

A periodic potential underdamped stochastic resonance method and its application for gear fault diagnosis

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Abstract: The vibration feature of weak gear fault is often covered in strong background noise, which makes it necessary to establish weak feature enhancement methods. Among the enhancement methods, stochastic resonance (SR) has the unique advantage of transferring noise energy to weak signals and has a great application prospect in weak signal extraction. But the traditional SR potential model cannot form a richer potential structure and may lead to system instability when the noise is too great. To overcome these shortcomings, the article presents a periodic potential underdamping stochastic resonance (PPUSR) method after investigating the potential function and system signal-to-noise ratio (SNR). In addition, system parameters are further optimized by using ant colony algorithm. Through simulation and gear experiments, the effectiveness of the proposed method was verified. We concluded that compared with the traditional underdamped stochastic resonance (TUSR) method, the PPUSR method had a higher recognition degree and better frequency response capability.

Keywords: fault diagnosis, stochastic resonance, periodic potential, underdamped, weak signal

1. Introduction

Gearboxes are widely used in mechanical equipment transmission, which health working can effectively reduce costs and energy consumption. However, gearboxes are often affected by the component failures, which reduce the transmission efficiency of the equipment, thereby increasing energy consumption and possibly even fatal effects. [1]. Therefore, timely detection and accurate extraction of fault characteristics are particularly important [2, 3]. Vibration signal detection is usually used to extract fault

characteristics, but machinery gearboxes often work coupled with other components amid a strong background noise. Therefore, extracting early faults amid strong noise is very difficult [4, 5].

To obtain important characteristics information from the vibration signals of mechanical equipment, many scholars have done much research on such signal processing methods such as local mean decomposition [6, 7], singular value decomposition [8, 9], ensemble empirical mode decomposition [10, 11], and wavelet transform [12, 13]. Those methods are widely used in the fault diagnosis of mechanical equipment and have achieved fruitful results, but they all improve detection by filtering noise. When using those methods to reduce noise, a useful signal may be reduced or even destroyed. However, stochastic resonance (SR) is a recent noise-assisted data processing method. Compared with the traditional weak signal detection method, SR does not eliminate noise to detect weak signals, but transfers noise energy to weak signals and improves weak signal while eliminating noise. SR was first proposed in the study of paleometeorological glaciers by Benzi [14]. Then with increasing research by experts, SR began in physics [15], chemistry [16], and biology [17], and has been applied in many engineering and technical fields. To overcome the limitation of small parameters under the condition of adiabatic approximation, Tan et al. proposed an adaptive frequency-shifted and rescaling transform SR method for a time-frequency index to detect high-frequency signals in actual engineering [18]. Leng et al. studied the SR of a two-dimensional Duffing oscillator with large parameters [19]. Subsequently, Lei et al. proposed an adaptive overdamped SR weak fault diagnosis method based on an ant colony algorithm [20]. Li et al. used a chaotic ant colony algorithm to optimize system parameters and applied it to planetary gearbox fault diagnosis [21]. Liu et al. proposed an adaptive SR detection method based on an improved artificial fish swarm algorithm, the numerical simulation and bearing experiments show that this method can effectively extract weak fault signals [22]. Then, Li et al. proposed a multicomponent population average SR of singular value decomposition and ensemble empirical mode decomposition; the effective components were selected to be SR one by one, and then the overall average reached the goal of

extracting weak signals [23, 24]. Qiao et al. proposed a piecewise linear SR weak fault diagnosis method, which solved the output saturation characteristics of bistable SR by establishing a piecewise potential model [25]. Han et al. proposed an adaptive variable-scale frequency-shifted band-pass SR denoising method based on empirical mode decomposition to detect multifrequency weak signals [26].

However, the above studies were based on overdamped SR. In the overdamped SR, the inertia and damping factor are neglected. When considering the system inertia and damping factor, the SR model becomes a second-order differential equation, which is equivalent to secondary filtering and has better signal output effect. Therefore, researchers have further studied underdamped SR. In terms of underdamped SR, Lu et al. proposed a bistable underdamped SR method, which adjusted the system parameters by variable step size to achieve the maximum SNR output [27]. López proposed underdamped SR based on the FitzHugh–Nagumo potential and used an improved particle swarm optimization algorithm to achieve optimal parameter matching, the effectiveness of that method was verified by simulation and bearing experiments [28]. Lei et al. used the synergistic effect of the vibration signal, potential structure, and damping factor and used the weighted SNR as the evaluation index of the output signal to achieve weak signal extraction [29]. The above studies achieved certain research results in underdamped SR but were limited to a bistable SR system. The potential structure adjustment was too singular and it was difficult to form a rich potential structure, so the optimal match between potential structure, noise, and the periodic signal could not be achieved. In addition, a bistable SR system achieves noise energy consumption by adjusting the barrier height and the well depth. However, excessive adjustment of a nonlinear potential system not only increases the response time of the system but also increases its instability.

To solve the above problems, we propose a weak fault diagnosis method for a periodic potential underdamped stochastic resonance (PPUSR) system. This method can not only independently adjust the barrier height, depth and width of the potential well, but also achieve a more abundant potential structure and can achieve coordinated matching of the potential structure, periodic signal, and noise based on ant colony

algorithm. At the same time, the continuous potential well can consume excessive noise energy, thereby reducing noise interference, improving the stability of the system, and enhancing weak signal detection ability.

The rest of the paper is as follows: Section 2 describes the PPUSR model, SNR of the PPUSR model, and parameters optimization based on ant colony algorithm. Section 3 proposes a weak fault diagnosis method for PPUSR and simulates the method. Section 4 describes the verification of the effectiveness of the proposed method through gear experiments in the lab and rolling mill gearbox engineering practice. Section 6 is the conclusion.

