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Keywords: Hydrogen; Methane; Methanol; Combustion; Engine



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Article

Influence of Fuel Types and Equivalence Ratios on NO_x Emissions in Internal Combustion Engines: A Comparative Analysis of Methane, Ethanol, and Hydrogen Blends

Amr Abbass

Abstract: This study investigates the emissions and combustion properties of methane and ethanol biofuels in internal combustion engines; emphasizing the impact of equivalency ratios and fuel types on NO_x emissions. A variety of numerical models utilizing chemical kinetics simulations through Cantera (David G. Goodwin, 2023.) were implemented to evaluate performance across different equivalency ratios. Results indicate that lean mixtures ($\phi < 1$) generate increased NO_x levels owing to greater combustion temperatures, but rich mixtures ($\phi > 1$) markedly diminish NO_x emissions. Ethanol blends with hydrogen reduce carbon emissions and improve flame stability, facilitating the shift from conventional diesel to hydrogen-powered engines. Validation was accomplished through experimental data, indicating that stoichiometric methane combustion and hydrogen-ethanol mixtures resulted in a 20% decrease in NO_x emissions and enhanced cycle efficiency. Hydrogen, while its poor volumetric energy density and elevated ignition requirements, demonstrates potential when amalgamated with ethanol or methane, providing a solution for cleaner engine performance while tackling its principal difficulties. The study's findings were verified and contrasted with those of other recent investigations regarding hydrogen engines emissions reduction (K. J. Lee, 2013) (Dimitriou, 2018).

Keywords: Hydrogen; Methane Methanol Combustion Engine

1. Introduction

Because of its unique combustion characteristics, hydrogen is a fuel with great potential for use in future energy systems (desert, 2001). It is also environmentally friendly. It is essential to comprehend these characteristics to enhance internal combustion engines and combustion management strategies. Hydrogen is suitable for advanced combustion strategies such as Homogeneous Charge Compression Ignition (HCCI) and Reactive Controlled Compression Ignition (RCCI), which aim to enhance engine performance and decrease harmful emissions (Efsthios, 2023). However, hydrogen-fueled engines face pre-combustion processes, knocking, and back-firing challenges. To address these issues, scientists design combustion chambers that improve air-fuel mixing, use swirling technologies to enhance fuel distribution and develop materials with solid resistance to hydrogen embrittlement. Developing hydrogen infrastructure and supply chains is crucial for the broader application of hydrogen-fueled engines.

2. Model and Methods

The central part of the model is a Python script that analyzes hydrogen combustion using the Cantera library, utilizing the "gri30" chemical mechanism. The model also calculates the same simulation for a methane fuel engine and compares the results. The combustion cycle consists of isentropic compression, volume combustion, and expansion stages, closely monitoring thermodynamic parameters and species evolution. The model determines combustion characteristics,

including peak temperature, pressure, and heat release, and calculates work output and exhaust temperature. Its flexibility allows for extensive research in varying operating windows.

The first part of the model is a simulation tool that analyzes the Ignition Delay for various fuels, including hydrogen, methane, methanol, propane, and ethylene. The goal is to model hydrocarbon combustion and NO_x formation under constant volume conditions, with the reactor starting with stoichiometric air-fuel blends at several initial temperatures and a pressure of 1 atm. The model records the time when a sudden temperature rise is observed, allowing for numerical accuracy and computational efficiency (Dimitriou, 2023).

The model's second section tracks the ignition delay throughout the combustion of various fuels. This is crucial for attaining combustion stability and is the primary aspect in numerical modeling, which gives each fuel the appropriate amount of time to carry out actual combustion and produce tangible results.

The Third part of the model investigates the burning characteristics of hydrogen-methane (H₂-CH₄) fuel blends. It assesses thermal and pollution metrics such as CO, heat release, and NO concentrations by changing the hydrogen percentage in the fuel mix at stoichiometric combustion conditions in a manner that resembles the process in Reactive Controlled Compression Ignition (RCCI). This study aims to investigate the trade-offs between NO and CO concentrations and heat release to draw a picture concerning the feasibility of hydrogen and methane in different percentages of methane in energy applications to reach the best clean mixture (Kiverin, 023).

The model's fourth section examines the impact of substituting hydrogen for various hydrocarbon fuels, altering the equivalency ratio, and monitoring the impact on NO_x emissions. Propane and methane are the fuels that are examined. The model forecasts the highest NO concentration that will happen during the combustion stage and clearly connects the usage of hydrogen, methane, or propane with the removal of NO_x during the combustion phases..

The model's fifth section examines how the equivalency ratio and hydrogen blend with a biofuel of methanol affect the amount of NO_x emissions produced.

The final section presented the idea of adding methane to the hydrogen methanol blend in determined percentages to track the NO_x emissions level by changing the equivalency ratio.

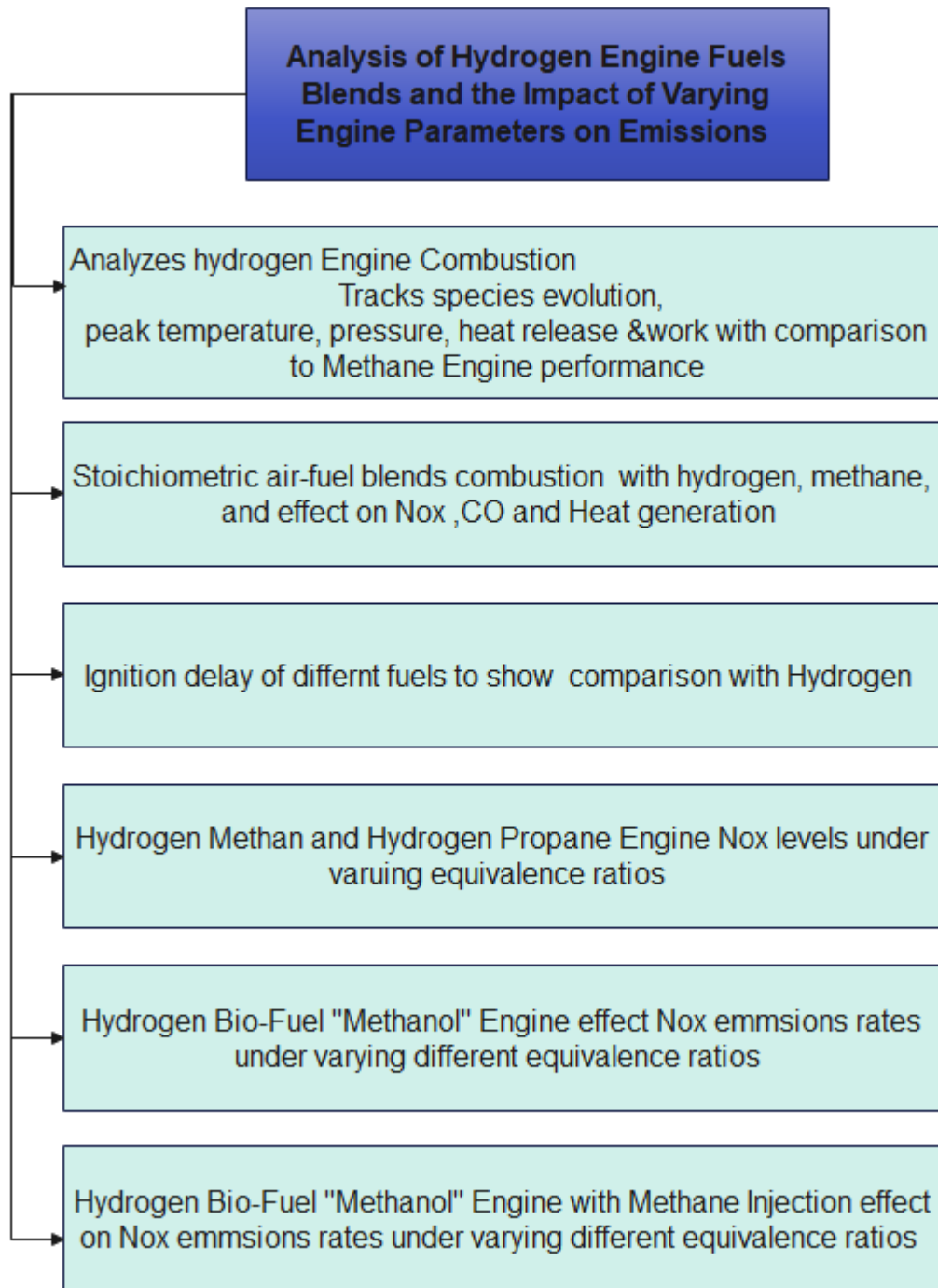


Figure 1. An overview of the hydrogen engine models examined in this study.

