$  extracted from the left thermal image  
 338 in Fig. 18 along the midline of the glove. This example demonstrates the mutual positions of points  
 339 characterizing transition zone: onset transition, end of transition and highest temperature gradient. As  
 340 described in Ref. [30] and seen in the example presented, the transition line detected in accordance  
 341 with the highest-gradient criterion corresponds approximately to the midpoint of the transition zone.  
 342 A comprehensive characterization of the transition zone facilitates comparison with results obtained  
 343 independently by different experimental methods and numerical simulations. The actual midpoints of  
 344 the transition zone detected in both thermal images along the microphone-array line were found to be  
 345 at  $x_{tr,IR}/c = 0.370 \pm 0.005$  and  $0.080 \pm 0.005$ , being very close to the transition locations detected by  
 346 microphone-array method ( $x_{tr,m}/c = 0.34 \pm 0.01$ , see Fig. 20 and  $0.07 \pm 0.01$ ).

#### 347 4.7. Discussion

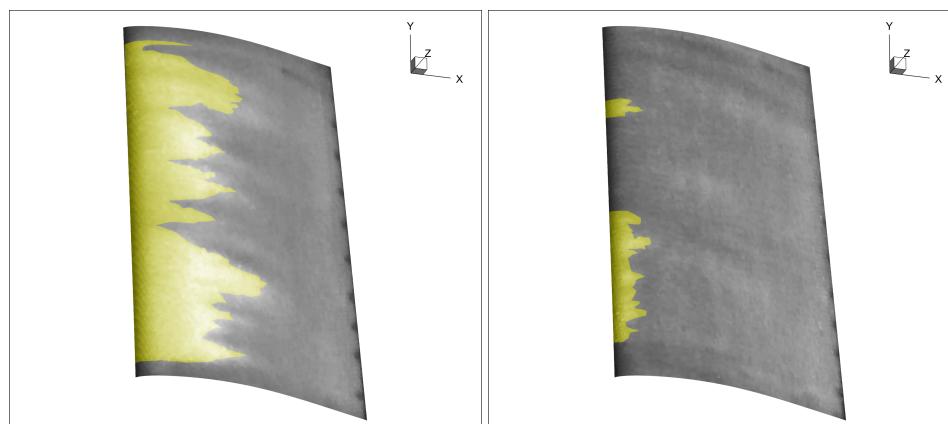
348 Main objective was a quantitative comparison of detection of laminar-turbulent transition  
 349 positions with two different methods: Thermographic pictures and microphones. It is interesting to  
 350 note that both methods agree in their results within their limitations: Thermography takes maximum  
 351 temperature gradient as the *location* of transition as Figs 13 and 15 indicate. In contrast (see Fig 14)  
 352 transition from SPL seems to be very sharp within one unit of distance ( $= 1/40 = 0.025$  in terms of  $x/c$ ).

353 If one uses CFD (see Fig. 21) to determine laminar to turbulent transition then results compare  
 354 more to a region with the end of the laminar part ( $(x/c)_{lam-end} = 0.29$ ) given by the minimum of  $c_f$  and  
 355 the beginning of the fully turbulent region by the the following maximum ( $(x/c)_{fully-turb-start} = 0.42$ ).