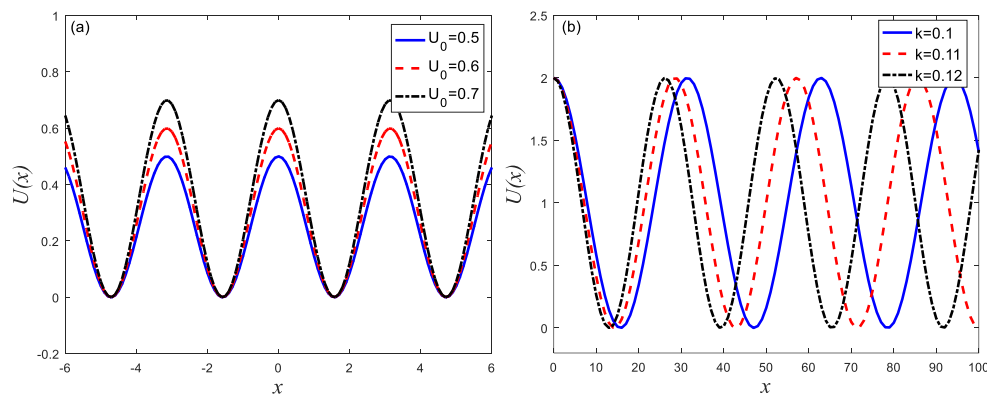
2. PPUSR method

2.1. The introduction of PPUSR model

SR uses the synergy of the input signal and noise in a nonlinear system to enhance the signal energy and identify weak signals. The three conditions generated by SR are nonlinear systems, input signals, and noise. When the three all achieve optimal matching relations, SR has the most obvious amplification effect on the signal. The traditional SR is bistable or tristable, affecting the enhancement of weak signals, in order to improve the weak signal enhancement ability for SR. In this paper, PPUSR potential model is studied, the nonlinear system that is potential function [30] introduced is

$$U(x) = U_0 \cos^2(kx) \quad (1)$$

The periodic potential model curve is shown in Fig. 1.



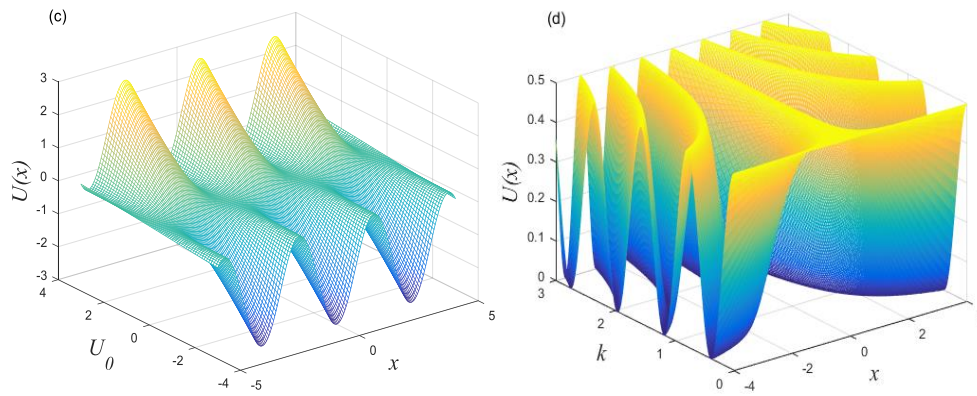


Fig. 1. Potential function $U(x)$ versus x . (a) Different parameter k with fixed $U_0 = 1$. (b) Different parameter k with fixed $U_0 = 2$. (c) Three-dimensional potential function $U(x)$ versus U_0 and x with fixed $k = 1$. (d) Three-dimensional potential function $U(x)$ versus k and x with fixed $U_0 = 0.5$.

The potential model changes with the parameters, as shown in Fig. 1. Fig. 1a and 1c show that as the potential system parameter U_0 increases, the bottom position of the potential well is fixed, and the barrier height and well depth increase continuously. In Fig. 1b and 1d, when the potential system is adjusted with the increase of k , the barrier height and well depth are unchanged, but the width of the potential well gradually increases. As the parameters U_0 and k increase, the potential depth, the barrier height, and the width increase, and vice versa. Therefore, the change of the barrier height and the well depth can be achieved by adjusting U_0 , and the adjustment k can independently adjust the potential well width. When the noise and periodic signals are small, the particles oscillate inside a potential well; at this time the nonlinear system is a monostable system. As the noise increases, the particles break through the barrier and oscillate between two or more potential wells; at this time the nonlinear system is a multistable system. Therefore, when the external driving signal is constant, the synergistic effect of the parameters enables the particles to acquire an optimal motion form, thereby producing a better SR effect.

To further analyze the PPUSR mechanism, and the SNR of the PPUSR potential model is analyzed here. In accordance with the potential model, we considered particle motion in a PPUSR system to get

$$\frac{dx^2}{dt^2} = -\frac{dU(x)}{dx} - \gamma \frac{dx}{dt} + S(t) + N(t) \quad (2)$$

where $U(x)$ is the periodic potential function. γ is the damping factor. $S(t) = A \cos 2\pi f_d t$ is a periodic signal, where A is amplitude, f_d is frequency. $N(t) = \sqrt{2D}\xi(t)$ is the Gauss white noise with zero mean satisfies $E[N(t)n(t-\tau)] = 2D\delta(\tau)$, D is noise intensity. Combining Eq. (1) with Eq. (2) yields

$$\frac{dx^2}{dt^2} = 2kU_0 \cos(kx) \sin(kx) - \gamma \frac{dx}{dt} + A \cos 2\pi f_d t + \sqrt{2D}\xi(t) \quad (3)$$

The PPUSR system model is shown in Fig. 2.