Equations of The Model (Q.T. Dam, 2024)

Equivalence ratio:

$$\phi = \frac{\text{Fuel-to-air ratio (actual)}}{\text{Fuel-to-air ratio (stoichiometric)}} \quad (1)$$

Final pressure after isentropic compression:

$$P_{\text{compression}} = P_{\text{inlet}} \cdot CR^{\frac{\gamma}{\gamma-1}} \quad (2)$$

Work during compression:

$$W_{\text{compression}} = C_v \cdot (T_{\text{compression}} - T_{\text{inlet}}) \quad (3)$$

Heat addition:

$$Q_{\text{ignition}} = C_v \cdot (T_{\text{Combustion}} - T_{\text{compression}}) \quad (4)$$

Temperature after isentropic expansion:

$$T_{\text{exhaust}} = T_{\text{combustion}} \cdot \left(\frac{P_{\text{expansion}}}{P_{\text{combustion}}} \right)^{\frac{\gamma-1}{\gamma}} \quad (5)$$

Work during expansion:

$$W_{\text{expansion}} = C_v \cdot (T_{\text{combustion}} - T_{\text{exhaust}}) \quad (6)$$

Net work:

$$W_{\text{net}} = W_{\text{expansion}} - W_{\text{compression}} \quad (7)$$

Thermal efficiency:

$$\eta = \frac{W_{\text{net}}}{Q_{\text{total}}} \quad (8)$$

Species mole fraction:

$$X_i = \frac{n_i}{n_{\text{total}}} \quad (9)$$

Ignition Delay Recording:

$$t_{\text{ignition}} = \text{time at which } T_{\text{reactor}} \geq T_{\text{initial}} + 50 \quad (10)$$

Energy Conservation:

$$\frac{dU}{dt} = \dot{Q} - P \frac{dV}{dt}, \quad \frac{dV}{dt} = 0 \quad (11)$$

Species Conservation:

$$\frac{dY_i}{dt} = \dot{\omega}_i W_i \quad (12)$$

3. Results

3.1. Comparative Analysis of NOx Emissions in Hydrogen and Methane Combustion: The Impact of Equivalence Ratio and Fuel Characteristics

The model begins by outlining its initial values, which include the equivalence ratio, temperature, pressure, and the type of fuel, which is either CH₄ or H₂. Combustion of methane (CH₄) with an equivalence ratio of 1 produces greater NO_x emissions than hydrogen (H₂) with the same equivalence ratio. Methane combustion has a theoretically calculated temperature of 3171.49 K at a

maximum temperature and combustion pressure of 118.93 bar, while hydrogen combustion occurs at 3341.46 K at a slightly lower combustion pressure. This higher temperature is due to higher flame speed and energy produced by hydrogen. Methane combustion produces more NO_x emissions than hydrogen, which is 11,479 ppm. This is due to the longer burn time of methane combustion and the presence of nitrogen in the fuel-air mixture. Hydrogen combustion occurs more quickly, but its short time limits NO_x emissions. When comparing hydrogen combustion with an equivalence ratio of 1, the influence of a vibrant mixture (more fuel and less oxygen) on combustion and NO_x formation is observed. At an equivalence ratio of 2, the maximum temperature reduces to 3156.66 K, and the combustion pressure decreases to 107.51 bar. The NO_x concentration decreases significantly from 11,479 ppm at equivalence 1 to only 684 ppm at equivalence 2.

In conclusion, higher equivalence ratios are preferable for hydrogen combustion, as fewer NO_x emissions are formed. Suppress NO_x emissions.

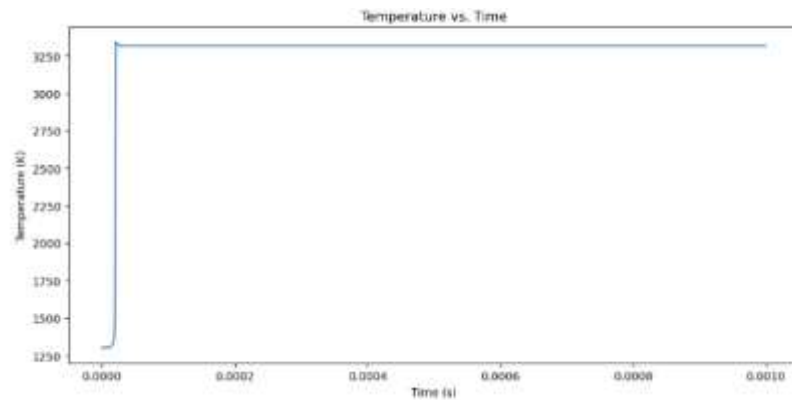


Figure 2. Temperature increase for hydrogen combustion when the equivalency value is 1.

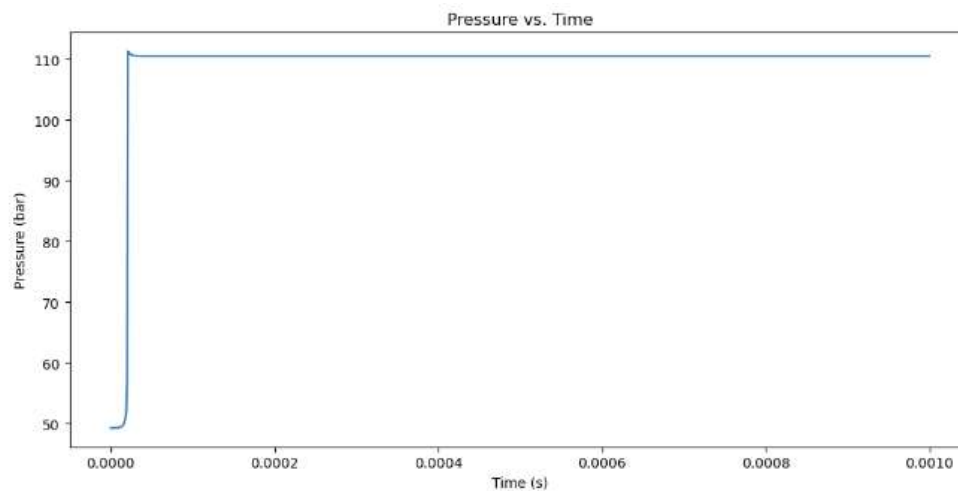


Figure 3. Pressure increase for hydrogen combustion when the equivalency value is 1.

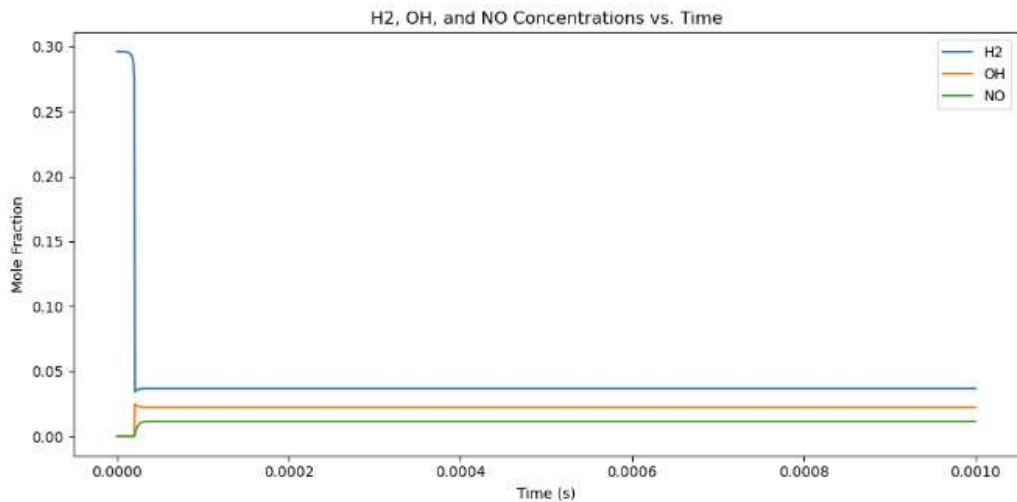


Figure 4. Mole fractions for hydrogen combustion when the equivalency value is 1.

