

Finite Element Modeling of A phenomenological Constitutive Model for Super-elastic Shape Memory Alloy and its Application for Preload Process Analysis of Bolted Joint

Xiangjun Jiang¹; Huang Jin¹; Yongkun Wang^{1*}; Fengqun Pan¹; Baotong Li²; and Peng Hao³

¹ Key Laboratory of Electronic Equipment Structural Design, Xidian University, Xi'an, Shaanxi, 710071, P.R.C. E-mail: xjjiang@xidian.edu.cn

² State Key Laboratory for Manufacturing System, Xi'an Jiaotong University, Xi'an, Shaanxi, 710049, P.R.C. E-mail: baotong.me@mail.xjtu.edu.cn

³ State Key Laboratory of Structural Analysis for Industrial Equipment, Dalian University of Technology, Dalian, Liaoning, 116023, P.R.C. E-mail: haopeng@dlut.edu.cn

Abstract

A phenomenological constitutive model is developed to describe the uniaxial transformation ratcheting behaviors of super-elastic shape memory alloy (SMA) by employing a cosine-type phase transformation equation with the initial martensite evolution coefficient that can capture the feature of the predictive residual martensite accumulation evolution and the nonlinear hysteresis loop on a finite element (FE) analysis framework. The effect of the applied loading level on transformation ratcheting are considered in the proposed model. The evolutions of transformation ratcheting and transformation stresses are constructed as the function of the accumulated residual martensite volume fraction. The FE implementation of the proposed model is carried out for the numerical analysis of transformation ratcheting of the SMA bar element. The integration algorithm and the expression of consistent tangent modulus are deduced in a new form for the forward and reverse transformation. The numerical results are compared with those of existing model and the experimental results to show the validity of the proposed model and its FE implementation in transformation ratcheting. Finally, a FE modeling is established for a repeated preload analysis of SMA bolted joint

keywords: SMAs; Super-elasticity; FE implementation; Phase transformation ratcheting; Preload process analysis of bolt

1. Introduction

SMA has been widely applied in MEMS, actuators, biomedical devices, organ transplantation, transportation, aerospace and civil engineering due to its well-known super-elasticity, shape memory effect, excellent biocompatibility and wear resistance [1-3]. In the last two decades, with the deeper understanding of thermo-mechanical coupling behavior of SMA, more and more scholars pay attention to the response of thermo-mechanical coupling behavior of the alloy undergoing cyclic loading. The cyclic deformation behavior of SMA, as described in earlier research, was followed with interested by the research group led by Miyazaki [4-5]. The group's research concluded that, the residual irrecoverable deformation of SMA would increase gradually with the increase of cycle times under the mechanical loading cycles. The phase transformation stress would also decrease with the increase of cycle numbers. Strnadel et al. [6-7] had studied the effect of alloy elements and components on mechanical cyclic deformation behavior of super-elastic NiTi alloy by experiments, and revealed the inhibition effect of high nickel content on cyclic residual deformation. Nemat-Nasser had also observed the deformation features of super-elastic NiTi alloy under cyclic loadings by experiments [8]. It was revealed that the increment of residual martensite strain and the dissipation energy decreases with the increase of cycle numbers, and tend to be stable following several cycles.

The super-elasticity of NiTi SMA gradually deteriorates during the process of cyclic deformation. This phenomenon could mainly be expressed as four aspects: the residual martensite strain accumulation, the gradual decrease of transformation stresses, the gradual change of phase transformation modulus, and the gradual decrease of dissipation energy with the cyclic loadings [5, 9-13]. The researches done by [14-20] show that the mechanical behavior of the alloy is strongly dependent on the loading rate. The non-monotonic relationship can be found between the dissipation energy and the loading rate. The mechanical dissipation and the latent heat of phase transformation cause the heat generation of the alloy. Kan et al. [21] had also studied experimentally the cyclic deformation behavior of the untrained super-elastic NiTi SMA at different loading rates. The results show that all of the phase transformation modulus, the starting stress of martensitic transformation, the hysteresis loop and the residual strain of NiTi SMA are strongly dependent on the loading rate.

Based on existing experimental studies, in recent years, a series of phenomenological

constitutive models had been built by different scholars [22–25] to describe the thermo–mechanical coupling deformation characteristics of SMA. As a further work, the FE implementation of the proposed model was to be carried out to calculate an accurate stress-strain responses for SMA-based devices. In the generalized plasticity frame, the return-mapping algorithm was utilized for the analysis of super-elastic NiTi alloy. [26, 27]. However, it was not yet possible to describe the deterioration of super–elasticity and shape memory effect of untrained materials during cyclic deformation. In view of the shortcomings of the above work, some scholars had further extended the above constitutive model. Based on the existing experimental observations with cyclic deformation, the corresponding cyclic constitutive model were established by Lagoudas and Auricchio [28] to describe the accumulation of interfacial defects and residual martensite in cyclic deformation. Though the Lagoudas’s model shows a good capability in the prediction of the nonlinear hysteresis loop shape with a lot of material parameters, that model can only give a reasonable description for the cyclic deformation characteristic of a specific loading conditions, instead of predicting the dependence of that one on the applied loading level. Kan and Kang [29-30] made a further expansion of the cyclic constitutive model based on the experimental observation by Kang [31] that is able to reasonably describe the applied loading level related to super–elastic deterioration phenomenon. Furthermore, the Kan-Kang's model accounts for the asymmetric effect of tension and compression of the material on the phase transformation ratcheting behavior and the effect of applied loading level with just a small number of material parameters. Nevertheless, the linear phase transformation hardening rule employed by Kan-Kang's model cannot predict the nonlinear hysteresis loop shape well.

This paper thus develops a phenomenological constitutive model to describe the uniaxial transformation ratcheting behaviors of super–elastic SMA. A cosine–type phase transformation equation with the initial martensite evolution coefficient that can capture the feature of the predictive residual martensite accumulation evolution and the nonlinear hysteresis loop is employed in the proposed model on a FE analysis framework. The return-mapping algorithm and the consistent tangent modulus are deduced in a new form for the phase transformation. The validity of the proposed model and its FE implementation in transformation ratcheting is finally examined by comparison between the proposed model and the existing model and experimental results. Finally, a numerical example is given to analyze the repeated preload process of SMA

bolted joint.

2 Constitutive modeling for NiTi SMA under cyclic loading

In the last one decade, many researchers focused on the constitutive modeling to describe the cyclic deformation of NiTi SMA. The phenomenological constitutive model shows good candidate to be integrated into the structure computational methods, such as FE method, to predict the cyclic deformation of SMA structure. However, these models with FE implementation seem to be difficult to analyze efficiently the complex structure due to relatively low computation efficiency and nonlinearity.

Therefore, in this work, an one-dimensional constitutive model for phase transformation ratcheting of super-elastic SMA is proposed based on the Brinson-model, and then implemented into FE model of one-dimensional bar element to describe the cyclic deformation of some mechanical structures, such as SMA bolt joint, SMA washer, and so on.

2.1 Constitutive equation and internal variables

In the proposed model, the total strain $\boldsymbol{\varepsilon}$ can be decomposed into an elastic strain tensor $\boldsymbol{\varepsilon}^e$ and an inelastic strain tensor $\boldsymbol{\varepsilon}^{in}$ with infinitesimal strain assumption.

$$\boldsymbol{\varepsilon} = \boldsymbol{\varepsilon}^e + \boldsymbol{\varepsilon}^{in} \quad (1)$$

The residual deformation of SMA under cyclic loadings is considered to be attributed to the residual martensite deformation due to the phase transformation ratcheting. In order to characterize this deformation mechanisms, there are two internal variables are introduced here in Helmholtz free energy, which is assumed to be additively decomposed into elastic and inelastic parts, as follow:

$$\Psi = \Psi_e(\boldsymbol{\varepsilon} - \boldsymbol{\varepsilon}^{in}, \xi, \delta, T) + \Psi_{in}(\xi, \delta, T) \quad (2)$$

The internal variable ξ as the martensite volume fraction depicts the stress-induced martensite phase transformation, and is constrained by $0 \leq \xi \leq 1$. δ characterizes the accumulative martensite transformation, including the accumulated martensite volume fraction δ_c .

From the principle of thermodynamics, Clausius–Duhem inequality can be expressed as

$$\boldsymbol{\sigma} : \dot{\boldsymbol{\varepsilon}} - \dot{\Psi}_e(\boldsymbol{\varepsilon} - \boldsymbol{\varepsilon}^{in}, \xi, \delta, T) - \dot{\Psi}_{in}(\xi, \delta, T) \geq 0 \quad (3)$$

where

$$\dot{\Psi}_e = \boldsymbol{\sigma} : (\dot{\boldsymbol{\varepsilon}} - \dot{\boldsymbol{\varepsilon}}^{in}) = \frac{\partial \Psi}{\partial (\boldsymbol{\varepsilon} - \boldsymbol{\varepsilon}^{in})} : (\dot{\boldsymbol{\varepsilon}} - \dot{\boldsymbol{\varepsilon}}^{in}) \quad (4a)$$

$$\dot{\Psi}_{in} = \frac{\partial \Psi}{\partial \xi} : \dot{\xi} + \frac{\partial \Psi}{\partial \delta} : \dot{\delta} = \frac{\partial \Psi}{\partial \xi} : \dot{\xi} + \frac{\partial \Psi}{\partial \delta_c} : \dot{\delta}_c \quad (4b)$$

The incomplete phase transformation between martensite and austenite could be observed during the cyclic phase transformation [29–31]. Furthermore, the amount of residual martensite would increase with the increasing number of loading cycles. Therefore, the total induced–martensite volume fraction ξ that represents the progressive increase of residual martensite strain is set as internal variable and divided into two parts, i.e., reversible martensite volume fraction ξ^r and irreversible residual one ξ^{ir} .

$$\xi = \xi^r + \xi^{ir} \quad (5)$$

According to the experimental observations, the phase transformation deformation evolves with the increasing number of loading cycles, and reaches a stable value after a certain cycle [29–31]. In order to account these evolution process, the internal variable δ_c is picked as the accumulated martensite volume fraction ξ^c that represents the evolution process of some variables and material parameters with the increasing number of cycles, and is written by

$$\xi^c = \int_0^t |\dot{\xi}^r(\tau)| d\tau \quad (6)$$

where, t is a kinematic time.