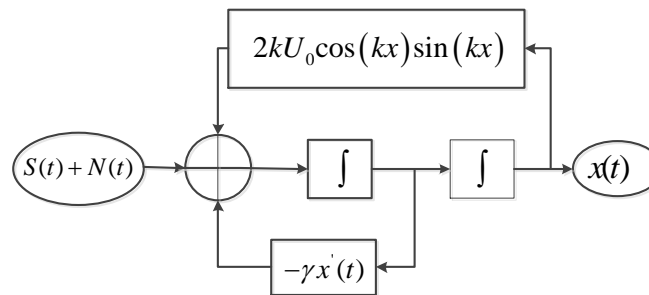


Fig. 2. PPUSR system model

2.2. SNR analysis of PPUSR potential model

To analyze the output SNR of the PPUSR model under small parameters, first let $\gamma=0$, $\varphi=0$, and $dx/dt = y$. Eq. (3) then becomes

$$\begin{cases} \frac{dx}{dt} = y \\ \frac{dy}{dt} = 2kU_0 \cos(kx) \sin(kx) + A \cos(\Omega t) + \sqrt{2D} \xi(t) \end{cases} \quad (4)$$

In addition, let $A=0$, $D=0$, and $\frac{dy}{dt}=0$. In the range $(0, \frac{\pi}{2})$, Eq. (4) can get the two

singular points as follows: $(x_1, y_1) = (\frac{\pi}{2k}, 0)$ and $(x_2, y_2) = (\frac{\pi}{k}, 0)$. The Jacobian

matrix obtained by linearizing Eq. (4) at the singularity (x_1, y_1) is $R_1 = \begin{bmatrix} 0 & 1 \\ -2k^2 U_0 & 0 \end{bmatrix}$

, and its corresponding singular value is $\beta_{1,2} = \pm \sqrt{-2k^2 U_0}$. Similarly, the Jacobian

matrix obtained by linearizing Eq. (4) at the singularity (x_2, y_2) is $R_2 = \begin{bmatrix} 0 & 1 \\ 2k^2 U_0 & 0 \end{bmatrix}$

, and its corresponding singular value is $\lambda_{1,2} = \pm \sqrt{2k^2 U_0}$. Then the steady-state

probability density function $\rho(x, y, t)$ of the particle motion can be derived by using

the Fokker–Planck equation [31]

$$\begin{aligned} \frac{\partial}{\partial t} \rho(x, y, t) = & -\frac{\partial}{\partial x} [y \rho(x, y, t)] - \frac{\partial}{\partial y} [(2kU_0 \cos(kx) \sin(kx) + A \cos \Omega t) \rho(x, y, t)] \\ & + D \left(\frac{\partial}{\partial x^2} + \frac{\partial}{\partial y^2} \right) \rho(x, y, t) \end{aligned} \quad (5)$$

According to the adiabatic approximation theory, the SPD function corresponding to Eq. (5) can be further derived as

$$\rho_{st}(x, y, t) = \bar{N} \exp\left[-\frac{U_e(x, y, t)}{D}\right] \quad (6)$$

where \bar{N} is the normalized constant and $U_e(x, y, t)$ is the generalized potential energy function [27] obtained by the small parameter expression

$$U_e(x, y, t) = \frac{1}{2}y^2 + U_0 \cos^2(kx) - Ax \cos \Omega t \quad (7)$$

To further analyze the mechanism of the SR potential model, we concluded that the escape rate of particles is

$$R_{\pm}(t) = \frac{\sqrt{\beta_1 \beta_2}}{2\pi} \sqrt{-\frac{\lambda_1}{\lambda_2}} \exp\left(\frac{-U_e(x_2, y_2, t) + U_e(x_1, y_1, t)}{D}\right) \quad (8)$$

Eq. (7) is substituted into Eq. (8) to get

$$R_{\pm}(t) = \frac{\sqrt{\beta_1 \beta_2}}{2\pi} \sqrt{-\frac{\lambda_1}{\lambda_2}} \exp\left(-\frac{U_0}{D}\right) \exp\left(\frac{\pi A}{2Dk} \cos \Omega t\right) \quad (9)$$

The Taylor expansion of Eq. (9) can be obtained by

$$R_{\pm}(t) = \frac{\sqrt{\beta_1 \beta_2}}{2\pi} \sqrt{-\frac{\lambda_1}{\lambda_2}} \exp\left(-\frac{U_0}{D}\right) \left[1 + \frac{\pi A}{2Dk} \cos \Omega t + \frac{1}{2} \left(\frac{\pi A}{2Dk}\right)^2 \cos^2 \Omega t\right] \quad (10)$$

Let $R_0 = \frac{\sqrt{\beta_1 \beta_2}}{2\pi} \sqrt{-\frac{\lambda_1}{\lambda_2}} \exp\left(-\frac{U_0}{D}\right)$. Due to the output power spectrum of the system is

$$S(\omega) = S_1(\omega) + S_2(\omega) \quad (11)$$

where let $R_1 \beta = R_0 \frac{A\pi}{2Dk}$, so we can get

$$S_1 = \frac{\pi^3 R_1^2 \beta^2}{8k^2 (R_0^2 + \Omega^2)} [\delta(\omega - \Omega) + \delta(\omega + \Omega)] \quad (12)$$

$$S_2 = \left[1 - \frac{R_1^2 \beta^2}{2(R_0^2 + \Omega^2)}\right] \frac{2\pi^2 R_0^2}{4k^2 (R_0^2 + \Omega^2)} \quad (13)$$

Finally, we get the SNR

$$SNR = \frac{\int_0^\infty S_1(\omega) d\omega}{S_2(\omega = \Omega)} = \frac{A^2 \pi^2}{32k^2 D^2} \exp\left(-\frac{U_0}{D}\right) \left[1 - \frac{U_0 A^2 \exp\left(-\frac{2U_0}{D}\right)}{16D^2 \left(\frac{k^2 U_0}{2\pi^2} \exp\left(-\frac{2U_0}{D}\right) + \Omega^2 \right)} \right]^{-1} \quad (14)$$

To show the advantage of using the proposed method to process the signal, under the same noise intensity, the SNRs with a change of the potential parameter are shown in Fig. 3. The SNR curve of the PPUSR method is also obtained, and the SNR is much higher than that of the traditional underdamped stochastic resonance (TUSR) method in [27]. Therefore, it can be concluded that for the same noise intensity and parameters, the ability of PPUSR to extract weak faults is much better than that of TUSR method.