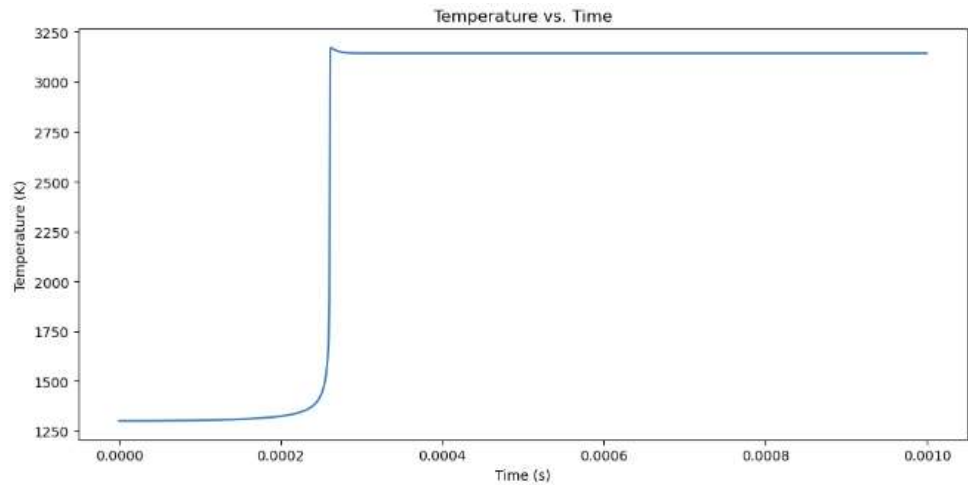


Figure 5. Temperature increase for Methane combustion when the equivalency value is 1.

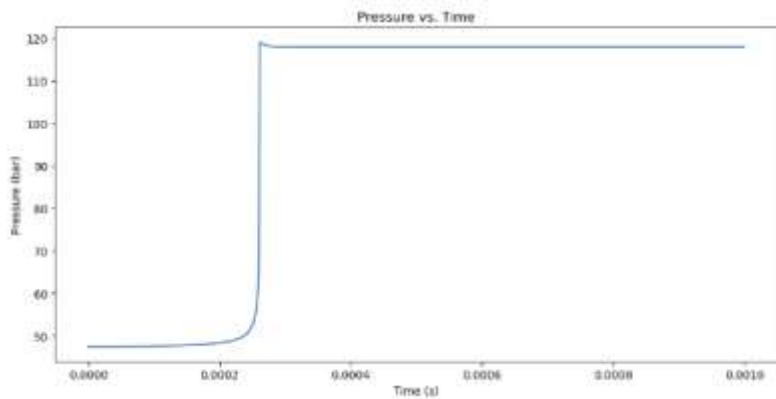


Figure 6. Pressure increase for Methane combustion when the equivalency value is 1.

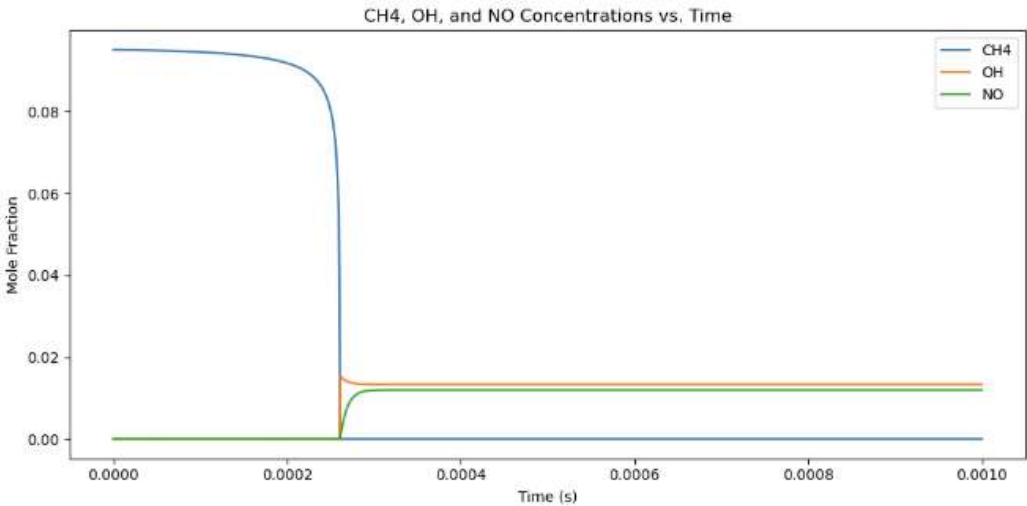


Figure 7. Mole Fraction for Methane combustion when the equivalency value is 1.

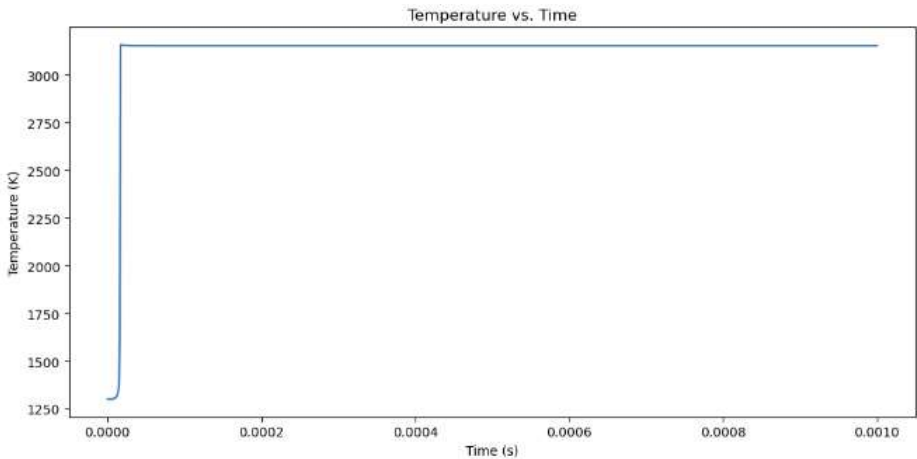


Figure 8. Temperature increase for hydrogen combustion when the equivalency value is 2.

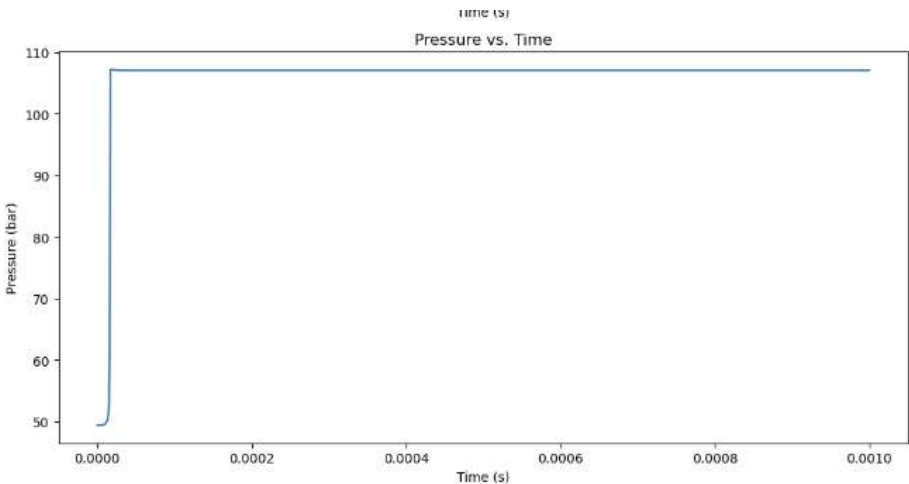


Figure 9. Pressure increase for hydrogen combustion when the equivalency value is 2.