Based on the frame of generalized plasticity, the total inelastic strain $\boldsymbol{\varepsilon}^{in}$ in Eq. (1) equals to the transformation strain $\boldsymbol{\varepsilon}^{tr}$ from stress–induced martensite phase and its reverse.

$$\boldsymbol{\varepsilon}^{in} = \boldsymbol{\varepsilon}^{tr} \quad (7)$$

The SMA constitutive model used here is based on a model originally formulated by [22–23], which is a phenomenological macro–scale one–dimensional constitutive model and can be written as

$$\sigma - \sigma_0 = E^s(\xi)\varepsilon - E^s(\xi_0)\varepsilon_0 + \Theta(T - T_0) + \Omega(\xi)\xi - \Omega(\xi_0)\xi_0 \quad (8)$$

where E^s is the Young's modulus, Ω is the transformation coefficient, Θ is the thermal elastic coefficient; T is the temperature. The subscript '0' indicates the initial values. Young's modulus E^s and transformation coefficient Ω are the function of the martensite volume fraction ξ , which are given as

$$E^s(\xi) = E_A + \xi(E_M - E_A) \quad (9a)$$

$$\Omega(\xi) = -\varepsilon_L E^s(\xi) \quad (9b)$$

It is assumed that the equivalent transformation strain and the recoverable martensite volume fraction shows proportional relationship (irreversible martensitic transformation does not participate in the transformation process). Then, the following relationship can be obtained

$$\xi = \frac{\varepsilon^{tr}}{\varepsilon_L} \quad (10)$$

Eq. (5) illustrates that the total induced-martensite volume fraction ξ consists of reversible martensite volume fraction ξ^r and irreversible residual one ξ^{ir} . Since the irreversible martensitic transformation does not take part in the phase transformation process, the stress is only the function of reversible martensite volume fraction, which is irrelevant to the residual martensite volume fraction in the phase transformation. The expression forms of ξ^r are as follows:

(i) Transformation to martensite phase

if $T > M_s$ and $\sigma_s^{cr} + C_M(T - M_s) < \sigma < \sigma_f^{cr} + C_M(T - M_s)$:

$$\xi_{A \rightarrow M}^r = \frac{1 - \xi_f^{ir}}{2} \cos \left\{ \frac{\pi}{\sigma_s^{cr} - \sigma_f^{cr}} [\sigma - \sigma_f^{cr} - C_M(T - M_s)] \right\} + \frac{1 + \xi_f^{ir}}{2} \frac{M \rightarrow A}{2} \quad (11)$$

(ii) Transformation to austenite phase

if $T > A_s$ and $C_A(T - A_f) < \sigma < C_A(T - A_s)$:

$$\xi_{M \rightarrow A}^r = \frac{\xi_f^r - \xi^{ir}}{2} \left\{ \cos \left[a_A \left(T - A_s - \frac{\sigma}{C_A} \right) \right] + 1 \right\} \quad (12)$$

where, σ_s^{cr} and σ_f^{cr} are starting and finishing stress of martensite transformation, C_A and

C_M are slope for the relation between critical transformation stress and temperature, A_s and A_f are starting and finishing temperature of austenite transformation, M_s and M_f are starting and finishing temperature of martensite transformation, $\xi_{A \rightarrow M}^r$ is the volume fraction of martensite transformation at the end of transformation to martensite phase, $\xi_{M \rightarrow A}^{ir}$ is the residual volume fraction of martensite transformation at the end of transformation to austenite phase. The parameters a_A and a_M are expressed as

$$a_A = \frac{\pi}{A_f - A_s}, \quad a_M = \frac{\pi}{M_f - M_s} \quad (13)$$

Now, Eq. (8) can be rewritten for the no initial values case with considering the irrecoverable feature as follow

$$\sigma = E^s(\xi)\varepsilon + \Omega(\xi)\xi^r \quad (14)$$

2.2 Evolution law of parameters governed by accumulated martensite volume fraction

As described in [28-30], the evolution of transformation ratcheting and transformation stresses with the number of loading cycles are related to the accumulated martensite volume fraction δ_c . It was found by Lagoudas [28] that the evolution of peak strain and valley strain was owing to the transformation ratcheting, and provided an evolution equation by taking the accumulated martensite volume fraction as governing variable. However, the further research by Kan [29] found that the parameters of transformation ratcheting and transformation stress associated with the loading stress level. The detailed evolution equations can be referred from the previous work [29]. The evolution equations are outlined as follows to satisfy the integrity of the content in this work.

(i) Evolution equation for residual martensite volume fraction

$$\dot{\xi}^{ir} = \xi_{\max}^{ir} c_{AM}^f(\sigma) e^{-b\xi^c} \dot{\xi}^c \quad (15)$$

where ξ_{\max}^{ir} is the maximum irreversible residual martensite volume fraction corresponding to the maximum load of stable cycle phase transformation. The material parameter b is to govern the saturation rate of residual martensite volume fraction ξ^{ir} . The revised function $c_{MA}(\sigma)$ is

introduced in the evolution law associated with the loading stress level to consider the correlation between that level and the residual martensite fraction, and written as

$$c_{AM}(\sigma) = \left(\frac{\langle Q_f^{AM} - Q_s^{AM} - \langle Q_f^{AM} - \sigma \rangle \rangle}{Q_f^{AM} - Q_s^{AM}} \right)^n \quad (16a)$$

$$c_{AM}^f(\sigma) = \max(c_{AM}^f(\sigma)) \quad (16b)$$

where $Q_f^{AM} = \sigma_f^{AM}$, $\langle x \rangle = \frac{1}{2}(x + |x|)$, c_{AM}^f is the value of c_{AM} at the endpoint of the forward transformation. The material parameter n is to describe the nonlinear relationship between the residual martensite volume fraction and the loading stress level. It can be found according to Eq. (19a) that in the forward phase, the value of c_{AM} increases with the increase of the loading stress level in the interval of $Q_s^{AM} \leq \sigma \leq Q_f^{AM}$, and reaches its maximum at the endpoint of forward transformation.

(ii) Evolution law of transformation stress

Due to incomplete phase transformation during loading cycles, the super-elastic NiTi SMA shows the mixture state of the austenite phase and residual martensite phase. The transformation stresses decrease with the increasing number of loading cycles. Thus, based on the experimental observations [31], the evolution equations with an exponential formulation were proposed by [29] to describe the progressive evolution of the transformation stresses with increasing number of loading cycles from their initial values to stable ones, and are introduced here as

$$\sigma_s^{AM} = \sigma_{s_0}^{AM} - (\sigma_{s_0}^{AM} - \sigma_{s_1}^{AM})(1 - e^{-c_s^{AM} \xi^c}) \quad (17a)$$

$$\sigma_f^{AM} = \sigma_{f_0}^{AM} - (\sigma_{f_0}^{AM} - \sigma_{f_1}^{AM})(1 - e^{-c_f^{AM} \xi^c}) \quad (17b)$$

$$\sigma_s^{MA} = \sigma_{s_0}^{MA} - (\sigma_{s_0}^{MA} - \sigma_{s_1}^{MA})(1 - e^{-c_s^{MA} \xi^c}) \quad (17c)$$

$$\sigma_f^{MA} = \sigma_{f_0}^{MA} - (\sigma_{f_0}^{MA} - \sigma_{f_1}^{MA})(1 - e^{-c_f^{MA} \xi^c}) \quad (17d)$$

where $\sigma_{s_0}^{AM}$, $\sigma_{f_0}^{AM}$, $\sigma_{s_0}^{MA}$ and $\sigma_{f_0}^{MA}$ are the transformation stresses of initial cyclic loading, and $\sigma_{s_1}^{AM}$, $\sigma_{f_1}^{AM}$, $\sigma_{s_1}^{MA}$ and $\sigma_{f_1}^{MA}$ are the transformation stresses of stable phase transformation. c_s^{AM} , c_f^{AM} , c_s^{MA} and c_f^{MA} are the parameters to govern the saturated rates of the transformation stresses.

3 FE modeling for one-dimensional bar element of SMA ratcheting behavior

3.1 Numerical integration algorithm

Based on the infinitesimal strain assumption, the increment of substep $n+1$ of total strain $\Delta \varepsilon_{n+1}$ can be defined as the summation of the increments of elastic strain, and transformation strain,

$$\Delta \varepsilon_{n+1} = \Delta \varepsilon_{n+1}^e + \Delta \varepsilon_{n+1}^{tr} \quad (18)$$

According to Eq. (10), the increment formulation of transformation strain can be written as

$$\Delta \varepsilon_{n+1}^{tr} = \varepsilon_L \Delta \xi_{(n+1)}^r \quad (19)$$

Based on Eq. (14), the stress in incremental calculation can be obtained as

$$\sigma_{n+1} = E_{n+1}^s(\xi_{n+1}) \left(\varepsilon_{n+1} - \varepsilon_L \Delta \xi_{(n+1)}^r \right) \quad (20)$$

in which, $\varepsilon_{n+1} = \varepsilon_n + \Delta \varepsilon_{n+1}$ is the strain in the current increment step, and ε_n is the strain in the substep n .

In this research, the total martensite volume fraction is divided into two parts, i.e., reversible martensite volume fraction ξ^r and irreversible residual one ξ^{ir} . Hence, the increment of reversible martensite volume fraction can be written as:

$$\Delta \xi_{(n+1)}^r = \Delta \xi_{(n+1)} - \Delta \xi_{(n+1)}^{ir} \quad (21)$$

According to Eqs. (11-12, 17), the phase transformation conditions can be introduced as:

$$F_{AM}(\sigma, \xi) = \sigma - \sigma_s^{AM}(\xi) = 0, \text{ forward phase transformation} \quad (22a)$$

$$F_{MA}(\sigma, \xi) = \sigma - \sigma_s^{MA}(\xi) = 0, \text{ reverse phase transformation} \quad (22b)$$

Substituting Eq. (20) into Eq. (22), it gives

$$E_{n+1}^s(\xi_{n+1}) \left(\varepsilon_{n+1} - \varepsilon_L \Delta \xi_{(n+1)}^r \right) - \sigma_s(\xi) = 0 \quad (23)$$

in which, $\sigma_s(\xi) = \sigma_s^{AM}(\xi)$ for forward phase transformation, and $\sigma_s(\xi) = \sigma_s^{MA}(\xi)$ for reverse phase transformation.

Solving Eq. (23) by the Newton-Raphson method yields

$$c^{tr} = \frac{E_{n+1}^s(\xi_{n+1}) \left(\varepsilon_{n+1} - \varepsilon_L \Delta \xi_{(n+1)}^r \right) - \sigma_s(\xi)}{\varepsilon_{n+1} \left(\frac{dE_{n+1}^s}{d\xi_{n+1}} \right) - \varepsilon_L \left(\frac{dE_{n+1}^s}{d\xi_{n+1}} \right) \Delta \xi_{(n+1)}^r - \varepsilon_L \lambda_{(n+1)}^{tr} E_{n+1}^s(\xi_{n+1}) - H_{tr}} \quad (24)$$

in which, $H_{tr} = H_{for}$ for forward transformation, $H_{tr} = H_{rev}$ for reverse transformation. \square

The increment of total martensite volume fraction can be updated in substep $k+1$ of Newton iteration by Eq. (20) until the convergence conditions is reached, i.e., $\left|c^{tr}\right|_{k+1}/\left|\Delta\xi_{(n+1)}\right|_{k+1} < Toler$.

$$\Delta\xi_{(n+1)}\Big|_{k+1} = \Delta\xi_{(n+1)}\Big|_k - c^{tr}\Big|_{k+1} \quad (25)$$

Derived from Eqs. (5, 9, 15), the incremental formulation of some variables in Eq. (24) can be obtained as

$$\frac{dE^s}{d\xi} = E_M - E_A \quad (26)$$

$$\lambda^{tr} = \begin{cases} \lambda_{A \rightarrow M}^{tr} = \frac{d\xi^{r}}{d\xi_{A \rightarrow M}} = 1 \\ \lambda_{M \rightarrow A}^{tr} = \frac{d\xi^{r}}{d\xi_{M \rightarrow A}} = \frac{1}{1 + \xi_{ir \max}^r c_{AM}^f b e^{-b\xi^r}} \end{cases} \quad (27)$$

After $\Delta\xi_{(n+1)}$ and $\Delta\xi_{(n+1)}^r$ are obtained, the increment of transformation strain and the stress and can be updated by Eqs. (19) and (20).