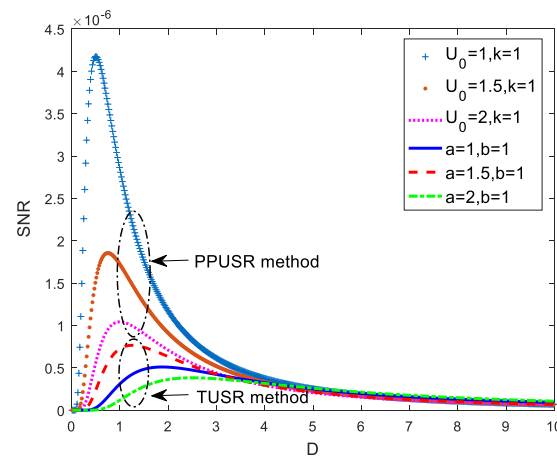


Fig. 3. Comparison of SNR between PPUSR method and TUSR method

2.3. Parameters optimization based on ant colony algorithm

According to the above analysis, the parameters can be optimally matched by adjusting the parameters U_0 and k . This paper optimizes the system parameter adjustment potential model based on the ant colony algorithm to achieve the best match between the potential model, the input signal and noise, thus obtaining the best SNR output. The ant colony system was first proposed by the Italian scholars Dorigo and Maniezzo [32]. The basic idea of applying the ant colony algorithm to solve the optimization problem is that the ants with shorter paths release more pheromone. With the advancement of time, the concentration of pheromone accumulated on the shorter path is gradually increasing, and the number of ants selecting the path is increasing. In

the end, the whole ants will concentrate on the best path under the action of positive feedback, and the corresponding solution is the optimal solution to be optimized.

In order to compare the SNR output of PPUSR method and TUSR method, the ant colony algorithm is used to optimize the parameters of two SR systems. Simulate a small parameter signal, $S(t) = A \cos(2\pi f_0 t)$, where the sampling frequency $f_s = 5$, signal amplitude $A = 0.01$, signal frequency $f_0 = 0.008$, noise intensity $D = 0.5$, data length $N = 5000$, number of ants $m = 100$, pheromone evaporation coefficient $\rho = 0.9$, maximum iteration number $n = 20$, pheromone increase intensity coefficient $Q = 200$, pheromone concentration $q = 200$, damping factor $\gamma = 1$. The parameters optimization range are $[0, 10]$, the matching parameters and SNR are obtained by the ant colony algorithm optimization system parameters of PPUSR method and TUSR method in Fig. 3. Comparing Figs. 3a and 3b, the output SNR of PPUSR method is 23.19 dB, and the output SNR of TUSR method is 21.15 dB. Therefore, it can be judged that the PPUSR method has a better SNR output under the same conditions, that is, has better SR effect.

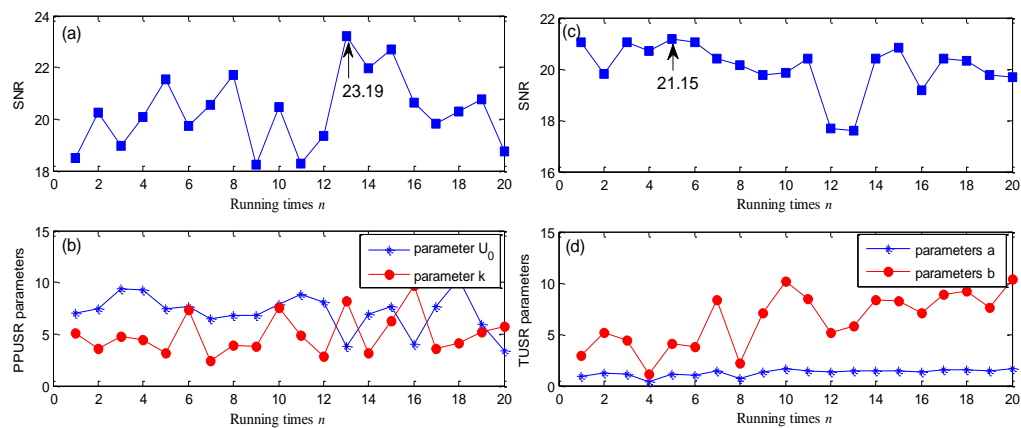


Fig. 4. Output SNR of PPUSR and TUSR method for ant colony

3. The weak fault diagnosis strategy based on PPUSR model

3.1. PPUSR weak fault diagnosis method

In accordance with the above analysis, we propose a weak fault diagnosis method for the PPUSR model. The main process is described as follows:

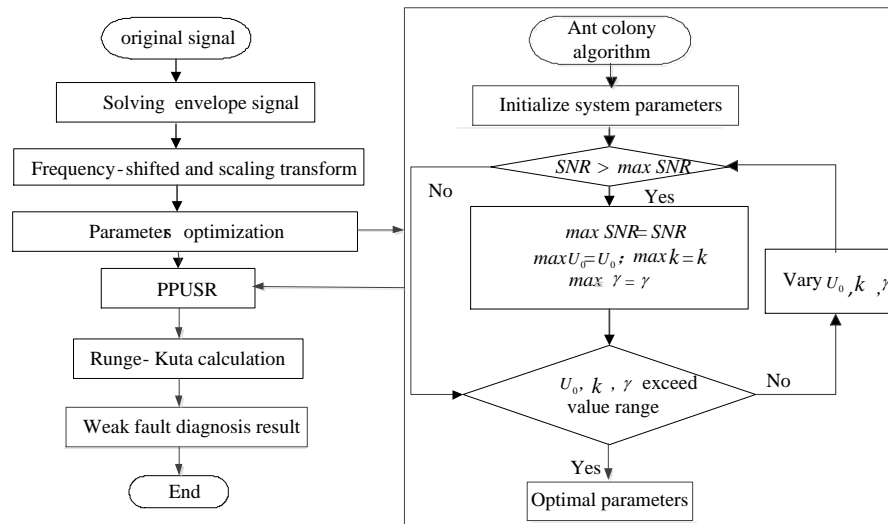


Fig. 5. PPUSR weak fault diagnosis process

- (1) Signal preprocessing. The collected vibration signal is processed to obtain an envelope signal. Then the input signal is initially processed by using a shifting-frequency and rescaling transform to meet small parameter requirements under approximate adiabatic conditions.
- (2) Initializing of the parameters of the system. The parameters include ant colony size, step size, optimization range, maximum iteration number, pheromone evaporation coefficient, and the pheromone increasing intensity coefficient.
- (3) Parameter optimization. Calculate the corresponding probability according to the prior knowledge and pheromone concentration, and find the optimal path of the ant; that is, find the optimal combination of parameters U_0 and k .
- (4) Output signal calculation. Input the optimized parameters into the PPUSR system, calculate the system output using the Runge–Kutta equation for Eq. (15), and use the SNR as the evaluation index of the output signal for Eq. (16).

$$\left\{ \begin{array}{l} y_1 = y[n]; \quad x_1 = -U'(x[n]) - \gamma y_1 + S[n] + N[n]; \\ y_2 = y[n] + x_1 h / 2; \quad x_2 = -U'(x[n]) + y_2 h / 2 - \gamma y_2 + S[n] + N[n]; \\ y_3 = y[n] + x_2 h / 2; \quad x_3 = -U'(x[n]) + y_3 h - \gamma y_3 + S[n+1] + N[n+1] \\ y_4 = y[n] + x_3 h / 2; \quad x_4 = -U'(x[n]) + y_4 h - \gamma y_4 + S[n+1] + N[n+1] \\ x[n+1] = x[n] + (y_1 + 2y_2 + 2y_3 + y_4)h / 6; \\ y[n+1] = y[n] + (x_1 + 2x_2 + 2x_3 + x_4)h / 6; \end{array} \right. \quad (15)$$

where $f(t, x)$ is the two derivatives of the potential function, and h is the calculated interval.

$$SNR = 10 \log_{10} \left(\frac{A_d}{\sum_{i=1}^{N/2} A_i - A_d} \right) \quad (16)$$

where N is the data length, A_d is amplitude of the fault characteristic frequency in the power spectrum, $\sum_{i=1}^{N/2} A_i - A_d$ is amplitude of the power spectrum for sum noise.

(5) Evaluation of output results. Find the optimal SNR and the corresponding optimal parameter match. If the optimal parameter is within the optimization range, expand the range of the corresponding parameter and return to step 3; otherwise, go to step 6.

(6) Signal post processing. Substituting the obtained optimal parameters into the PPUSR system, the fault characteristic frequency of the vibration signal is finally extracted.

3.2. Simulation verification

The parameters of PPUSR method and the TUSR method were set to $U_0 = k = 1$, $a = b = 1$ respectively, and the frequency response curves were obtained in Fig. 6. We can get the SNR curves of both methods decreasing with increasing frequency. However, the high-frequency SNR of PPUSR method was better than that of the TUSR method, and the SNR was more stable under different frequency conditions, indicating that the stability of the PPUSR potential model is better.

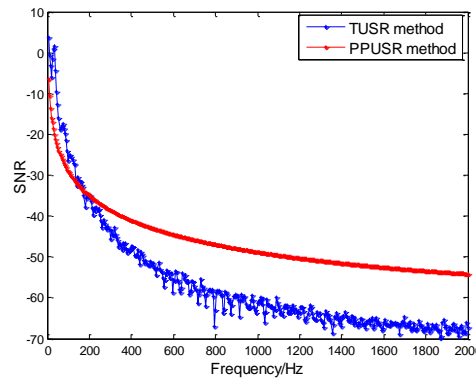


Fig. 6. Frequency response of PPUSR and TUSR methods

To verify the effectiveness of the proposed method, we processed the simulated gear fault signal by the PPUSR weak fault diagnosis method. The noise was added as signal interference. The simulated gear fault signal was expressed by

$$S(t) = A \sin(2\pi ft) \cdot \exp\left\{-d[t - n(t)T_d]^2\right\} + \sqrt{2D}\varepsilon(t) \quad (17)$$

The PPUSR equation for the simulated gear fault signal was

$$\frac{dx^2}{dt^2} = 2kU_0 \cos(kx) \sin(kx) - \gamma \frac{dx}{dt} + S(t) \quad (18)$$

where $A = 1$ is the signal strength, $f = 1$ KHz is the modulation frequency, $d = 6f_s$ ($f_s = 10$ KHz is the sampling frequency) is the attenuation rate, $n(t) = [t/T_d]$ is the control pulse, $\xi(t)$ is the Gaussian white noise, $\sqrt{2D} = 1$ is the noise intensity, $T_d = 0.01786$ is the interval pulse ($f_d = 1/T_d = 56$ Hz is the drive frequency), and the sampling time is 0.5 s. Figs. 7a and 7b show the simulated gear fault signals without and with noise respectively.

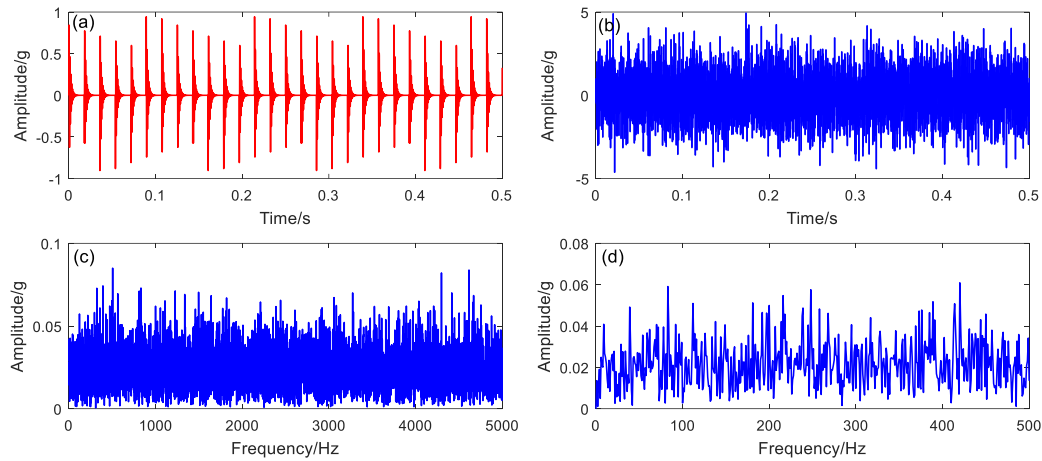


Fig. 7. Simulated gear signals. (a) Periodic signal without noise. (b) Periodic signal with noise. (c) Spectrum. (d) Envelope spectrum.