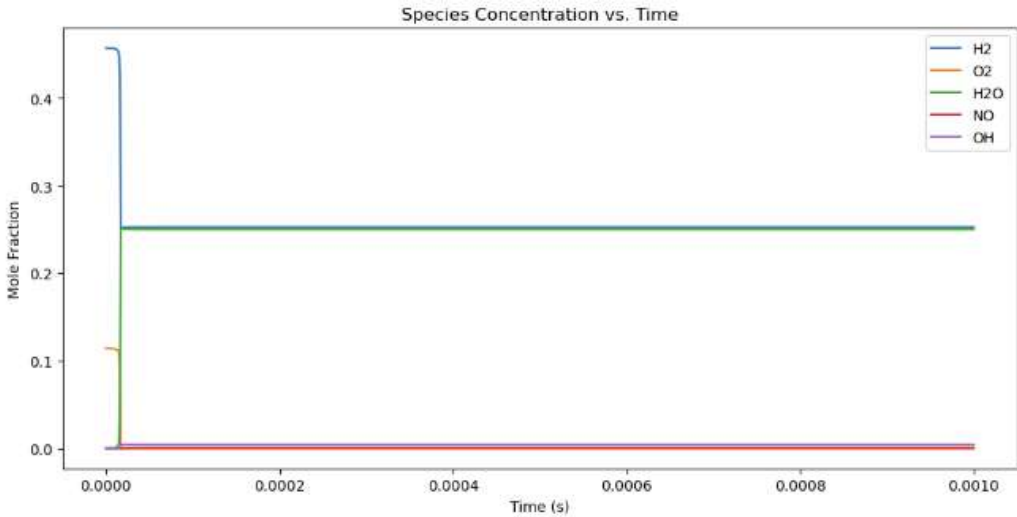


Figure 10. Mole Fractions for hydrogen combustion when the equivalency value is 2.

Table 1. Comparison of Combustion Parameters for Hydrogen and Methane at Various Equivalence Ratios.

Parameter	H ₂ Equivalence 1	H ₂ Equivalence 0.8	H ₂ Equivalence 0.5	H ₂ Equivalence 2	CH ₄ Equivalence 1
Maximum Temperature (K)	3,341	3,188	2,739	3,037	3,080
Combustion Pressure (bar)	111	107	95	121	136
Combustion Energy (MJ/kg)	2.75	2.45	1.82	2.98	2.2
Compression Temperature (K)	880	879	877	886	820
Compression Pressure (bar)	49	49	49	49	47
Expansion Work (MJ/kg)	2.23	2.03	1.63	2.58	1.85
Net Work (MJ/kg)	1.58	1.42	1.06	1.77	1.35
Cycle Efficiency (%)	49	49	47	54	55
Exhaust Temperature (K)	1,346	1,277	1,078	1,177	1,173
NO Concentration (ppm)	11,480	18,384	18,726	389	9,953

The equivalency ratio significantly influences NO concentration and temperature in hydrogen combustion. As the equivalency ratio for hydrogen declines from 1 to 0.5 (indicating a leaner mixture), the concentration of NO rises from 11,480 ppm to 18,726 ppm, attributed to increased oxygen availability and elevated combustion temperatures facilitating NO production. In contrast, a rich mixture (H₂ equivalency 2) significantly decreases NO concentration to 389 ppm, as insufficient oxygen inhibits NO synthesis. In contrast, methane combustion at an equivalency ratio of 1 yields a moderate NO concentration of 9,953 ppm, which is lower than that of lean hydrogen mixes but

greater than that of rich hydrogen mixtures. The peak temperature in hydrogen combustion exhibits a comparable pattern. At an equivalent ratio of 1, the temperature reaches its numerical theoretical peak at 3,341 K. Still, leaner mixtures with an equivalence ratio of 0.5 provide lower temperatures of 2,739 K. Rich mixtures (equivalence 2) marginally reduce the peak temperature to 3,037 K because of incomplete combustion. The combustion of methane, reaching a peak temperature of 3,080 K, is marginally lower than that of stoichiometric hydrogen but exceeds that of rich hydrogen mixtures. Hydrogen typically attains elevated temperatures owing to its superior energy content and rapid burning rates, which is challenging when determining the engine material to support hydrogen operations. In comparing pressure rise, hydrogen combustion at an equivalency ratio of 1 yields a combustion pressure of 111 bar, which diminishes with leaner mixtures, resulting in 95 bar at an equivalence ratio of 0.5. Rich hydrogen combustion at 121 bar surpasses stoichiometric values owing to elevated fuel energy content. The burning of methane attains a markedly higher pressure (136 bar) compared to hydrogen due to its slower combustion rate and denser energy release. Ignition delay is critical in these comparisons. Hydrogen exhibits a significantly shorter igniting delay than methane owing to its reduced activation energy and accelerated flame propagation—the fast ignition of hydrogen combustion results in elevated peak temperatures and pressures. Methane, characterized by an extended ignition delay, demonstrates a more gradual combustion process, leading to reduced temperatures and pressures. Concerning network output, hydrogen at an equivalence ratio of 1 produces 1.58 MJ/kg, which diminishes with leaner mixtures, yielding 1.06 MJ/kg at an equivalence ratio of 0.5. Rich hydrogen combustion (1.77 MJ/kg) surpasses stoichiometric hydrogen due to enhanced fuel energy accessibility. Methane, at an equivalency ratio of 1, generates a network of 1.35 MJ/kg, which is inferior to hydrogen across all equivalence ratios due to its reduced specific energy. Transitioning from methane to hydrogen in engine operation entails substantial alterations. The elevated combustion energy and flame velocity of hydrogen necessitate altered injection systems to regulate ignition timing and prevent pre-ignition (Efsthios, 2023). The elevated peak temperatures and pressures require materials that endure significant thermal and mechanical strains. Moreover, hydrogen's tendency to produce reduced NO_x emissions at rich equivalency ratios presents environmental advantages but at the cost of fuel economy; nonetheless, its lean mixtures may necessitate sophisticated approaches to minimize NO_x generation. Hydrogen offers a cleaner and more efficient substitute for methane, yet it requires significant engine design and operation modifications to accommodate its distinct combustion properties.

3.2. Ignition Delay for Different Fuels Compared to Hydrogen

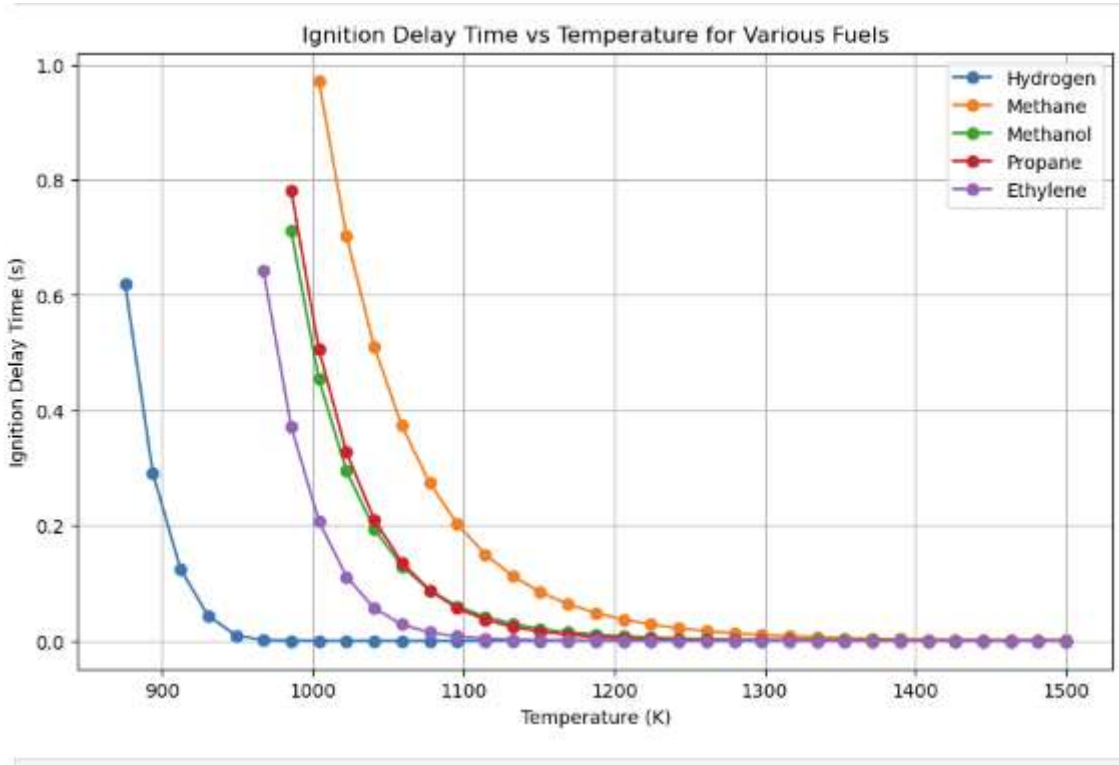


Figure 11. Different fuels' ignition delays.