3.2 Solving for incremental stiffness matrix

In order to construct the FE model of one-dimensional bar element of super-elastic NiTi SMA ratcheting behavior, the constitutive equation as Eq. (14) should be expressed as the incremental formulation. The increment stiffness matrix of bar element should be further derived. Therefore, Eq. (14) needs first to be expressed as the relationship of the change in nodal force and the elemental deformation in length. Then it is carried through a resolution of nonlinear equation by the variational method to obtain the stiffness matrix. The relationship between of force and deformation is expressed as

$$\mathbf{F}^s = E^s(\xi) A^s \ln\left(\frac{L^s}{L_0^s}\right) \frac{\mathbf{x}}{L^s} + \Omega(\xi) \xi^r A^s \frac{\mathbf{x}}{L^s} \quad (28)$$

where, $\mathbf{F}^s = \boldsymbol{\sigma}^s A^s = \sigma^s A^s \frac{\mathbf{x}}{L^s}$ is the force vector of bar element, σ^s is the axial stress of SMA bolt bar, A^s is the cross sectional area of bar element, L_0^s is the unstressed initial length

of bar element, L^s is the real-time length of bar element after deformation; \mathbf{x} is the nodal coordinates of bar element, and \mathbf{x}/L^s is the nodal position vectors of bar element, $\boldsymbol{\sigma}^s = \sigma^s A^s \mathbf{x}/L^s$ is the vector formula of axial stress.

Since the phase transformation strain is a relative large value, the total strain is calculated in the logarithmic formulation $\varepsilon = \ln(L^s/L_0^s)$ to approximate the real one.

The variation formula of \mathbf{F}^s can be expressed as

$$\Delta \mathbf{F}^s = \frac{d\mathbf{F}}{d\xi^r} \frac{d\xi^r}{d\mathbf{x}} \Delta \mathbf{x} = \mathbf{K}_U^{sma} \Delta \mathbf{x} \quad (29)$$

in which,

$$\frac{d\mathbf{F}}{d\xi^r} = A^s \frac{d\boldsymbol{\sigma}}{d\xi^r} = A^s \frac{d\sigma}{d\xi^r} \frac{\mathbf{x}}{L^s} \quad (30)$$

According to the variation formula of Eq. (28), the following expression can be obtained as

$$\frac{d\xi^r}{d\mathbf{x}} = \frac{\boldsymbol{\beta}}{\boldsymbol{\alpha}} \quad (31)$$

and

$$\boldsymbol{\alpha} = \left[\frac{d\sigma}{d\xi^r} - \frac{\xi^r}{\lambda^{tr}} \ln\left(\frac{L^s}{L_0^s}\right) \frac{dE^s}{d\xi} - \frac{\xi^r}{\lambda^{tr}} \frac{d\Omega}{dE^s} \frac{dE^s}{d\xi} - \Omega \right] \frac{\mathbf{x}}{L^s}$$

$$\boldsymbol{\beta} = \left\{ \left[1 + \ln\left(\frac{L^s}{L_0^s}\right) \right] \frac{E^s}{L^s} - \ln\left(\frac{L^s}{L_0^s}\right) \frac{E^s}{L^s} \frac{\mathbf{xx}^T}{L^{s2}} \right\} + (\Omega \xi^r - \sigma) \left(\frac{1}{L^s} - \frac{\mathbf{xx}^T}{L^{s3}} \right)$$

From Eq. (9), it can be derived

$$\begin{cases} \frac{dE^s}{d\xi} = E_m - E_a \\ \frac{d\Omega}{dE^s} = -\varepsilon_L \end{cases} \quad (32)$$

Now, the stiffness matrix \mathbf{K}_U^{sma} cannot be obtained yet, unless the increment of the phase transition $d\sigma/d\xi^r$ is given. The derivation process of the increment of the phase transition will be discussed as follow.

(i) Transition to martensite phase

According to the model discussed in [36], the forward transition stresses can be expressed as

$$\sigma_s^{AM} = \sigma_s^{cr} + C_M(T - M_s) \quad (33a)$$

$$\sigma_f^{AM} = \sigma_f^{cr} + C_M(T - M_s) \quad (33b)$$

Eq. (11) can be rewritten as

$$\sigma = \sigma_f^{AM} + \left[(\sigma_s^{AM} - \sigma_f^{AM}) / \pi \right] \arccos \left\{ 2 \left[\frac{\xi_{A \rightarrow M}^r - \left(1 + \frac{\xi_{M \rightarrow A}^{ir}}{2} \right)}{1 - \frac{\xi_{M \rightarrow A}^{ir}}{2}} \right] \right\} \quad (34)$$

The differential formulation of Eq. (34) can be further expressed as

$$\frac{d\sigma}{d\xi_{A \rightarrow M}^r} = \Gamma_0^{AM} \frac{(\sigma_s^{AM} - \sigma_f^{AM})}{(\Gamma_1^{AM} + \Gamma_2^{AM} - 1)} \quad (35)$$

in which,

$$\Gamma_0^{AM} = \left[2 \left(1 - \frac{\xi_{M \rightarrow A}^f}{\pi} \right) \right] \left\{ 1 - \left[2 \frac{\xi_{A \rightarrow M}^r - \left(1 + \frac{\xi_{M \rightarrow A}^f}{2} \right)}{1 - \frac{\xi_{M \rightarrow A}^f}{2}} \right]^2 \right\}^{(1/2)}$$

$$\Gamma_1^{AM} = -c e^{-c\xi_c} (\sigma_s^{AM} - \sigma_f^{AM})$$

and

$$\Gamma_2^{AM} = c e^{-c\xi_c} \left[(\sigma_{f_0}^{AM} - \sigma_{f_1}^{AM}) - (\sigma_{s_0}^{AM} - \sigma_{s_1}^{AM}) \right] \arccos \left\{ \left[2 \frac{\xi_{A \rightarrow M}^r - \left(1 + \frac{\xi_{M \rightarrow A}^{ir}}{2} \right)}{1 - \frac{\xi_{M \rightarrow A}^{ir}}{2}} \right] \right\} / \pi$$

(ii) Transition to austenite phase

For reverse transition, Eq. (12) can be rewritten as

$$\sigma = C_A(T - A_s) - (C_A/a_A) \arccos \left[2 \frac{\xi_{M \rightarrow A}^r}{\left(\frac{\xi_{f_1}^r - \xi_{f_0}^r}{2} - 1 \right)} \right] \quad (36)$$

With the description in [36], the reverse transition stresses can be expressed as

$$\sigma_s^{MA} = C_A(T - A_s) \quad (37a)$$

$$\sigma_f^{MA} = C_A(T - A_f) \quad (37b)$$

Another expression can be written as

$$A_s = T - \sigma_s^{MA} / C_A \quad (38a)$$

$$A_f = T - \sigma_f^{MA} / C_A \quad (38b)$$

Therefore, Eq. (13) can be expressed as

$$a_A = \frac{\pi}{A_f - A_s} = \frac{\pi C_A}{\sigma_s^{MA} - \sigma_f^{MA}} \quad (39)$$

Therefore, the differential formulation of Eq. (36) can be derived as

$$\frac{d\sigma}{d\xi_{M \rightarrow A}^r} = \Gamma_0^{MA} \frac{(\sigma_s^{MA} - \sigma_f^{MA})}{(1 - \Gamma_1^{MA} - \Gamma_2^{MA})} \quad (40)$$

in which,

$$\Gamma_0^{MA} = -2 / \left[\pi \left(\xi_{A \rightarrow M}^r - \xi_{M \rightarrow A}^{ir} \right) \right] \left\{ 1 - \left[2 \xi_{A \rightarrow M}^r / \left(\xi_{A \rightarrow M}^r - \xi_{M \rightarrow A}^{ir} \right) - 1 \right]^2 \right\}^{(1/2)}$$

$$\Gamma_1^{MA} = -c e^{-c \xi^c} (\sigma_{s_0}^{MA} - \sigma_{s_1}^{MA})$$

and

$$\Gamma_2^{MA} = c e^{-c \xi^c} \left[(\sigma_{s_0}^{MA} - \sigma_{s_1}^{MA}) - (\sigma_{f_0}^{MA} - \sigma_{f_1}^{MA}) \right] \arccos \left\{ 2 \xi_{M \rightarrow A}^r / \left(\xi_{A \rightarrow M}^r - \xi_{M \rightarrow A}^{ir} \right) - 1 \right\} / \pi$$

Combined with Eqs. (29-32, 35, 40), the stiffness matrix \mathbf{K}_U^{sma} of single bar element of SMA can be obtained.

4 Numerical simulation and model verification

In this section, cyclic responses for ratcheting associated with one dimensional cyclic behavior of SMA by current FE model are verified by some typical experimental results outlined in this section [31] and additional experimental observations made in the current work to describe the transformation ratcheting during the stress-controlled cyclic loading at room temperature. The typical tensile-unloading stress-strain curve of super-elastic SMA is shown in Fig. 1. It is shown an apparent super-elastic feature of SMA. However, its curve presents a little bit different from the description in the referred literature for the NiTi SMA manufactured by other companies (e.g., SMA, San Jose, CA, USA). It exhibits an apparent hardening behavior during the stress-induced martensite transformation. After unloading, it could be seen a relative high residual strain ($\varepsilon_r=1.5\%$) remained. The high residual strain implies that there is an incomplete phase transformation from the stress-induced martensite to original austenite after unloading, which leads to some remained amount of martensite. It is further found that the amount of remained martensite increases progressively with the cyclic loadings. It is different from the ratcheting of ordinary metals without phase transformation, this phenomenon of accumulation deformation had been named as “phase transformation ratcheting”, and discussed with more details in [28–31]. It

illustrates that the pure austenite replaced by the mixture of austenite and remained martensite in the metal after the cycle loadings. Certainly, the stresses are no longer the phase transformation stresses of the pure austenite and martensite.

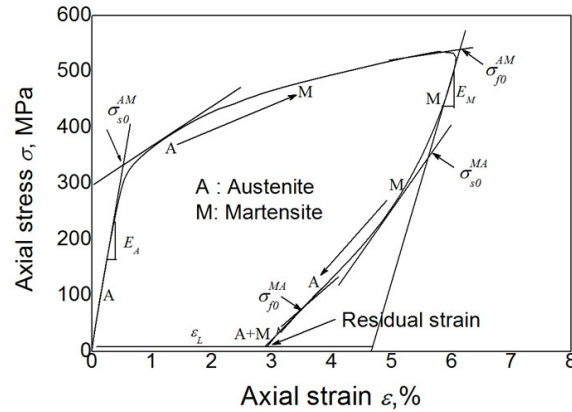


Fig. 1. Typical tension–unloading stress–strain curve of materials [29]

Some parameters should be defined prior to discuss the ratcheting deformation of super-elastic SMA, as shown in Fig. 1. Those parameters include the elastic modulus of austenite E_A and martensite E_M , the start stresses of austenite to martensite σ_s^{AM} and martensite to austenite σ_s^{MA} , the finish stress austenite to martensite σ_f^{AM} and martensite to austenite σ_f^{MA} . The subscript ‘0’ here indicates the first phase transformation cycle of cyclic tension–unloading. ϵ_L is the uniaxial maximum phase transformation strain. It should be noted that the parameters of elastic modulus and transformation stress are nominal variables due to the existence of residual martensite and its change during the cyclic loadings. Moreover, the dissipation energy W_d is defined as the area around by the stress–strain curve in each loading–unloading cycle,

$$W_d = \oint \sigma d\epsilon$$

This parameter reflects the damping feature of NiTi SMA, which is a unique property of the metals, and has been extensively applied in the engineering applications.

The capacity of the proposed model to describe the uniaxial transformation ratcheting of NiTi SMA at the temperature with pure austenite phase is firstly verified by comparing the predicted results with the corresponding experimental ones by Kang et al. [31] and the simulated ones by Kan et al. [30]. The material parameters of the used NiTi SMA cited from Kang et al. [31] are listed in Table 1.