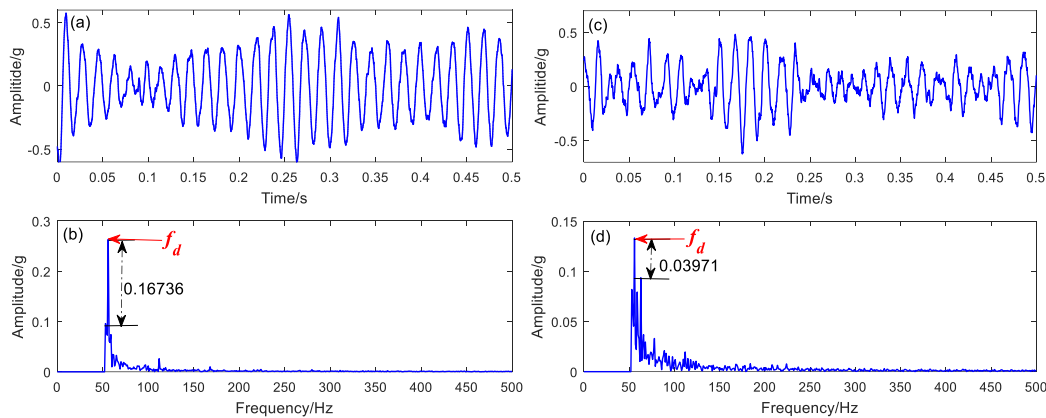


Fig. 8. PPUSR and TUSR methods to extract simulated gear faults. (a) PPUSR method waveform. (b) PPUSR method spectrum. (c) TUSR method waveform. (d) TUSR method spectrum.

We can be seen from Fig. 7c that the simulated gear fault signal does not detect any fault information in the spectrum due to strong noise interference. In Fig. 7d, the target signal is also completely covered by the strong noise, so the fault frequency of the simulated gear signal is not recognized. To verify the effectiveness of the proposed method, the envelope signal was processed by the PPUSR system. To meet the requirements of small parameters, the envelope signal was a shifting-frequency and rescaling transform, and then the ant colony algorithm was used to adaptively optimize the system parameters and damping factor; the parameter combination was $U=0.4723$, $k=1.2$, and $\gamma=0.06$. When the parameters were put into the PPUSR system, the time domain waveform and spectrum were as shown in Figs. 8a and 8b. As

can be seen from Fig. 8a, the time domain waveform exhibited a periodic variation. In Fig. 8b, it can be seen that the simulated fault characteristic frequency was prominent, and its amplitude was 0.16736 higher than the surrounding noise. As a comparison with the proposed method, we used the TUSR method to process the simulated gear signals. The ant colony algorithm was also used to optimize the system parameters and damping factors to obtain $a = 1.102$, $b = 4.96$, and $\gamma = 0.0075$. The parameters were put into the TUSR system, and the time domain and spectrum were obtained, as shown in Figs. 8c and 8d. The fault characteristic frequency could also have been extracted by the TUSR method, but the characteristic frequency peak was seriously interfered with by the surrounding noise, and its amplitude was only 0.03971 higher than that of the surrounding noise. The above analysis shows that when the noise was too large, the periodic potential could consume too much noise interference through multiple potential wells, and the bistable system was singular due to the adjustment of the system potential structure. When the noise was driven too much, the instability of the bistable system was enhanced, which affected the enhancement of the weak signal.

4. Experimental verification

4.1. Gear experimental verification

To further verify the effectiveness of the proposed method, we detected a gear fault characteristic frequency using the power transmission fault diagnosis test bench shown in Fig. 9. We cut a crack 1 mm deep and 0.15 mm wide in the root of the gear in the root direction to simulate an early failure. The modulus of the gear was 1.5. The parameters of the gear were $Z_1 = 100$, $Z_2 = 29$, $Z_3 = 90$, and $Z_4 = 36$. The motor speed was 840 r/min, the sampling frequency was 5120 Hz, and the data length was 4,096 points. Because the speed of the motor was reduced by a planetary gearbox, the transmission ratio was 4.571, so the input shaft speed of the parallel shaft gearbox was 183.767 r/min; that is, the rotation frequency was 3.063 Hz. According to vibration analysis theory, the fault characteristic frequency of the faulty tooth was $f_d = 10.561$ Hz. The time domain waveform and spectrum of the vibration data collected by the acceleration sensor were as shown in Fig. 10. It can be seen from the

time domain waveform that although multiple impacts can be seen, the target frequency could not be determined due to noise interference, and there were multiple low-frequency and high-frequency noises in the spectrum; the fault characteristic frequency components were completely covered by noise. In Fig. 10c, the fault characteristic frequency can be seen, but the interference frequency was too high. The fault characteristic frequency could not be accurately judged. The fault characteristic frequency was extracted by the PPUSR method and the TUSR method. To meet the small parameter requirement, by using the ant colony algorithm to optimize the parameters and damping factors of the two systems, we got $U_0 = 1.627$, $k = 0.684$, $\gamma = 0.019$, and $a = 1.275$, $b = 2.064$, $\gamma = 0.0021$. The parameters and damping factors were substituted into the SR system. The time domain waveforms and spectrum are shown in Fig. 11. Comparing Fig. 11a with 11c, the PPUSR method clearly had greater periodicity than did the TUSR method. Comparing Fig. 11b with 11d, it can be seen that the fault characteristic frequency extracted by the PPUSR method was obviously manifested, the doubling frequency component appeared, and the fundamental frequency amplitude was 1.0217 higher than that of the surrounding noise. The TUSR method also extracted the fault characteristic frequency, but it was seriously interfered with by the surrounding noise. The fundamental frequency amplitude was only 0.1054 higher than the surrounding noise, and the doubling frequency was also interfered with by the noise. Therefore, the gear failure experiment demonstrated again that the proposed method was superior to the TUSR method and had better recognition.