The graph shows the ignition delay time of various fuels, including hydrogen, methane, methanol, propane, and ethylene, based on temperature. Hydrogen has the shortest delay, indicating its higher reactivity and lower activation energy, resulting in faster flame propagation and improved combustion efficiency. Methane has longer delayed, especially at lower temperatures, indicating slower reaction rates and higher energy thresholds. Propane and ethylene show intermediate behaviors, with their delays decreasing more steeply with rising temperatures. Methanol, closer to propane and ethylene, still lags hydrogen in terms of reactivity. This suggests hydrogen's suitability for high-efficiency combustion systems.

3.3. The Hydrogen-Methane Combination Impacts Heat Release and Emissions

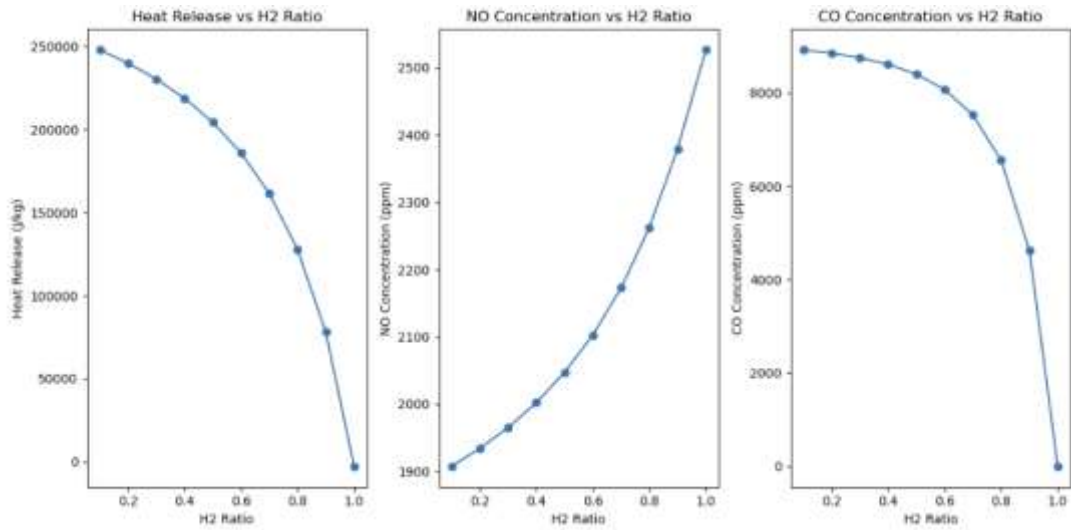


Figure 12. Emissions and heat release Impact of the percentage of hydrogen in the H2/Ch4 fuel blend.

Heat Release: Methane (CH_4) exhibits a significantly higher volumetric energy density (36 MJ/m^3) compared to hydrogen (H_2), which has only 10.8 MJ/m^3 . This indicates that, for an equivalent volume of fuel, the burning of Methane will produce roughly 3.3 times more energy. Hydrogen possesses a mass-based energy density of 120 MJ/kg , almost 2.4 times greater than that of methane, which is 50 MJ/kg . Volumetric systems, under stoichiometric conditions, combust less heat due to the low density of hydrogen despite its large mass energy. As the hydrogen ratio increases, it diminishes overall volumetric energy contribution, which explains the observed reduction in heat release in the reported simulations.

Nitrogen Oxide emissions: Hydrogen combustion produces a higher temperature than methane when comparing hydrogen and methane. Specifically, burning hydrogen at a 1:1 equivalent ratio achieves a temperature of 3,341 Kelvin, but the combustion of an equal amount of methane reaches a maximum of only 3,080 Kelvin. The elevated temperature produces significant quantities of nitrogen dioxide molecules during hydrogen combustion, exceeding those of methane combustion at stoichiometric ratios

Carbon dioxide emissions: The combustion of hydrogen produces no additional carbon-based emissions, reducing carbon monoxide concentrations. However, the combustion of methane produces emissions due to carbon oxidation at stoichiometric ratios. If a hydrogen ratio of one is maintained, carbon monoxide is absent, suggesting that the increase in carbon-based emissions may cease if hydrogen combustion systems are implemented.

3.4. NO_x Performance of Methane-Hydrogen Dual Fuel Engines

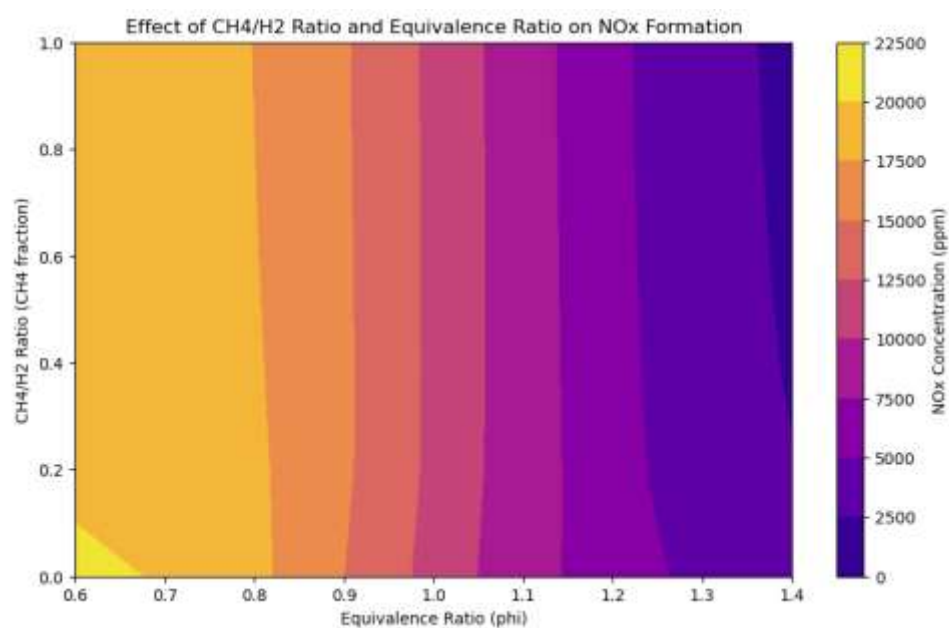


Figure 13. Heat Map of Dual Fuel Engine Equivalence and Methan to Hydrogen Ratio Emissions Effect.

The heatmap comprehensively depicts the correlation between the CH_4/H_2 ratio and equivalency ratio (ϕ) regarding NO_x production in a dual-fuel combustion system. NO_x concentrations peak at approximately 22,500 ppm under CH_4 -dominant ratios ($\text{CH}_4/\text{H}_2 \approx 1.0$) and stoichiometric circumstances ($\phi \approx 1.0$). This is due to the combustion properties of methane, which generate elevated peak flame temperatures under these conditions, promoting NO_x generation. As the H_2 ratio increases (resulting in a lower CH_4/H_2 ratio), NO_x concentrations markedly diminish, especially in fuel-rich mixes ($\phi > 1.2$). This reduction transpires as affluent mixes restrict oxygen availability, inhibiting the high-temperature reactions essential for thermal NO_x generation, notwithstanding increased flame temperatures. Conversely, lean mixes ($\phi < 0.9$) demonstrate diminished NO_x concentrations at all CH_4/H_2 ratios owing to lower flame temperatures. Excess air

in lean mixes serves as a thermal ballast, absorbing heat during combustion and diluting the reaction zone, hence lowering peak temperatures. Moreover, reduced fuel availability diminishes the total heat release, while slower reaction rates further constrain the energy output per unit of time, leading to substantial suppression of the Zeldovich mechanism. The heatmap identifies a significant NO_x hotspot for stoichiometric methane-rich blends, underscoring the necessity of meticulously balancing the equivalency ratio and fuel content to optimize emissions. Increasing the hydrogen ratio can significantly diminish NO_x emissions, particularly in fuel-rich settings with low oxygen. This illustrates hydrogen's capability to function as a more environmentally friendly substitute for methane in dual-fuel engine applications, yielding reduced emissions while preserving performance. The analysis highlights the necessity of regulating equivalency ratios and fuel mixtures to get optimal combustion properties and adhere to environmental standards.