Table 1 Material parameters observed by Kang et al. [31].

$$E_A=72 \text{ GPa}; E_M=45 \text{ GPa}; \nu_A=0.3; \nu_M=0.3; \varepsilon_L=0.043; T=295 \text{ K};$$

$$\sigma_{s0,T}^{AM}=320 \text{ MPa}; \sigma_{f0,T}^{AM}=550 \text{ MPa}; \sigma_{s0,T}^{MA}=405 \text{ MPa}; \sigma_{f0,T}^{MA}=120 \text{ MPa};$$

$$\sigma_{s1,T}^{AM}=120 \text{ MPa}; \sigma_{f1,T}^{AM}=550 \text{ MPa}; \sigma_{s1,T}^{MA}=317 \text{ MPa}; \sigma_{f1,T}^{MA}=50 \text{ MPa};$$

$$c_s^{AM}=0.05; c_f^{AM}=0.05; c_s^{MA}=0.05; c_f^{MA}=0.05; n=2; \xi_{ir\max}=0.75; b=0.2.$$

The results obtained with various peak stresses, e.g., 450, 500 and 550 MPa are shown in Figs. 2-4. It is seen from the figures that the proposed model provides reasonable predictions to the uniaxial transformation ratcheting of super-elastic NiTi SMA, compared with the experiments observed by Kang et al. [31], as shown in Figs. 2(a)–4(a). The simulated results obtained from simulated results with various peak stresses are shown in Figs. 2(b)–4(b). The figures show the reasonability by the current FE model to predict the uniaxial transformation ratcheting of super-elastic SMA, including the hysteresis loop curve, and the predicted peak and valley strain. The peak and residual strains of super-elastic SMA increase progressively during the initial several loading cycles, and to a stable value since then. The nearly-total closed hysteresis loop means that the strain increment happens in the tension loading is completely recoverable under the following unloading. The closed hysteresis loop that still becomes smaller and smaller during the further cyclic loading means that the dissipation energy decreases with the increase number of loading cycles. Compared with the Kan-Kang's model with linear hardening law, as describe in [29], the introduction of the cosine-type nonlinear function in the constitutive model enables the predicted hysteresis loop with an apparent nonlinear feature that is more identical with the experimental results .

Due to the introduced power function $c_{AM}^f(\sigma)$ that is associated with the applied loading level in the phase transformation ratcheting simulation, the model can reasonably predict the variation of peak and valley strain with different applied loading level. It should be noted that the introduction of the power equation with the coefficient of m and n , as described by Eqs (20a) and (22a), can reasonably improves the prediction of peak strain and valley residual strain changing with different applied loading level, though there exists difference between the simulated and

experimental results yet, especially for larger loading levels. This indicates that the power equation introduced in the proposed model does still not fully reflect the highly nonlinear relationship between the transformation ratcheting and the applied loading level. To achieve better prediction results, it should be introduced a more complex function formulation with added more material parameters.

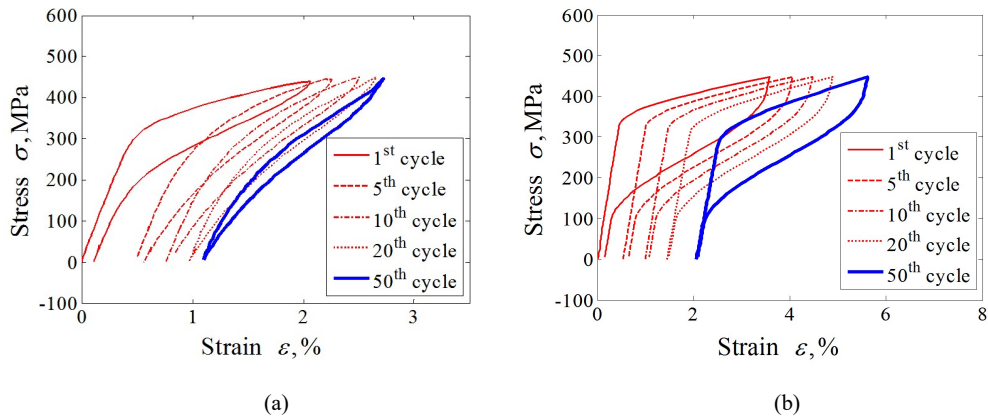


Fig. 1. Experiments and simulations for cyclic tension-tension with loading stress of 450 MPa: (a) experimental result; (b) simulated result by proposed model.

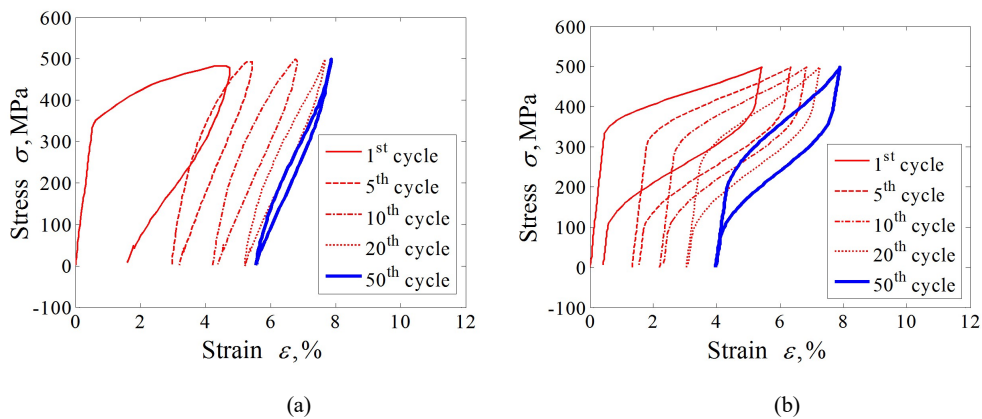


Fig. 2. Experiments and simulations for cyclic tension-tension with loading stress of 500 MPa: (a) experimental result; (b) simulated result by proposed model.

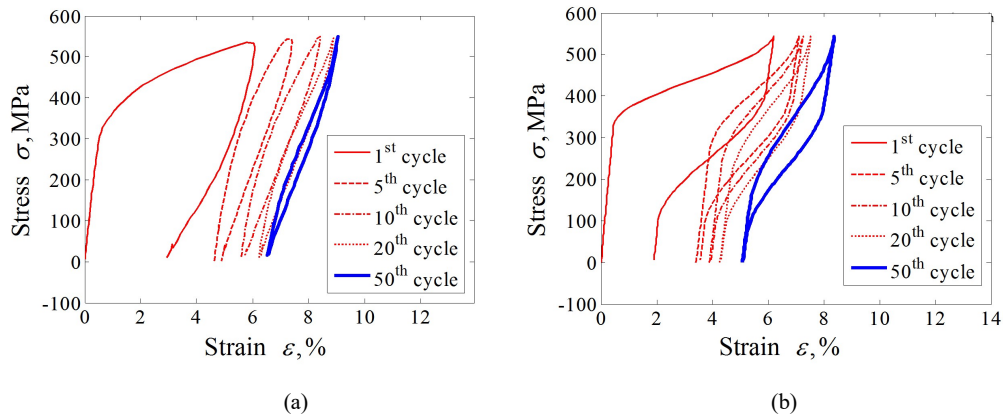


Fig. 3. Experiments and simulations for cyclic tension–tension with loading stress of 550 MPa: (a) experimental result; (b) simulated result by proposed model.

To evaluate the current FE model, some experiments under the uniaxial cyclic loading conditions is carried out and then are used to verify the validity of the proposed model in the following discussions. The materials used in the new experiments are the super-elastic NiTi SMA micro-tubes (Ni, 55.89% at mass, Xi'an Saite Metal Materials Development Co., Ltd., China). The heat treatment process is different from the SMA used in Kang et al. [31], and then the start temperature of martensite transformation A_f is 282K lower than the test temperature 298K, the original phase of the alloy is the pure austenite phase. Three different kinds of the uniaxial nominal are controlled by axial load under cyclic tension–unloading with positive mean loads, including cyclic tension–unloading tests with various applied peak stresses, e.g., 325, 365 and 405 at room temperature. The number of cycles is prescribed as 50, and the stress rate is prescribed as 20 MPa/s. The material parameters used in the current FE model are determined by trial and error method from the test data, as described in Section 2.2, and listed in Table 1 for the current FE simulation.

Fig. 5(a)-7(a) show the experimental results of the uniaxial tension-unloading, which is used to understand the basic performance of the NiTi SMA. It can be found from Fig. 6 that the start and finish stresses of forward transformation shows decrease from 285 MPa to 225 MPa with the increase number of loading cycles; while the start and finish stresses of reverse transformation are from 345 MPa to 320 MPa, respectively. After the unloading, a small residual strain is observed. It is mainly caused by the martensite residual during phase transformation.

It is seen from the figures that the proposed model provides reasonable predictions to the

uniaxial transformation ratcheting of super-elastic NiTi SMA. The predicted peak and residual strains, and their evolution characteristics are in fairly good agreement with the experimental results obtained in the cyclic tension-unloading tests by current tests. Also, the dependence of transformation ratcheting on the applied peak stress is reasonably predicted by the model due to the employment of stress-dependent power function.

Table 2 Material parameters determined by tests of current work

$$E_A = 48 \text{ GPa}; E_M = 35 \text{ GPa}; \nu_A = 0.3; \nu_M = 0.3; \varepsilon_L = 0.063; T = 295 \text{ K};$$

$$\sigma_{s0,T}^{AM} = 285 \text{ MPa}; \sigma_{f0,T}^{AM} = 550 \text{ MPa}; \sigma_{s0,T}^{MA} = 345 \text{ MPa}; \sigma_{f0,T}^{MA} = 164 \text{ MPa};$$

$$\sigma_{s1,T}^{AM} = 225 \text{ MPa}; \sigma_{f1,T}^{AM} = 550 \text{ MPa}; \sigma_{s1,T}^{MA} = 320 \text{ MPa}; \sigma_{f1,T}^{MA} = 35 \text{ MPa};$$

$$c_s^{AM} = 0.05; c_f^{AM} = 0.05; c_s^{MA} = 0.05; c_f^{MA} = 0.05; n = 3; \xi_{ir\max} = 0.75; b = 0.5.$$

From Figs. 5(c, d), 6(c, d) and 7(c, d), it is found that the peak and valley accumulation strains increase with an exponential function law during the loading cycles, and the dissipation energy W_d decreases on the contrary. The relative high degree of uniformity illustrates that the proposed model could reasonably simulate the transformation ratcheting of SMA. After a certain cycles, both of accumulation strains and dissipation energy show apparent change and quickly to a stable value. These findings also agree with the conclusions simulated by Kan-Kang's model [33].

To further demonstrate the reasonability of the proposed constitutive model to predict the transformation ratcheting of super-elastic NiTi alloy, two representative constitutive models developed by [28-29] are discussed about their capability to predict the transformation ratcheting in Fig. 8, respectively. The experimental data is obtained from [7]. The proposed model shows the capability to predict the nonlinear feature of hysteresis loop by compared with the Kan-Kang's model with linear hardening feature and the identical number of material parameters. Compared with the Lagoudas's model, the proposed model can also predict the nonlinear feature of hysteresis loop with the smaller number of material parameters. In addition, the proposed model can further give a reasonable description for the cyclic deformation characteristic on a specific applied loading level.