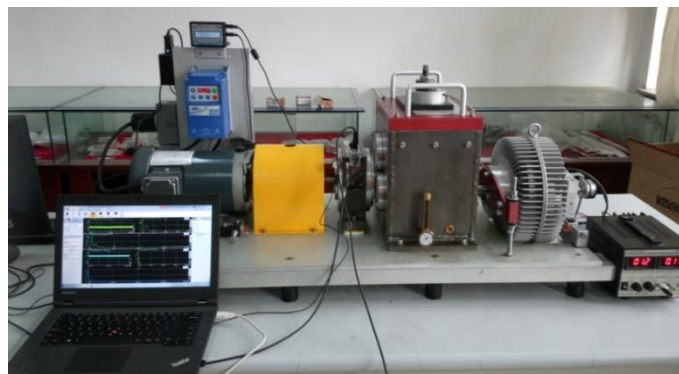


Fig. 9. Power transmission fault diagnosis test bench

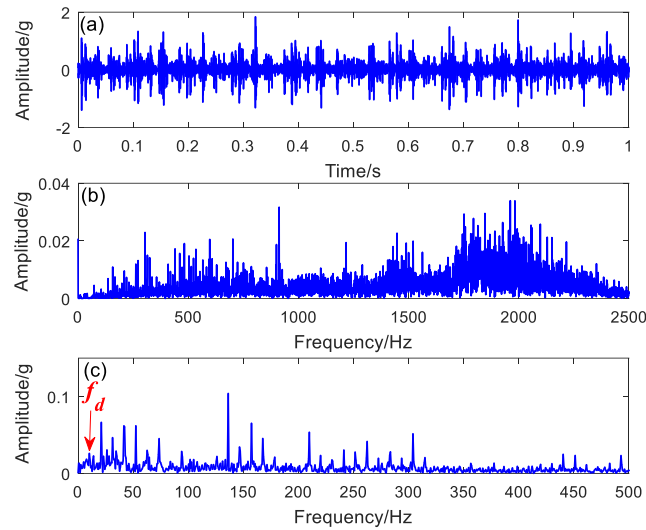


Fig. 10. Gearbox faults: (a) Waveform. (b) Spectrum. (c) Envelope spectrum.

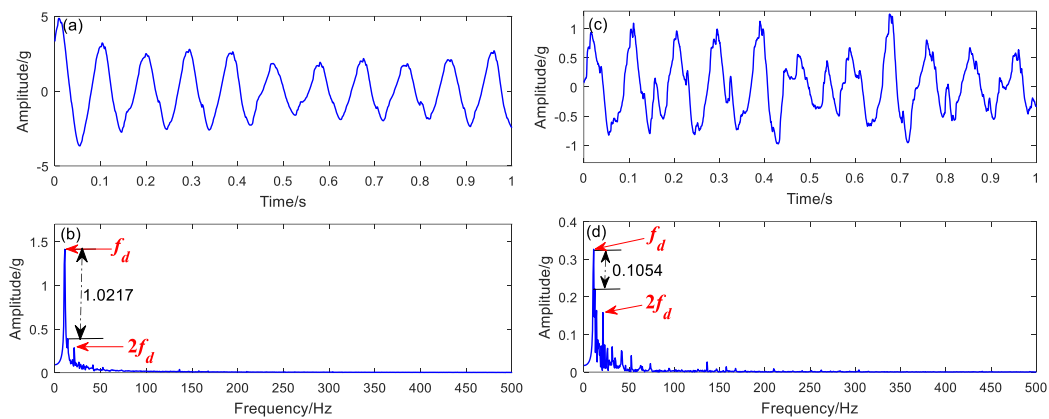


Fig. 11. PPUSR and TUSR methods to extract gearbox faults: (a) PPUSR waveform. (b) PPUSR spectrum. (c) TUSR waveform. (d) TUSR spectrum.

4.2. Rolling mill Gear experimental verification

When the rolling mill was inspected, it was found that the gearbox had slight abnormal vibration, but the equipment failure detection system did not issue an alarm, and the equipment was still in normal operation. The rolling mill gearbox is shown in Fig. 12a. In order to detect and prevent fault in the early stage, the measuring point of the acceleration sensor was arranged in the axial position of the input shaft of the rolling mill gearbox. The ZonicBook/618E tester was used to collect the vibration signal of the measuring point. The sampling frequency is 2560 Hz. The numbers of sampling points were 2048 points, the motor speed was 1300r/min, and the corresponding input shaft frequency was 21.67Hz. The time domain waveform, spectrum, and envelope spectrum were shown in Fig. 13. In Fig. 13a, the original signal was completely covered by the

surrounding noise, and the impact signal cannot be seen. In Fig. 13b, the frequency of 715 Hz is the maximum energy, and a dense sideband occurs. According to Table 2, the fault may appear on the transmission axis II, but the energy of the sideband is small, and the interval of the sideband cannot be determined. In Fig.13c, although we can see the fault characteristic frequency of the gear, the surrounding noise is seriously disturbed. Therefore, we cannot judge the specific faulty gear. The gear signals were processed by the proposed method and the TUSR method. The ant colony algorithm is used to optimize the system parameters and damping ratio, and the combination of parameters were $U_0=0.25$, $k=1.423$, $r=0.197$ and $a=2.7$, $b=0.0109$, $r=0.402$, respectively. Comparing with Fig. 14a and Fig. 14c, the time domain waveform curve has better periodicity in Fig. 14a. In Fig. 14b and 14d, both methods yield a characteristic frequency of 19.2 Hz. According to Table 1, it can be judged that the gear on the transmission shaft II is faulty. Through disassembly, we find that the gear on the transmission shaft II has broken tooth in Fig. 12b, which was consistent with the actual situation. Comparing Fig. 14b and 14d, Fig. 14b not only has the characteristic frequency but also the frequency multiplication was seen, and the recognition degree of the gear fault characteristic frequency was better. Therefore, the effectiveness of the proposed method was proved again by the gear fault, and it had better effect than the TUSR method.