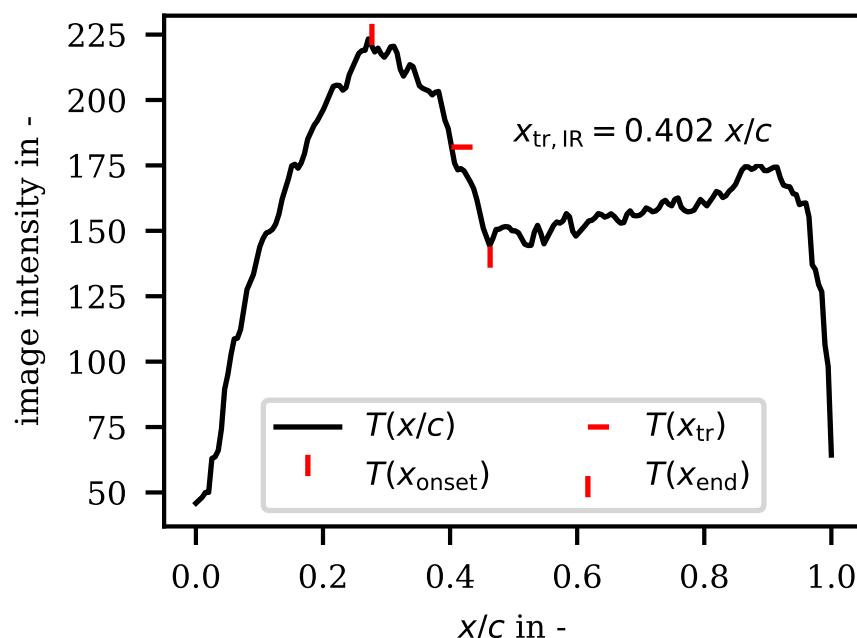
356 Therefore it can be stated that thermographic imaging provides a non-intrusive method which  
 357 can be used from a distance to detect laminar to turbulent transition its accuracy proved to be in  
 358 accordance with data both from microphones and transitional CFD.



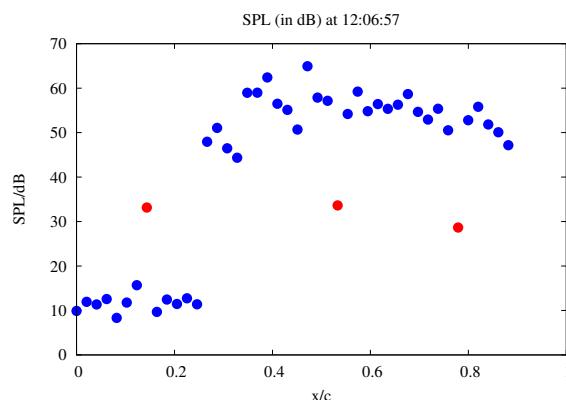
**Figure 17.** Thermal images from Figure 16 after reconstruction and mapping onto the 3D grid of the aerodynamic glove. Images are from 11:22:27 (left) and 11:23:13 (right, both SCADA time). Here the glove are is shown only.