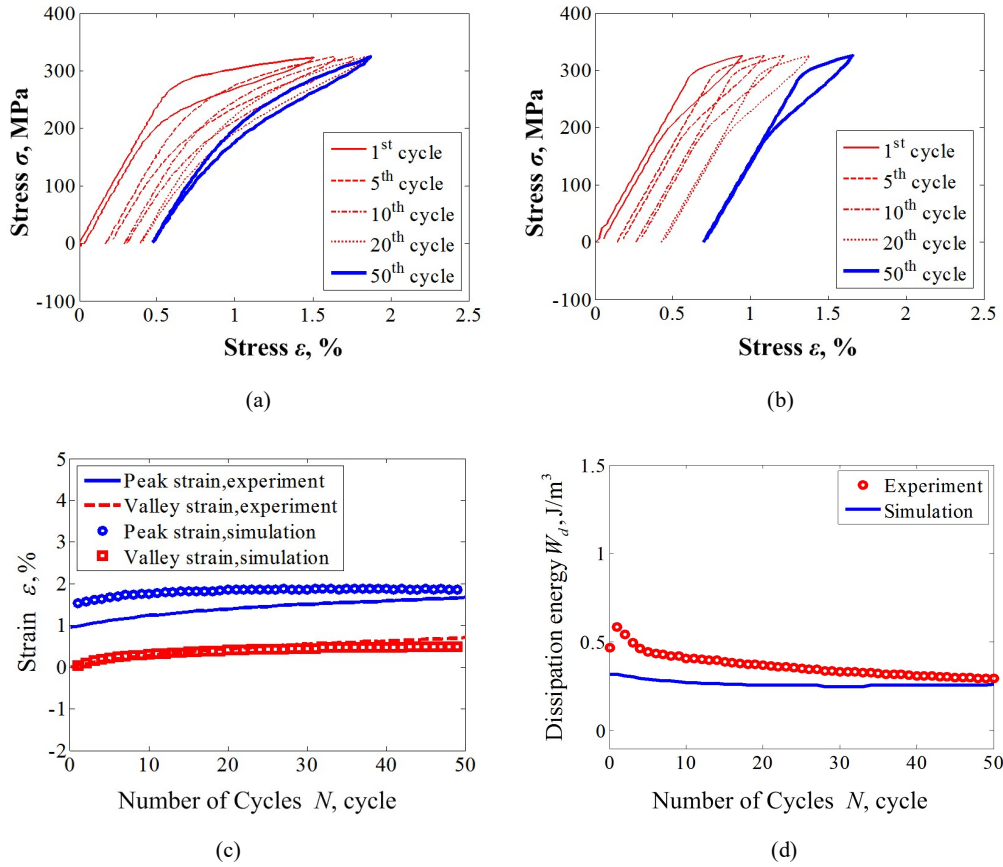
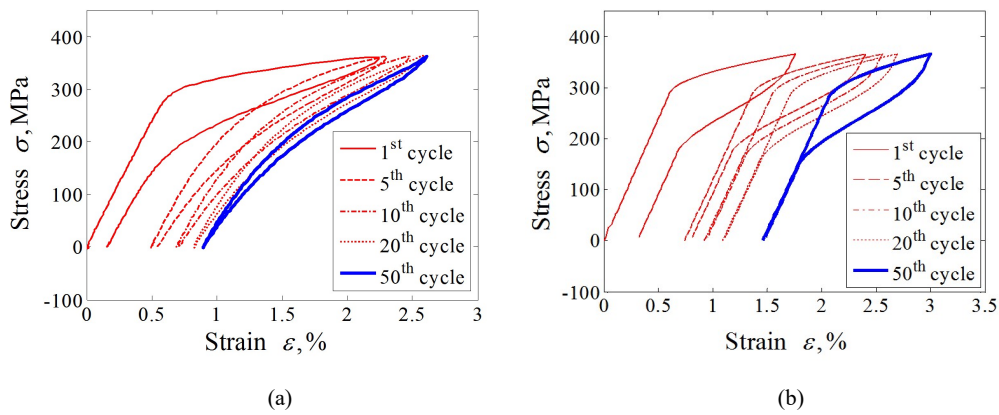


Fig. 5. Experiments and simulations for cyclic tension–unloading transformation ratchetting with constant loading stress of 325 MPa: (a) experimental results; (b) simulated results; (c) curves of peak and valley strains vs. cycle numbers; (d) curves of dissipation energy vs. cycle numbers.



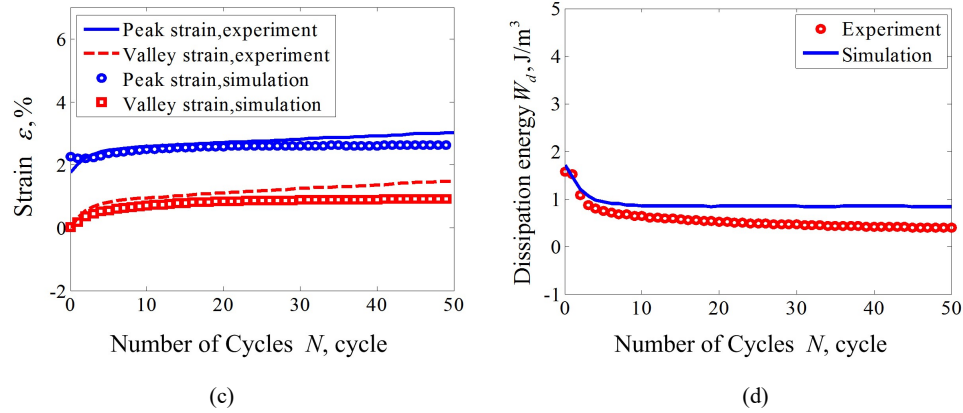


Fig. 6. Experiments and simulations for cyclic tension–unloading transformation ratchetting with constant loading stress of 365 MPa: (a) experimental results; (b) simulated results; (c) curves of peak and valley strains vs. cycle numbers; (d) curves of dissipation energy vs. cycle number.

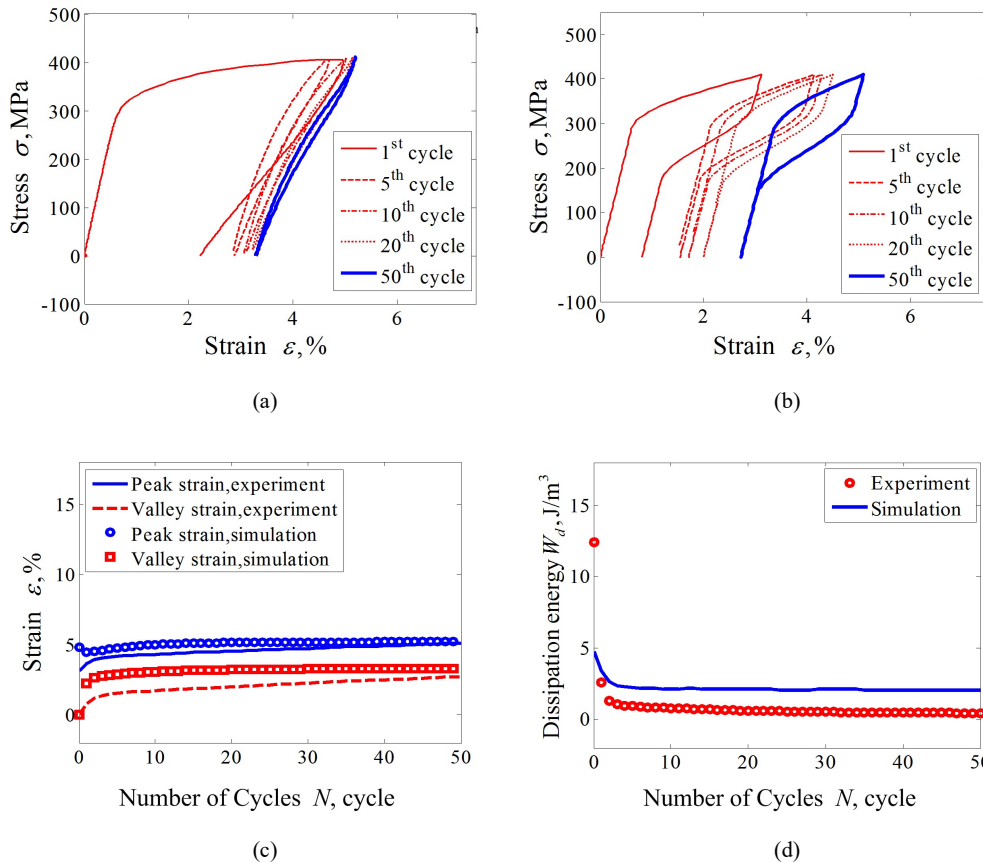


Fig. 7. Experiments and simulations for cyclic tension–unloading transformation ratchetting with constant loading stress of 405 MPa: (a) experimental results; (b) simulated results; (c) curves of peak and valley strains vs. cycle numbers; (d) curves of dissipation energy vs. cycle number.

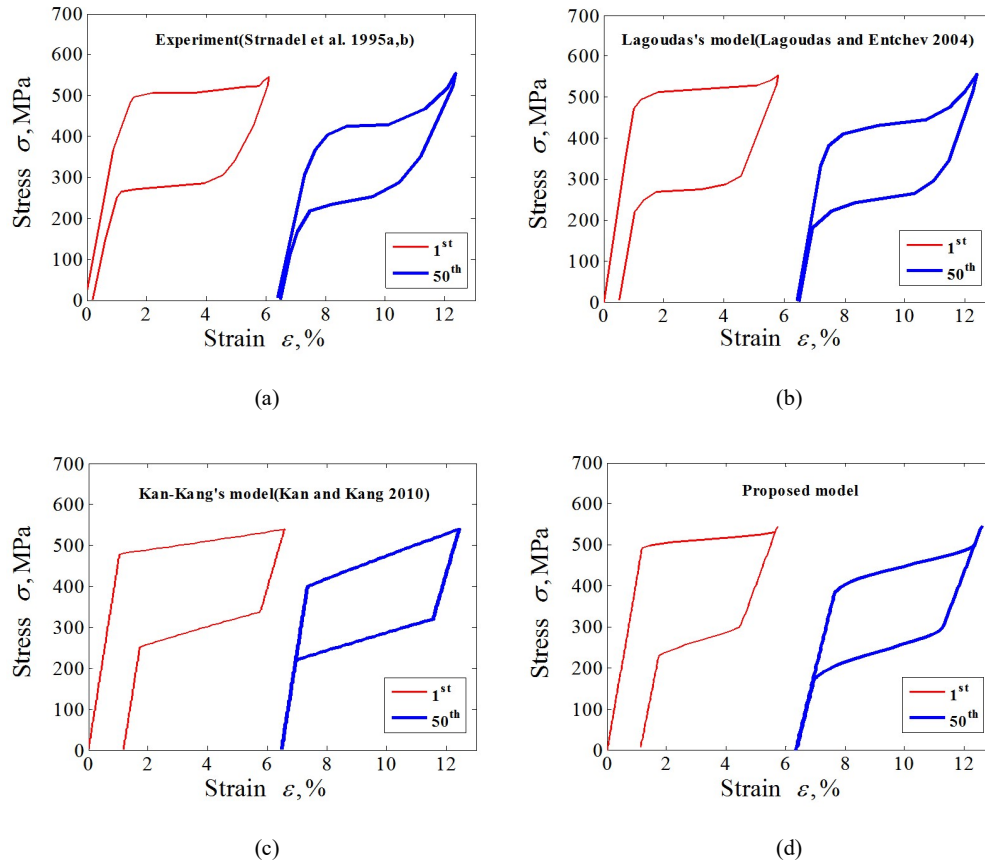


Fig. 8. Stress–strain response of SMA to cyclic loading up to a constant value of stress: curves for the first and 50th cycles: (a) experimental results; (b) simulated results by Lagoudas's model; (c) simulated results by Kan-Kang's model; (d) simulated results by proposed model.