3.5. NO_x Performance of Propane-Hydrogen Dual Fuel Engines

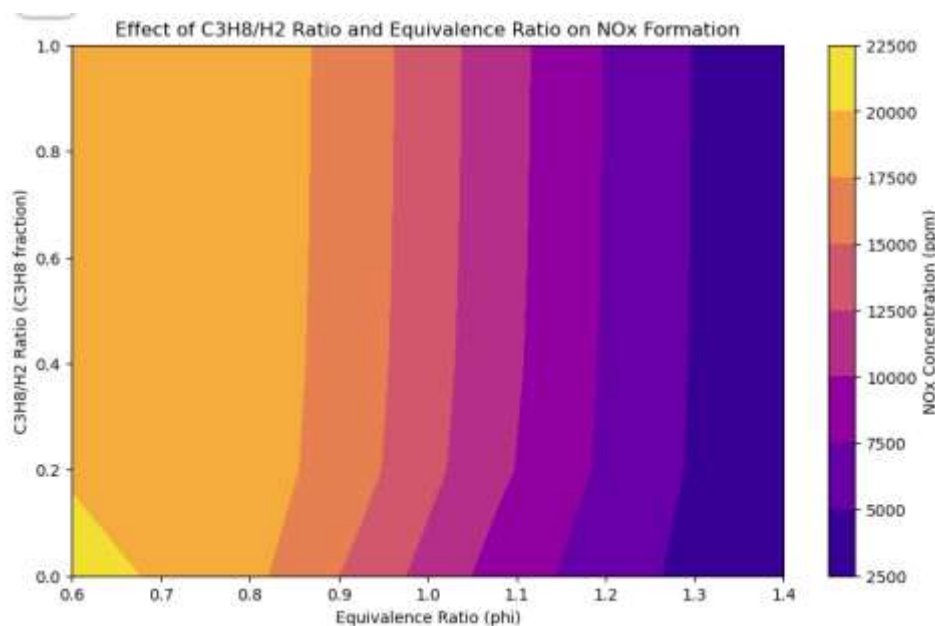


Figure 14. Heat Map of Dual Fuel Engine Equivalence and Propane to Hydrogen Ratio Emissions Effect.

The heatmap shows the effect of propane-to-hydrogen ratio and equivalence ratio on NO_x formation in dual-fuel combustion. The highest NO_x concentrations are found in propane-dominant mixtures under stoichiometric conditions, driven by higher flame temperatures. As the hydrogen ratio increases, NO_x concentrations decrease, especially in fuel-rich mixtures. Lean mixtures have lower NO_x levels due to reduced flame temperatures. Hydrogen-rich blends achieve the lowest NO_x emissions across all equivalence ratios, demonstrating hydrogen's ability to reduce NO_x formation in high-temperature combustion. This highlights a NO_x hotspot at stoichiometric propane-rich conditions, emphasizing the importance of hydrogen enrichment and non-stoichiometric equivalence ratios in propane-hydrogen dual-fuel systems.

3.6. Methane, Hydrogen, and Methanol Blend vs Fuel Percentages and Equivalence Ratio

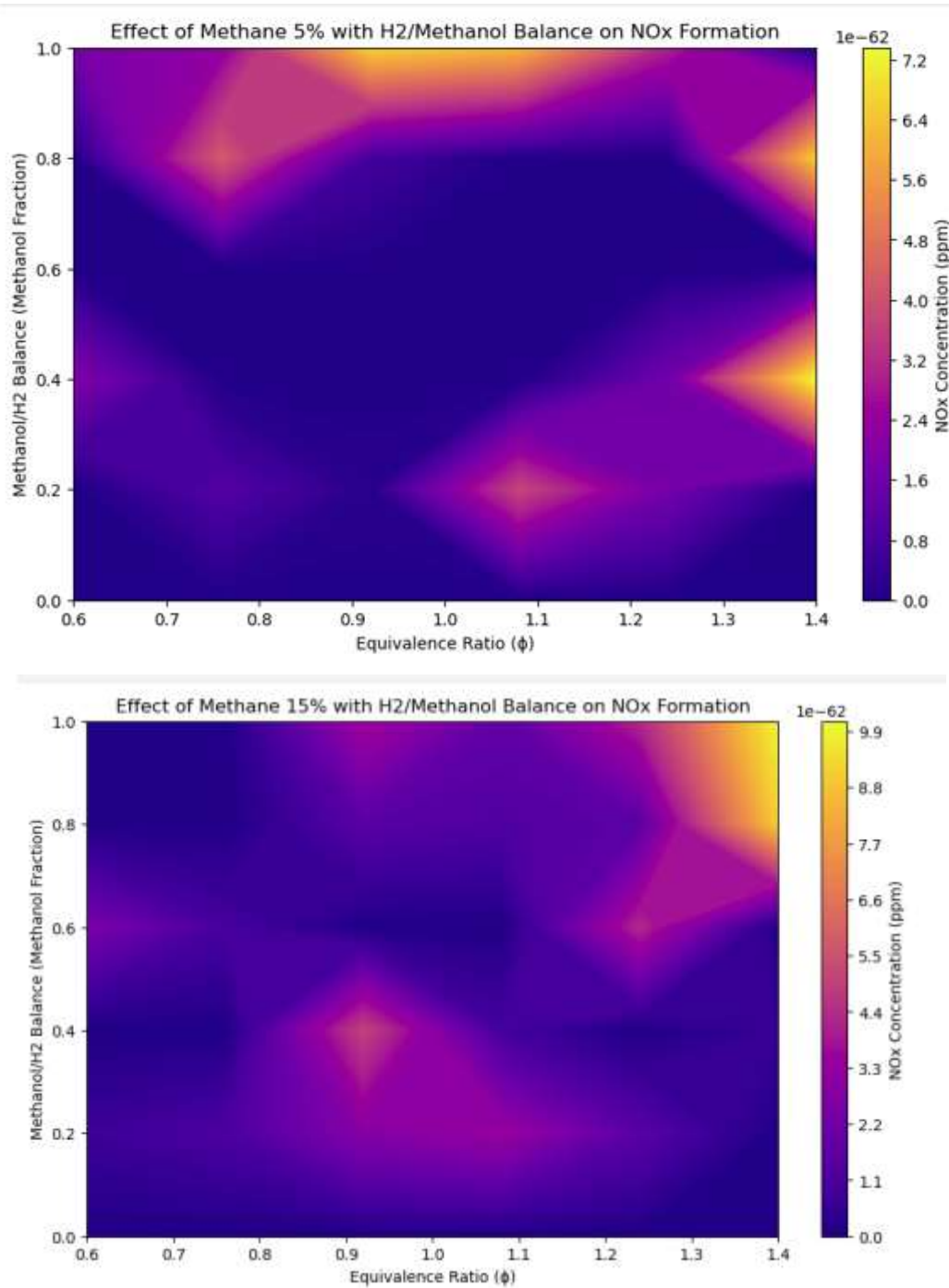


Figure 15. Combustion characteristics of three fuel blends: methane, hydrogen, and methanol vs equivalency ratio and fuel percentages .

The augmentation of the methane fraction from 5% to 15% results in a significant elevation of NOx concentrations throughout the fuel-to-air ratio and methanol fraction ranges. This signifies that methane combustion markedly increases NOx production owing to its elevated flame temperature relative to methanol. Increased temperatures amplify the thermal NOx process, resulting in heightened NOx production, particularly under conditions near the ideal fuel-to-air ratio. The Influence of the Fuel-to-Air Ratio: For both methane fractions, NOx concentrations reach their maximum at the optimal fuel-to-air ratio, where the equilibrium between fuel and oxygen generates the highest combustion temperatures. Under lean conditions (insufficient fuel), NOx formation diminishes due to lower temperatures, but in rich conditions (enough fuel), the lack of oxygen

constrains NO_x production despite elevated temperatures. The balance of methanol and hydrogen indicates that increased methanol percentages result in a reduction in NO_x emissions, especially under lean conditions. The reduced flame temperature of methanol, in contrast to methane and hydrogen, inhibits the thermal NO_x process. Conversely, augmenting the hydrogen proportion (by diminishing methanol) results in elevated NO_x emissions owing to hydrogen's remarkably high flame temperature, which accelerates thermal NO_x generation more vigorously than methane. The results underscore the specific contributions of methane, methanol, and hydrogen to NO_x emissions. Methane increases NO_x emissions owing to its elevated flame temperature, whereas methanol reduces NO_x generation through its cooling impact inside the mixture. The elevated flame temperature of hydrogen exacerbates NO_x production as its proportion rises. The results underscore the necessity of adjusting fuel composition and combustion parameters to reduce NO_x emissions, providing critical insights for the development of cleaner and more efficient fuel blends in advanced energy systems.