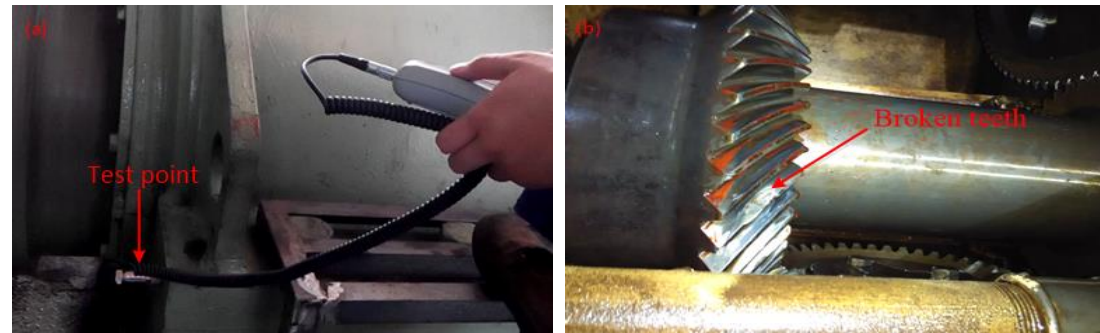


Fig. 12 Rolling mill gearbox faults: (a) Fault detection site. (b) Gear broken tooth

Table 1. Frequency conversion of each axis and meshing frequency of each gear pair

Transmission shaft	Tooth ratio	Rotating speed (r/min)	Rotation frequency(Hz)	Meshing frequency(Hz)
I	--	1300	21.67	--

II	33/38	1129	18.82	715
III	42/89	533	8.89	790
IV	21/74	151	2.52	186
V	24/85	43	0.72	61
VI	38/38	43	0.72	27

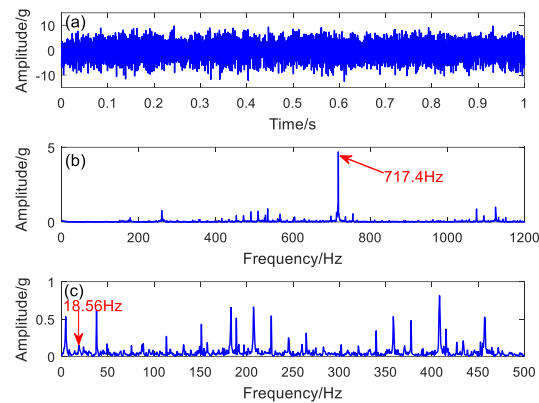


Fig. 13. Rolling mill gearbox faults: (a) Waveform. (b) Spectrum. (c) Envelope spectrum.

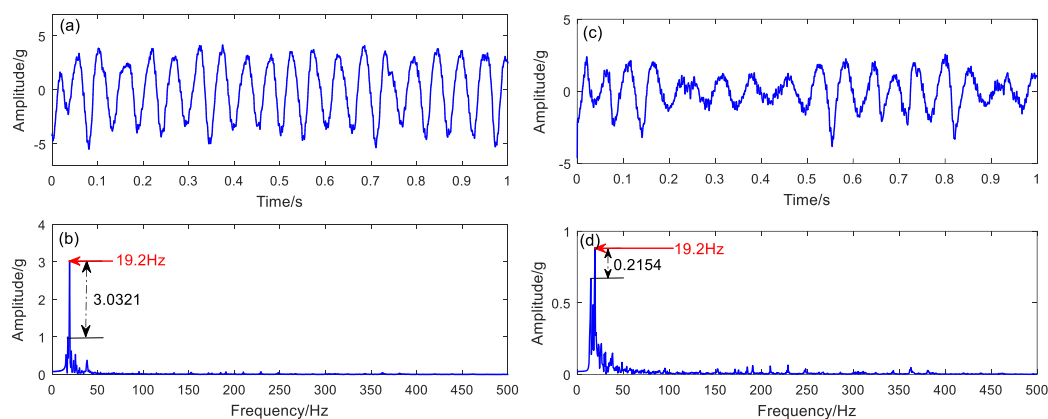


Fig. 14. PPUSR and TUSR methods to extract rolling mill gearbox faults: (a) PPUSR waveform.

(b) PPUSR spectrum. (c) TUSR waveform. (d) TUSR spectrum.

5. Conclusion

To overcome disadvantage of TUSR fault detection method, this paper proposes a PPUSR method and studies the potential function, and SNR of the simulated system based on the PPUSR phenomenon. After presented method was validated by theoretical and experimental, the following conclusions can be drawn:

(1) The PPUSR model can not only adjust the potential well width independently by parameter k but can also synchronously adjust the barrier height and the depth of the

well by adjusting the parameter U_0 . Therefore, it has a richer potential model structure and can achieve the best match between potential structure, periodic signal, and noise.

(2) When the noise is too great, the oscillation of the particles between potential wells in the TUSR system is too fast due to the limitation of the potential model. The excessive noise will cause noise interference in a finite bistable system, which will easily lead to instability of the system. PPUSR can make the particles transition through multiple potential wells, thus consuming excessive noise energy and improving the stability of the system.

(3) According to experimental and engineering application verification results, the PPUSR weak fault diagnosis method can obtain larger fault characteristic frequency amplitude than can the TUSR method. In addition, PPUSR method had a better SNR for extracting high-frequency signals than the TUSR method.

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