It should be noted that the exponential function introduced in the proposed model correlated the phase transformation ratcheting to the loading stress level still cannot fully reflect the high nonlinear correlation between this ratcheting behavior and the loading stress level (e.g. Figs. 3 and 7). In order to achieve the better prediction results, it is necessary to consider the introduction of more complex functional forms, and increase more material parameters. The evolution of the phase transformation ratcheting strain is the focus of this paper. Therefore, this paper adopts a simpler exponential function which can describe its general rule.

For some cases with unsatisfactory simulation results, the following are explained:

(1) In the phase transformation process, the mismatch of the internal strain of austenite and martensite produces large local stress between the interface of austenite and martensite. The local stress promotes the dislocation slip of austenite to reach the starting critical stress, that is, the

induced plasticity due to phase transformation is produced. However, because the grain size of the selected material for experiment is relatively large, only the austenite near the interface between austenite and martensite can be motivated the induced plasticity due to phase transformation at some certain stress level. When the outer stress level reaches the starting critical stress of austenite dislocation slip, that slip of the whole grain can be motivated. Therefore, by the comparison between the experimental results of Figs. 2(a)-4 (a), it is known that the valley value strain of the first circle show a significant nonlinear growth, and the same phenomenon can be observed from the experimental results of Figs. 5-7.

(2) The phenomenological constitutive model proposed in this paper shows reasonability as the amount of the induced plasticity is small. However, it does not consider the critical motivation conditions of the starting stress of the dislocation slip of austenite. Therefore, when the stress level of the external load reaches a certain value, for example, more than 500MPa, there exists a certain difference between the proposed model and the experimental value in the prediction of the hysteresis loop.

(3) Because of the above reasons, the prediction of ratcheting strain has a certain deviation in the initial cycles when the external load is larger, and since then a better prediction result is obtained. Although the proposed model has a certain numerical difference from the experimental results in the prediction of individual conditions, the main characteristics of the described phase transformation ratcheting behavior are in accordance with the experimental results. For this kind of cyclic deformation with so strong nonlinearity, the accurate description of the hysteresis loop should increase a lot of material parameters, which is not conducive to the application of the engineering for the constitutive model.

5. Numerical examples for the analyses of preload force of SMA bolted joint

The preload force of SMA bolted joint is an important factor for the service properties of bolted joint, which also plays a crucial role in protecting the static and dynamic characteristics and the locking and sealing properties of the whole SMA joint. However, in some conditions, it is inevitable to reload the preload force of SMA bolt. This repeated process might decay the excellent super-elastic of SMA bolt due to its phase transformation ratcheting of materials, especially for large applied preload force. In this paper, based on the one-dimension FE model of

initial value of Δu . The target of the iterative calculation is to reach the force equilibrium conditions, i.e. $F_b = F_m = F_{p0}$ from initial force conditions $F_b = F_m = 0$. For each sub-step of iterative calculation, a sub tension force is applied on the bolt, and the sub compression force is used on the member. The amount of deformation for each iterative sub-step are set as Δu^b and Δu^m for bolt and member, respectively. For the absolute coordinates value of nodes 2 and 4 for each iterative sub-step calculation, the following deformation equation gets to work:

$$\begin{cases} u_n^b = u_n^b + \Delta u_{n-1}^b \\ u_n^m = u_n^m + \Delta u_{n-1}^m \\ \Delta u_n = u_n^b - u_n^m \end{cases} \quad (42)$$

in which, n is number of iterative sub-step.

The amount of interference Δu decreases gradually with iterative calculation in progress. For the case of $\Delta u = 0$, the amount of penetration between the bearing surfaces of the bolt or nut and the member is zero, and the iteration is terminated. It should be noted that the target preloading force F_{p0} should be continuously adjusted the amount of initial interference Δu_0 by trial and error method. The specific implementation process is shown in the flow chart of Fig. 10.

It should be noted that the SMA bolt element and the member element are independent for each other in FE modeling. It means that the force increments of iteration for ΔF_n^m and ΔF_n^e are applied on the SMA bolt element and the member element, respectively, in spite of $\Delta F_n^m = \Delta F_n^e$. Then the deformations for each element are calculated independently until the conditions of convergence $\Delta u = 0$ is satisfied.

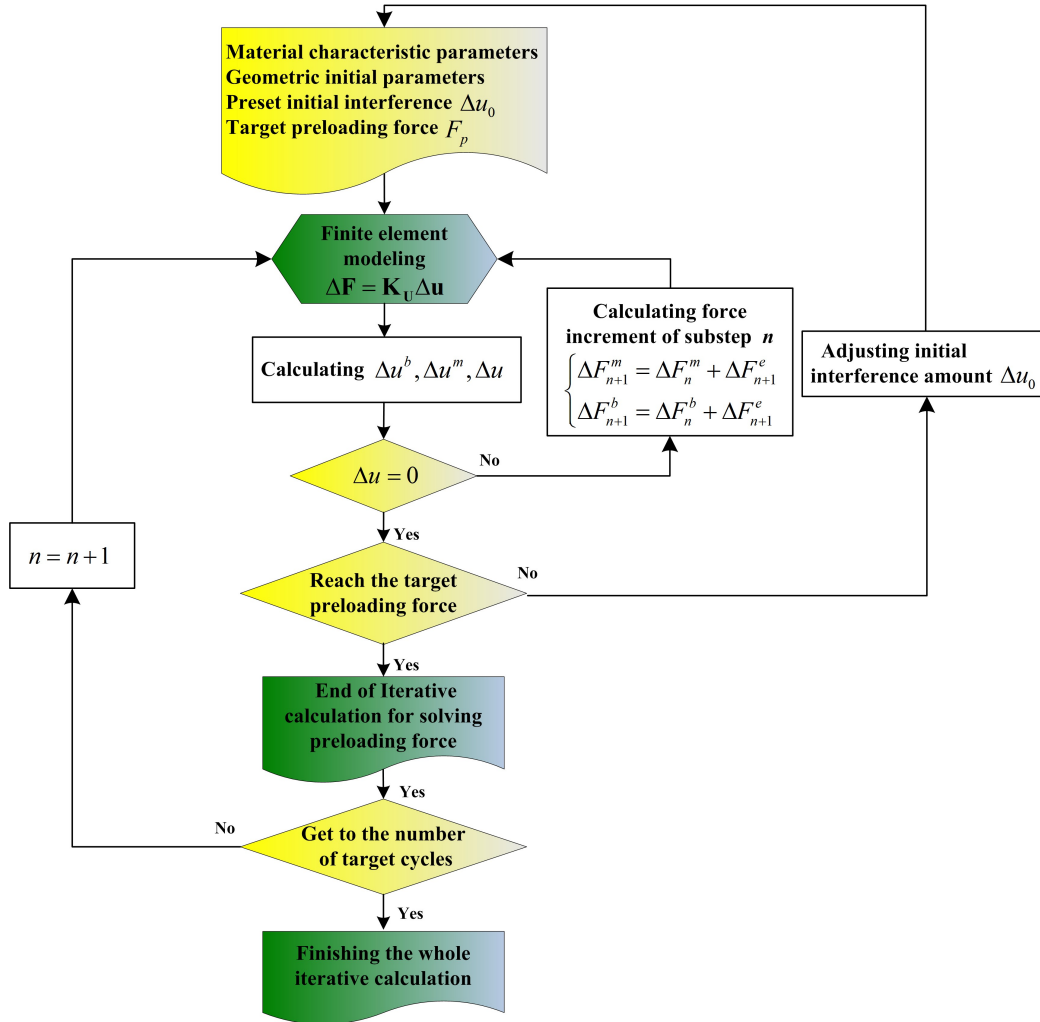


Fig. 10. Flow chart of analysis of the repeated cyclic preloading

The equilibrium equation of SMA bolt or member can be expressed using the standard FE assembly operation:

$$\Delta \mathbf{F} = \mathbf{K}_U \Delta \mathbf{u} \quad (43)$$

Where, $\Delta \mathbf{F}$ indicates the increment of nodal force vectors of bar elements of the FE model, $\Delta \mathbf{u}$ is the nodal displacement vectors, \mathbf{K}_U is the stiffness matrices.

The FE model of a member is shown in Fig. 9. It can be seen that the member elements consist of the nodes 1 and 2. The node 2 is the restrained end, and the node 1 is the force applied end of member element. Thus, Eq. (43) can be further decomposed into

$$\begin{bmatrix} \Delta \mathbf{F}_1 \\ \Delta \mathbf{F}_2 \end{bmatrix} = \begin{bmatrix} \mathbf{K}_u^{11} & \mathbf{K}_u^{12} \\ \mathbf{K}_u^{21} & \mathbf{K}_u^{22} \end{bmatrix} \begin{bmatrix} \Delta \mathbf{u}_1 \\ \Delta \mathbf{u}_2 \end{bmatrix} \quad (44)$$

where, \mathbf{u}_1 , \mathbf{u}_2 and \mathbf{F}_1 , \mathbf{F}_2 are the displacements and the forces of the node 1 and 2, respectively. \mathbf{F}_1 and \mathbf{F}_2 are a pair of equilibrium forces with equal magnitude in opposite directions. \mathbf{K}_u^{11} , \mathbf{K}_u^{12} , \mathbf{K}_u^{13} , \mathbf{K}_u^{21} and \mathbf{K}_u^{22} are the partitioned matrices of matrix \mathbf{K}_u^m .

Since the member is equivalent to a bar element in the FE model, the relationship between force vector and displacement vector of the node i can be written as

$$\mathbf{F}_m = \frac{E_m A_m}{L_{m0}} (L_m - L_{m0}) \frac{\mathbf{x}}{L_m} = k_m (L_m - L_{m0}) \frac{\mathbf{x}}{L_m} \quad (45)$$

where, A_m is the equivalent cross section area of the member element. L_{m0} are the initial unstressed length of member element, E_m is the elastic modulus of member element, k_m is the compression stiffness of member element. \mathbf{F}_m is the force vector of member element, $\mathbf{x} = \mathbf{x}_i - \mathbf{x}_j$ depicts the relative nodal position denoted by \mathbf{x}_i and \mathbf{x}_j in the global frame. $i, j=1$ or 2 are the node numbers of member element. $L_m = \left[(\mathbf{x}_i - \mathbf{x}_j)^T (\mathbf{x}_i - \mathbf{x}_j) \right]^{1/2}$ is the stressed length of the member element after deformed.

With the first order Taylor expansion of Eq. (45), it can be derived as

$$\Delta \mathbf{F}_m = \frac{\partial \mathbf{F}_m}{\partial \mathbf{x}} \Delta \mathbf{x} \quad (46)$$

Since $\Delta \mathbf{F}_j = -\Delta \mathbf{F}_i$, the incremental relationship between nodal force and element length can be expressed as

$$\Delta \mathbf{F}_m = \mathbf{K}_m \Delta \mathbf{u}_m$$

$$\text{where, } \Delta \mathbf{F}_m = \begin{bmatrix} \Delta \mathbf{F}_i \\ \Delta \mathbf{F}_j \end{bmatrix}, \Delta \mathbf{u}_m = \begin{bmatrix} \Delta \mathbf{u}_i \\ \Delta \mathbf{u}_j \end{bmatrix} = \begin{bmatrix} \Delta \mathbf{x}_i \\ \Delta \mathbf{x}_j \end{bmatrix}, \text{ and } \mathbf{K}_m = \begin{bmatrix} k_m & -k_m \\ -k_m & k_m \end{bmatrix}$$

The stiffness matrix of one node can be obtained by the differential calculation as

$$k_m = \frac{\Delta \mathbf{F}_m}{\Delta \mathbf{x}} = \frac{E_m A_m}{L_m} \frac{\mathbf{x} \mathbf{x}^T}{L_m^2} + \frac{E_m A_m}{L_m} \frac{L_m - L_{m0}}{L_{m0}} \mathbf{I}_3 = \frac{k_m L_{m0}}{L_m} \frac{\mathbf{x} \mathbf{x}^T}{L_m^2} + k_m \frac{L_m - L_{m0}}{L_m} \mathbf{I}_3 \quad (47)$$

where, \mathbf{I}_3 is 3×3 is identity matrix.