3.7. Emission levels of Hydrogen And Methanol Mixes.

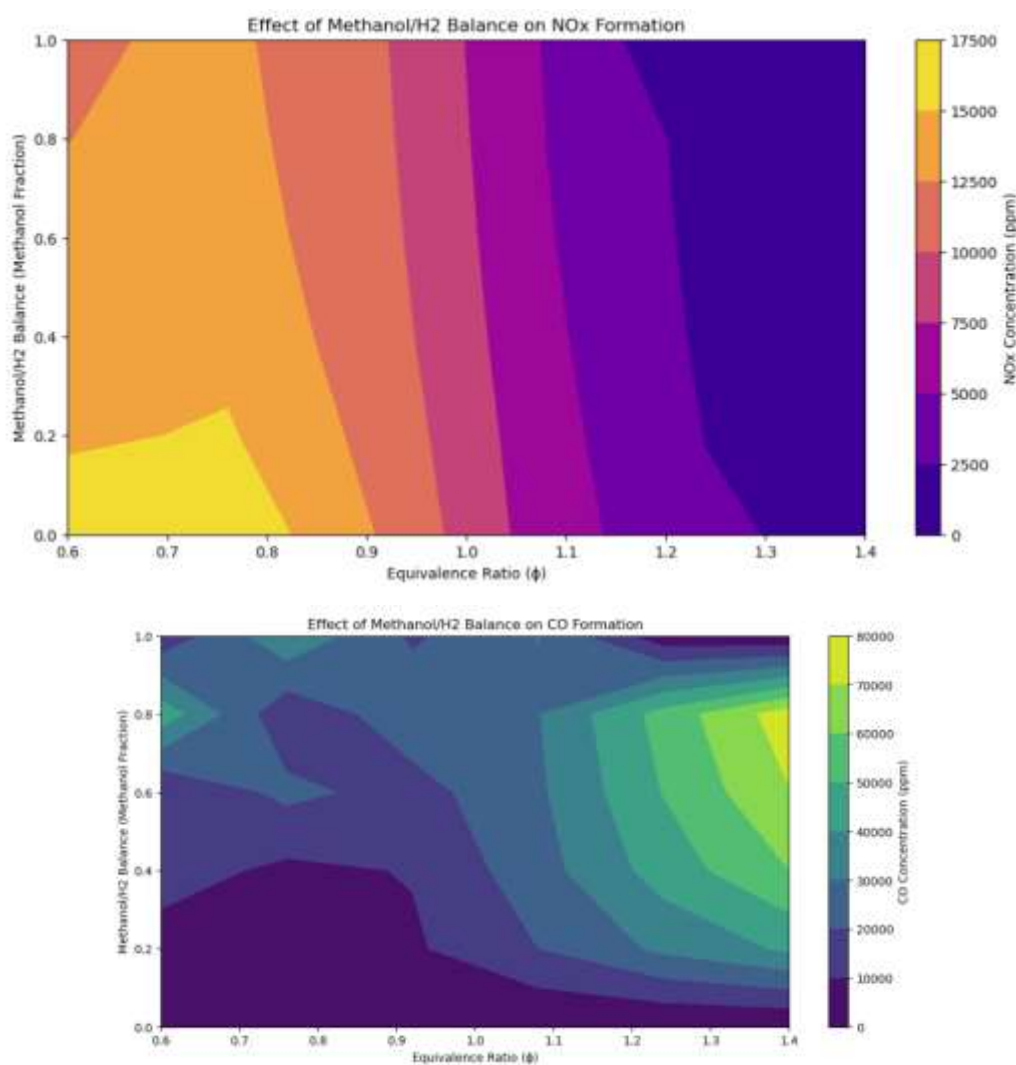


Figure 16. The impact of the equivalency ratio interactions between methanol and hydrogen mixtures on emissions levels.

The study examines the impact of methanol/hydrogen fuel content and combustion equivalence ratio on the generation of NO_x and CO. It highlights the interaction between oxygen availability (from methanol and air) and temperature effects, which dictate emission trajectories. NO_x production is maximized in areas where the equivalency ratio approaches stoichiometric levels, and the

methanol percentage exceeds 0.6. This is due to methanol's molecular structure containing oxygen, which improves combustion efficiency and increases flame temperature. Reduced NO_x emissions are observed in lean conditions ($\varphi < 0.8$), where surplus air reduces combustion temperature, inhibiting thermal NO_x generation. Abundant conditions ($\varphi > 1.2$) show that restricted air supply and surplus fuel diminish flame temperature, inhibiting NO_x generation due to inadequate nitrogen oxidation. As the methanol fraction diminishes and the hydrogen fraction escalates, NO_x emissions decline due to hydrogen combustion typically generating lower flame temperatures than methanol-dominant combustion. CO formation is maximized at rich combustion conditions ($\varphi > 1.2$) and elevated methanol fractions (> 0.6). In these conditions, they restricted oxygen supply from air, resulting in incomplete combustion, leaving a substantial amount of carbon as CO. The elevated carbon content of methanol intensifies this impact relative to hydrogen-rich fuels. The study also highlights the influence of temperature on NO_x production, with elevated temperatures facilitated by near-stoichiometric conditions and substantial methanol concentrations enhancing NO_x production. Oxygen availability affects CO production, with inadequate oxygen leading to incomplete combustion and elevated CO emissions. Increased methanol fraction often results in elevated NO_x emissions and heightened CO emissions under rich conditions. The study provides valuable insights into emission control tactics for blended methanol-hydrogen fuels.

4. Validation and Literature Review

(Jingyun Sun, 2023)The study on hydrogen-methanol blends with varying methane percentages (5% and 20%) aligns with Sun et al.'s 2023 study on pollutant formation mechanisms in hydrogen/ammonia/methanol ternary carbon-neutral fuel blend combustion. Both studies highlight the dual role of hydrogen and methanol in influencing NO_x emissions and combustion dynamics. Higher hydrogen fractions lead to elevated NO_x emissions due to increased flame temperatures, while methanol effectively mitigates NO_x emissions through lower combustion temperature and heat-absorbing characteristics. Both studies also highlight that NO_x emissions peak near stoichiometric or slightly rich equivalence ratios due to maximum flame temperatures. Methanol's contribution to reducing NO_x emissions in ternary and binary blends underscores its critical role in enabling low-emission combustion strategies. The inclusion of methane in the study further refines the understanding of multi-fuel blends, demonstrating the complementary effects of hydrogen's high reactivity and methanol's emission-reducing properties.

(Dimitriou, 2018) A study on a multi-cylinder compression ignition engine tested under different hydrogen-diesel energy share ratios found that it can reduce harmful emissions and BTE by up to 90% at low load conditions. However, the maximum H₂ energy share ratio drops to 85% at medium loads, leading to pre-ignition and unbalanced operation. At medium loads, NO_x emissions increase by up to four times compared to conventional engines due to high in-cylinder temperatures. Exhaust gas recirculation reduces NO_x emissions by up to 75%, but high EGR rates can deteriorate soot oxidation. Improved fuel atomization shifts the combustion phase closer to the TDC, reducing NO_x emissions and soot formation. Alternative technologies like water injection and low-temperature combustion techniques are necessary to reduce NO_x, soot, and carbon emissions simultaneously. The study reveals high flame temperatures and ample oxygen availability primarily drive high NO_x emissions in part-load operations. This is consistent with trends observed in hydrogen and propane blends under stoichiometric conditions. However, transitioning to rich hydrogen mixtures ($\phi > 1.2$) effectively reduces NO_x emissions by limiting oxygen availability, offering a potential solution to mitigate NO_x emissions in part-load operations. This highlights the importance of stoichiometric or near-lean equivalence ratios in optimizing efficiency.

(Li, 2023) The study confirms the strong correlation between flame temperature and NO_x emissions, highlighting the role of high temperatures in thermal NO_x formation. It also demonstrates that adjusting hydrogen equivalence ratios effectively controls NO_x emissions, emphasizing the importance of equivalence ratio management.

(Q.T. Dam, 2024)This paper presents a dynamic model for an internal combustion engine powered by hydrogen, focusing on engine efficiency, torque, and emissions reduction. The model

shows high efficiency in hydrogen combustion, but high combustion temperatures necessitate effective cooling systems. Simulations show torque output improvements and optimized fuel consumption, but the fuel's low density necessitates careful fuel injection control.