**Figure 18.** Thermal images from Figure 16 after transition detection procedure.



**Figure 19.** Thermal image intensity extracted along the microphone-array line for the time instance 11:22:27.

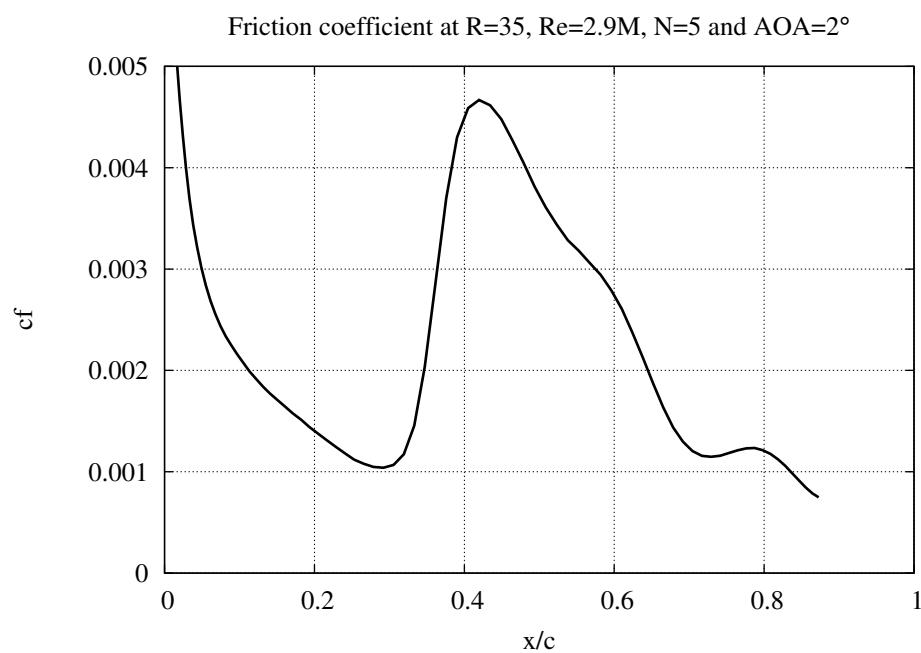


**Figure 20.** Development of SPL vs chord at 12:06:57. If one discards outliers (in red)  $x_{tr, M}/c$  may be deduced to  $x_{tr, M}/c = 0.34 \pm 0.01$

### 359 5. Summary and Conclusions

360 An aerodynamic glove known from free-flight experiments with air planes was designed,  
 361 manufactured and used on a 2 MW state-of-the-art wind turbine to investigate laminar to turbulent  
 362 transition. Local inflow conditions were recorded by a 5-hole-probe and surface pressure measurements  
 363 were performed by an array of 64 pressure sensors from PSI-Initium. In addition an array containing 48  
 364 microphones was used for capturing flow sound near the wall. Two different types of thermographic  
 365 cameras were used to report the rotor surface temperature and thereby distinguish laminar (warmer)  
 366 and turbulent (colder) regions. Angle-of-attack could be determined from 5-hole probe and pressure  
 367 distribution within 1 degree accuracy.

368 Although the geometric surface did not met standards from wind tunnel models and during  
 369 the measurement blade and glove seemed to be fouled by insects and/or dust, the irregularly  
 370 shaped regions of laminar boundary layer were consistently identified by thermographic imaging and  
 371 microphones even during wind turbine start up and agreed within their respective accuracies.



**Figure 21.** Location of transition from wall shear stress ( $c_f$ ) by CFD-code FLOWer.  $N=5$  corresponds to a turbulence intensity ( $TI$ ) of 0.3 %. Onset of transition may be located at  $x/c = 0.29$  and flow is fully turbulent at  $x/c = 0.42$ . Slope is at maximum at  $x/c = 0.36$ .

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453 **Author Contributions:** conceptualization, A.P.S. and T.R.; methodology, T.R and A.P.S.; software, T.R. and A.P.S.;  
454 CFD, T.R. and A.P.S.; validation, T.R; formal analysis, T.R.; investigation, all; resources, C.D., N.B., E.S. and C.J.;  
455 data curation, T.R.; writing—original draft preparation, T.R., C.D. and A.P.S.; writing—review and editing, T.R.,  
456 A.P.S. and A.F.; visualization, T.R., C.D., E.S.; supervision, A.P.S.; project administration, T.R.; funding acquisition,  
457 A.P.S.

458 **Funding:** “This research was funded by Gesellschaft für Energie und Klimaschutz Schleswig-Holstein GmbH,  
459 Kiel, Germany grant number 8/12-26”

460 **Acknowledgments:** We acknowledge help from Senvion, Denker & Wulf, wingtec and others. We thank the  
461 University of Oldenburg and Jaroslaw Pucylowski, PhD for using their wind tunnel for the calibration of the  
462 five hole probe. Computations were performed with resources provided by the North-German Superconducting  
463 Alliance (HLRN).

464 **Conflicts of Interest:** “The authors declare no conflict of interest.”

## 465 Abbreviations

466 The following abbreviations are used in this manuscript:

467

AOA	angle of attack
BIMAQ	Bremer Institute for Metrology, Automation and Quality Science
CFD	Computational Fluid Dynamics
DLR	German Aerospace Center
DTC	Digital Temperature Compensation
DWGE	Deutsche WindGuard Engineering GmbH
L2D	lift-to-drag $c_L/c_D$
SCADA	Supervisory Control and Data Acquisition
SPL	sound pressure level
SS	steady-state phase
SSD	Solid state disk
SU	start-up phase
TI	turbulence intensity
TSR	tip-speed-ratio
WT	wind turbine