The equilibrium equation of member structure can be achieved using the standard FE assembly operation as

$$\Delta \mathbf{F} = \mathbf{K}_m \Delta \mathbf{u}_m = \mathbf{K}_U^m \Delta \mathbf{u} \quad (48)$$

The boundary condition of member element during the preloading of bolt is $\Delta \mathbf{u}_4 = \mathbf{0}$. Thus,

$$\begin{cases} \mathbf{K}_u^{33} \Delta \mathbf{u}_3 = \Delta \mathbf{F}_3 \\ \Delta \mathbf{F}_4 = -\Delta \mathbf{F}_3 \end{cases} \quad (49)$$

The further derivation gives

$$\Delta \mathbf{u}_3 = [\mathbf{K}_u^{33}]^{-1} \Delta \mathbf{F}_3 \quad (50)$$

According to the above calculation, the deformation of each element of member and the relationship of stress and strain are obtained.

As for the SMA bolt, the FE model of bolt is established simplistically to be a bar element. As shown in Fig. 9, the SMA bolt element consists of nodes 3 and 4. Therefore, Eq. (43) can be divided into

$$\begin{bmatrix} \Delta \mathbf{F}_3 \\ \Delta \mathbf{F}_4 \end{bmatrix} = \begin{bmatrix} \mathbf{K}_u^{33} & | & \mathbf{K}_u^{34} \\ \mathbf{K}_u^{43} & | & \mathbf{K}_u^{44} \end{bmatrix} \begin{bmatrix} \Delta \mathbf{u}_3 \\ \Delta \mathbf{u}_4 \end{bmatrix} \quad (51)$$

where, \mathbf{u}_3 , \mathbf{u}_4 and \mathbf{F}_3 , \mathbf{F}_4 are the displacements and the forces of the nodes 3 and 4, respectively. \mathbf{F}_3 and \mathbf{F}_4 are a pair of equilibrium forces with equal magnitude in opposite directions. \mathbf{K}_u^{33} , \mathbf{K}_u^{34} , \mathbf{K}_u^{43} and \mathbf{K}_u^{44} are the partitioned matrices of matrix \mathbf{K}_U^b .

The boundary condition of SMA bolt element during the preloading of bolt is $\Delta \mathbf{u}_4 = \mathbf{0}$. Thus,

$$\begin{cases} \mathbf{K}_u^{33} \Delta \mathbf{u}_3 = \Delta \mathbf{F}_3 \\ \Delta \mathbf{F}_4 = -\Delta \mathbf{F}_3 \end{cases} \quad (52)$$

The further derivation gives

$$\Delta \mathbf{u}_3 = [\mathbf{K}_u^{33}]^{-1} \Delta \mathbf{F}_3 \quad (53)$$

Since the FE model of SMA bolt is established as just one bar element, it is no need to carry out the assembly calculation of FE modeling. Therefore,

$$\Delta \mathbf{F} = \mathbf{K}_b \Delta \mathbf{u}_b = \mathbf{K}_U^b \Delta \mathbf{u} \quad (54)$$

$$\text{where, } \mathbf{K}_b = \begin{bmatrix} k_b & | & -k_b \\ -k_b & | & k_b \end{bmatrix}$$

and $k_b = \mathbf{K}_U^{sma}$. \mathbf{K}_U^{sma} is the stiffness matrix of node in SMA bolt element, and can be seen in Eq. (48). It is given by comparing Eq. (51) and Eq. (54),

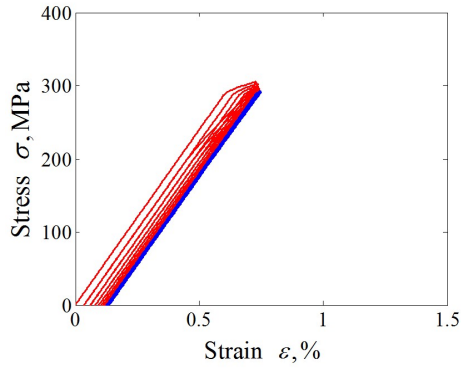
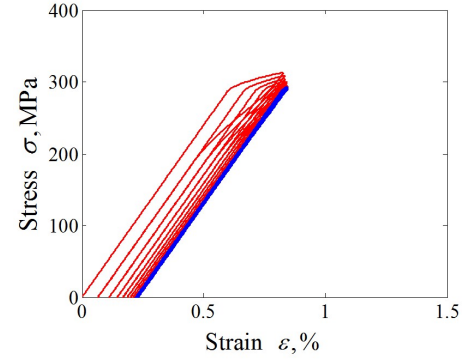
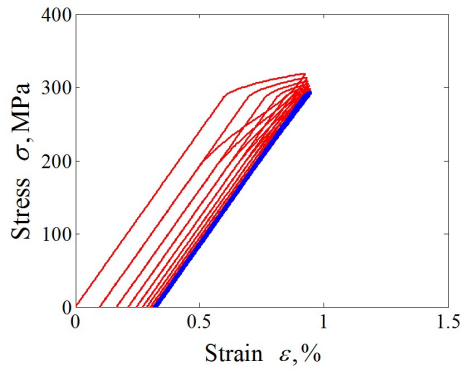
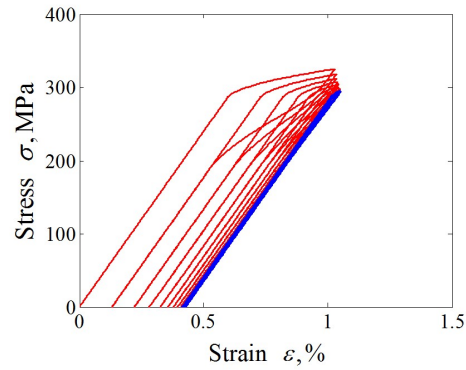
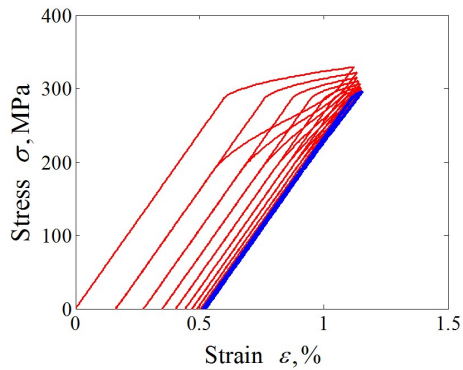
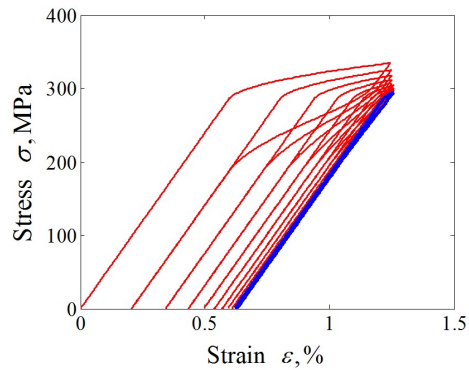
$$\Delta \mathbf{F}_b = \begin{bmatrix} \frac{\Delta \mathbf{F}_3}{\Delta \mathbf{F}_4} \end{bmatrix}, \Delta \mathbf{u}_b = \begin{bmatrix} \frac{\Delta \mathbf{u}_3}{\Delta \mathbf{u}_4} \end{bmatrix}, \mathbf{K}_U^b = \begin{bmatrix} \mathbf{K}_u^{33} & \mathbf{K}_u^{34} \\ \mathbf{K}_u^{43} & \mathbf{K}_u^{44} \end{bmatrix} \quad (55)$$

5.2 Simulations and Results

In this section, it is given a numerical example about the repeated preload process of SMA bolted joint based on the above FE model. The M6 NiTi SMA bolt is selected in this research. Upper and lower members are completely identical, including dimensional size and materials. The calculation approach of the member stiffness can be cited from the references in Motosh and Nassar [35-36] who had developed the analytical model to describe the accurate member stiffness relatively. The stiffness value is calculated as $k_m=4 \times 10^8 \text{ N} \cdot \text{m}^{-1}$ as a reference with the dimensional sizes of height 50mm and diameter 30mm, and the elastic modulus $70 \times 10^9 \text{ Pa}$ and Poisson ratio 0.28 of selected aluminum.

FE simulations are conducted for ten transverse loading cycles. Figure 11 shows the stress-strain hysteresis loops obtained from the bolt bar for different amount of preset interference, i.e. $\Delta u_0=0.5\text{mm}$, $\Delta u_0=0.55\text{mm}$, $\Delta u_0=0.6\text{mm}$, $\Delta u_0=0.65\text{mm}$ and $\Delta u_0=0.7\text{mm}$, that correspond to the initial preload forces of SMA bolt as $F_{p0}=8.65\text{kN}$, $F_{p0}=8.86\text{kN}$, $F_{p0}=9.04\text{kN}$, $F_{p0}=9.19\text{kN}$ and $F_{p0}=9.32\text{kN}$. It can be found that the SMA material of the bolt experienced the cyclic ratcheting with increasing number of repeated preloading cycles. The repeated cyclic preloading will produce lower clamping force of bolt with the same amount of preset interference until tension stress value of bolt is close to the martensite start stress. Similar to the materials ratcheting behavior, the clamping force of SMA bolt experiences the great attenuation during the initial several loading cycles and tends to be stable with a very small value since then. It should be noted that although the attenuation rate of the clamping force of bolt increases with the increase of initial preload force of bolt, the larger initial preload force of bolt shows higher residual clamping force after experienced cycle preloading. After experienced nearly five loading cycles, the martensite phase produced in initial preload will be close to disappearance. It can also be found from Fig. 12 that the higher member stiffness would improve the reduction of preload force under repeated preloading cycles that may help the engineer to improve his design. However, as the member

stiffness in this research is higher than $k_m=7 \times 10^8 \text{ N} \cdot \text{m}^{-1}$, the initial preload and its attenuation rate are not changed obviously. It means that the effectiveness of the optimum design from the standpoint of adjustment of k_m will not be too obvious.