(Verhelst, 2023) (Wallnerb, 2009) The findings of this study regarding hydrogen-methane mixes corroborate and enhance our results by illustrating the trade-offs among NO_x emissions, energy density, and engine efficiency. Our findings demonstrate that hydrogen-rich blends produce more significant NO_x emissions owing to increased flame temperatures, aligning with this study's observation that higher methane concentrations diminish NO_x emissions by tempering flame temperatures. The reduction in engine performance with elevated methane percentage corresponds with our observations that hydrogen's better mass-based energy density improves thermal efficiency, especially at low equivalency ratios. The vehicle-level measurements in the study indicate a 3% enhancement in brake thermal efficiency with hydrogen relative to gasoline, corroborating our results of increased net work and efficiency for hydrogen, especially under stoichiometric conditions. These comparisons highlight that although the amalgamation of methane with hydrogen enhances energy storage density and diminishes NO_x emissions, it undermines efficiency, underscoring the necessity for meticulous tuning of blend ratios to reconcile emissions, efficiency, and storage capacity in real systems.

(S. Molina, 2023) This study supports our investigation into the feasibility of hydrogen as a fuel and how it interacts with dilution techniques for NO_x reduction. Our results show comparable trade-offs when blending hydrogen with methanol and methane, highlighting the significance of regulating dilution rates to balance thermal efficiency and combustion stability. Our results are consistent with the documented shrinking of stable combustion ranges at increasing dilution ($\lambda > 2$), when fuel blending is necessary to stabilize the high flame temperature of hydrogen. The possibility for improving injection tactics is also highlighted in both studies. Our research shows how methanol and hydrogen blends improve efficiency in lean situations, which is consistent with this study's emphasis on direct injection for performance gains.

(S. K. V, 2014) Both experiments demonstrate that, as a result of improved combustion properties, adding hydrogen to fuel mixes increases brake thermal efficiency and lowers CO and HC emissions. Both also note that, particularly at larger hydrogen fractions, the high flame temperature of hydrogen causes a rise in NO_x emissions. Additionally, both studies highlight how performance can be maintained and emission trade-offs minimized by blending hydrogen in controlled proportions, such as up to 20%.

(M. Huang, 2024.)The study investigates the impact of injector design and timing on hydrogen combustion in internal combustion engines. It reveals that larger injectors increase fuel delivery but increase pre-ignition risk. Early injection improves stability, while late injection reduces NO_x emissions but increases knock risk. Optimized injector configurations improve engine efficiency and reduce NO_x emissions.

(R.A. Bakar, May 2022)The study examines the efficacy of a direct injection diesel engine utilizing hydrogen in a dual-fuel configuration. Essential findings encompass the assessment of hydrogen flow rates, engine efficiency, NO_x emissions, and combustion stability. Both studies forecast a 15-20% enhancement in efficiency, attributable to hydrogen's superior flame velocity and reduced ignition energy. Both emphasize the difficulty of regulating NO_x emissions, with the experimental study revealing no substantial decrease in NO_x despite augmented hydrogen flow, but CO, CO₂, and particulate emissions were markedly diminished. Combustion stability is typical, especially in lean conditions, where pre-ignition and knock occur. Both underscore the potential of hydrogen to enhance engine performance yet illustrate the challenges in controlling emissions without adversely affecting other engine parameters.

(A. Keromnes, 2013) Our research examines the comparative examination of NO_x emissions and combustion properties of hydrogen and methane across different equivalency ratios and operational situations. Hydrogen combustion exhibits markedly reduced NO_x emissions in rich mixes (e.g., equivalence ratio 2.0) owing to restricted oxygen availability, whereas stoichiometric methane combustion produces elevated NO_x levels due to prolonged burn duration and increased combustion

pressures. Hydrogen's elevated flame temperature (e.g., 3341 K) and swift combustion rates lead to reduced ignition delays, accelerated combustion, and diminished NO_x emissions under specific conditions relative to methane. Nonetheless, methane's superior volumetric energy density results in elevated peak combustion pressures, highlighting its unique combustion dynamics. The cited study corroborates our results by illustrating hydrogen's decreased NO_x emissions in fuel-rich syngas mixtures, attributable to the inhibition of oxygen-dependent NO_x pathways and its swift chemical kinetics. Both investigations confirm hydrogen's enhanced reactivity and combustion efficiency, especially at rich equivalency ratios, while emphasizing the influence of temperature and pressure on emissions. The cited study supports our finding that leaner mixes elevate NO_x concentrations due to enhanced oxygen availability and elevated combustion temperatures. Both investigations confirm hydrogen's promise as a cleaner combustion fuel, highlighting its environmental advantages and unique properties relative to methane.

5. Conclusions

This work employed Cantera, an open-source computational tool, to examine ideal fuel mixes for improved energy efficiency and diminished emissions by simulating chemical kinetics, thermodynamics, and transport processes. The code enabled comprehensive study of dual and multi-fuel combustion, including comparison of ignition delay, equivalence ratio impacts, and pollutant emissions across different fuel combinations. The equivalency ratio greatly affects NO_x emissions and combustion properties of hydrogen (H₂) and methane (CH₄). For H₂, lean mixtures (equivalence ratio = 0.5) enhanced NO concentrations owing to surplus oxygen and greater combustion temperatures, while rich mixtures (equivalence ratio = 2.0) diminished NO_x to 389 ppm due to oxygen constraints. Conversely, methane burning under stoichiometric conditions ($\phi = 1.0$) yielded a moderate concentration of NO, which was lower than that produced by lean H₂ combustion but higher than that from rich mixes. Methane demonstrated a reduced reaction rate and lower peak combustion temperatures relative to H₂, which, at an equivalence ratio of 1.0, attained 3,341 K, whereas methane peaked at 3,080 K. These facts correspond with hydrogen's superior reactivity and energy density relative to methane.

The equivalency ratio and fuel blend composition substantially influence NO_x emissions in CH₄/H₂ dual-fuel combinations. NO_x reached its zenith under methane-dominant conditions (CH₄/H₂ = 1.0) at stoichiometric equivalency. Nevertheless, as the H₂ ratio augmented in rich mixes ($\phi > 1.2$), NO_x concentrations diminished owing to oxygen limitations constraining thermal NO_x production, despite greater flame temperatures. Lean mixes ($\phi < 0.9$) led to diminished NO_x concentrations across all CH₄/H₂ ratios, since extra air absorbed combustion heat, hence lowering flame temperatures.

In multi-fuel blends comprising CH₄, methanol, and hydrogen, methane generally represented 40-50% of the mixture. The incorporation of methanol enhanced energy density, whilst hydrogen diminished carbon emissions, so improving environmental performance. The burning of CH₄/methanol/H₂ mixtures at different equivalency ratios had substantial impacts on pollutant generation. Lean mixtures ($\phi < 0.9$) exhibited diminished NO_x emissions, whereas rich mixtures ($\phi > 1.2$) restricted oxygen availability, hence further inhibiting NO production. CO emissions peaked under stoichiometric conditions but diminished as hydrogen ratios grew, attributable to hydrogen's carbon-free combustion properties. These findings highlight the significance of optimizing the equivalency ratio and fuel blending to attain a balance between energy efficiency and pollution reduction.

This study illustrates that hydrogen-rich mixtures have significant environmental benefits, particularly when combined with methane or propane. Engine improvements and thermal management are essential to address the increased combustion temperatures and pressures linked to hydrogen. Utilizing techniques such as Cantera, researchers may determine ideal equivalency ratios and blend compositions for sustainable, high-performance combustion systems.

In conclusion, our findings are corroborated by other investigations. Research conducted by Jingyun Sun (2023) highlights hydrogen's contribution to increased NO_x emissions resulting from

elevated flame temperatures, while methanol demonstrates efficacy in their reduction, a phenomenon reflected in our findings. Dimitriou (2018) and Li (2023) affirm the significant influence of equivalency ratios on NO_x emissions, emphasizing hydrogen's capacity to diminish NO_x in rich settings. Research conducted by Verhelst (2023) and Wallnerb (2009) corroborates the trade-offs identified between NO_x emissions, energy density, and efficiency in hydrogen-methane mixtures, whereas S. Molina (2023) endorses the significance of dilution methods in achieving a balance between efficiency and combustion stability. The efficiency improvements and emission problems identified by S.K.V. (2014) and R.A. Bakar (2022) further validate our conclusions about NO_x reduction measures. Ultimately, A. Keromnes (2013) corroborates our findings on hydrogen's increased reactivity and diminished NO_x emissions in rich mixtures, although methane's superior volumetric energy density underscores specific trade-offs. These studies collectively underscore the importance of hydrogen-methanol-methane mixes in attaining cleaner and more efficient combustion.

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