(a) $\Delta u_0=0.5\text{mm}$, $k_m=4 \times 10^8 \text{ N} \cdot \text{m}^{-1}$ (b) $\Delta u_0=0.55\text{mm}$, $k_m=4 \times 10^8 \text{ N} \cdot \text{m}^{-1}$ (c) $\Delta u_0=0.6\text{mm}$, $k_m=4 \times 10^8 \text{ N} \cdot \text{m}^{-1}$ (d) $\Delta u_0=0.65\text{mm}$, $k_m=4 \times 10^8 \text{ N} \cdot \text{m}^{-1}$ (e) $\Delta u_0=0.7\text{mm}$, $k_m=4 \times 10^8 \text{ N} \cdot \text{m}^{-1}$ (f) $\Delta u_0=0.7\text{mm}$, $k_m=7 \times 10^8 \text{ N} \cdot \text{m}^{-1}$

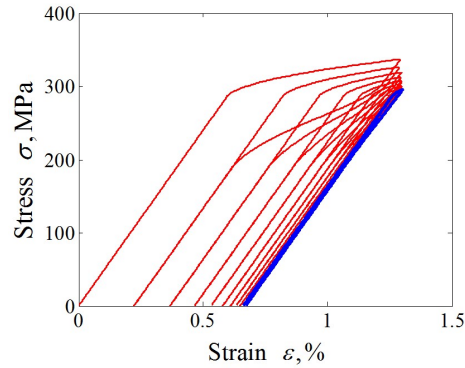
(g) $\Delta u_0=0.7\text{mm}$, $k_m=1\times 10^9 \text{ N}\cdot\text{m}^{-1}$

Fig. 11. Curves of stress–strain responses on bolt bar under different load cases

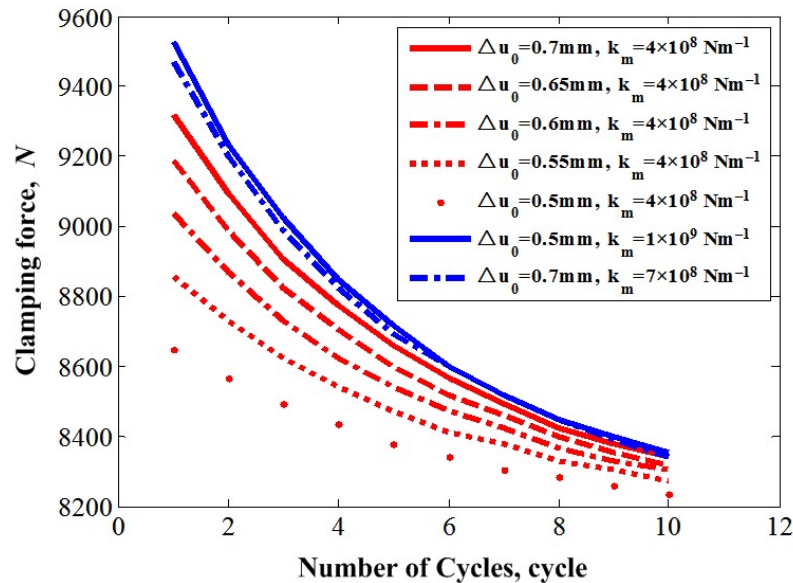


Fig. 12. Clamping force reduction of bolt with increasing loading cycles under different load cases

5. Conclusions

In this work, a phenomenological constitutive modeling and its FE implementation of super-elastic SMAs is carried out to simulate the transformation ratcheting behaviors of the super-elastic SMA undergoing cyclic loading, the conclusions are obtained as follows:

(1) The assumed cosine-type phase transformation function considering the initial martensite evolution is verified to be candidate for predicting the contributions of cyclic accumulation of residual martensite caused by incomplete reverse transformation, and the nonlinear behavior of hysteresis loop undergoing cyclic loading. The evolutions of transformation ratcheting, and

transformation stresses can be controlled by accumulated martensite volume fraction as a internal variable. The correlation between the applied loading level and the transformation ratcheting is also able to be captured by the proposed model.

(2) A FE model derived by return-mapping method is then to analyze the attenuation law of the clamping force of SMA bolt experiences as numerical example. It is found that the great attenuation of clamping force of SMA bolt happens during the initial several loading cycles and tends to be stable with a very small value since then. The conclusions that the larger initial preload force of bolt shows higher residual clamping force after experienced cycle preloading, and the higher member stiffness would improve the reduction of preload force under repeated preloading cycles would help the engineer to improve his design.

Acknowledgments

We would like to thank for the financial support of this work by the National Natural Science Foundation of China (NSFC) under Grant Number 51775406, 51405371, and 51675398, the Open Research Fund of State Key Laboratory of Structural Analysis for Industrial Equipment (Grant No. GZ1612), the Fundamental Research Funds for the Central Universities (Grant No. JB180412), the Natural Science Foundation of Shanxi Province of China (Grant No. 2017JM5035), 111 Project (B14042).

References

- [1] Humbeeck, J. V. (1999). "Non-medical applications of shape memory alloys." *Mater. Sci. Eng., A*, 273–275, 134–148.
- [2] Duerig, T., Pelton, A., and Stöckel, D. (1999). "An overview of nitinol medical applications." *Mater. Sci. Eng., A*, 273–275, 149–160.
- [3] Morgan, NB. (2004). "Medical shape memory alloy applications: the market and its products." *Mater. Sci. Eng., A*, 378, 134–148.
- [4] Miyazaki S., Otsuka K., and Suzuki Y. (1981). "Transformation pseudoelasticity and deformation behavior in a Ti–50.6 at% Ni alloy." *Scripta Metall.*, 15(3) : 287–292.

- [5] Miyazaki S., Imai T., and Lgo Y. (1986). "Effect of cyclic deformation on the pseudoelasticity characteristics of TiNi alloy." *Metall.Trans.*, 17A: 115–120.
- [6] Strnadel, B., Ohashi, S., Ohtsuka, H., Ishihara, T., and Miyazaki, S. (1995a). "Cyclic stress–strain characteristics of Ti–Ni and Ti–Ni–Cu shape memory alloys." *Mater. Sci. Eng. A*, 202: 148–156.
- [7] Strnadel, B., Ohashi, S., Ohtsuka, H., Miyazaki, S., and Ishihara, T. (1995b). "Effect of mechanical cycling on the pseudoelasticity characteristics of Ti–Ni and Ti–Ni–Cu alloys." *Mater. Sci. Eng. A*, 203: 187–196.
- [8] Nemat–Nasser S., and Guo W. G. (2006). "Super–elastic and cyclic response of NiTi SMA at various strain rates and temperatures." *Mech. Mater.*, 38 (5–6) :463–474.
- [9] Lagoudas, DC., and Bo, Z. (1999). "Thermomechanical modeling of polycrystalline SMAs under cyclic loading, Part II: material characterization and experimental results for a stable transformation cycle." *Int. J. Eng. Sci.*, 37 (9), 1141–1173.
- [10] Sehitoglu, H., Anderson, R., Karaman, I., Gall, K., and Chumlyakov, Y. (2001). "Cyclic deformation behavior of single crystal NiTi." *Mater. Sci. Eng., A*, 314(1–2), 67–74.
- [11] Gall, K., and Maier, HJ. (2002). "Cyclic deformation mechanisms in precipitated NiTi shape memory alloys." *Acta. Mater.*, 50(18), 4643–4657.
- [12] Wang, X., Xu, B., and Yue, Z.(2008). "Phase transformation behavior of pseudoelastic NiTi shape memory alloys under large strain." *J. Alloys Compd.*, 463(1), 417–422.
- [13] Song, D., Kang, G., Kan, Q., Yu, C., and Zhang, C. (2014). "Non–proportional multiaxial transformation ratchetting of super–elastic NiTi shape memory alloy: Experimental observations." *Mech. Mater.*, 70(1), 94–105.
- [14] Grabe, C., and Bruhns, O. T. (2008). "On the viscous and strain rate dependent behavior of polycrystalline NiTi." *Int. J. Solids Struct.*, 45(7), 1876–1895.
- [15] Morin, C., Moumni, Z., and Zaki, W. (2011). "A constitutive model for shape memory alloys accounting for thermomechanical coupling." *Int. J. Plast.*, 27(5), 748–767.
- [16] He, Y. J., and Sun, Q. P. (2011). "On non–monotonic rate dependence of stress hysteresis of superelastic shape memory alloy bars." *Int. J. Solids Struct.*, 48(11), 1688–1695.
- [17] He, Y. J., and Sun, Q. P. (2010). "Rate–dependent domain spacing in a stretched NiTi strip." *Int. J. Solids Struct.*, 47 (20), 2775–2783.
- [18] He, Y. J., and Sun, Q. P. (2010). "Frequency–dependent temperature evolution in NiTi shape memory alloy under cyclic loading." *Smart Mater. Struct.*, 19(19), 115014.
- [19] He, Y., Yin, H., Zhou, R., and Sun, Q. (2010). "Ambient effect on damping peak of NiTi shape memory

alloy." *Mater. Lett.*, 64(13),483–1486.

[20] Yin, H., He, Y., and Sun, Q. (2014). "Effect of deformation frequency on temperature and stress oscillations in cyclic phase transition of NiTi shape memory alloy." *J. Mech. Phys. Solids*, 67(1), 100–128.

[21] Kan, Q.H., Yu, C., Kang, G., Li, J., and Yan, W. (2016). "Experimental observations on rate-dependent cyclic deformation of super-elastic NiTi shape memory alloy." *Mech. Mater.*, 97, 48–58.

[22] Brinson, and L.C. (1993). "One-dimensional constitutive behavior of shape memory alloys: thermomechanical derivation with non-constant material functions." *J. Intell. Mater. Syst. Struct.*, 4(2), 229–242.

[23] Boyd, J.G., and Lagoudas, D.C. (1996). "A thermodynamical constitutive model for shape memory materials. Part I: The monolithic shape memory alloy." *Int. J. Plast.*, 12(6), 805–842.

[24] Auricchio, F., and Lubliner, J. (1997). "A uniaxial model for shape-memory alloys." *Int. J. Solids Struct.*, 34(27), 3601–3618.

[25] Zaki, W., and Moumni, Z. (2007). "A three-dimensional model of the thermomechanical behavior of shape memory alloys." *J. Mech. Phys. Solids*, 55(11), 2455–2490.

[26] Ortiz, M., Simo, J.C. (1986). "An analysis of a new class of integration algorithms for elastoplastic constitutive relations." *Int. J. Numer. Meth. Eng.*, 23: 353–366.

[27] Simo, J.C., Ortiz, M. (1985). A unified approach to finite deformation elastoplastic analysis based on the use of hyperelastic constitutive equations. *Comput. Meth. Appl. Mech. Eng.*, 49: 221–245.

[28] Lagoudas, D. C., and Entchev, P. B. (2004). Modelling of transformation-induced plasticity and its effect on the behavior of porous shape memory alloys. Part I: constitutive model for fully dense SMAs. *Mech. Mater.*, 36:

[29] Kan, Q., and Kang, G. (2010). Constitutive model for uniaxial transformation ratchetting of super-elastic NiTi shape memory alloy at room temperature. *Int. J. Plasticity*, 26 (3):441–465.

[30] Kan, Q. H., Kang, G. Z., and Guo, S. J. (2012). Finite element implementation of a super-elastic constitutive model for transformation ratchetting of NiTi alloy. *Int. J. Comput. Methods.*, 9(1): 1240022:1–12. 865–892.

[31] Kang, G. Z., Kan, Q. H., Qian, L. M., and Liu, Y.J. (2009a). Ratcheting deformation of superelastic and shape memory NiTi Alloys. *Mech. Mater.*, 41, 139–153.

[32] Jiang, X., Zhu, Y., Hong, J., Chen, X., and Zhang Y. (2013). Investigation into the loosening mechanism of bolt in curvic coupling subjected to transverse loading. *Eng. Fail. Anal.*, 32(3): 360–373.

[33] Jiang, X. J., Zhu, Y. S., Hong, J., and Zhu L. B. (2013). Development and Validation of an Analytical Model

for Stiffness Analysis of Curvic Coupling in Tightening. *J. Aerospace Eng.*, 6: 509-522.

[34] Jiang, X. J., Zhu, Y. S., Hong, J., and Zhu, L. B. (2016). Stiffness Analysis of Curvic Coupling in Tightening by Considering the Different Bolt Structures. *J. Aerospace Eng.*, 29 (3) :04015076.

[35] Motosh, N. (1976). Determination of Joint Stiffness in Bolted Connections. *ASME J. Eng. Ind.*, 983:858–861.

[36] Nassar, S. A., and Abboud A. (2009). An Improved Stiffness Model for Bolted Joints. *J. Mech. Design*, 131, (12): 1–10.